

Vehicle Level Preliminary Design Review

Spaceport America Cup 2024

IREC 30k SRAD Hybrid Engine, Experimental Payloads, and Airframe

10/30/23

Agenda

1. Purpose of PDR
2. Stakeholder Needs Identification
 - IREC Team Members
 - KXR Executive Board
 - Experimental Rocket Sounding Association
3. Concept Definition
 - Mission
 - Spaceport America Cup Competition
4. Element Architectures
5. System Architectures
6. Questions

Purpose of PDR

Reiterate and clarify stakeholder needs

Present Design Solutions for potential redirection

Receive feedback on progress and where to improve

Establish a baseline for our architecture

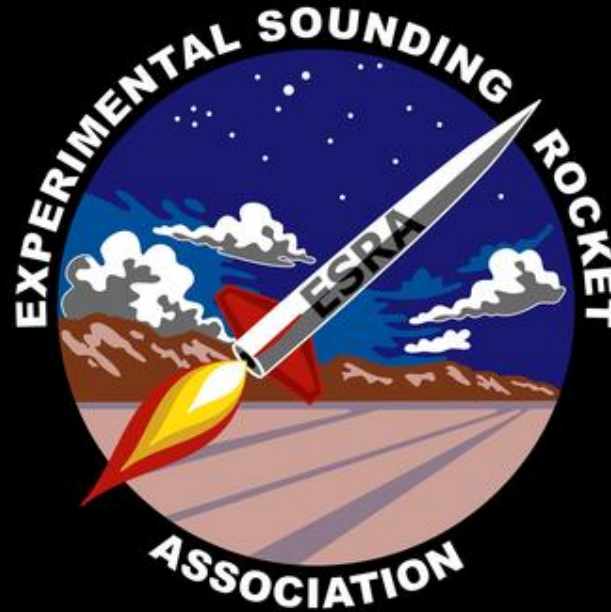
Ensure architecture is feasible within given requirements/constraints

Stakeholders



Our Team

Students striving to push themselves to prepare for industry through hands on experience.



ESRA

The platform to launch 30k rockets, competition, rules, and requirements.

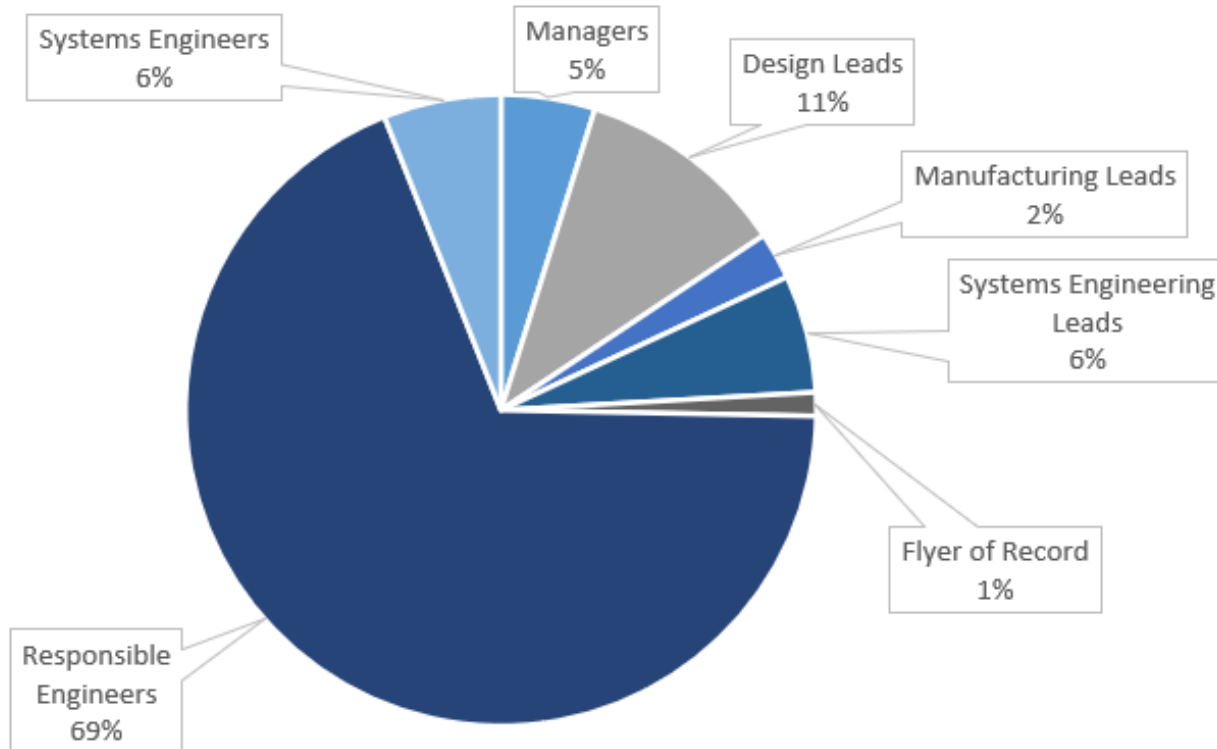


KXR Executive Board

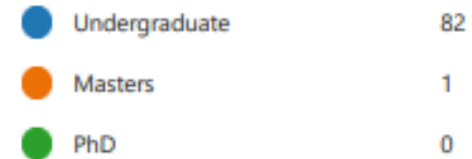
Provides students the opportunity to achieve their goals through funding.

Our Team - Demographics

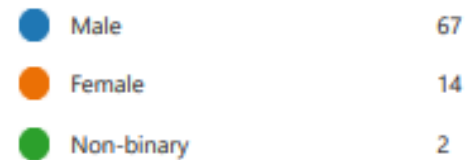
Team Responsibility Splits



1. What degree are you currently pursuing?



2. What do you identify as?



ESRA – Deliverables

| Deliverables | |
|-----------------------------|------------|
| Item | Deadline |
| 1st Interim Report | 12/15/2023 |
| 2nd Interim Report | 2/16/2024 |
| 3rd Interim Report | 4/19/2024 |
| Flight Readiness Review | 5/10/2024 |
| Technical Report | 5/10/2024 |
| Poster and Podium Materials | 5/10/2024 |
| School Participation Letter | 5/10/2024 |
| Final Launch Day | 6/22/2024 |

ESRA – Spaceport Cup Scoring Summary

| Deliverable | Category | Sub-Categories | Pts. Available |
|---|----------------------------|---|----------------|
| Early Deliverables (60 Points) | Entry Form | N/A | 15 |
| | 1st Interim Report | N/A | 15 |
| | 2nd Interim Report | N/A | 15 |
| | 3rd Interim Report | N/A | 15 |
| Technical Report (200 Points) | Completeness | N/A | 20 |
| | Style and Format | Style | 20 |
| | | Mechanics | 10 |
| | | Format | 10 |
| | Analysis | Depth of Analysis | 50 |
| | | Assumptions and Sensitivity Analysis | 30 |
| | | Verification and Validation tests | 40 |
| Use of Charts and Figures | | 20 | |
| Design Implementation (240 Points) | Design Quality & Decisions | Team Design Vision, Goals and Systems Engineering | 50 |
| | | SRAD components | 50 |
| | | Team Knowledge | 20 |
| | Build Quality | Design Quality and Robustness | 30 |
| | | Manufacturing and Construction Methods | 30 |
| | | Consistent Design | 30 |
| | | Compliance with DTEG | 30 |
| Flight Performance (500 Points) | Apogee Performance | See Equation | 350 |
| | Recovery Performance | N/A | 150 |
| Total | | | 1000 |

2.7.1.7 BONUSES FOR CUBESAT BASED PAYLOADS

Teams whose payload(s) qualify for the form factor exemption described in Section 2.3.5.2 of this document, yet still adopt the CubeSat standard form factor, will be awarded 50 bonus points in addition to their total earned score. This promotes ESRA and SDL's encouragement that teams adopt the CubeSat standard for their payload(s) whenever possible – either as the payload structure itself, or as an adapter which the payload is mated to prior to the combined assembly's integration with the launch vehicle (such an adapter could be included in the official payload mass).

2.7.1.8 BONUSES FOR EFFICIENT LAUNCH PREPARATIONS

Teams whose preparedness, efficient operations, and hassle-free design permit their being launched in a timely manner will be awarded bonus points in addition to their total earned score according to the following tiered system. Launch readiness is declared when competition officials managing Launch Control receive the team's completed Flight Card. No bonus points will be awarded for launch attempts ending in catastrophic failures (CATO).

- 50 bonus points will be awarded to teams declared launch ready by the end of the designated field preparation day and flown by the end of the first launch day. They remain eligible to receive these points until the end of the first launch day, or until their first launch attempt ends in a scrub – at which point the team is no longer eligible for the 50 point bonus, but may still achieve bonus points awarded for teams declared launch ready on the first launch day.
- 25 bonus points will be awarded to teams declared launch ready and flown during the second launch day. They remain eligible to receive these points until the end of the second launch day, or until their first launch attempt ending in a scrub – at which point the team may attempt to regain eligibility by attempting a return to launch readiness by the end of the day. Otherwise, the team is no longer eligible for bonus points.
- 0 bonus points will be awarded to teams declared launch ready and flown during the third launch day.

$$Points = 350 - \left(\frac{350}{0.3 \times Apogee_{Target}} \right) \times |Apogee_{Target} - Apogee_{Actual}|$$

where $Apogee_{Target}$ may equal either 10,000 ft AGL or 30,000 ft AGL

ESRA – SDL Payload Challenge Scoring

Awards:

- 1st Place Payload Award: \$1000
- 2nd Place Payload Award: \$750
- 3rd Place Payload Award: \$500
- SDL Technology Relevance Award
- Honorable mentions as warranted (judges discretion)

Judging Criteria (1000 points possible):

- Scientific or Technical Objective(s) (400 points)
 - How relevant and well-designed is your scientific or technical objective?
- Payload Construction and Overall Professionalism (250 points)
 - Includes make/buy decisions, craftsmanship, material usage, poster, handouts, reports, etc.
- Readiness / Turnkey Operation (50 points)
 - Will the payload interfere with launch operations? Will the payload operate after hours of launch preparation, rail time, heat, waiting for other launches, etc?
- Execution of Objective(s) (300 points)
 - Judges should be informed of results by Saturday at noon or a zero in this category will be assessed.
 - How well did it accomplish the objective(s)?
 - Note that rocket failure results in 150 points (half credit – don't know if payload would have worked or not)

SDL Technology Relevance Award

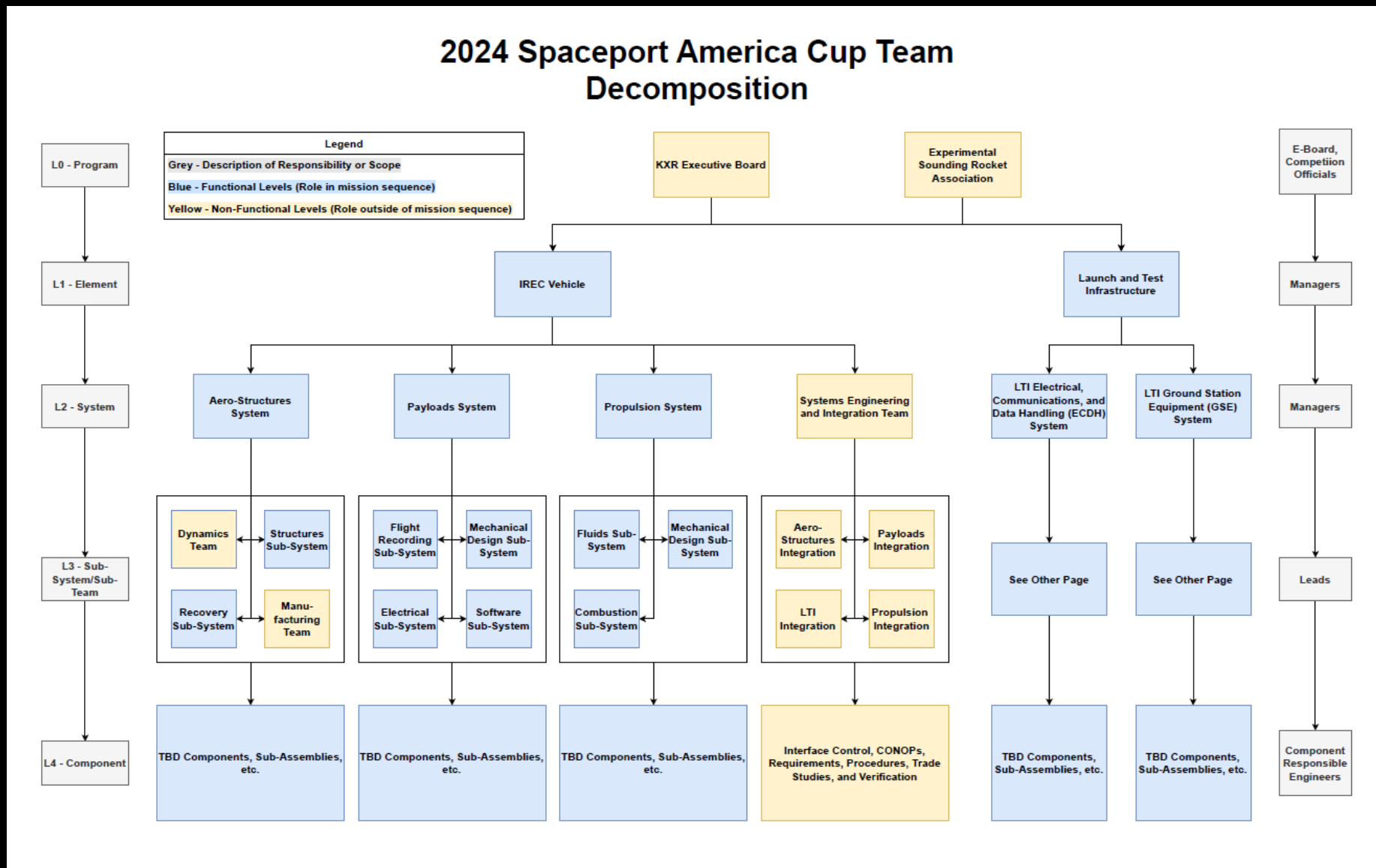
- This is a separate award and has no impact on the overall SDL payload challenge.
- This provides an opportunity for students to focus on and integrate technologies relevant to SDL's mission into their payload.
- 2024 technology areas: Robotics, Artificial Intelligence, and Infrared

IREC Team: The Mission

1. Launch a rocket carrying an Experimental Payload to 30,000 feet AGL.
2. Score points through successful flight, recovery, and payload deploy.
3. Meet industry professionals and student teams from across the world at Spaceport in Las Cruces, NM.



IREC Vehicle Team Decomposition



IREC Vehicle Architecture

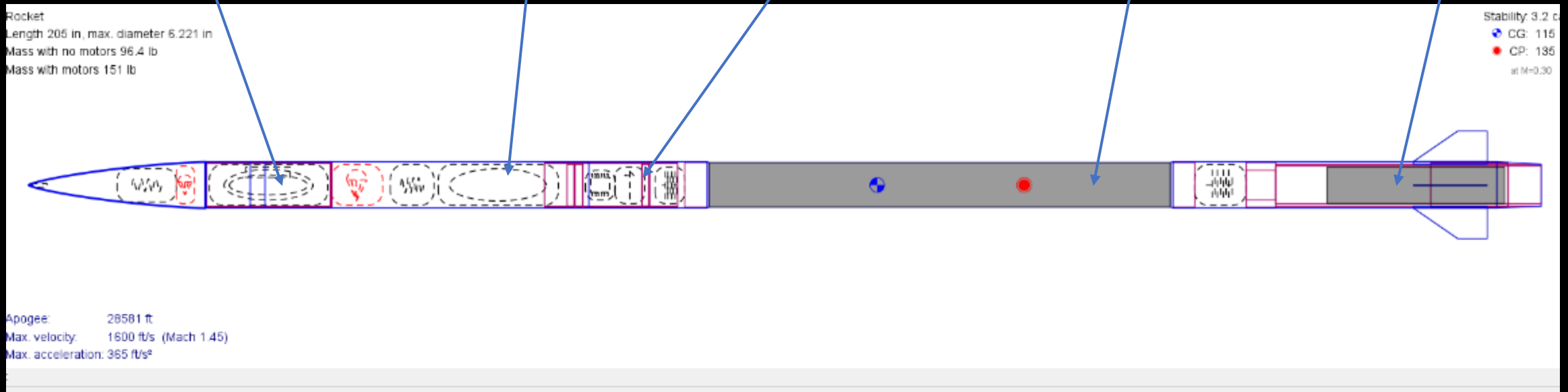
Recovery Coupler

Deployable Experiment

Avionics and Propulsion Control

Oxidizer Tank

Combustion Chamber

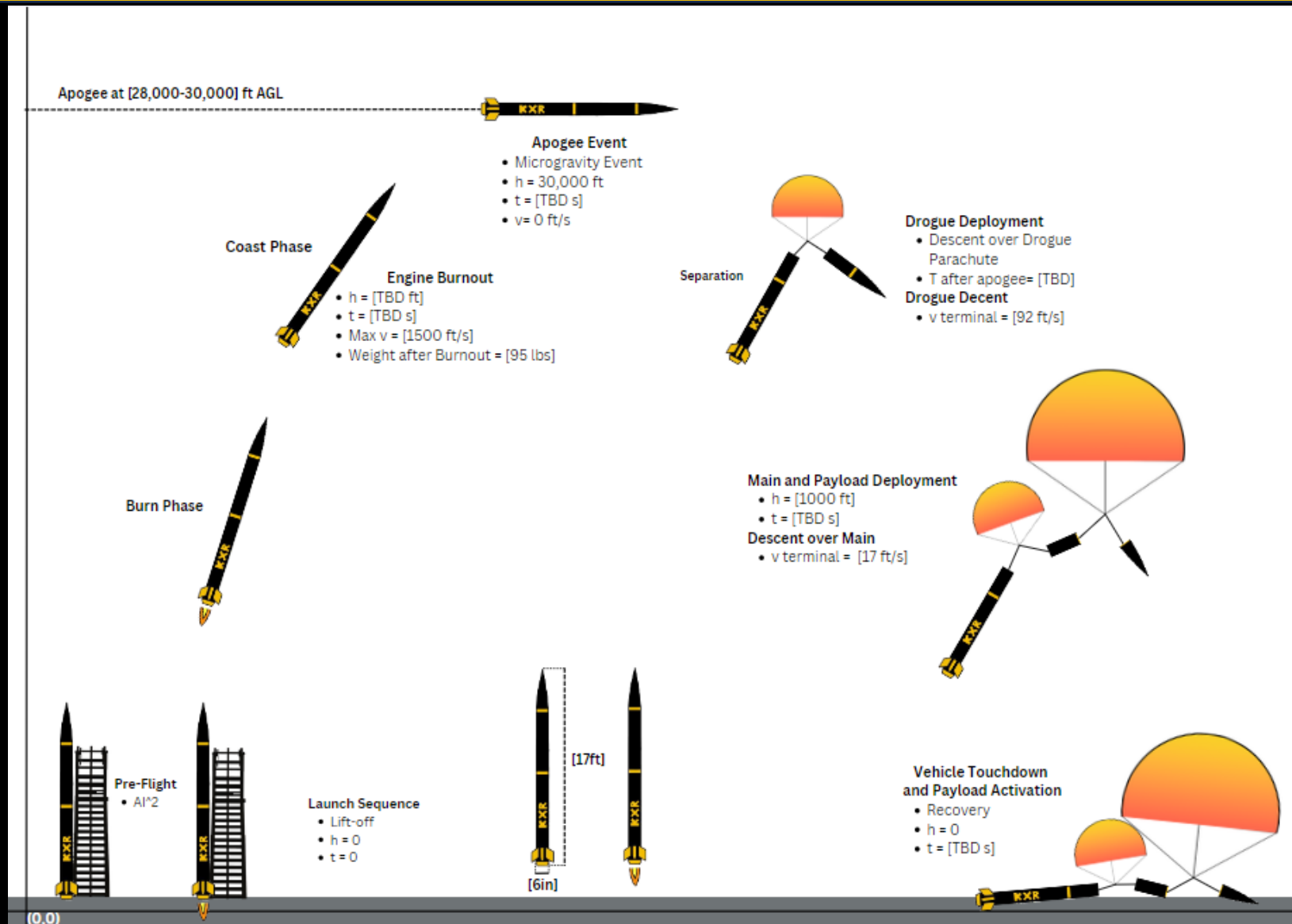


IREC Vehicle Requirements and TPMs

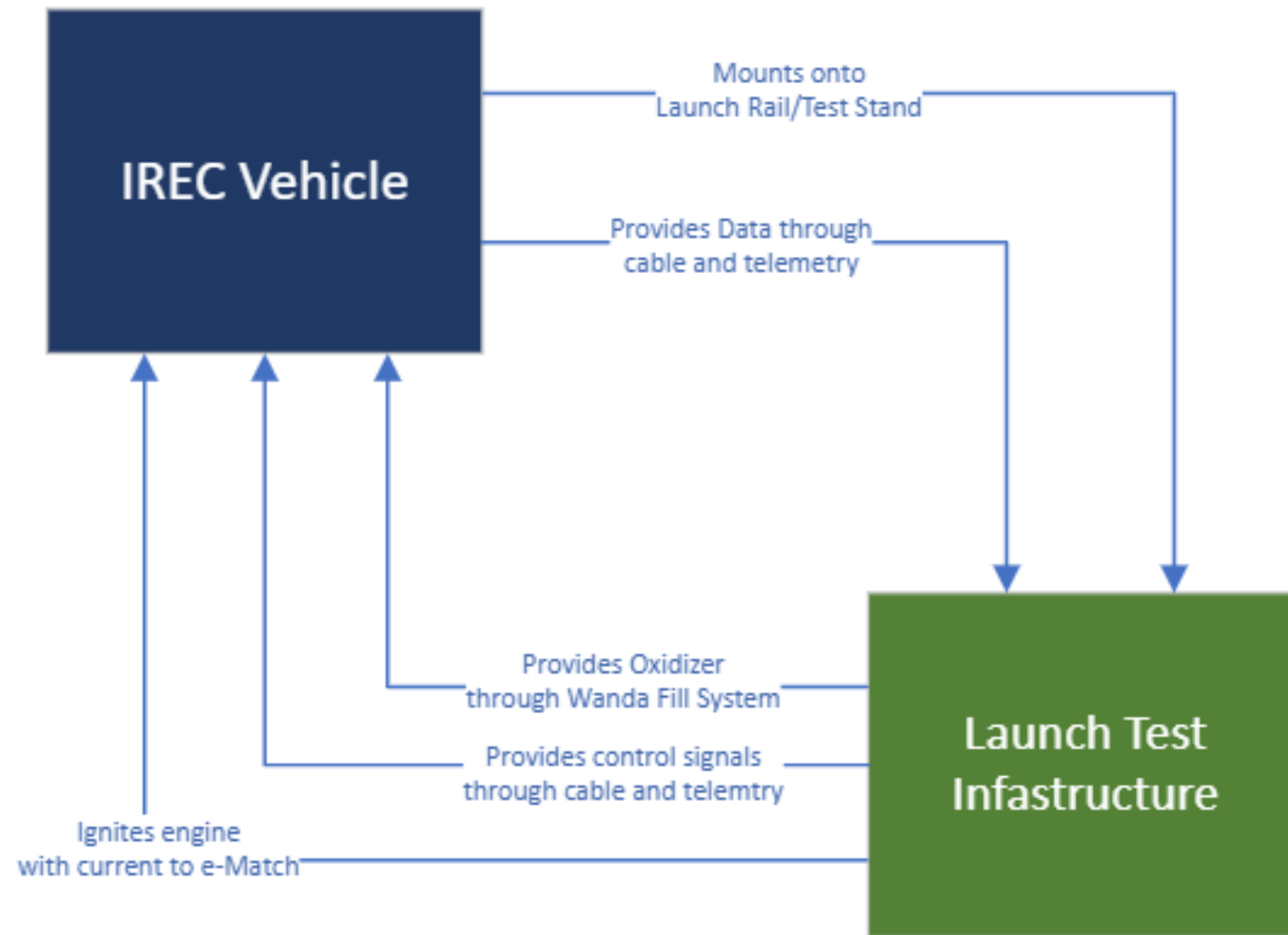
| Requirement | Verification Method |
|---|---------------------|
| The Vehicle shall launch to [30,000] feet. | Demonstration |
| The Vehicle shall have a weight of [150] lbs. | Inspection |
| The Vehicle shall be successfully recovered. | Demonstration |
| The Vehicle shall contain separate avionics, payload, aerostuctures, and propulsion systems. | Inspection |
| The Vehicle shall interface between external ground support equipment (GSE). | Inspection |
| The Vehicle shall withstand [120 degree farenheight] for [3 hours] at a time. | Demonstration |
| The Vehicle shall be reusable. | Demonstration |
| The Vehicle shall be prepared at the launch field within [2 hours]. | Demonstration |
| The Vehicle shall be launched using hybrid propulsion. | Demonstration |
| The Vehicle shall be transported using vehicles. | Demonstration |
| The Vehicle shall be verified through test launch by [April 11th, 2024]. | Demonstration |
| The Vehicle shall be designed by the [End of November 2023]. | Inspection |
| The Vehicle shall exceed a velocity off the rail of [120] ft/s. | Test |
| The Vehicle shall launch off a [30-foot] rail. | Demonstration |
| The Vehicle shall be designed, manufactured, and verified within a budget of [\$15,800]. | Demonstration |
| The Vehicle shall have a maximum height of 18 feet. | Inspection |

| Measure | TPM Value | Units | Verification |
|------------------|-----------------|-------|--------------|
| Length | [17] | ft | Inspection |
| Weight | [150] | lbs | Inspection |
| Maximum Speed | [1,500] | ft/s | Test |
| Apogee AGL | [28,000-30,000] | ft | Test |
| Engine Class | O-Class | N/A | Test |
| Thrust-to-Weight | [12.24:1] | N/A | Test |
| Outer Diameter | [6.22] | in | Inspection |

IREC Vehicle CONOPs



IREC Vehicle Interface Diagram



IREC Vehicle Cost

The Vehicle has a budget of \$15,800 that can break down into each System:

Propulsion: \$8,300

Aerostructures: \$5,500

Payloads: \$2,000

5-20% from each system will be used as a buffer for overhead or emergency costs.

IREC Vehicle Schedule – 9 Month Process

PI 1: August – Mid-December (5.5 months)

- System Requirements Reviews (Systems and Vehicle)
- Preliminary Design Reviews (Systems and Vehicle) (October 30th – November 9th)
- Design Solution Development Phase
- Critical Design Reviews (Systems and Vehicle) (November 27th)
- ERFs for long lead time items are created and approved.

PI 2: Mid-December – End-of-February (2.5 months including Winter Break)

- Finish all procurement.
- Finish initial and component-level simulation models for verification.
- Begin and Finish Manufacturing of all vehicle components necessary to proceed *to testing*.
- Book plane tickets and housing.

IREC Vehicle Schedule – Continued

PI 3: March – Mid-June (3.5 months)

- All assembly, testing, fill, launch, safety, recovery, etc. procedures shall be officially completed and released, pending changes after collecting testing data.
- Testing campaign, verification across entire vehicle
- Flight Readiness Review
- Technical Report submission, Poster creation, and paper deliverables are finalized.
- Itinerary is established and finalized for travel.

PI 4: Mid-June – End-of-July

- Attend competition and complete competition mission sequence.
- Post-competition debrief.
- Prepare documentation for future project cycle.

IREC Vehicle Risks

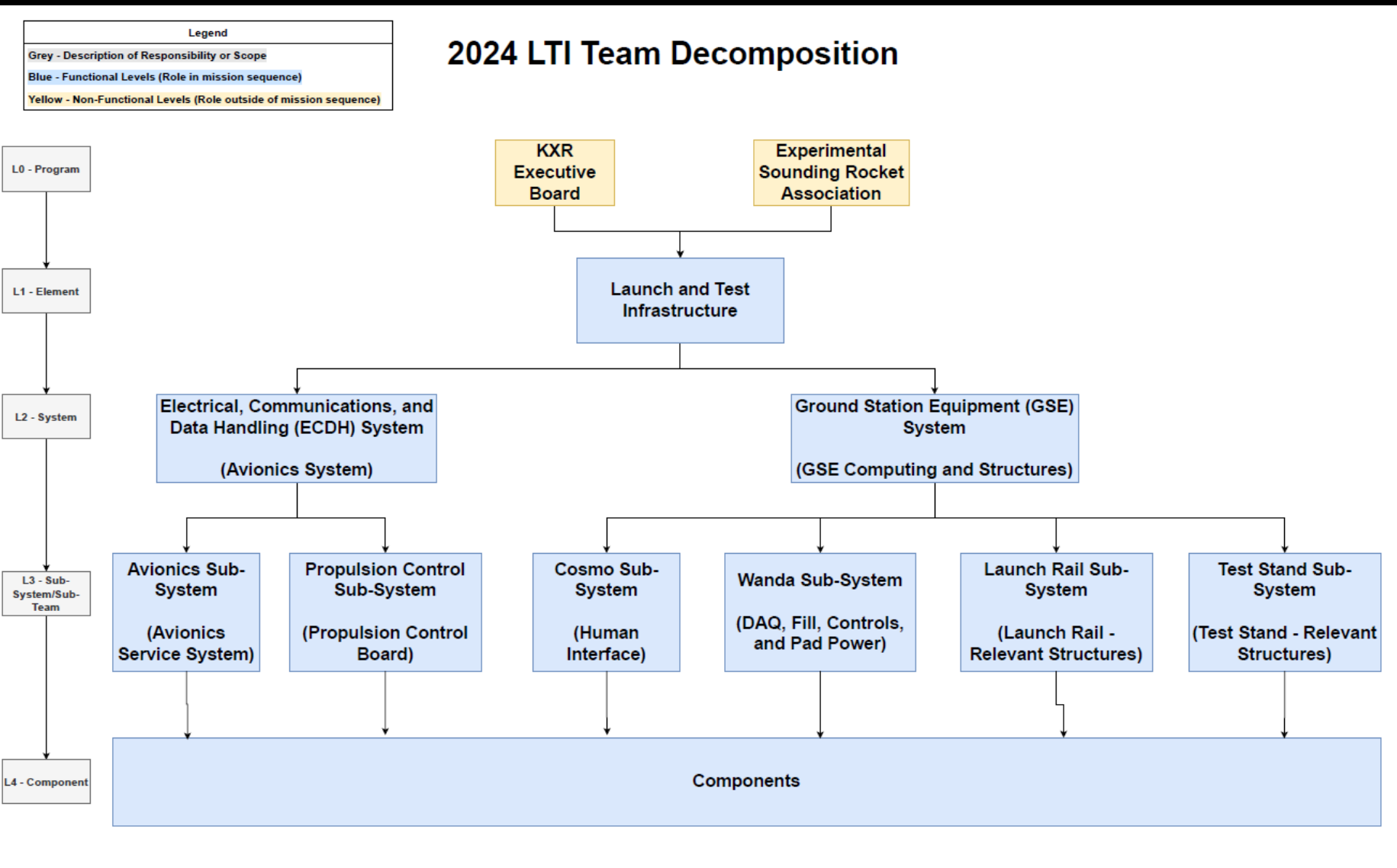
Possible Failure Modes:

- The vehicle does not reach desired altitude
- The vehicles stability fin system breaks
- The GSE does not ignite the propulsion system
- The vehicle loses connection to the ground station during fill
- The vehicle does not separate at main deployment
- The vehicle does not separate at drogue deployment

IREC Vehicle Verification Plans

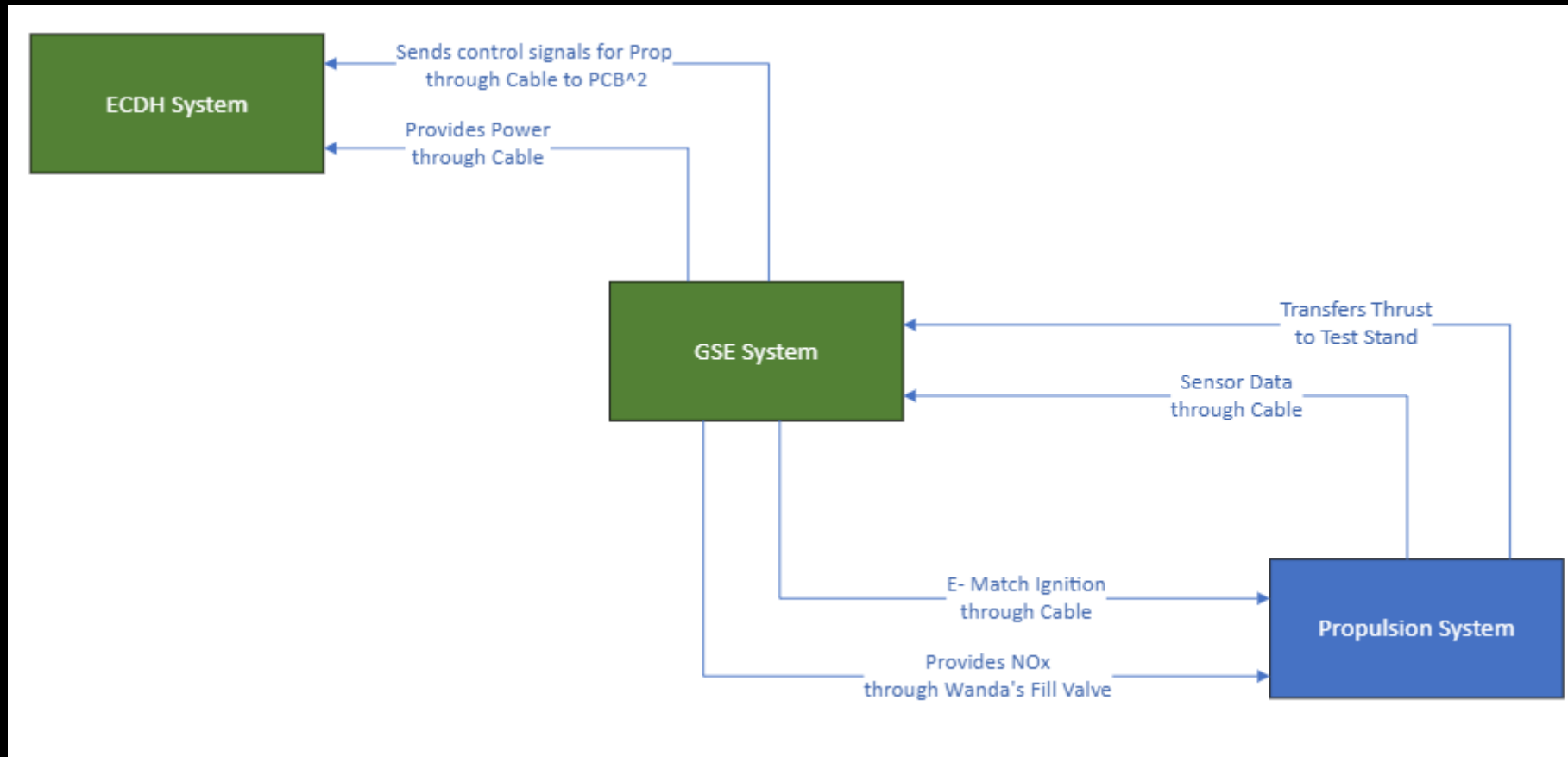
- Dry Fit Demonstration
 - Verify through demonstration that all interfacing components within the vehicle fit together and can be assembled with ease.
- Full Vehicle Verification Testing
 - Propulsion System Verification Testing
 - Payloads System Verification Testing
 - Aerostructures Verification Testing
- Test Launch
 - Verify and validate by demonstration that the vehicle and fulfills all functional requirements and mission sequence

IREC's Interpretation of LTI Team Decomp

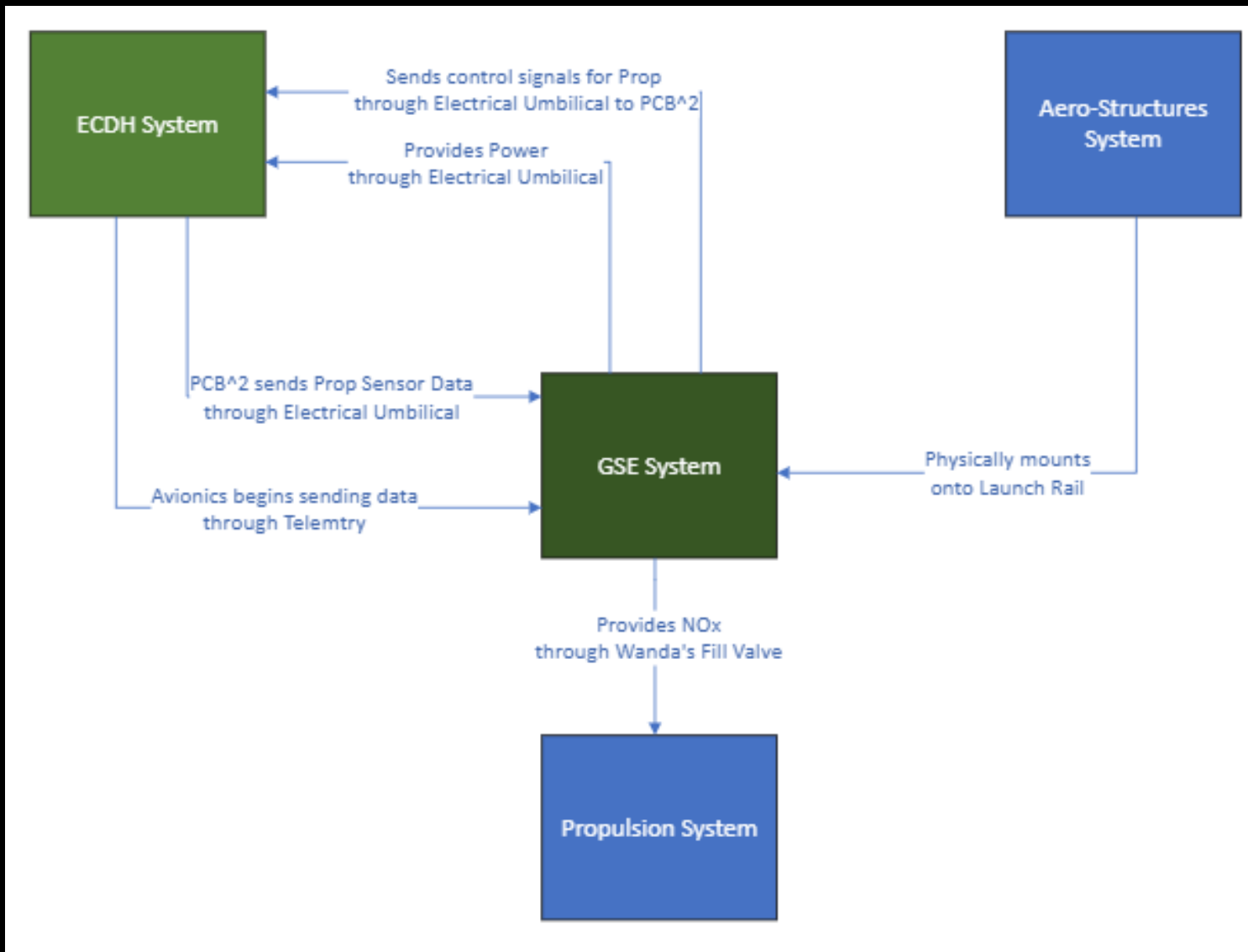


- Restructured within IREC strictly for requirements and interface writing
- **Please refer to LTI's own Team Breakdown if you are on their team!!**

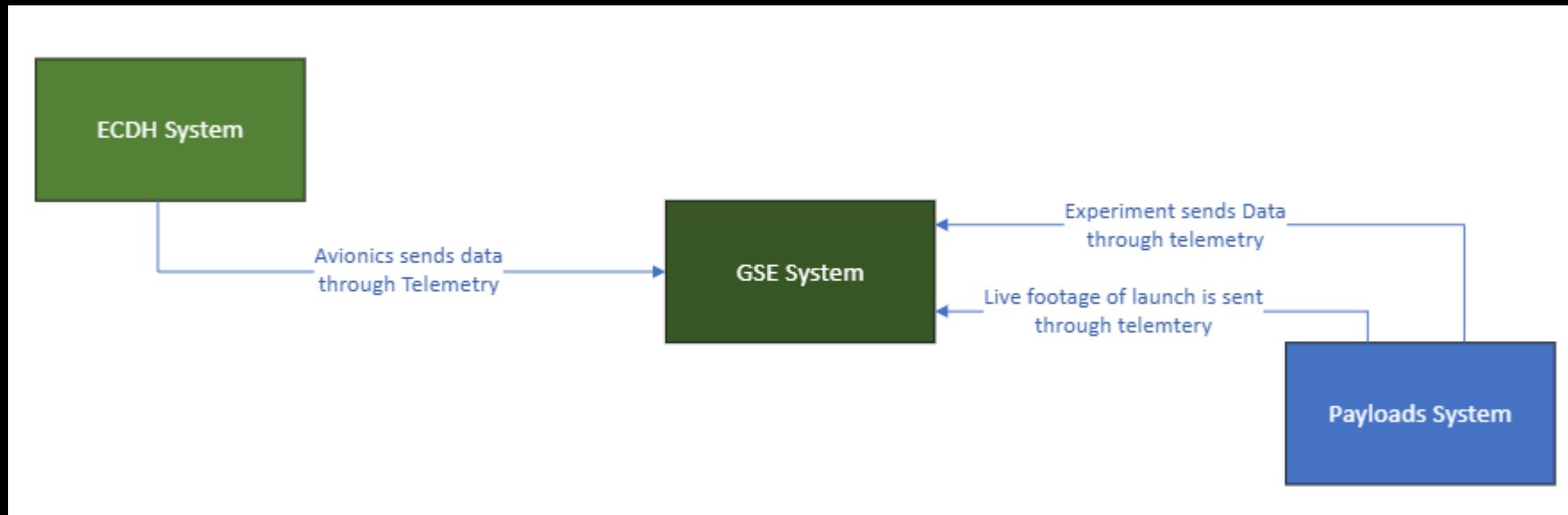
GSE – Test Configuration



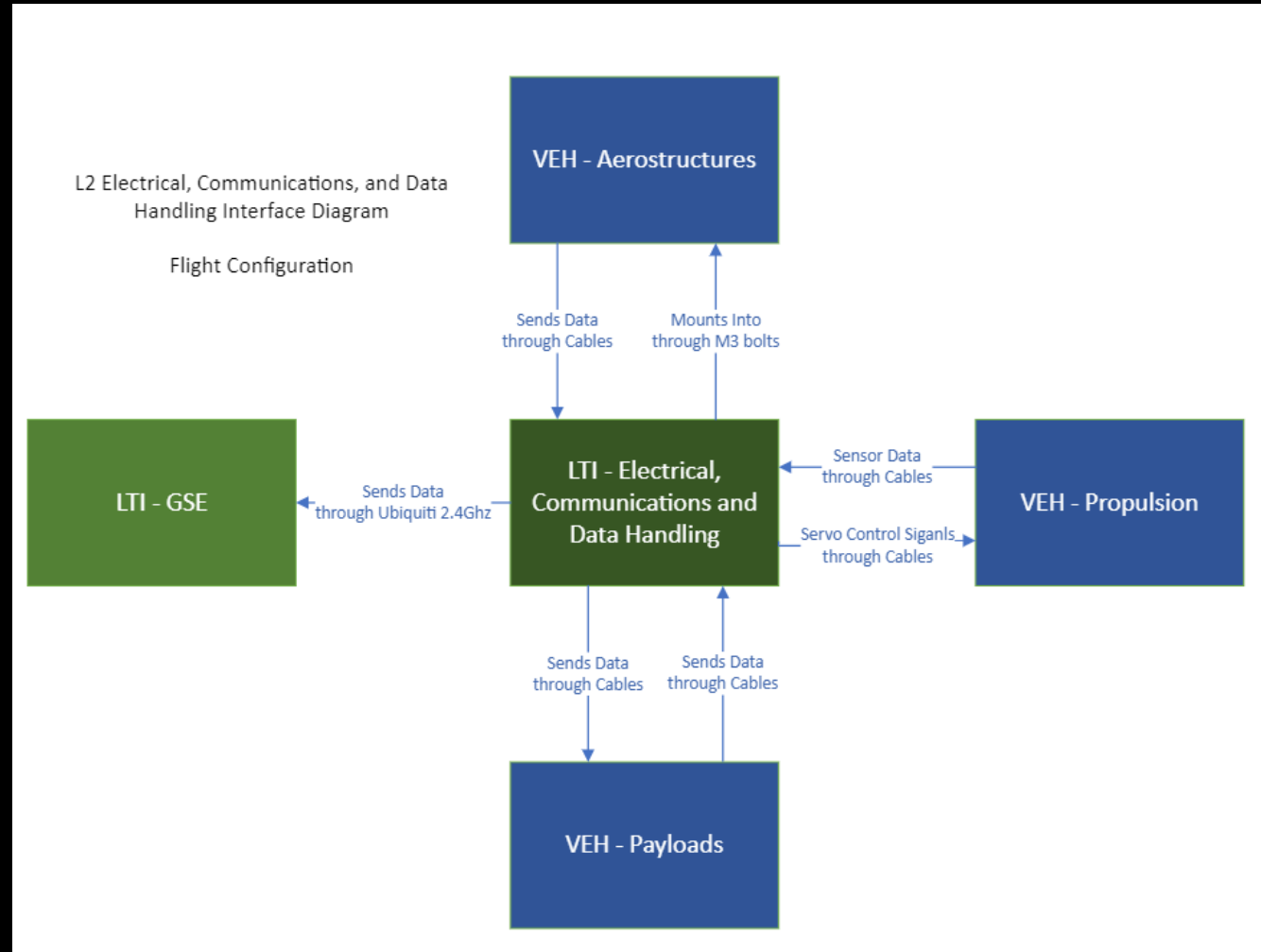
GSE – Launch Configuration



GSE – Flight Configuration



ECDH (Avionics and PCB²)– Flight Config



Questions?

A close-up photograph of a complex, dark-colored aerodynamic structure, likely a wing or tail section of a racing yacht. The structure features a prominent yellow, curved component and a textured, ribbed section. The background is blurred, showing more of the structure and some white markings.

Aero-Structures System Preliminary Design Review

Spaceport America Cup 2024

Project Helios

11/14/23

Agenda

1. General Aero-Structures Overview
2. Dynamics
3. Structures
4. Recovery
5. Manufacturing

Disclaimer: The question section is at the end of every sub-system section. Please hold questions until the designated question slide.

Aero-Structures Architecture

Target Apogee: 30,000ft

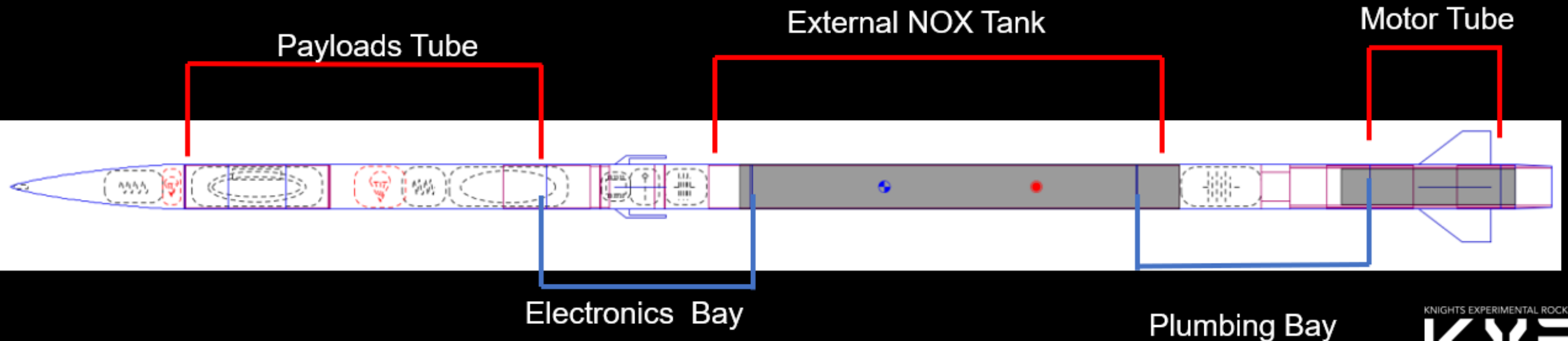
Max Speed :1700ft/s

Length :17 ft

OD: 6.22 inches

Max Dynamic Pressure: 15.5 PSI

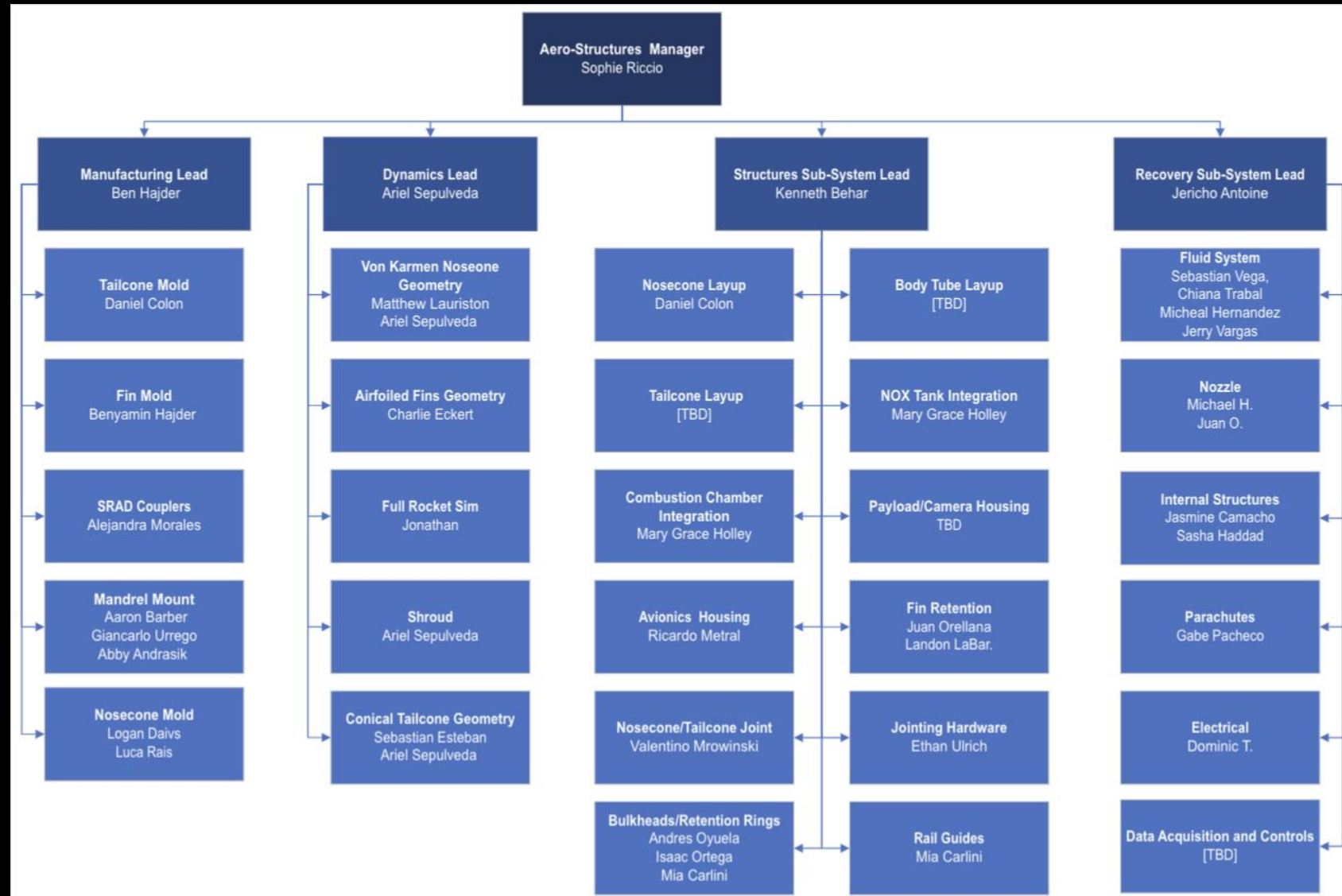
Dry Mass: 98 lbs



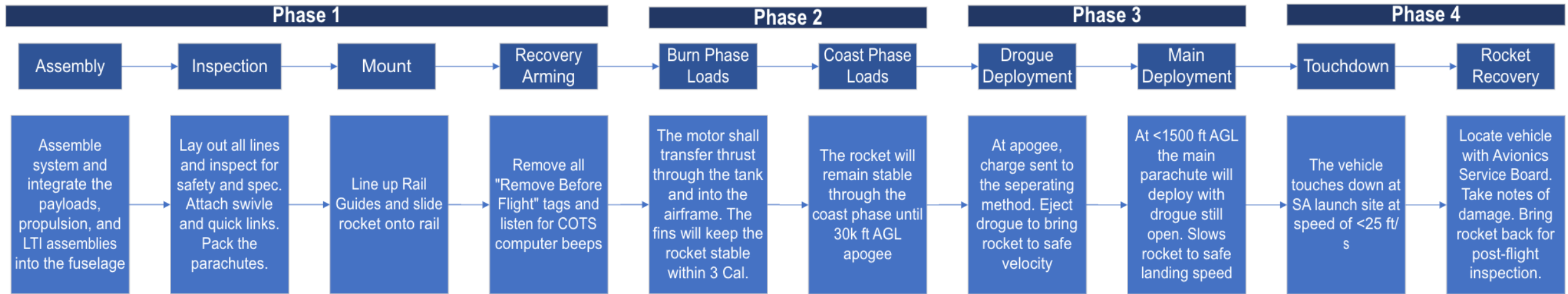
Aero-Structures Requirements

| Requirement | Verification Method |
|--|---------------------|
| The Aero-Structures System shall withstand a load of [12] G's. | Analysis |
| The Aero-Structures System shall have a Weight of [30] lbs. | Inspection |
| The Aero-Structures System shall deploy the Payload Experiment System during Main Deployment Event. | Inspection |
| The Aero-Structures System shall house the Avionics System. | Inspection |
| The Aero-Structures System shall house the Propulsion System | Inspection |
| The Aero-Structures System shall consist of Recovery and Structures Sub-Systems. | Inspection |
| The Aero-Structures System shall utilize non-functional Manufacturing and Dynamics Teams. | Inspection |
| The Aero-Structures System shall house all internal components. | Inspection |
| The Aero-Structures system shall have a minimum stability of [1.5] Calibers from launch and until apogee approach. | Analysis |
| The Aero-Structures System shall fully be designed by the [first week of December 2023]. | Inspection |
| The Aero-Structures System shall fully be procured by the [last week of January 2024]. | Inspection |
| The Aero-Structures System shall fully be manufactured by the [last week of February 2024]. | Inspection |
| The Aero-Structures System shall fully be tested by the [last week of March 2024]. | Inspection |
| The Aero-Structures System shall be designed to withstand their respective loads within a minimum Safety Factor of [2]. | Analysis |
| The Aero-Structures System shall be completely reusable. | Test |
| The Aero-Structures System shall successfully launch off the launch rail at an azimuth of [6] degrees. | Test |
| The Aero-Structures System shall be designed, manufactured, verified, and launched within a budget [\$5,500]. | Inspection |
| The Aero-Structures System shall have a total length of [17.083] ft. | Inspection |
| The Aero-Structures System shall be adequately vented throughout flight. | Inspection |

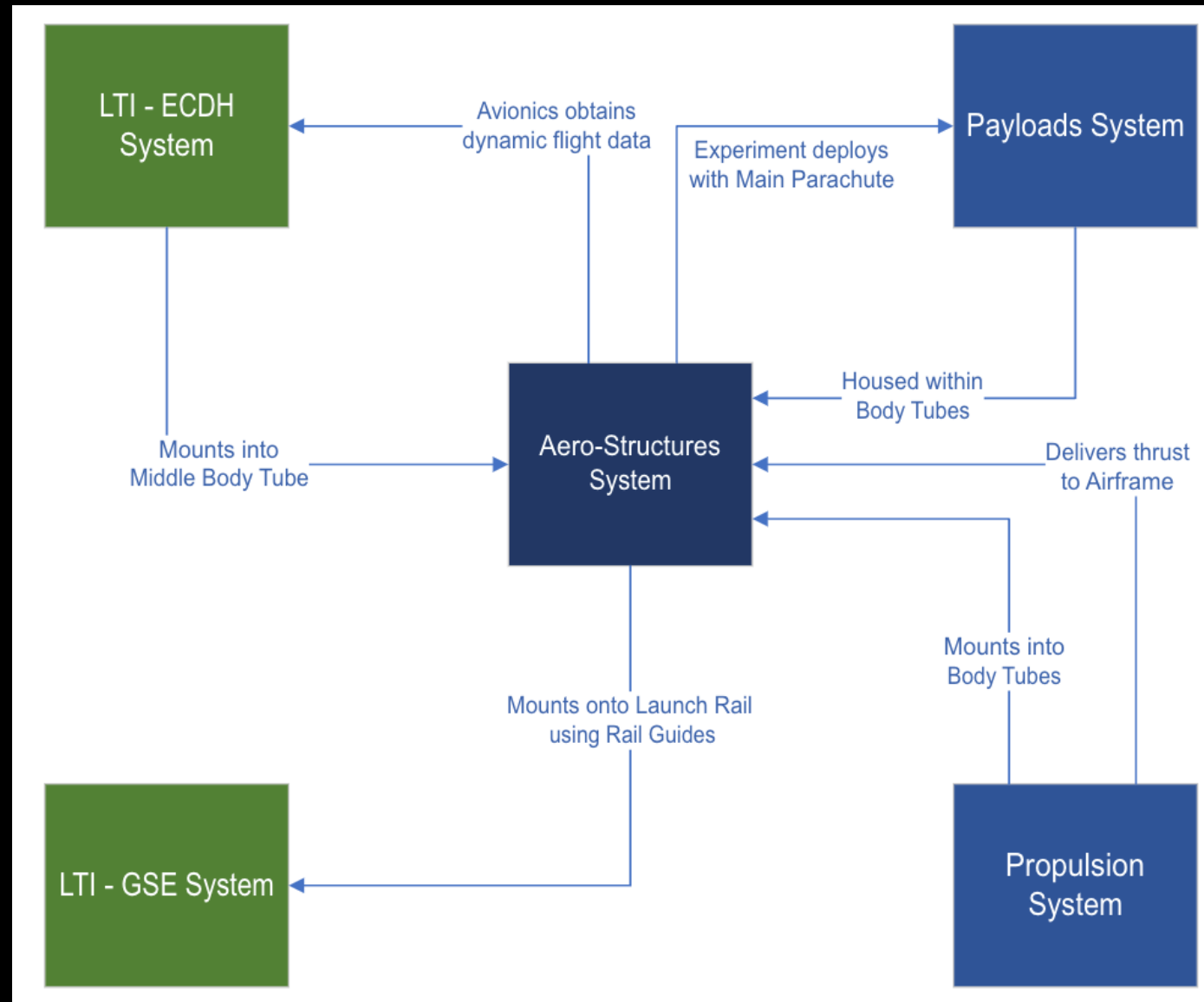
Aero-Structures Organization Chart



Aero-Structures CONOPS



Aero-Structures Interface Diagram



Aero-Structures Verification Plans

| Planned Tests | Details |
|--------------------------------|---|
| Carbon Fiber Composite Testing | Compressive, Tensile, Bending, and Bolt Tear Out |
| FEA | Bulkheads, Tubes, Nose Cone, Tail Cone, Fins, and Fin Brackets |
| Full Rocket Fluent Sim | Iteration of simulation with improvements to full rocket CAD assembly |
| Mold Testing | Nose Cone, Tail Cone, Fin, etc... |
| Manufacturing Test Article | 6-8 inch section of body tube |
| Parachute CD Testing | Done through UCF or neighbor universities. |
| Inspection of all Components | Damage, sizing, weight, etc... |
| Airframe Dry Fit | Full Dry Dress Assembly. |
| Recovery Ground Test | Rocket with ballast weight tested. |
| Launch | Full vehicle assembly, loading of propellants, and fire sequence. |

Aero-Structures Cost

Total Budget: \$5500

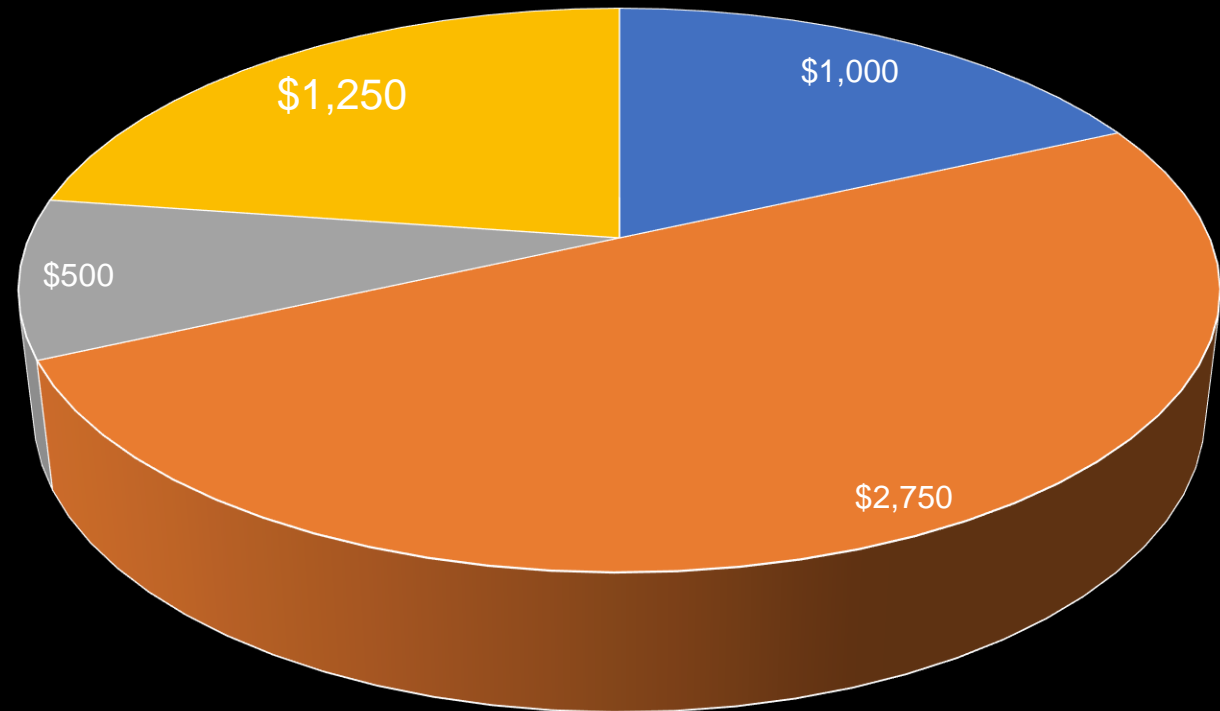
Structures: \$2750

Manufacturing: \$1000

Recovery: \$1250

\$500 Buffer

IREC Aero-Structures Budget



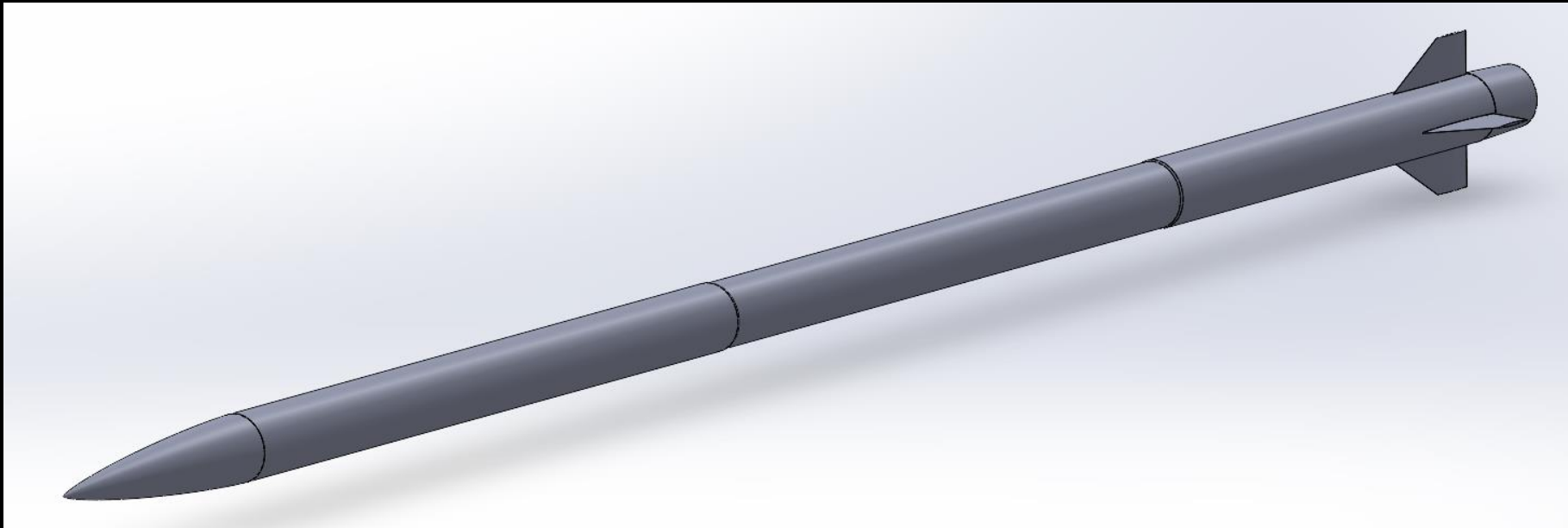
■ Recovery ■ Structures ■ Buffer ■ Manufacturing

Aero-Structures Risks

| Risks | Mitigation Strategies |
|--|--|
| Delay in materials arriving. | Order materials with long lead times earlier. |
| Integration Issues between Sub-Systems and Systems | Implement ICD to track all mission critical interfaces between Aero-Structures. |
| Manufacturing Setbacks | Create test articles and subscale models of full-scale components. |
| Rocket CATO from structural inadequacies | Perform rigorous hand calcs, simulations, and coupon testing, as well as designing with 2 times safety factor. |

Dynamics Team

The Dynamics Team is responsible for designing the geometries of Nosecone, Tailcone, Fins, and Antenna Shrouds. This includes determining the expected Aerodynamic loads that will be experienced by the Airframe.



Dynamics Requirements

| Requirement | Verification Method |
|--|---------------------|
| The Dynamics Team shall determine the maximum stresses acting on the rocket from aerodynamic loading to design around. | Analysis |
| The Dynamics Team shall verify drag coefficient for Aero-Structures System components through a variety of testing and CFD methods. | Analysis |
| The Dynamics Team shall verify sufficient stability throughout flight through aerodynamics analysis prior to testing. | Analysis |
| The Dynamics Team shall optimize the entire rocket to reduce Maximum Drag. | Analysis |

| Requirement | Verification Method |
|--|---------------------|
| The Nose Cone shall have a maximum Drag Coefficient of [] during flight. | Analysis |
| The Shroud shall minimize the drag contribution of the external antenna shrouds | Analysis |
| The Tail Cone shall have a maximum Drag Coefficient of [] during flight. | Analysis |
| The Tail Cone shall minimize drag experienced during flight. | Analysis |
| The Rail Guides shall induce a maximum Drag of [] lbs during flight. | Analysis |
| The Fins shall retain an airfoiled cross-sectional geometry. | Inspection |
| The Fins shall have a maximum Lift Coefficient of [] during flight. | Analysis |
| The Fins shall have a maximum Drag Coefficient of [] during flight. | Analysis |
| The Fins shall provide stability throughout flight. | Demonstration |

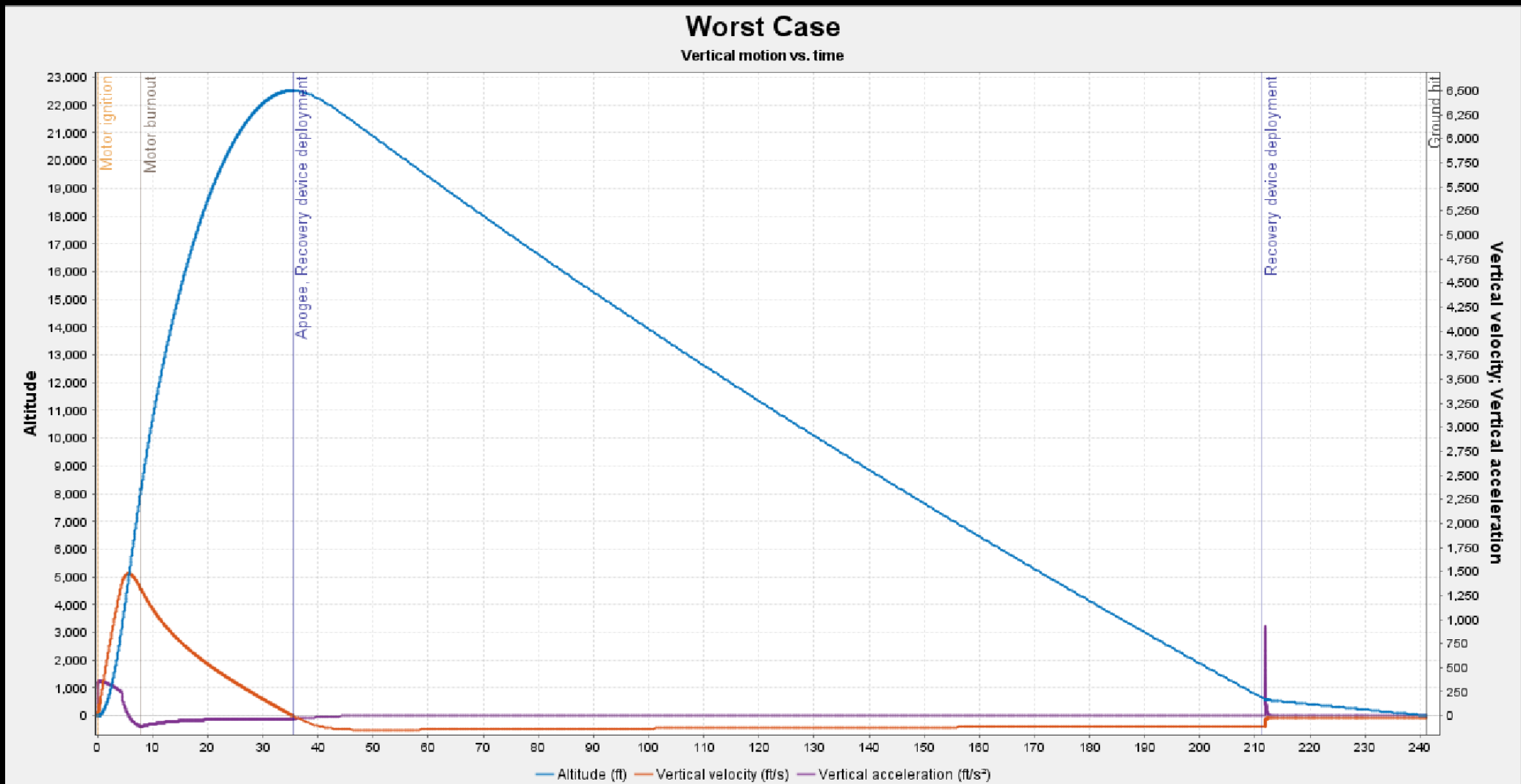
Dynamics TPMs

| Measure | TPM Value | Units | Verification Method |
|-----------------------------|-------------------------|-------------------|----------------------------|
| Maximum Coefficient of Drag | [0.7] | - | Analysis |
| Maximum Total Drag Force | [337] | lbf | Analysis, Hand Calculation |
| Maximum Acceleration | [370] | ft/s ² | Analysis |
| Maximum Moment | [7560] | lb*in | Analysis |
| Maximum Dynamic Pressure | [15.6] | psi | Analysis |
| Max Pitch Moment | [7560] | lb*in | Analysis |
| Max Pitch Rate | [0.081] | rad/s | Analysis |
| Max Yaw Rate | [-0.016] | rad/s | Analysis |
| Max Roll Rate | [1.8*10 ⁻⁷] | rad/s | Analysis |

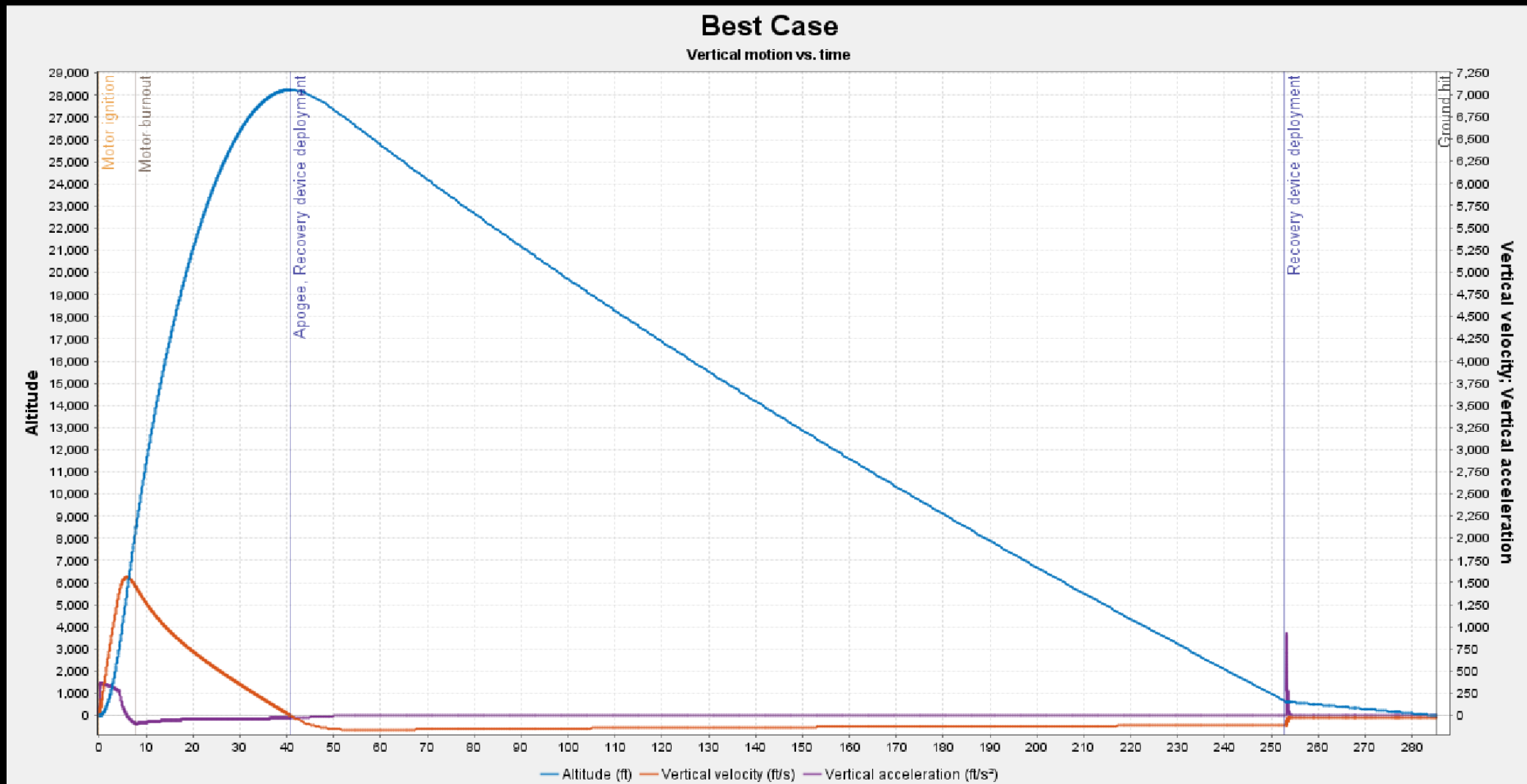
Dynamics Risks

| Risks | Mitigation Methods |
|---|--|
| Aerodynamic Loads are not calculated correctly, leading to failure during flight. | Ensure Aerodynamic Loads are calculated correctly through repeated verification. |
| Selected designs are not feasible. | Look for real-world examples from other team reports that have successfully launched for implementing new designs. |
| Selected designs are not able to integrate with other Sub-Systems. | Ensure constant communication between the other Sub-Systems. |

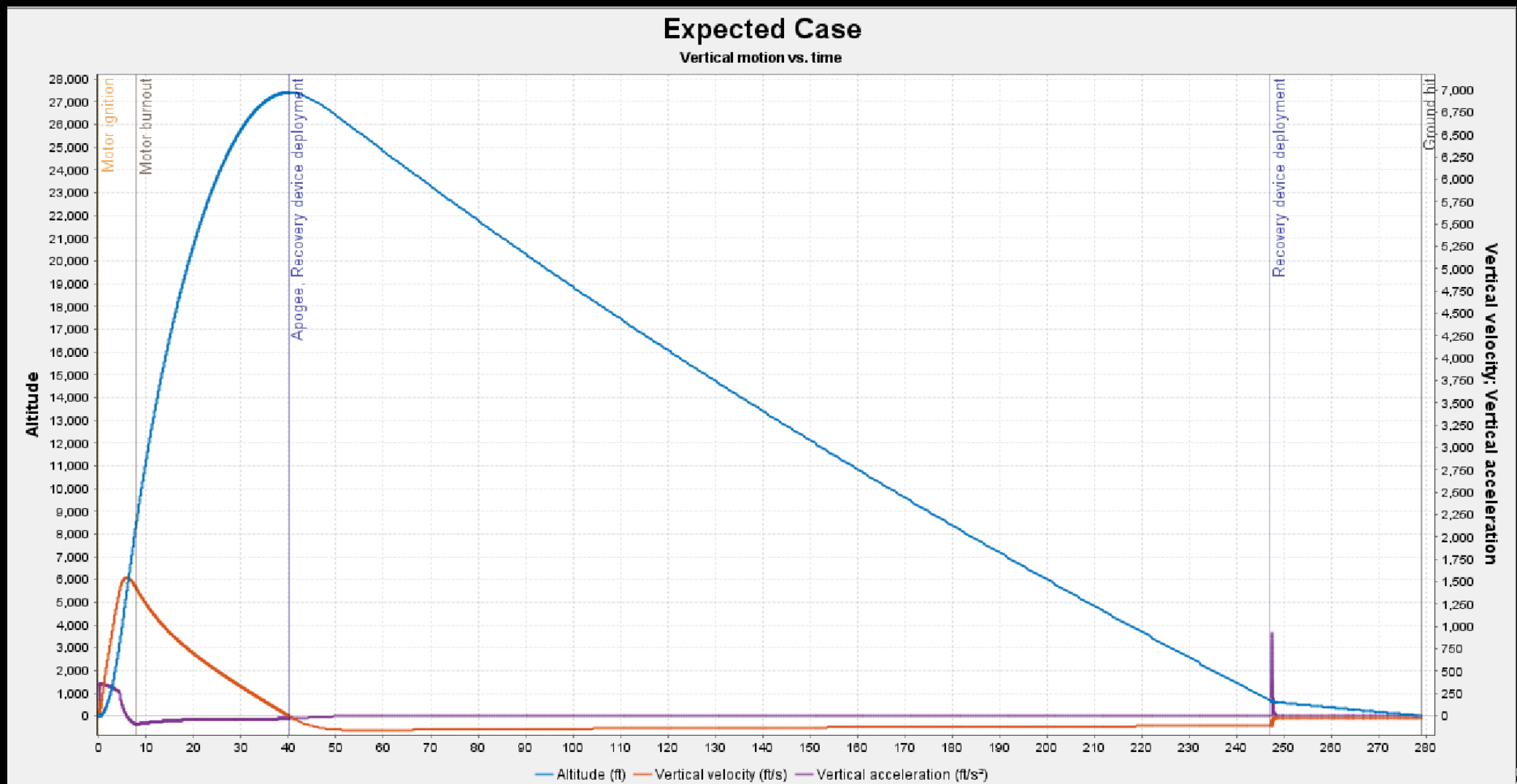
Worst Case - "Rough (19.7 mil)"



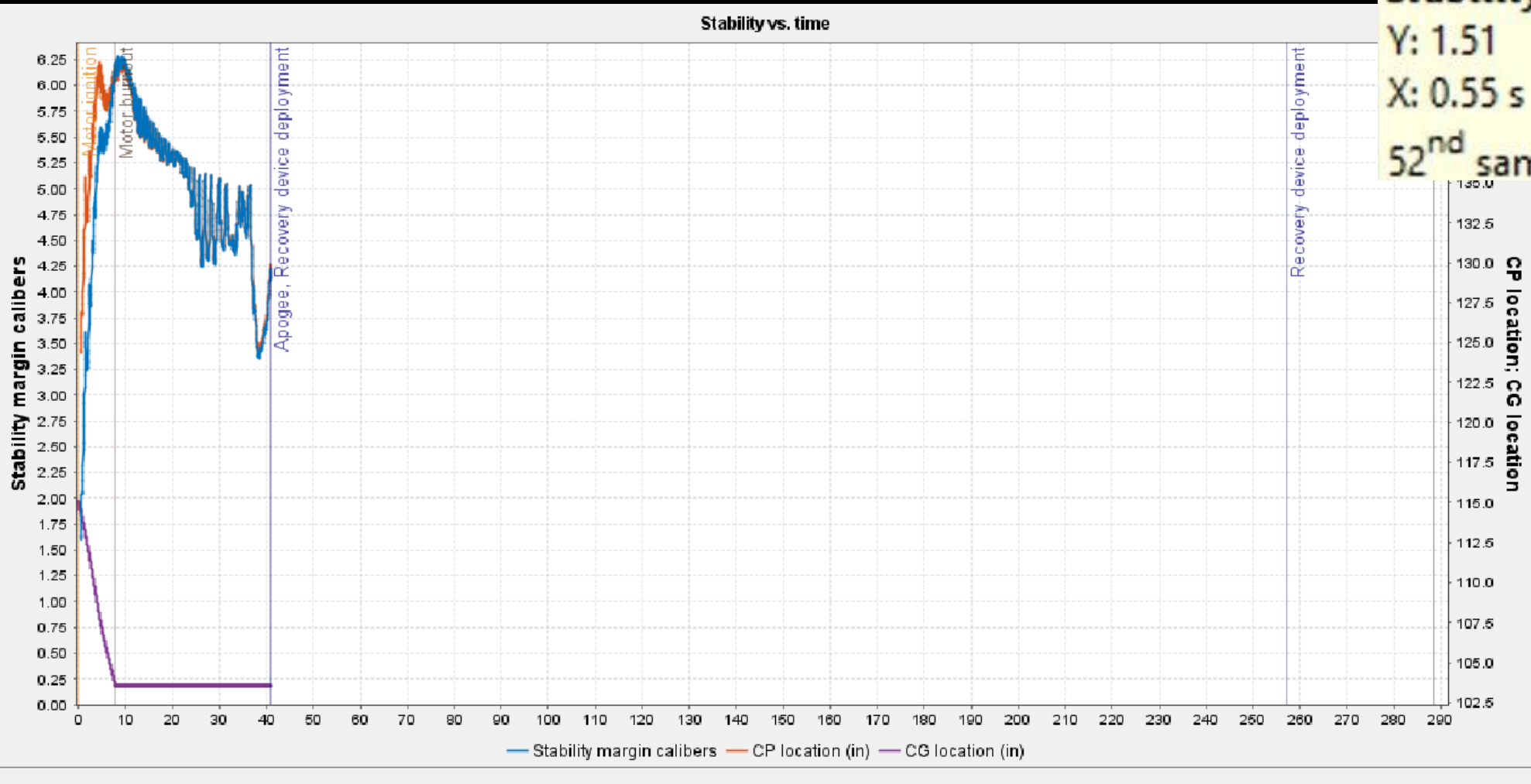
Best Case - "Aircraft Sheet-metal (0.079 mil)"



Expected Case - "Smooth Paint (0.787 mil)"



Stability Margin



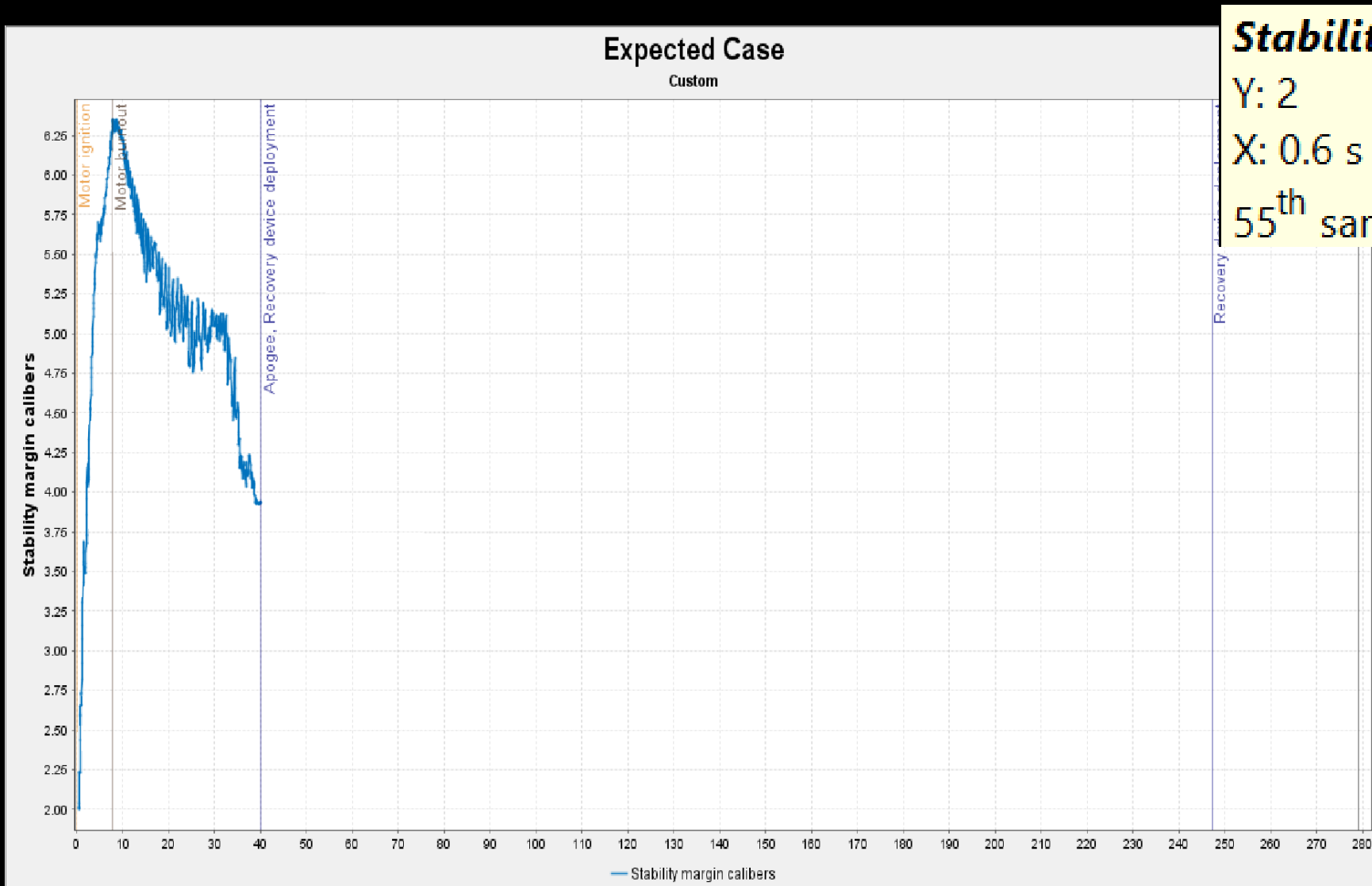
Stability margin calibers

Y: 1.51

X: 0.55 s

52nd sample

Stability Margin – Expected Case



Stability margin calibers

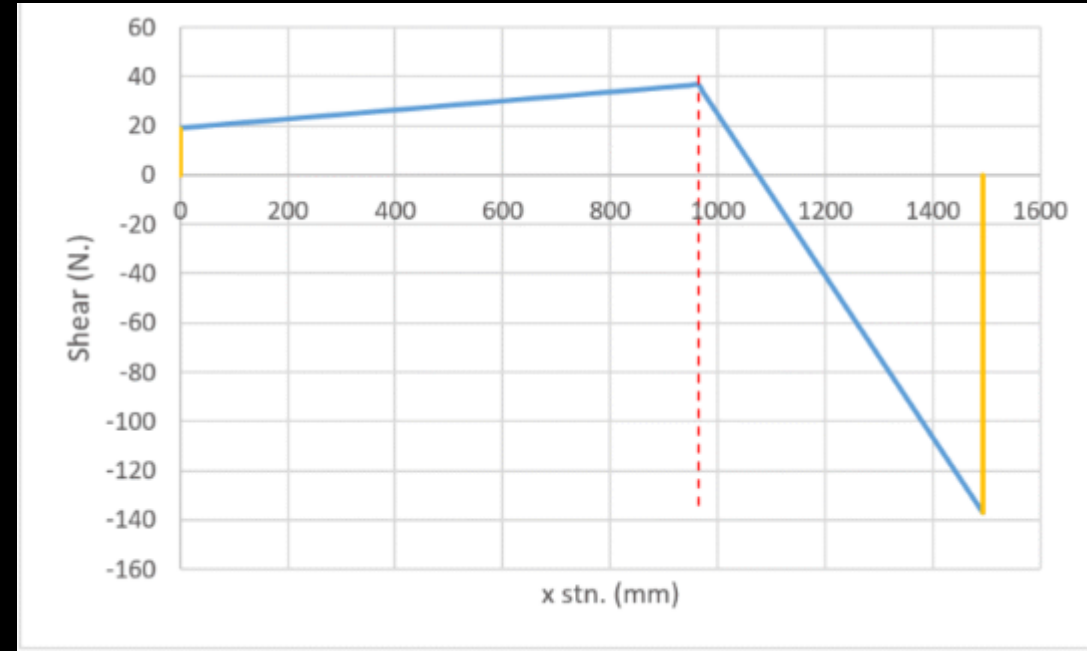
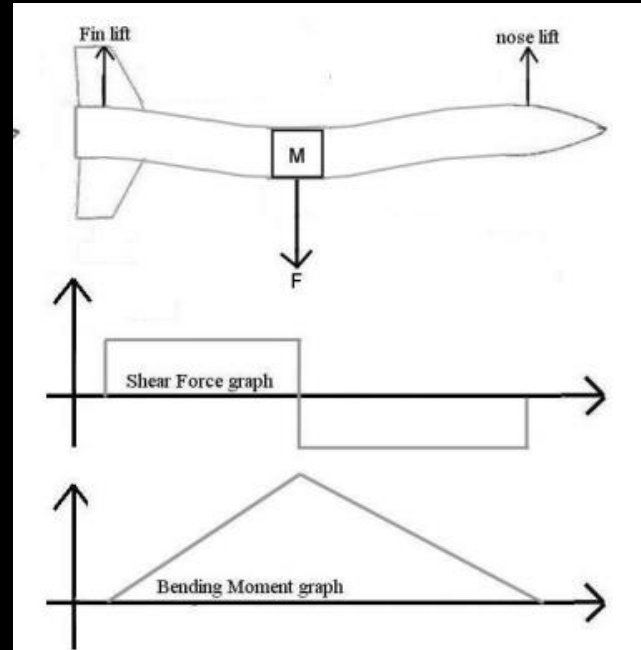
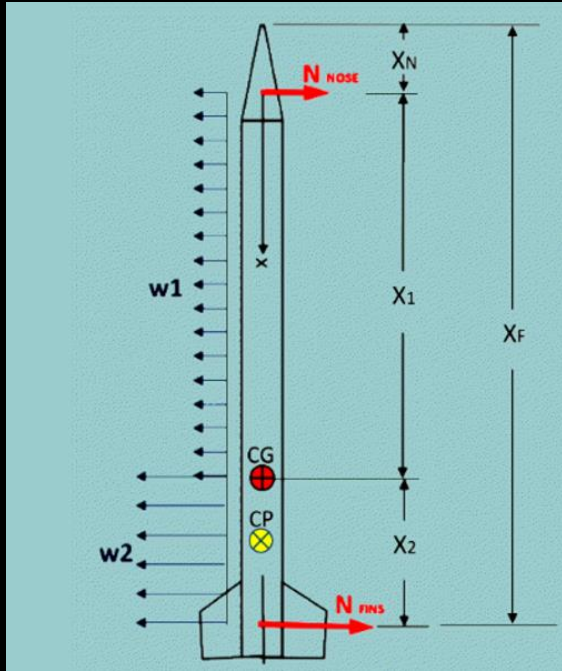
Y: 2

X: 0.6 s

55th sample

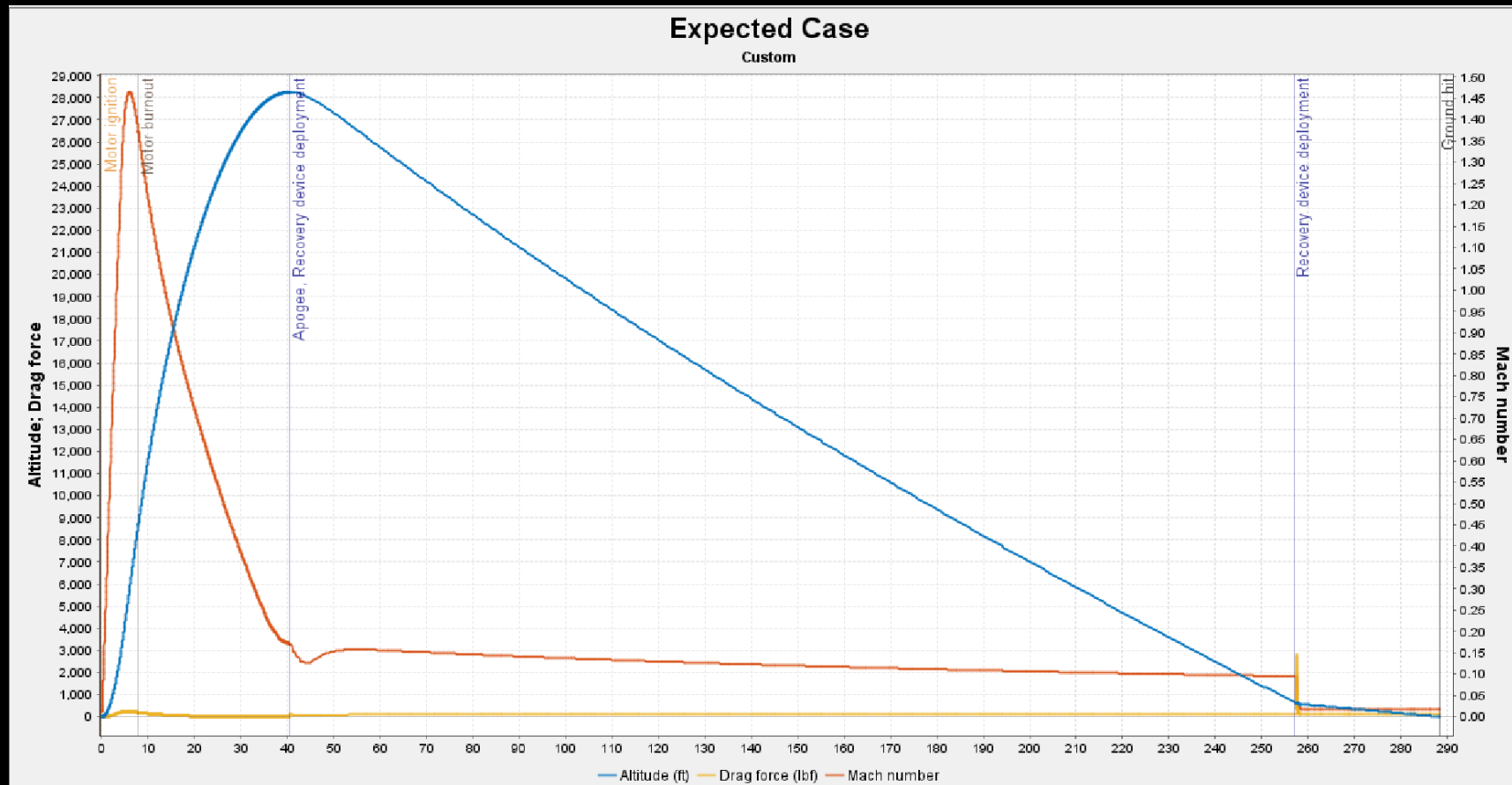
According to OpenRocket Flight Simulation, Stability stays above two for the duration of flight.

Aero Loads



| Max Q[Dynamic Pressure] (Psi) | Reference Area (in ²) | AoA (radians) | Normal Force on Nose Cone(lbf) | Normal Force on Fins(lbf) | Distributed load W1(lb/in) | Distributed load W2(lb/in) | Distance from NC for M1 (in) | Distance from NC from M2 (in) | Section Modulus(in ³) |
|-------------------------------|-----------------------------------|--------------------------|--------------------------------|---------------------------|----------------------------|----------------------------|--|-------------------------------|-----------------------------------|
| 15.56178729 | 30.39556944 | 0.104719755 | 99.06685421 | 243.7044614 | 0.468547479 | 3.745779337 | 211.4339712 | 113.9389167 | 2.896094428 |
| M1 Max Moment (lbin) | M2 Max Moment (lbin) | Lateral Shear (V1) (lbf) | Lateral Shear (V2) (lbf) | Max Stress on Body (psi) | Drag Force (lb) | Compressive Stress (psi) | Compressive Stress to Mass Inertia (psi) | Total Stress (psi) | Safety Factor |
| 7563.948025 | 7563.948025 | 52.21210629 | -243.7044614 | 2611.775346 | 337.4579929 | 11.10220993 | 599.433185 | 3222.310741 | 28.40160598 |

Full Rocket Sim



Apogee: 28250 ft
Max. velocity: 1612 ft/s (Mach 1.46)
Max. acceleration: 370 ft/s²

Stability: 3.21 cal
CG: 110 in
CP: 130 in
at M=0.30

Nose Cone

- Design Considerations: Ogive, *LD (Von-Karman) and *LV Haack Series, *X^{1/2} and X^{3/4} Power Series
- Honors Thesis by Chad O' Brien (supervised by principal Research Engineer Dr. David Lineberry)
- Gives insight on slenderness ratio and Haack Series drag comparisons

F. Axial Force from Pressure – Raw Data

Table 4: Axial Force from Dynamic Pressure

| Mach/File | LD-HAACK | | | | | | | | | LV-HAACK | | | | | | | | |
|-----------|----------|--------|--------|--------|--------|--------|--------|--------|--------|----------|--------|--------|--------|--------|--------|--------|--------|--------|
| | SF_0.5 | | | SF_1.0 | | | SF_2.0 | | | SF_0.5 | | | SF_1.0 | | | SF_2.0 | | |
| | F3 | F4 | F5 | F3 | F4 | F5 | F3 | F4 | F5 | F3 | F4 | F5 | F3 | F4 | F5 | F3 | F4 | F5 |
| 0.3 | 0.0025 | 0.0015 | 0.0011 | 0.0085 | 0.0055 | 0.0039 | 0.0340 | 0.0219 | 0.0156 | 0.0029 | 0.0018 | 0.0013 | 0.0102 | 0.0066 | 0.0048 | 0.0417 | 0.0273 | 0.0196 |
| 0.5 | 0.0074 | 0.0045 | 0.0031 | 0.0261 | 0.0166 | 0.0117 | 0.1056 | 0.0677 | 0.0479 | 0.0087 | 0.0054 | 0.0038 | 0.0313 | 0.0203 | 0.0144 | 0.1296 | 0.0845 | 0.0603 |
| 0.7 | 0.0176 | 0.0108 | 0.0074 | 0.0643 | 0.0406 | 0.0285 | 0.2667 | 0.1696 | 0.1192 | 0.0207 | 0.0130 | 0.0091 | 0.0771 | 0.0498 | 0.0353 | 0.3250 | 0.2108 | 0.1495 |
| 0.9 | 0.0470 | 0.0290 | 0.0199 | 0.1894 | 0.1184 | 0.0817 | 0.8197 | 0.5114 | 0.3515 | 0.0543 | 0.0343 | 0.0239 | 0.2191 | 0.1408 | 0.0987 | 0.9485 | 0.6093 | 0.4257 |
| 0.92 | 0.0534 | 0.0330 | 0.0226 | 0.2189 | 0.1366 | 0.0939 | 0.9529 | 0.5926 | 0.4055 | 0.0615 | 0.0389 | 0.0271 | 0.2513 | 0.1612 | 0.1126 | 1.0912 | 0.6991 | 0.4866 |
| 0.94 | 0.0613 | 0.0379 | 0.0259 | 0.2556 | 0.1592 | 0.1089 | 1.1185 | 0.6934 | 0.4719 | 0.0703 | 0.0445 | 0.0308 | 0.2910 | 0.1863 | 0.1296 | 1.2671 | 0.8093 | 0.5607 |
| 0.96 | 0.0710 | 0.0439 | 0.0299 | 0.3011 | 0.1871 | 0.1273 | 1.3230 | 0.8177 | 0.5532 | 0.0810 | 0.0512 | 0.0354 | 0.3399 | 0.2169 | 0.1502 | 1.4832 | 0.9439 | 0.6504 |
| 0.98 | 0.0828 | 0.0511 | 0.0347 | 0.3563 | 0.2209 | 0.1496 | 1.5685 | 0.9672 | 0.6506 | 0.0939 | 0.0593 | 0.0408 | 0.3991 | 0.2539 | 0.1748 | 1.7429 | 1.1052 | 0.7570 |
| 1 | 0.0965 | 0.0596 | 0.0403 | 0.4203 | 0.2603 | 0.1754 | 1.8484 | 1.1382 | 0.7620 | 0.1089 | 0.0687 | 0.0471 | 0.4677 | 0.2968 | 0.2033 | 2.0403 | 1.2901 | 0.8788 |
| 1.02 | 0.1114 | 0.0690 | 0.0466 | 0.4890 | 0.3030 | 0.2037 | 2.1431 | 1.3198 | 0.8814 | 0.1253 | 0.0790 | 0.0540 | 0.5419 | 0.3436 | 0.2345 | 2.3567 | 1.4879 | 1.0098 |
| 1.04 | 0.1264 | 0.0786 | 0.0531 | 0.5565 | 0.3456 | 0.2324 | 2.4255 | 1.4958 | 0.9994 | 0.1417 | 0.0896 | 0.0613 | 0.6156 | 0.3905 | 0.2663 | 2.6654 | 1.6822 | 1.1404 |
| 1.06 | 0.1403 | 0.0876 | 0.0594 | 0.6175 | 0.3845 | 0.2594 | 2.6735 | 1.6519 | 1.1069 | 0.1572 | 0.0997 | 0.0683 | 0.6836 | 0.4342 | 0.2965 | 2.9441 | 1.8584 | 1.2612 |
| 1.08 | 0.1528 | 0.0958 | 0.0652 | 0.6702 | 0.4182 | 0.2831 | 2.8814 | 1.7825 | 1.1990 | 0.1715 | 0.1090 | 0.0749 | 0.7441 | 0.4728 | 0.3235 | 3.1860 | 2.0101 | 1.3670 |
| 1.1 | 0.1641 | 0.1030 | 0.0704 | 0.7160 | 0.4468 | 0.3035 | 3.0571 | 1.8909 | 1.2760 | 0.1845 | 0.1174 | 0.0808 | 0.7980 | 0.5064 | 0.3471 | 3.3971 | 2.1397 | 1.4576 |
| 1.3 | 0.2526 | 0.1570 | 0.1073 | 1.0578 | 0.6512 | 0.4420 | 4.3526 | 2.6626 | 1.7996 | 0.2903 | 0.1818 | 0.1247 | 1.2174 | 0.7565 | 0.5154 | 5.0206 | 3.0999 | 2.1026 |
| 1.5 | 0.3343 | 0.2072 | 0.1415 | 1.3800 | 0.8486 | 0.5761 | 5.6179 | 3.4388 | 2.3277 | 0.3891 | 0.2425 | 0.1660 | 1.6127 | 0.9976 | 0.6787 | 6.5840 | 4.0539 | 2.7489 |
| 1.7 | 0.4208 | 0.2609 | 0.1784 | 1.7255 | 1.0627 | 0.7230 | 6.9900 | 4.2901 | 2.9115 | 0.4940 | 0.3075 | 0.2105 | 2.0350 | 1.2587 | 0.8571 | 8.2663 | 5.0945 | 3.4604 |

In Table 4, the listed values are directly collected from the FMSUMparse.sh shell routine showing in Figure 29. These were manipulated into non-dimensional calculations and plotted in the results section of this report. The values listed are in units of Newtons (N), the "F" is the slenderness ratio, and the "SF" is the radial scaling factor.

G. Axial Force from Viscous Effects – Raw Data

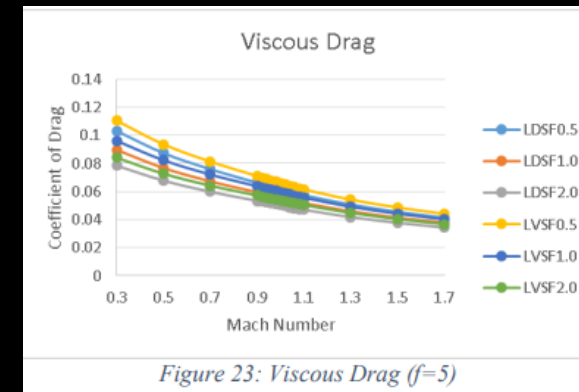
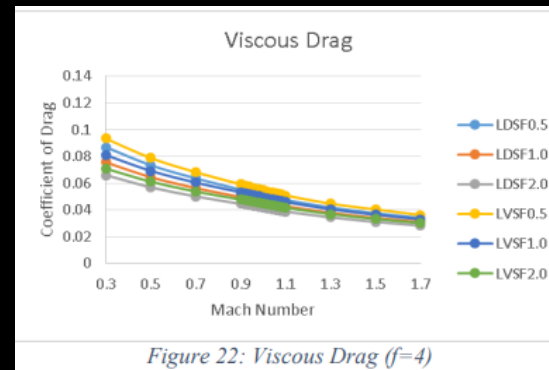
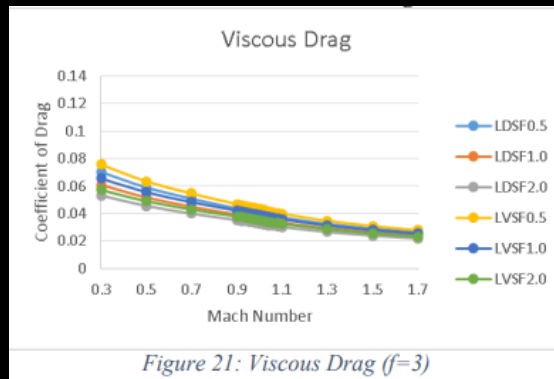
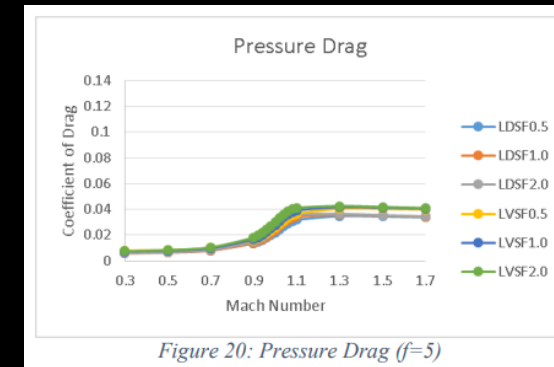
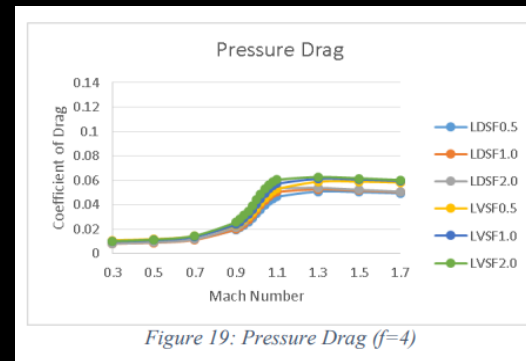
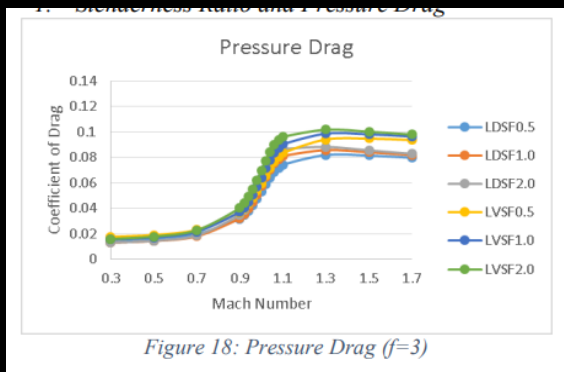
Table 5: Viscous Force

| Mach/File | LD-HAACK | | | | | | | | | LV-HAACK | | | | | | | | |
|-----------|----------|--------|--------|--------|--------|--------|--------|--------|--------|----------|--------|--------|--------|--------|--------|--------|--------|--------|
| | SF_0.5 | | | SF_1.0 | | | SF_2.0 | | | SF_0.5 | | | SF_1.0 | | | SF_2.0 | | |
| | F3 | F4 | F5 | F3 | F4 | F5 | F3 | F4 | F5 | F3 | F4 | F5 | F3 | F4 | F5 | F3 | F4 | F5 |
| 0.3 | 0.0115 | 0.0143 | 0.0170 | 0.0400 | 0.0497 | 0.0590 | 0.1398 | 0.1737 | 0.2064 | 0.0124 | 0.0153 | 0.0182 | 0.0430 | 0.0533 | 0.0632 | 0.1503 | 0.1864 | 0.2213 |
| 0.5 | 0.0269 | 0.0335 | 0.0399 | 0.0945 | 0.1178 | 0.1403 | 0.3332 | 0.4156 | 0.4954 | 0.0290 | 0.0360 | 0.0428 | 0.1015 | 0.1263 | 0.1504 | 0.3582 | 0.4462 | 0.5313 |
| 0.7 | 0.0455 | 0.0569 | 0.0680 | 0.1611 | 0.2017 | 0.2410 | 0.5729 | 0.7175 | 0.8575 | 0.0488 | 0.0610 | 0.0728 | 0.1730 | 0.2163 | 0.2583 | 0.6154 | 0.7700 | 0.9196 |
| 0.9 | 0.0645 | 0.0815 | 0.0980 | 0.2324 | 0.2935 | 0.3523 | 0.8361 | 1.0537 | 1.2637 | 0.0694 | 0.0875 | 0.1051 | 0.2496 | 0.3146 | 0.3775 | 0.8979 | 1.1304 | 1.3551 |
| 0.92 | 0.0664 | 0.0839 | 0.1010 | 0.2391 | 0.3024 | 0.3635 | 0.8618 | 1.0875 | 1.3052 | 0.0714 | 0.0901 | 0.1083 | 0.2570 | 0.3244 | 0.3895 | 0.9257 | 1.1668 | 1.3997 |
| 0.94 | 0.0682 | 0.0863 | 0.1039 | 0.2457 | 0.3112 | 0.3745 | 0.8868 | 1.1210 | 1.3465 | 0.0734 | 0.0926 | 0.1114 | 0.2643 | 0.3339 | 0.4014 | 0.9529 | 1.2029 | 1.4441 |
| 0.96 | 0.0700 | 0.0887 | 0.1067 | 0.2522 | 0.3198 | 0.3852 | 0.9108 | 1.1538 | 1.3875 | 0.0754 | 0.0952 | 0.1145 | 0.2713 | 0.3433 | 0.4131 | 0.9793 | 1.2384 | 1.4882 |
| 0.98 | 0.0718 | 0.0910 | 0.1096 | 0.2585 | 0.3283 | 0.3958 | 0.9338 | 1.1859 | 1.4279 | 0.0773 | 0.0977 | 0.1176 | 0.2781 | 0.3525 | 0.4246 | 1.0047 | 1.2731 | 1.5318 |
| 1 | 0.0735 | 0.0933 | 0.1125 | 0.2645 | 0.3365 | 0.4062 | 0.9560 | 1.2171 | 1.4677 | 0.0791 | 0.1002 | 0.1207 | 0.2847 | 0.3614 | 0.4359 | 1.0290 | 1.3070 | 1.5747 |
| 1.02 | 0.0752 | 0.0956 | 0.1153 | 0.2705 | 0.3447 | 0.4165 | 0.9777 | 1.2476 | 1.5069 | 0.0809 | 0.1026 | 0.1237 | 0.2911 | 0.3702 | 0.4470 | 1.0527 | 1.3402 | 1.6170 |
| 1.04 | 0.0768 | 0.0979 | 0.1182 | 0.2764 | 0.3528 | 0.4268 | 0.9993 | 1.2779 | 1.5457 | 0.0827 | 0.1050 | 0.1267 | 0.2974 | 0.3789 | 0.4580 | 1.0761 | 1.3730 | 1.6590 |
| 1.06 | 0.0785 | 0.1001 | 0.1210 | 0.2824 | 0.3611 | 0.4371 | 1.0214 | 1.3083 | 1.5844 | 0.0845 | 0.1074 | 0.1297 | 0.3037 | 0.3876 | 0.4690 | 1.0997 | 1.4059 | 1.7008 |
| 1.08 | 0.0802 | 0.1024 | 0.1238 | 0.2885 | 0.3694 | 0.4475 | 1.0443 | 1.3392 | 1.6232 | 0.0862 | 0.1098 | 0.1328 | 0.3102 | 0.3965 | 0.4801 | 1.1239 | 1.4390 | 1.7426 |
| 1.1 | 0.0820 | 0.1047 | 0.1267 | 0.2948 | 0.3779 | 0.4580 | 1.0678 | 1.3706 | 1.6621 | 0.0881 | 0.1123 | 0.1358 | 0.3168 | 0.4055 | 0.4913 | 1.1487 | 1.4724 | 1.7847 |
| 1.3 | 0.1003 | 0.1290 | 0.1566 | 0.3624 | 0.4670 | 0.5675 | 1.3186 | 1.6998 | 2.0660 | 0.1072 | 0.1379 | 0.1675 | 0.3873 | 0.5000 | 0.6078 | 1.4113 | 1.8211 | 2.2147 |
| 1.5 | 0.1194 | 0.1541 | 0.1872 | 0.4336 | 0.5593 | 0.6804 | 1.5813 | 2.0404 | 2.4818 | 0.1272 | 0.1657 | 0.2001 | 0.4615 | 0.5986 | 0.7273 | 1.6852 | 2.1809 | 2.6571 |
| 1.7 | 0.1394 | 0.1796 | 0.2185 | 0.5065 | 0.6556 | 0.7985 | 1.8504 | 2.3872 | 2.9033 | 0.1476 | 0.1911 | 0.2329 | 0.5393 | 0.6960 | 0.8505 | 1.9651 | 2.5463 | 3.1031 |

In Table 5, the listed values are directly collected from the FMSUMparse.sh shell routine showing in Figure 29. These were manipulated into non-dimensional calculations and plotted in the results section of this report. The values listed are in units of Newton (N), the "F" is the slenderness ratio, and the "SF" is the radial scaling factor.

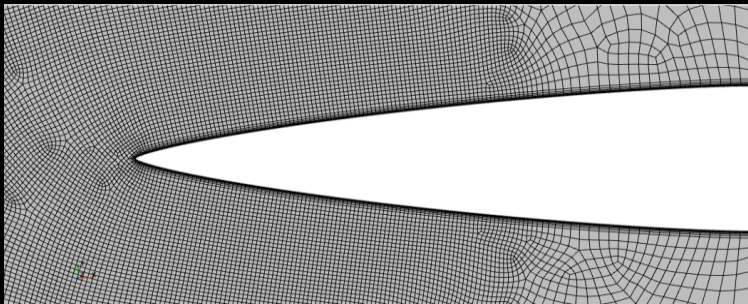
Nose Cone

- Pressure drag decreases with an increase in slenderness ratio, however it results in a significant increase in viscous drag
- Take this into account when considering best optimization for nosecone

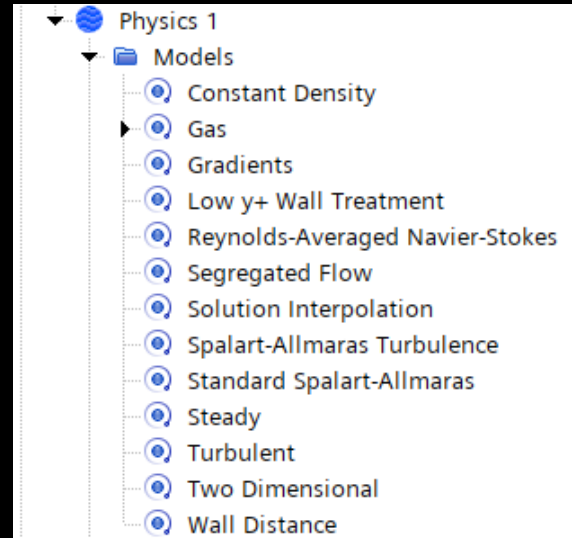
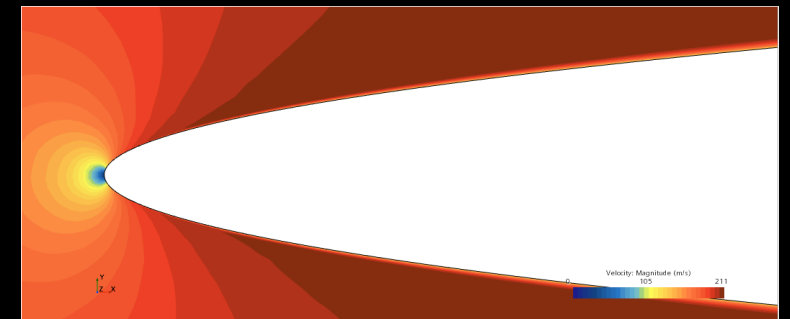
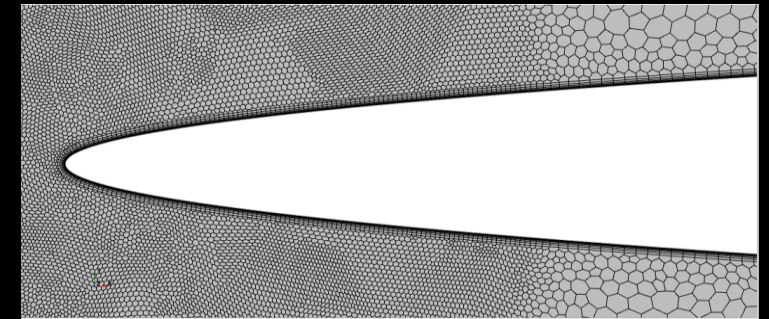


Nose Cone

Von-Karman



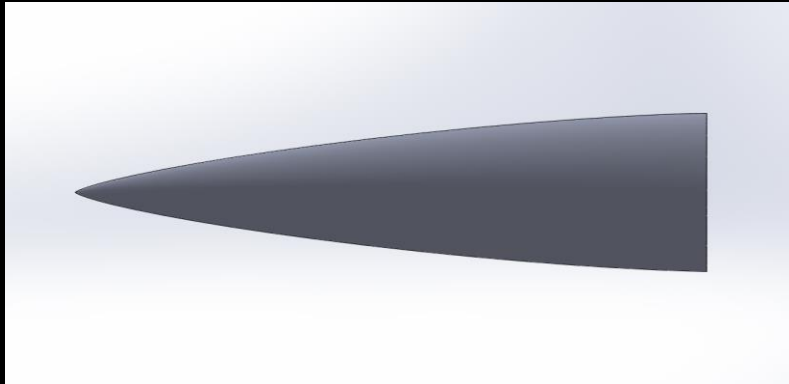
$X^{1/2}$ Power Series



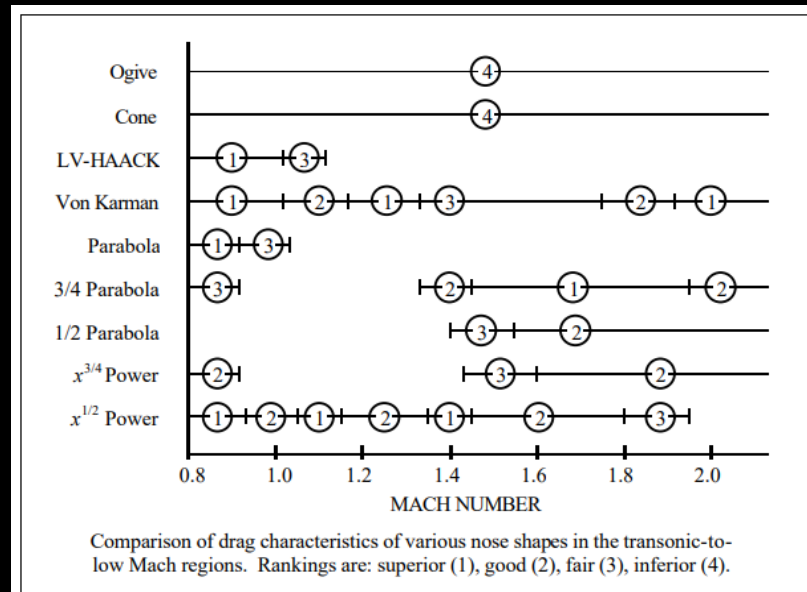
- Simulation done at Mach .6
- Von-Karman minimizes flow separation at the tip
- Velocity increases at a faster rate along the surface of the $X^{1/2}$ Power Series nosecone

Nose Cone

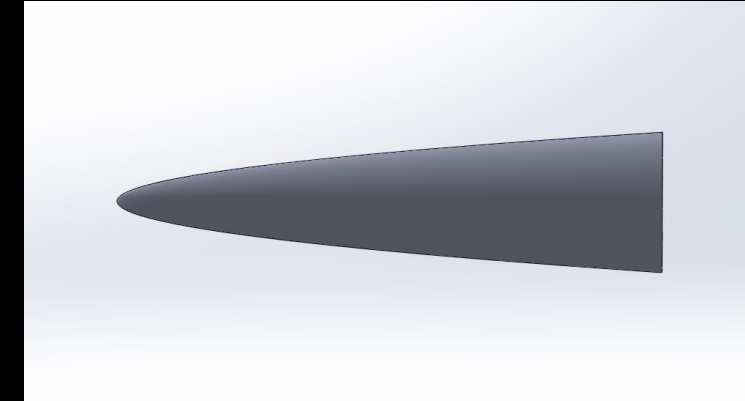
Von-Karman



Chinn, S.S.; Missile Configuration Design



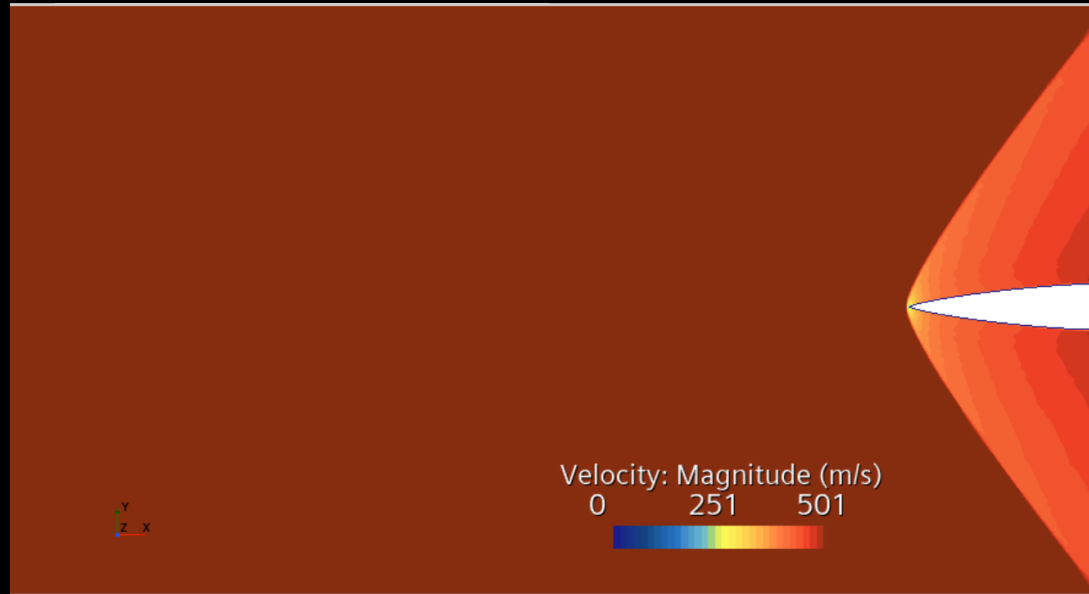
$X^{1/2}$ Power series



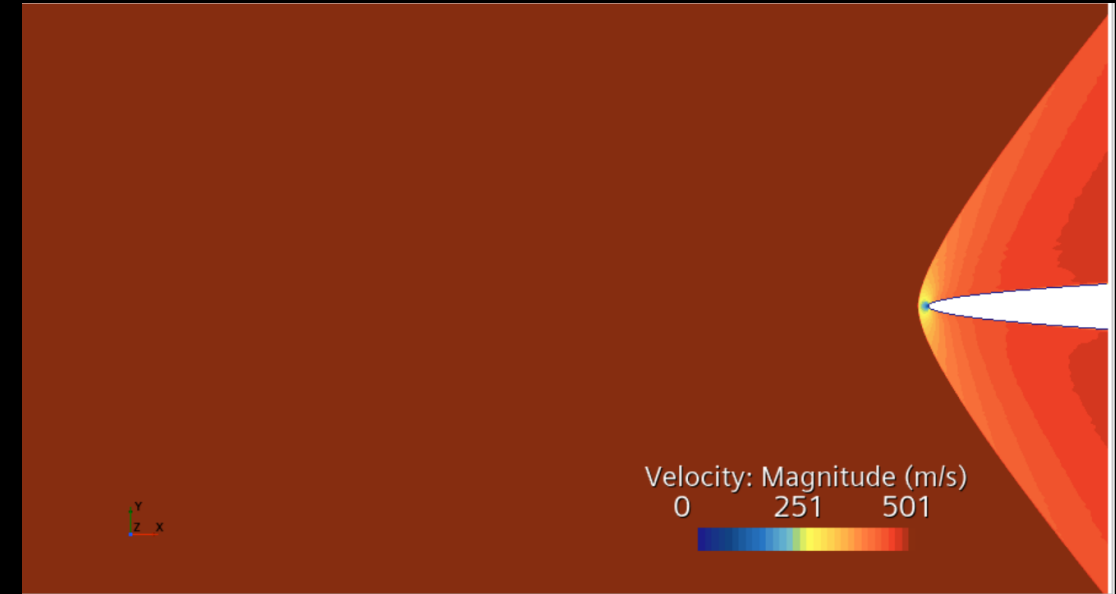
- Current design considerations are 24in L / 6.221in D (approximate slenderness ratio of 4)
- Supersonic- 10 s Transonic- 15 s Subsonic- remainder (Open Rocket)
- Von-Karman performs better in the Transonic region, while the $X^{1/2}$ Power Series performs better in the Supersonic region
- Max drag coefficients at Mach 1.53 (Open Rocket) - Von-Karman(.09) , $X^{1/2}$ Power Series(.08)

Nose Cone

Von-Karman



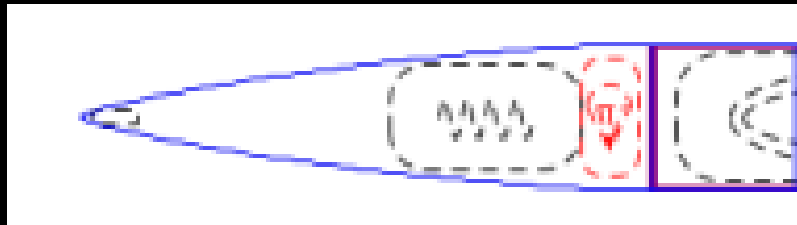
X^{1/2} Power Series



- Oblique shock wave for von-karman occurs at a smaller angle so the shock wave is weaker than the shock produced by $x^{1/2}$ Power Series

Nose Cone

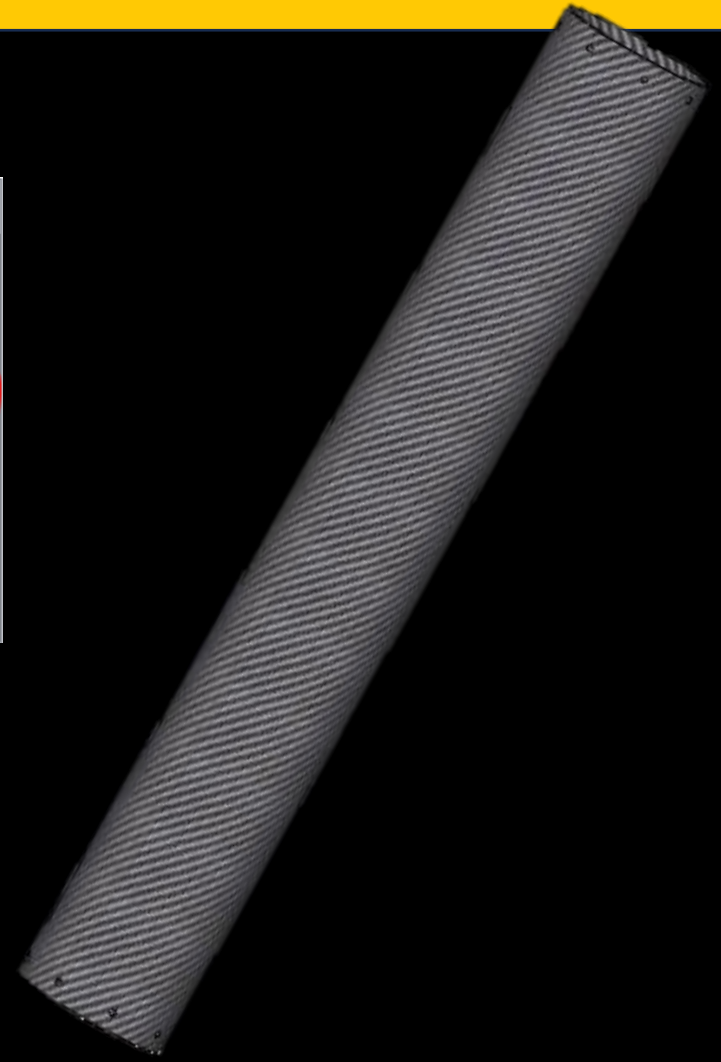
- Currently we are going with the Von-Karman considering the most time spent is in the Transonic region, and it performs fairly well or superior in the other regions
- Component Analysis in Open Rocket has the drag coefficient for $X^{1/2}$ Power Series and Von-Karman NC at .03 all throughout the subsonic region
- Conduct sims in the Transonic and Supersonic regions to get a better idea of how they perform (Open Rocket analysis not highly reliable in supersonic region)
- Conduct 2d/3d sims to research the most effective slenderness ratio
- Utilize Open Rocket to see affects on apogee
- Take into account volume needed for the drogue shock cord and parachute to fit inside, as well as manufacturing issues that could result from lengthening the nosecone



Body Tubes

| Component | Pressure C_D | Base C_D | Friction C_D | Total C_D |
|------------------------|--------------------|--------------------|--------------------|---------------------|
| Total | 0.127 (24%) | 0.144 (28%) | 0.249 (48%) | 0.520 (100%) |
| Nose Cone | 0.066 (13%) | 0.000 (0%) | 0.020 (4%) | 0.086 (17%) |
| nosecone straight | 0.000 (0%) | 0.000 (0%) | 0.004 (1%) | 0.004 (1%) |
| switchband | 0.000 (0%) | 0.000 (0%) | 0.002 (0%) | 0.002 (0%) |
| upper Tube | 0.000 (0%) | 0.000 (0%) | 0.040 (8%) | 0.040 (8%) |
| mid Tube | 0.000 (0%) | 0.000 (0%) | 0.030 (6%) | 0.030 (6%) |
| Transition forward nox | 0.009 (2%) | 0.000 (0%) | 0.000 (0%) | 0.010 (2%) |
| nox tank | 0.000 (0%) | 0.000 (0%) | 0.076 (15%) | 0.076 (15%) |
| Transition aft nox | 0.029 (6%) | 0.000 (0%) | 0.000 (0%) | 0.030 (6%) |
| lowest Tube | 0.000 (0%) | 0.000 (0%) | 0.055 (11%) | 0.055 (11%) |
| Trapezoidal Fin Set | 0.005 (1%) | 0.000 (0%) | 0.004 (1%) | 0.009 (2%) |
| Tailcone | 0.000 (0%) | 0.144 (28%) | 0.007 (1%) | 0.151 (29%) |

The Overall Coefficient of Drag for the carbon fiber body tubes is 0.257 (summation of coefficients of components).

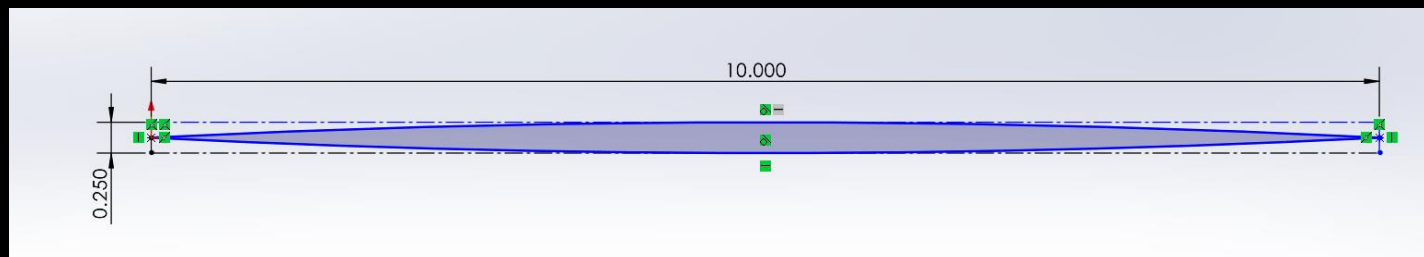
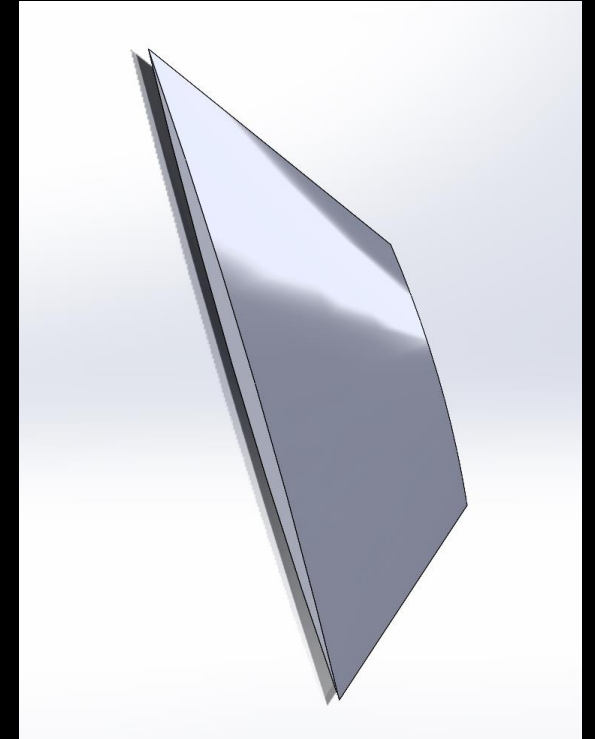
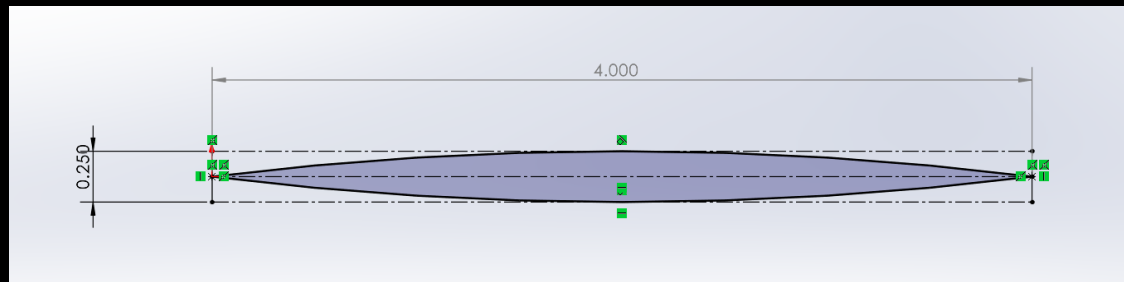
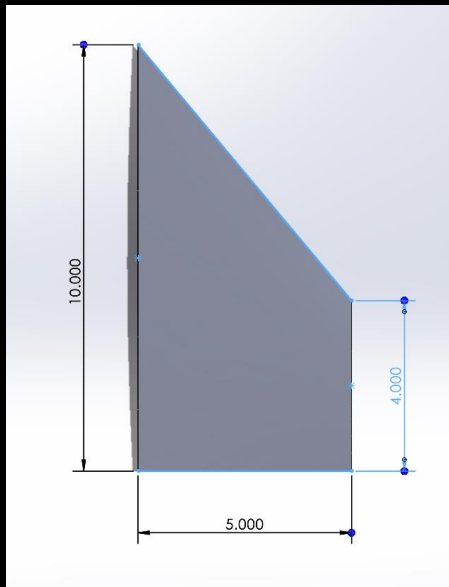


Fins

Current planform is a clipped delta with an airfoiled cross section.

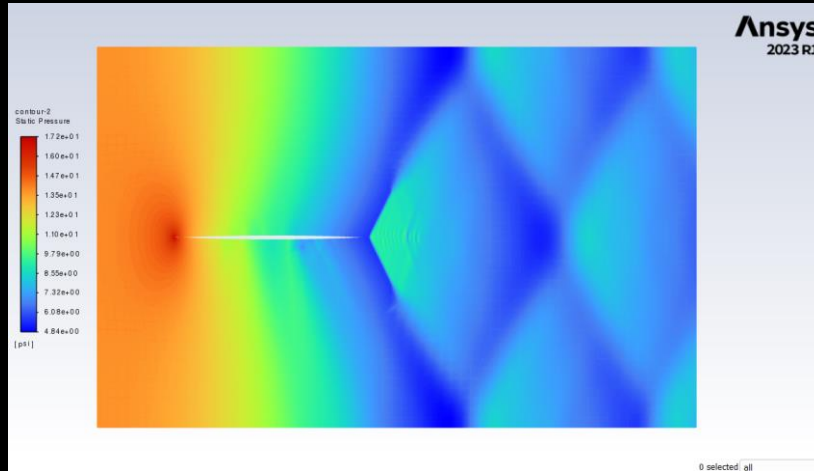
This has been chosen to help reduce drag caused by supersonic shocks.

Preliminary simulations show a drag coefficient of around .0075, which is lower than the OpenRocket prediction of .009.

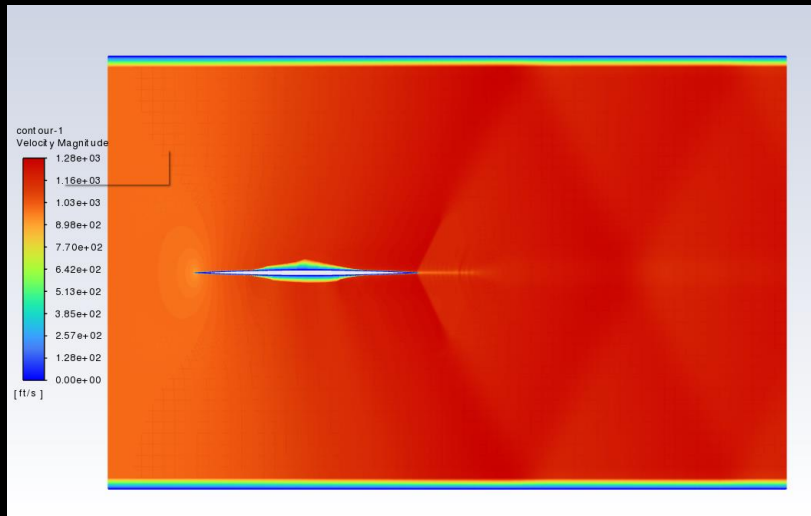


Fins

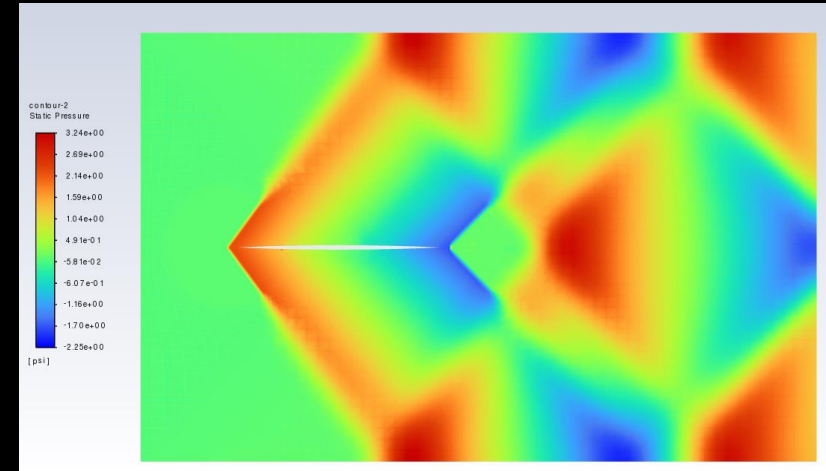
Static Pressure at 1000 ft/s



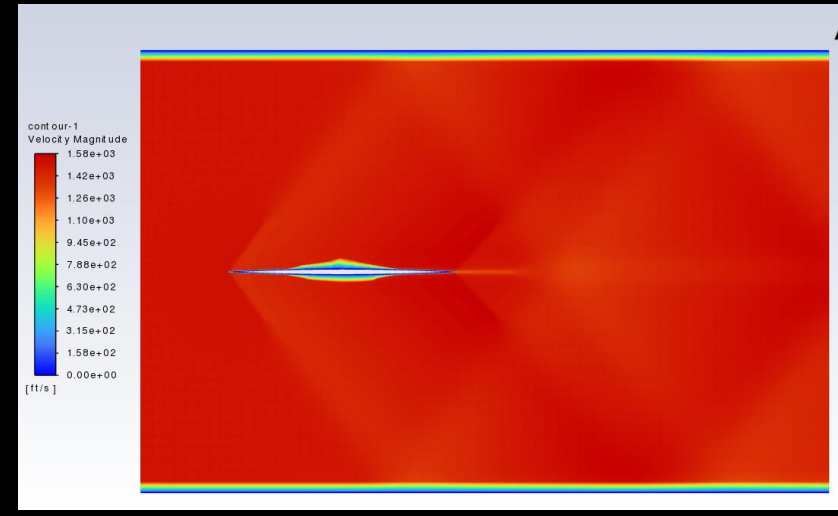
Velocity at 1000 ft/s



Static Pressure at 1500 ft/s



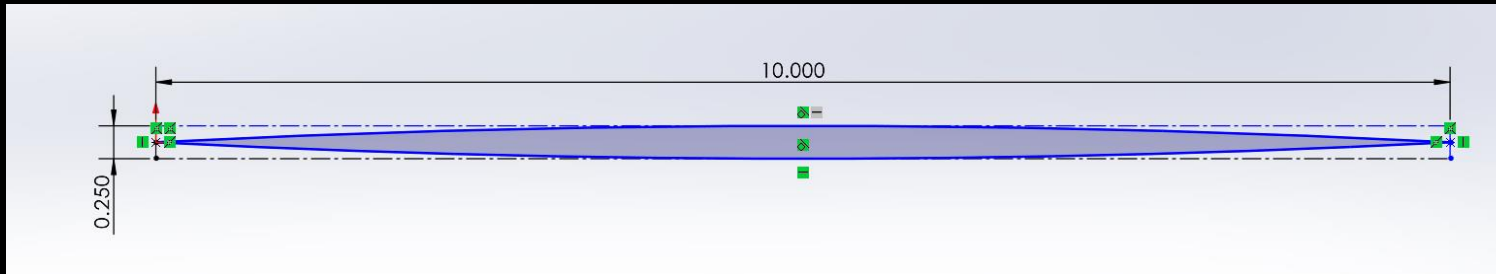
Velocity at 1500 ft/s



Fins

Fin flutter is a major risk of failure for fins, which could lead to their complete failure.

By choosing airfoiled fins, calculating fin flutter velocity became more difficult, as the thickness is not constant. To find a thickness, the area of the base of the fin was found. The area was then divided by the root chord. In essence, this method finds the flutter velocity of a rectangular fin with similar dimensions.



$$\text{Thickness} = 1.667 / 10 = .1667 \text{ in}$$

Ansys simulations could provide a more accurate method of calculation flutter velocity.

Fin Flutter Velocity Calculations

$$a := 1125.33 \frac{ft}{s} \quad \text{Speed of Sound}$$

$$AR := .5714 \quad \text{Aspect Ratio}$$

$$P := 11.77 \text{ psi} \quad \text{Air Pressure}$$

$$\lambda := .4 \quad \text{Taper Ratio}$$

$$t := 0.1667 \text{ in} \quad \text{Thickness}$$

$$G := 7498450 \text{ psi} \quad \text{Shear Modulus}$$

$$+ c := 10 \text{ in} \quad \text{Root Chord}$$

$$V_f := a \cdot \sqrt{\frac{G}{\left(\frac{1.337 \cdot (AR)^3 \cdot P \cdot (\lambda + 1)}{2 \cdot (AR + 2) \cdot \left(\frac{t}{c}\right)^3} \right)}}$$

$$V_f = (7.419 \cdot 10^3) \frac{ft}{s}$$

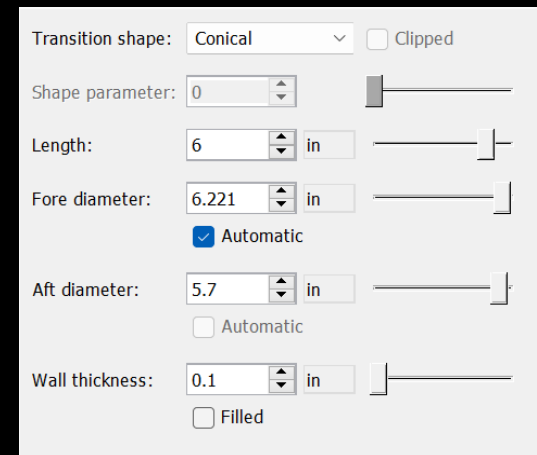
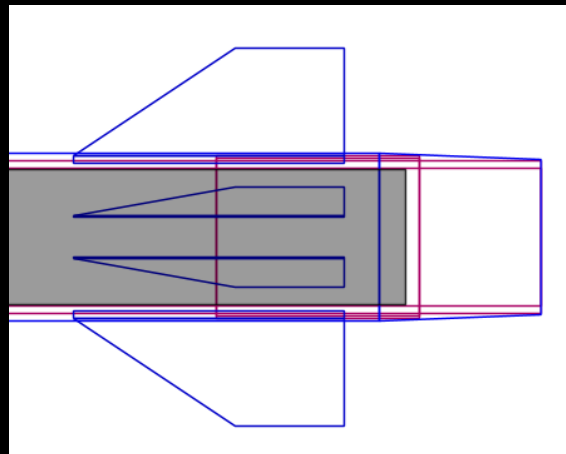
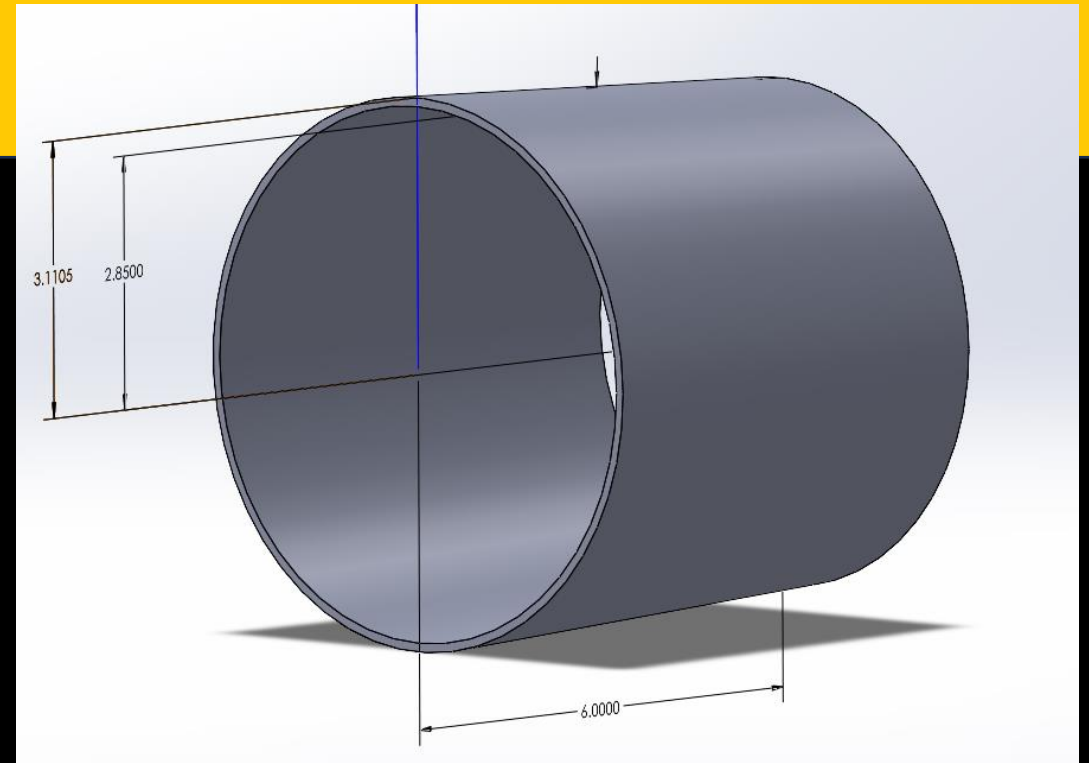
Tail Cone

Current design is a Conical Straight Tail Cone.

Purpose is to reduce the wake region behind vehicle during flight, reducing drag when compared to exclusion of a Tail Cone.

Conical Tail Cone chosen due to ease of manufacture and combustion chamber size constraints, which make the performance differences between different designs minimal.

Coefficient of Drag is currently 0.155 for this component, according to OpenRocket Component Analysis.



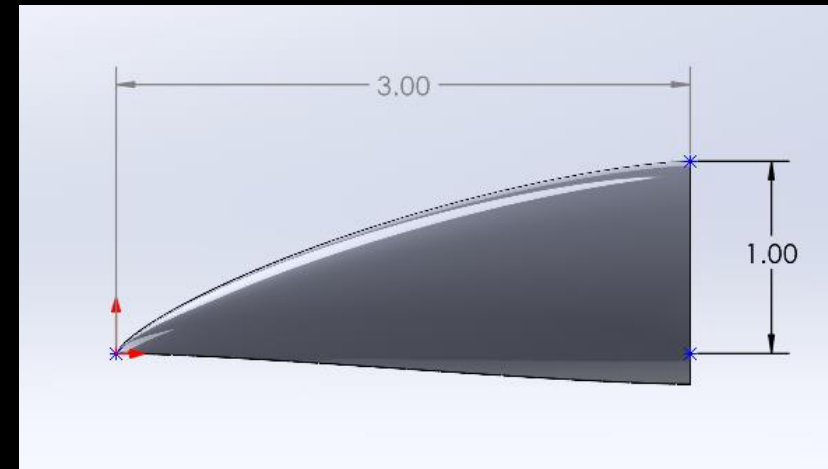
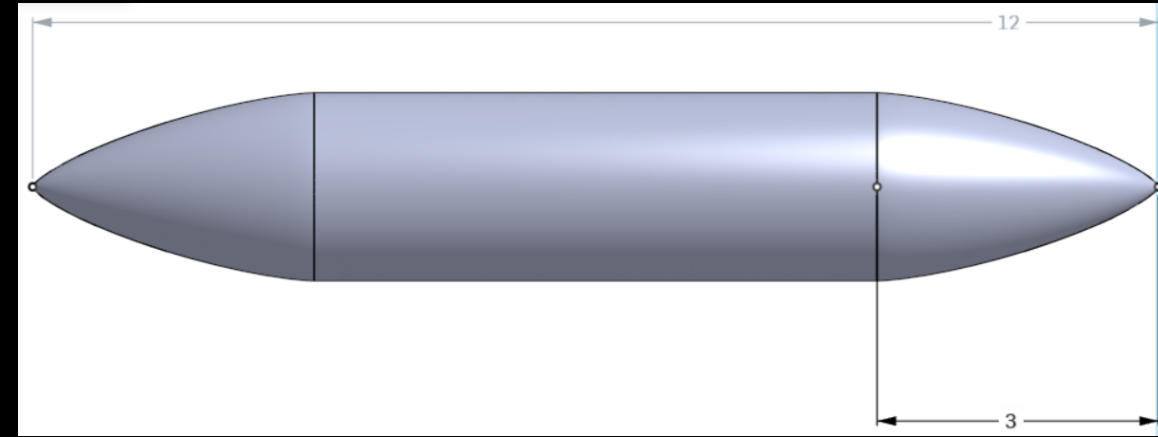
*units
are
inches

Shroud

Purpose: protects external antennas from ground impact

The current design uses a Von Karman geometry for its nose and tail sections, with a 6-inch cylindrical section for the antenna

Plastic or fiberglass will be used, allowing for an RF transparent enclosure



Shroud

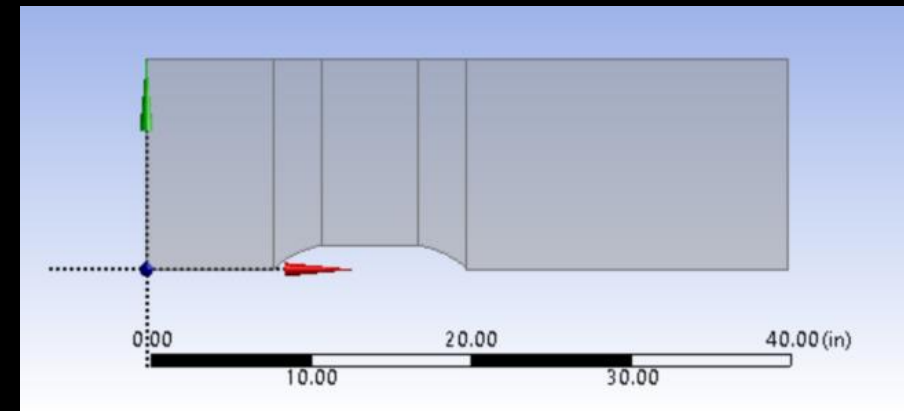
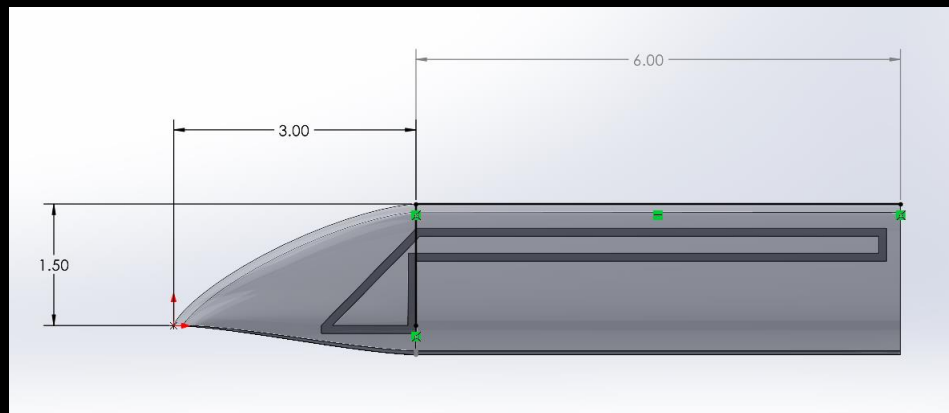
What's next:

A hook for attachment to the body of the rocket will be included in later iterations

The design will be further optimized for varying sizes of the antennas

Different cone designs will be tested and compared

Testing will be done on the current design

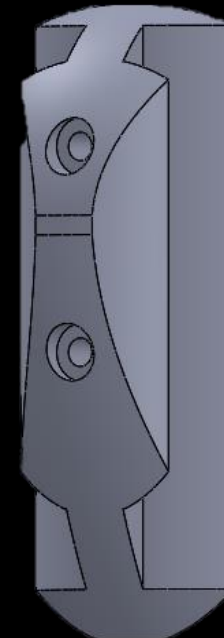
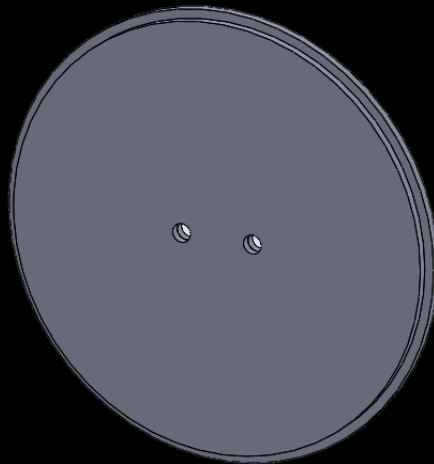


Structures Requirements

| Requirement | Verification Method |
|--|---------------------|
| Launch vehicles shall be adequately vented to prevent internal pressures developing during flight and causing either damage to the airframe or any other unplanned configuration changes. | Inspection |
| Joints shall be designed such that the coupling tube extends no less than 1 body tube diameter (1 caliber) into the airframe section from which the coupler will separate during flight. | Inspection |
| Joints shall be designed such that the coupler tube extends into the nosecone/tailcone/transition to the lesser of 1 body tube diameter (1 caliber) or the maximum depth possible by the design of the nosecone/tailcone/transition. | Inspection |
| Joints shall be designed such that the coupling tube extends into the mating component to the lesser of 1 body tube diameter (1 caliber) or the maximum depth possible by the design of the mating component. | Inspection |
| Joints shall be affixed by mechanical fasteners and/or permanent adhesive. | Inspection |
| Regardless of implementation (e.g., RADAX or other join types) airframe joints shall prevent bending, see https://www.osti.gov/biblio/5007820 . | Analysis |

Structures Sub-System

The Structures Team is responsible for the robust and nominal design of an airframe that can withstand all applied stresses, pressures of supersonic flight, and integrate all components of the rocket from other systems.



Structures Requirements

| | |
|---|------------|
| All load bearing eye bolts shall be of the closed-eye, forged type. | Inspection |
| The load bearing eye bolts, U-bolts, links, and any bolt and eye-nut assembly used in place of an eyebolt SHALL be steel. | Inspection |
| Load bearing U-bolts shall have mounting plates to ensure proper force distribution | inspection |
| The rail guide shall integrate with the LTI Launch Rail | Inspection |
| Rail buttons shall be attached using at least one metallic fastener through the reinforced airframe. | Inspection |
| Rail buttons shall implement “hard points” for sliding mechanical attachment of the rocket to the SA Cup supplied 1515 launch rail, serving to guide the rocket during the initial phase of boost until the rocket achieves sufficient velocity for the fins to provide aerodynamic stabilization | Inspection |
| The aft most launch rail button shall support the launch vehicle’s fully loaded launch weight while the rocket is in a vertical orientation. | Inspection |

Structures Requirements

| | |
|---|------------|
| The IREC Team WILL either lift the vehicle by the rail guides and/or demonstrate that the bottom guide can hold the vehicle's weight when vertical before permitting them to proceed with launch preparations. | Testing |
| The body tubes shall withstand bolt tear out with a safety factor of 2. | Analysis |
| The fins shall not be sheared off the rocket during flight | Analysis |
| The fins shall be securely attached to the fuselage. | Inspection |
| All bolts torqued down to proper spec. | Inspection |

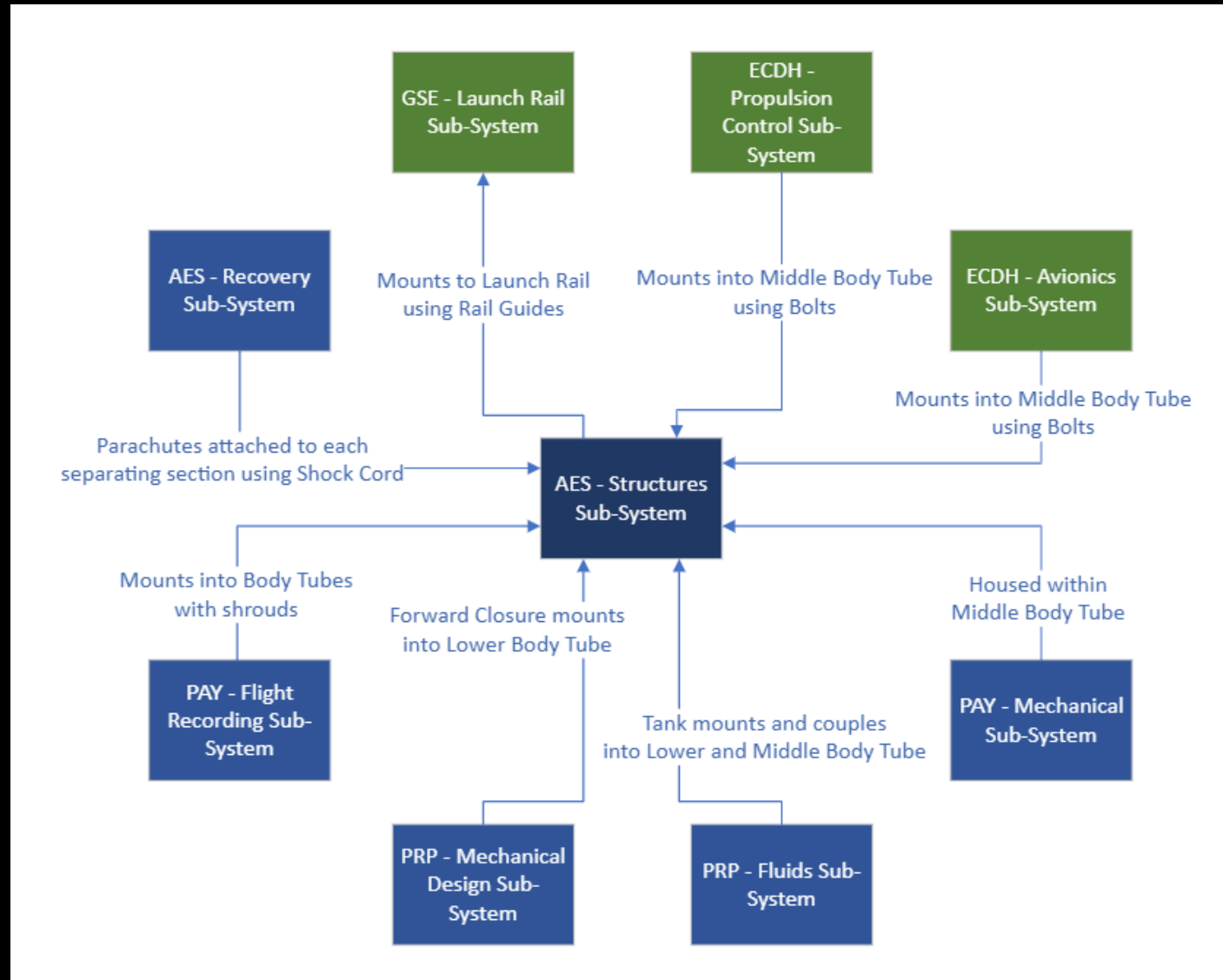
Structures TPMs

| Measure | TPM Value | Units | Verification Method |
|--|-----------|-----------------|----------------------------|
| Max load on airframe | [] | lbf | Ansys Fluent |
| Max size of internal volume (upper/middle tube) | 1710 | In ³ | SolidWorks, Open Rocket |
| Max size of internal volume (nosecone) | 340 | In ³ | SolidWorks, Open Rocket |
| Max allowable snatch force | 1406 | lbf | Handcalcs |
| Max shear force per bolt | 608 | lbf | calculator |

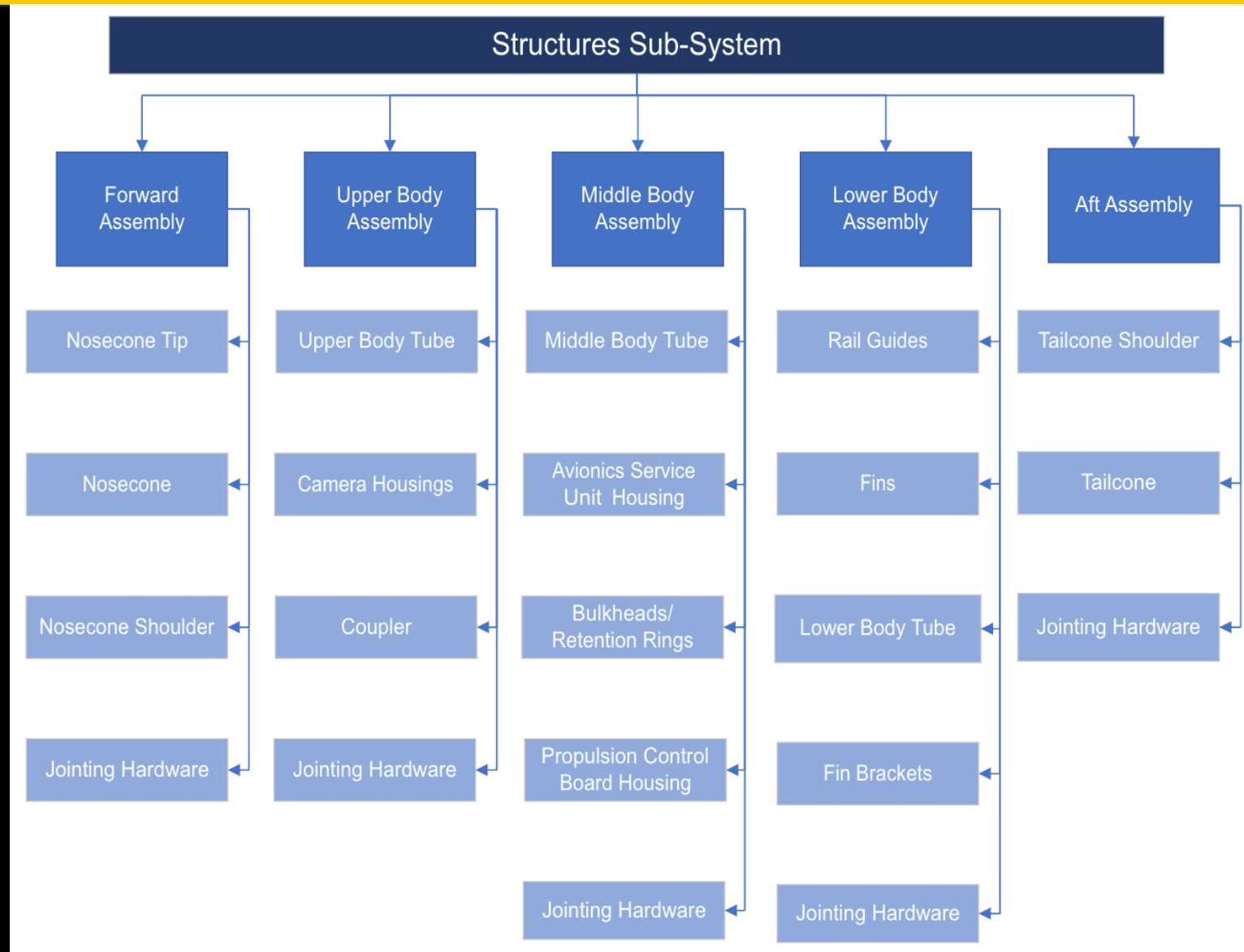
Structures TPMs

| | | | |
|--|-------|-----------------|--|
| Total body stress | 3222 | psi | Calculator |
| Max bending moment | 7564 | Lb-in | Calculator |
| Max strain | [] | in/in | Ansys Mechanical/ Composite Prepost |
| Max size of internal volume (combustion chamber) | 210 | In ³ | SolidWorks, Open Rocket |
| Component fit tolerancing | [] | In | Component testing |
| Max allowable bearing stress | 29000 | psi | Calculator |
| Max body shear force | 245 | lbf | Calculator |

Structures Interface Diagram

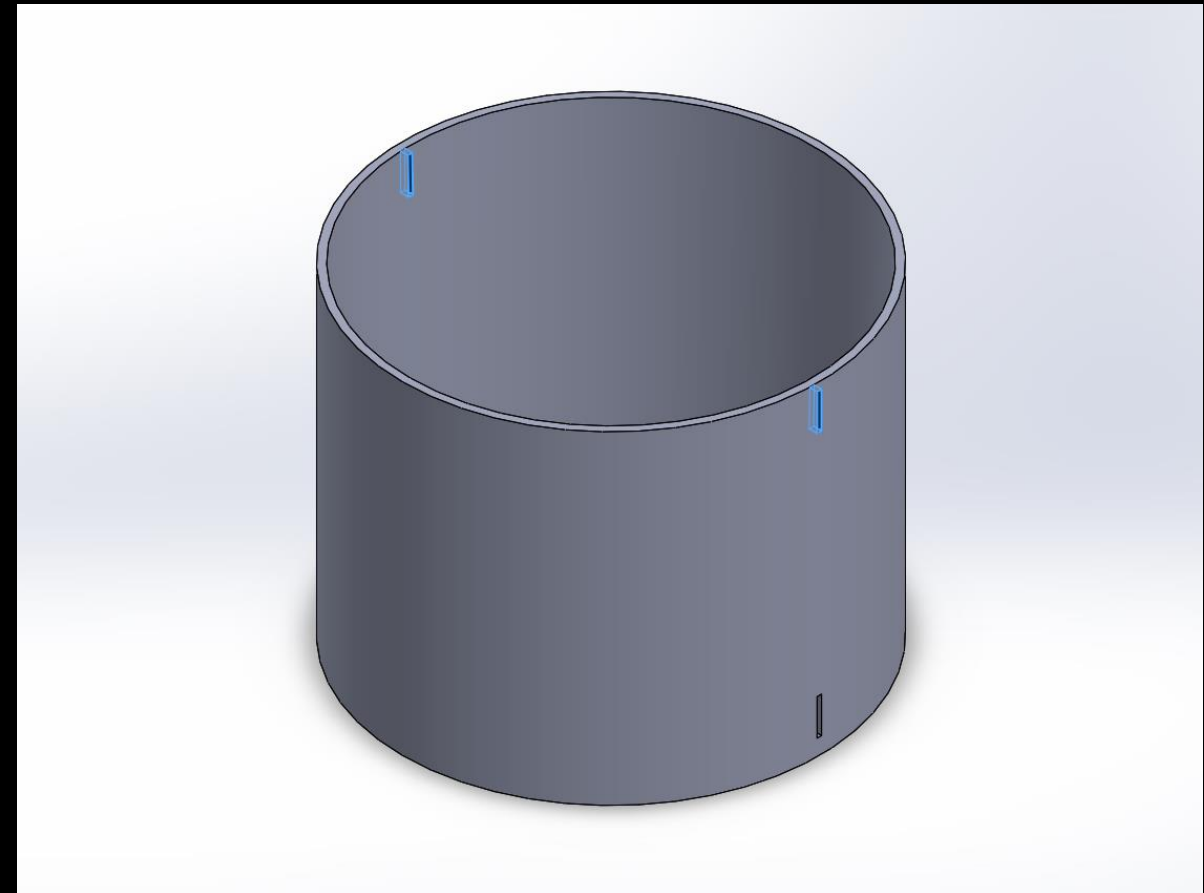
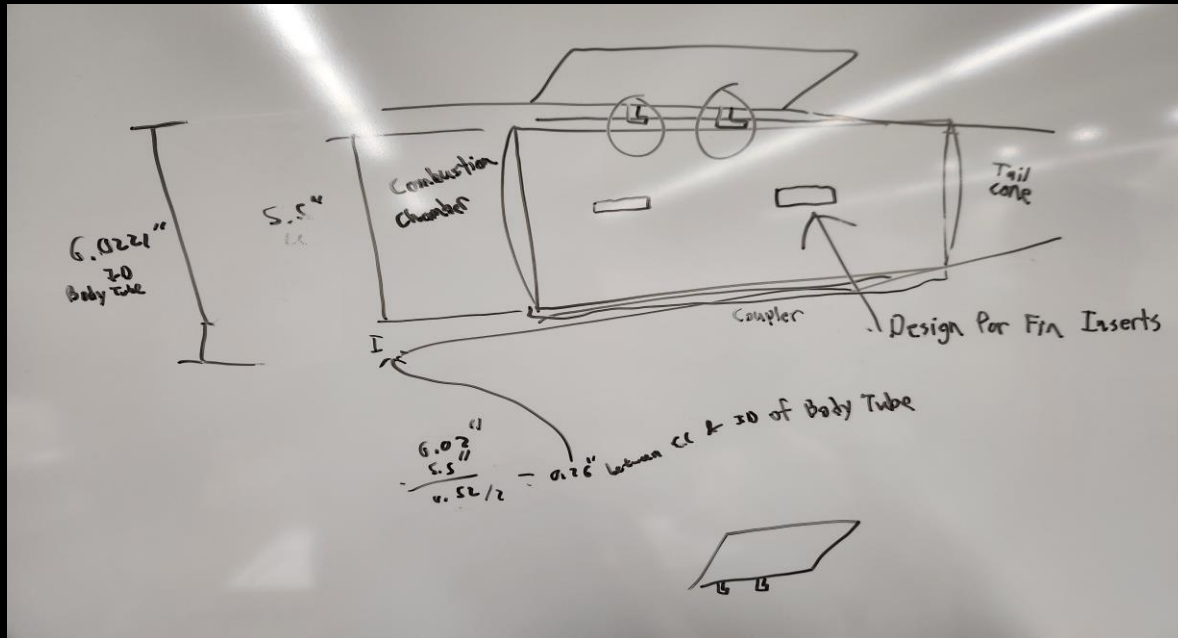


Structures Component Breakdown



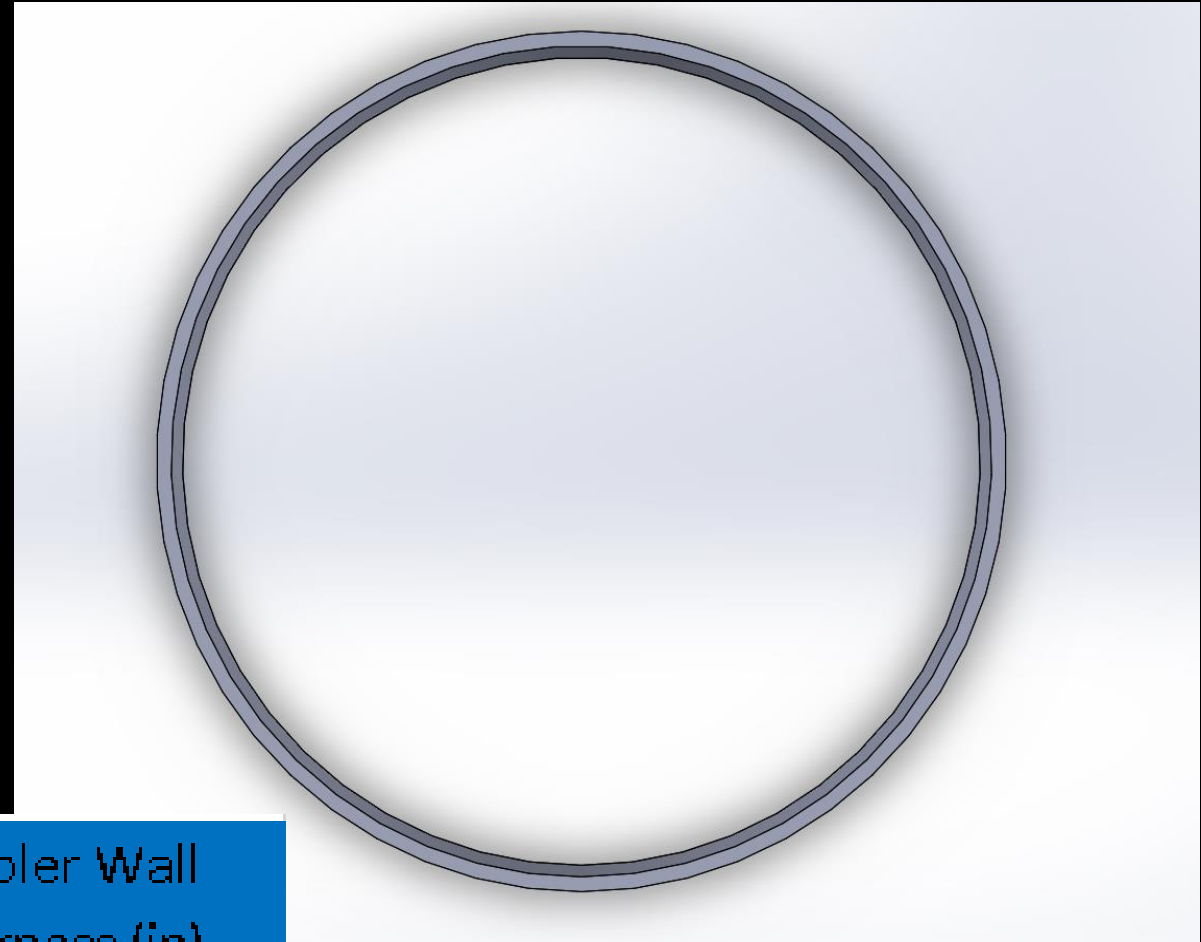
Tail Cone Shoulder Design

- Creating the Coupler to Fit Between the Body Tube ID (Inner Diameter) and Combustion Chamber, Holes will be Created Allowing Mounting Positions for Fin Inserts (Hole Shape TBD)
- Tail cone shoulder is sandwiched between combustion chamber and fins
- Coupler would have slits to align fin attachment points



Coupler's Design

- Outer Diameter: 6.021"
- Coupler Thickness Approx 0.111"
- DTEG Coupler Requirement: 1 Cal
- Rocket Diameter is Approx 6 Inches, Requiring Recovery Coupler Minimum Length to be 6 inches plus switch band length
- Internal couplers length
- Composite Envision Cost :
2x2' \$115.90/ lzd

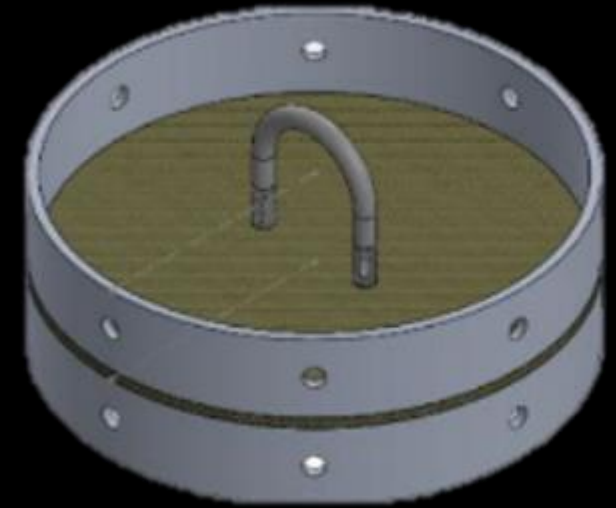


Coupler Wall
Thickness (in)

0.111093263

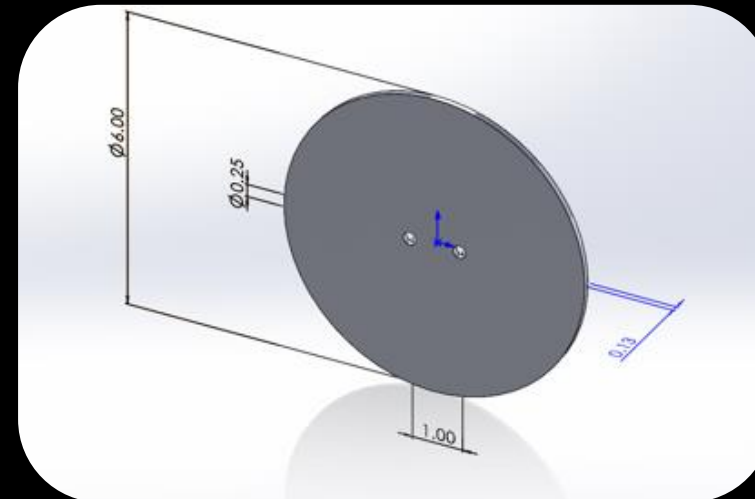
Bulkheads & Retaining Rings

| Component | Materials |
|-----------------|-----------------------|
| Retaining Rings | Aluminum Rings |
| Bulkhead | Pre-Preg Carbon Fiber |



Schedule

- ANSYS simulations
- Optimize current design for load cases

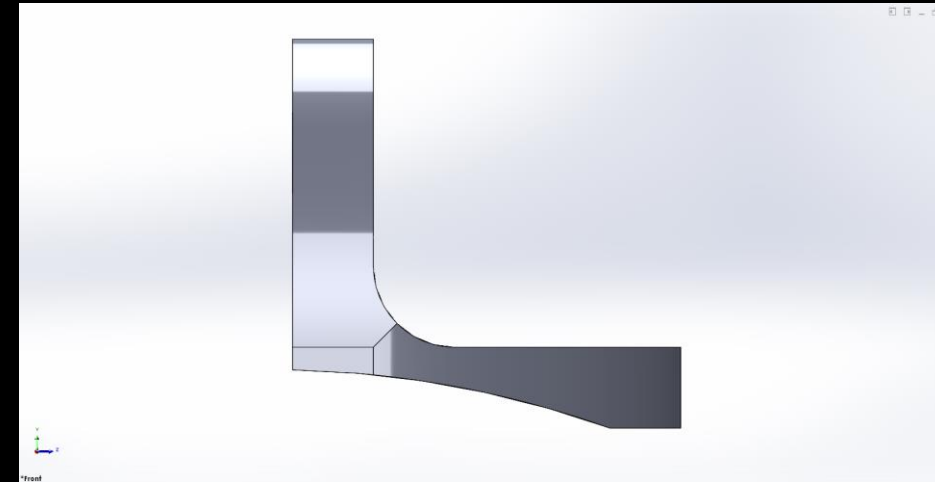
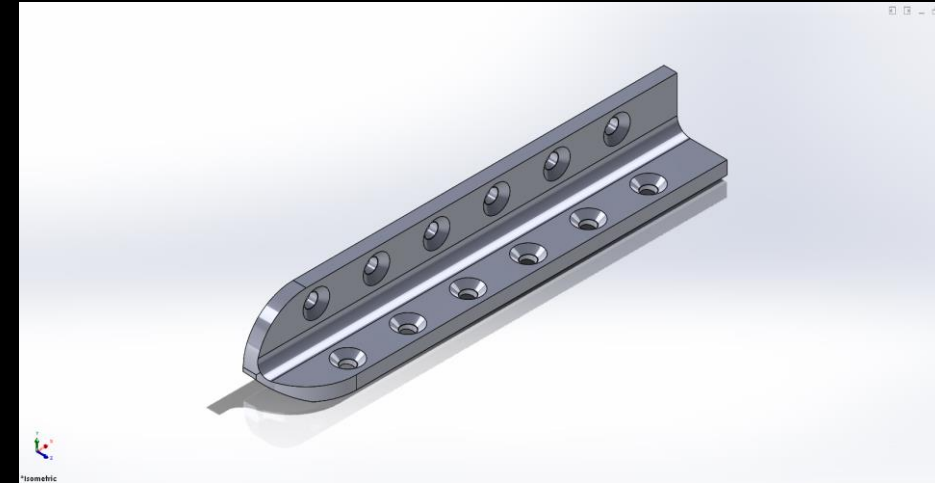


Fin Reinforcement

- Fin brackets attach to the fins and body tube.
- 6061 Aluminum fin brackets
- 12 ¼" screw holes to attach to the airframe and fins, 6 in each side
- 10 in. long Fin tabs, 8 in. x 1.2 in. Brackets with ¼ in thickness.
- Countersink holes and fillets for dynamic efficiency and stability

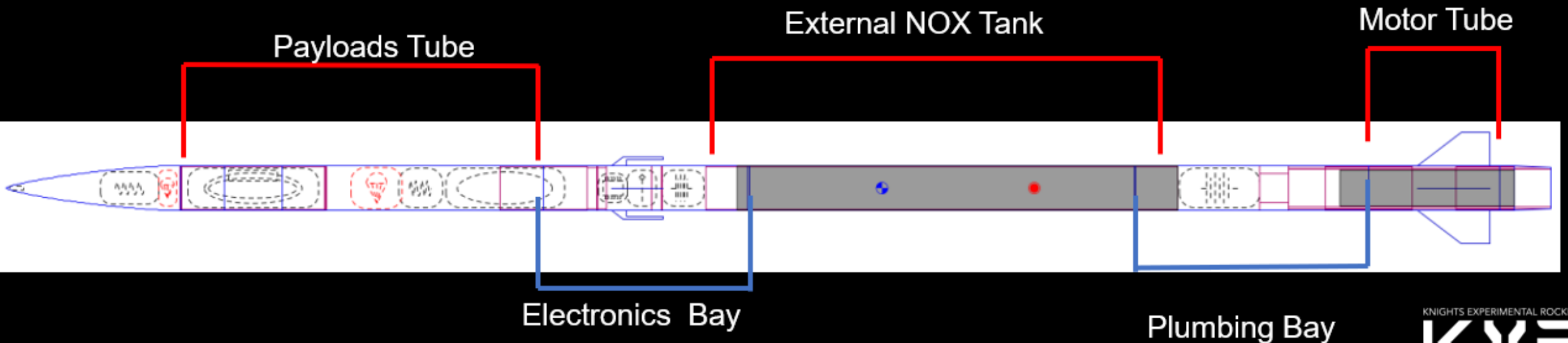
Schedule

- ANSYS Simulation to verify and optimize structural design and efficiency to ensure bracket capable to withstand forces apply to the fin.



Body Tubes

| Name of Body Tube | Unit | Payload Tube | Electronics Bay | Plumbing Bay | Motor Tube |
|-------------------|------|--------------|-----------------|--------------|------------|
| Length | in | 36 | 24 | 32 | 18 |
| Weight | lb | 2.2 | 1.5 | 2 | 1 |
| Thickness | in | 0.1 | 0.1 | 0.1 | 0.1 |



Composite Layup

Material

- 3k Bi-Axial 2x2 Twill Pre-Impregnated Carbon Fiber

Possible Layups

- “Roll” Wrap
- 0/90

| Description | Value | Units |
|------------------------------|--------|--------|
| Amount of Fabric to Purchase | ≈40 | Yards |
| Layers of Carbon Fiber | 6 to 8 | Plys |
| Tube Thickness | 0.1 | Inches |

Future Considerations

- UTM Testing
- Run ANSYS Sims for Layup



Rail Guides

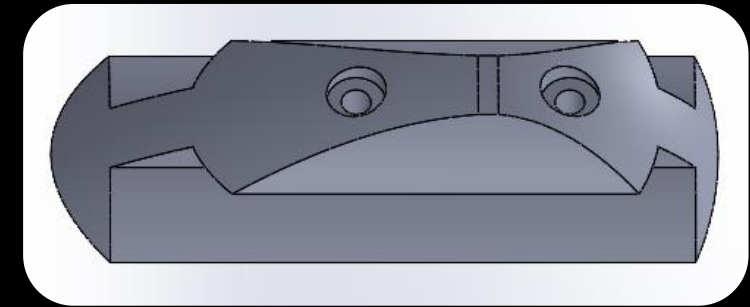
- SRAD Aluminum Rail Guides
- Optimized for better air flow

Material:

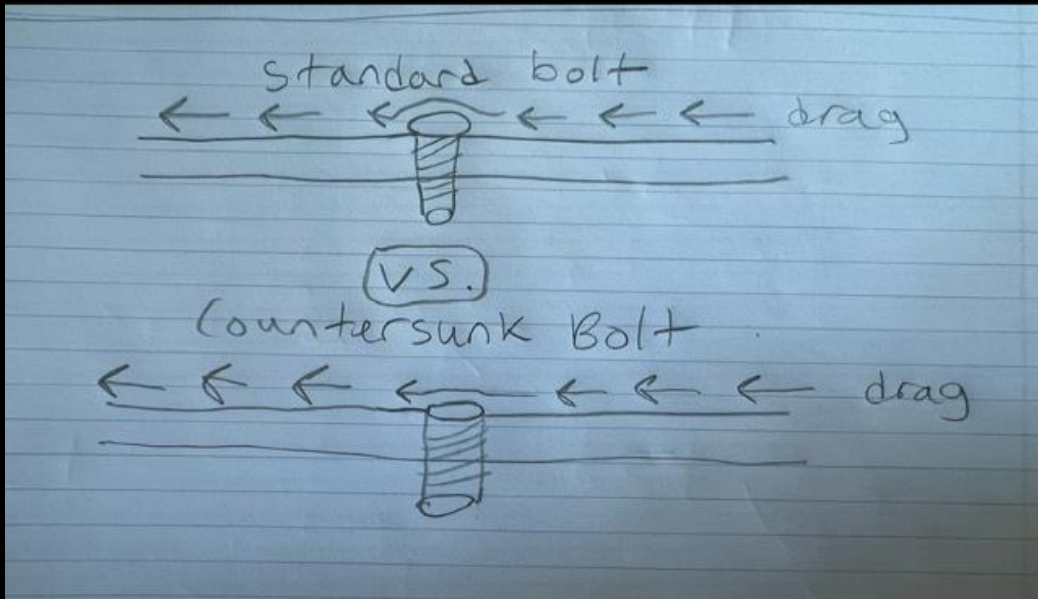
- Scrap Aluminum from machine shop
- Machined through UCF machine Shop

Next Steps:

- Iterative design to optimize air flow over guide
- More Subscale Models to test on KXR 1515
- Create Internal Component

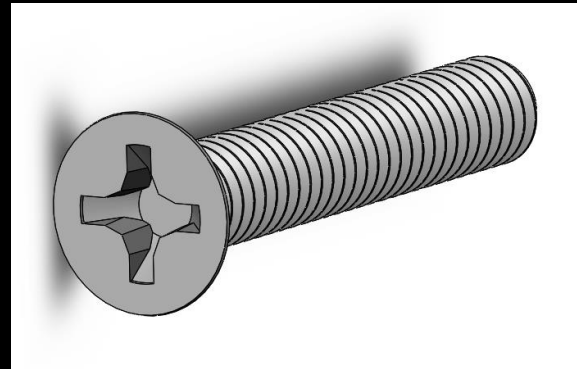


Airframe Jointing Hardware

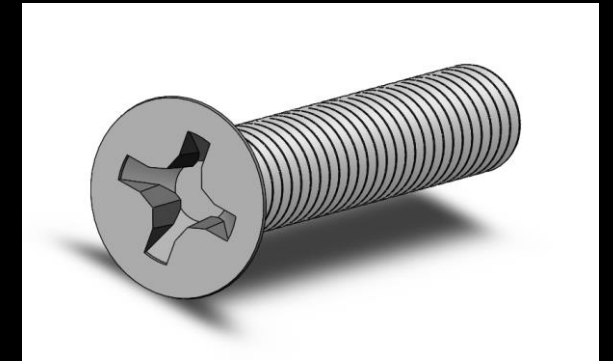


Final bolt length subject to change*

We are choosing ¼-20s, ballpark cost = \$40-50 including drill bit

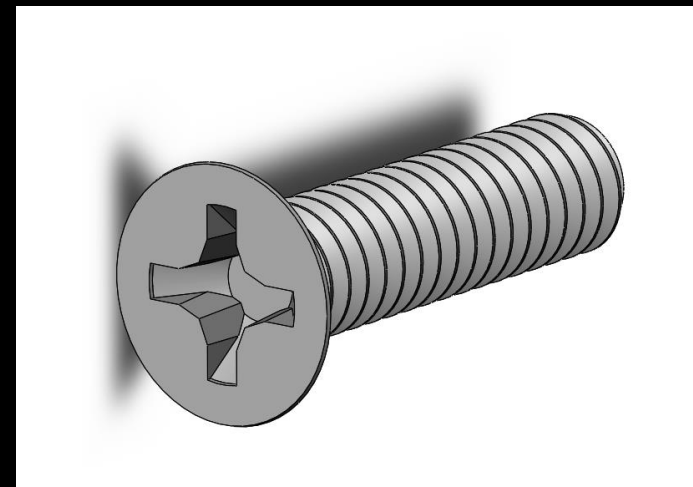


#8-32

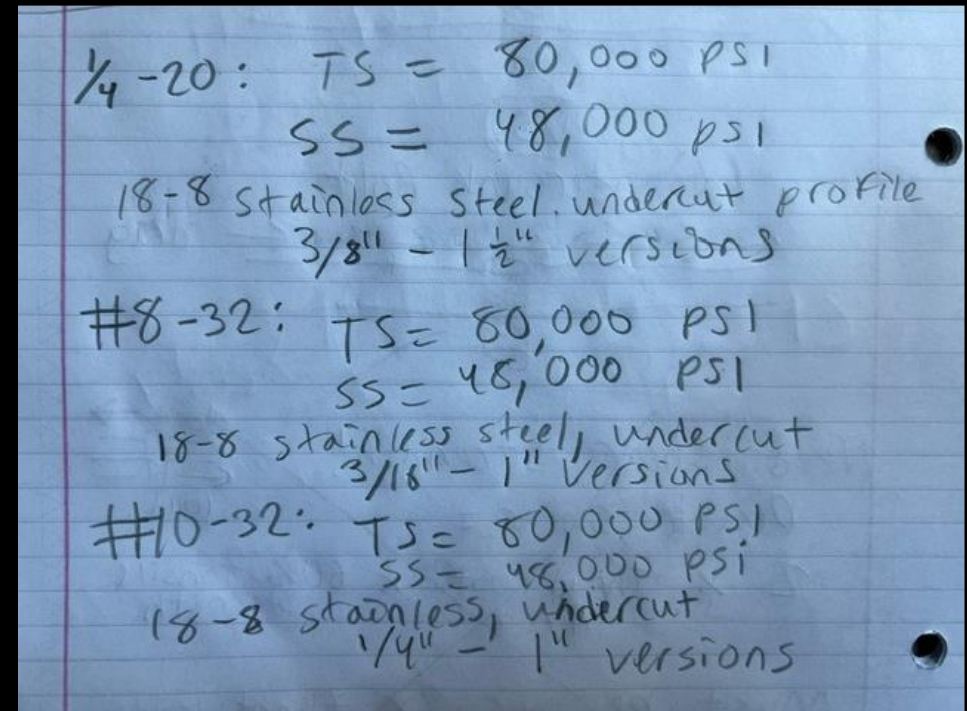
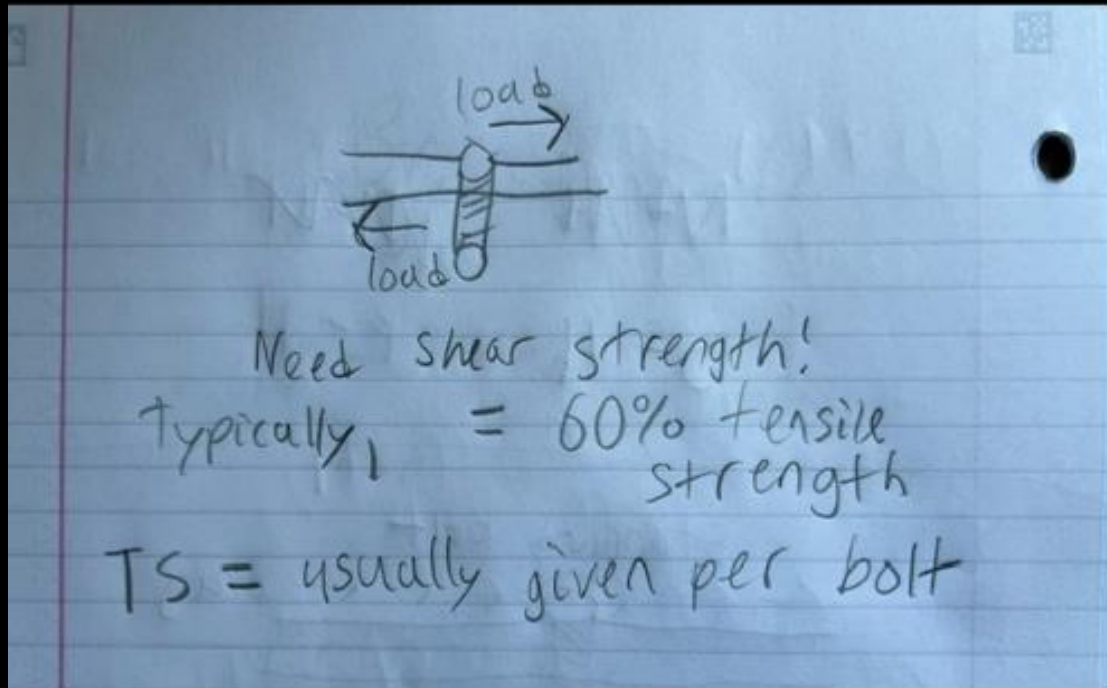


#10-32

¼-20



Jointing Hardware cont.



Essentially, the bolts will be fine. What we really need to worry about is bolt tear out from stress on the airframe.

Jointing Hardware cont.

Bolt Tearout:

Occurs when the force of the bolt pushing against its hole causes the material to fail.

Calculated by:

$$\sigma_{\text{bearing}} = \frac{F_{\text{bolt}}}{d_{\text{bolt, major}} \times t}$$

Force acting on each bolt
 major bolt diameter \times material thickness

| number of bolts | bolt major diameter (in) |
|-----------------|--------------------------|
| 8 | 0.25 |

Ideal numbers from 1/4"-20s

| Max Shear Force per Bolt (lbf) | Bearing Stress (psi) |
|--------------------------------|----------------------|
| 607.9366682 | 24317.46673 |

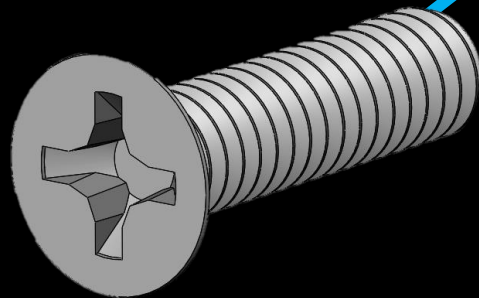
| SAE | | | |
|----------------|---------------|-------------------------|------------------------|
| Screw size/tpi | Dmajor (inch) | Ultimate Strength (lbf) | |
| | | Shear ^[1] | Tensile ^[2] |
| #2-56 | 0.086 | 354 | 540 |
| #4-40 | 0.112 | 577 | 880 |
| #5-40 | 0.125 | 753 | 1150 |
| #6-32 | 0.138 | 865 | 1320 |
| #8-32 | 0.164 | 1330 | 2030 |
| #8-36 | 0.164 | 1402 | 2140 |
| #10-24 | 0.190 | 1664 | 2540 |
| #10-32 | 0.190 | 1900 | 2900 |
| 1/4-20 | 0.250 | 3020 | 4610 |
| 1/4-28 | 0.250 | 3456 | 5275 |

Example calculations for multiple bolt types
 Max force pending final verification*

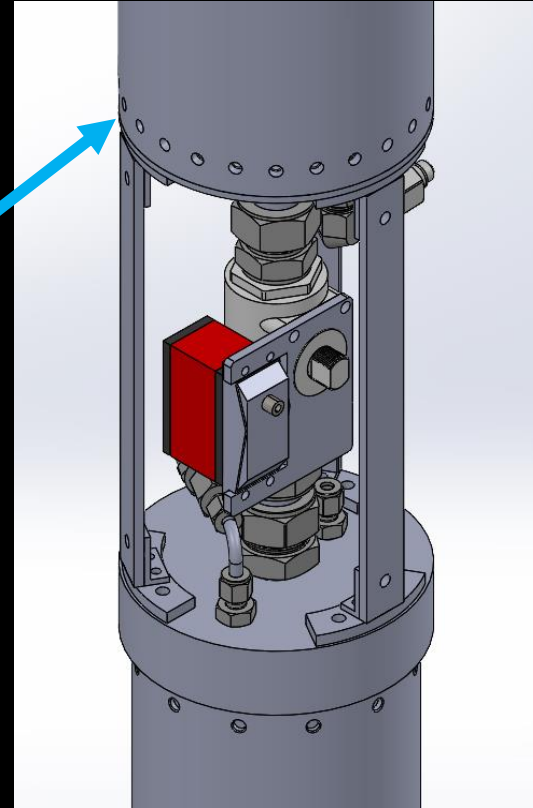
What's next: further testing and calculations to verify the number of bolts relative to max force on the airframe

NOX Tank Interface

- Tube attaches to NOX Tank Bulkhead
 - Threaded $\frac{1}{4}$ -20 holes
 - Slides over tank 1 Caliber or 6 inches
- Approx 60 $\frac{1}{4}$ -20 bolts in a staggered pattern
 - 30 Forward
 - 30 AFT

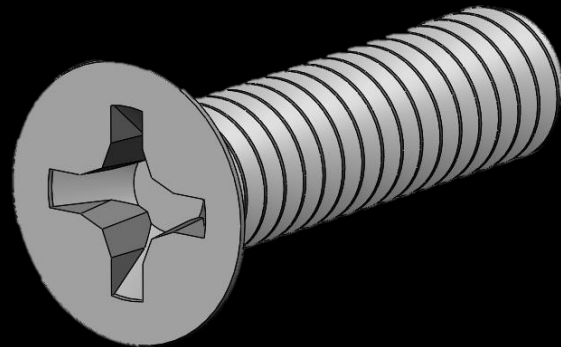


$\frac{1}{4}$ -20

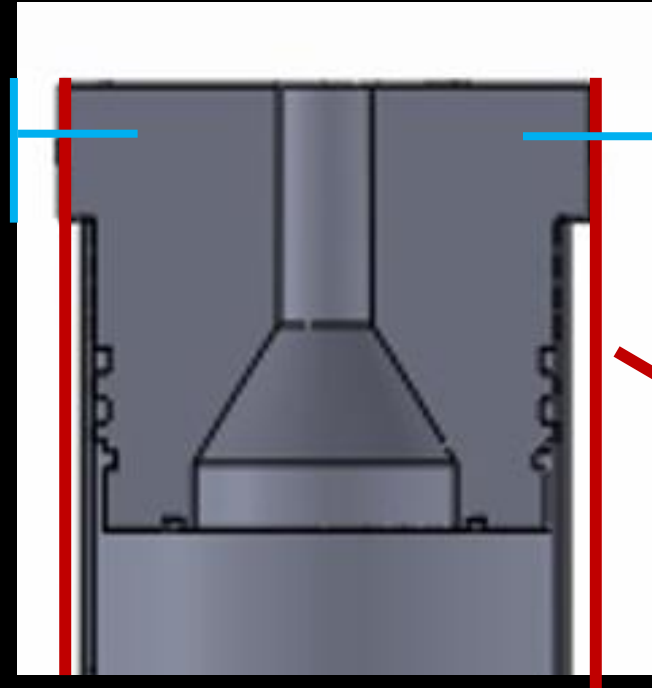


Combustion Chamber Interface

- Stretches from center of lower body tube to end of tail cone
- Retained within airframe with thrust plate
- Centering Ring on Combustion chamber at 1 CAL down

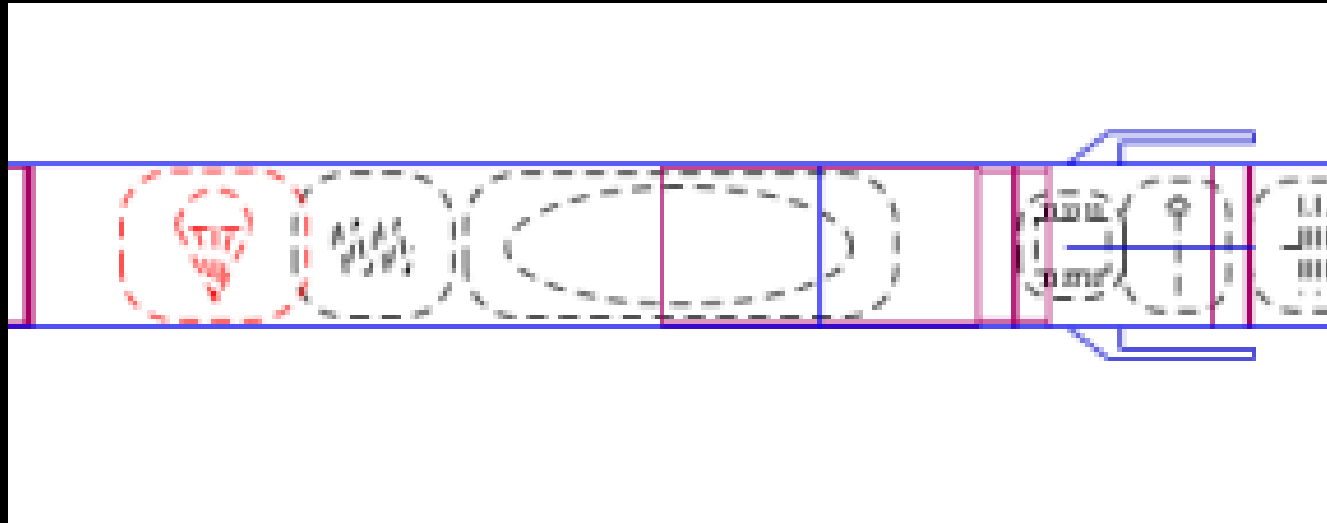


1/4-20



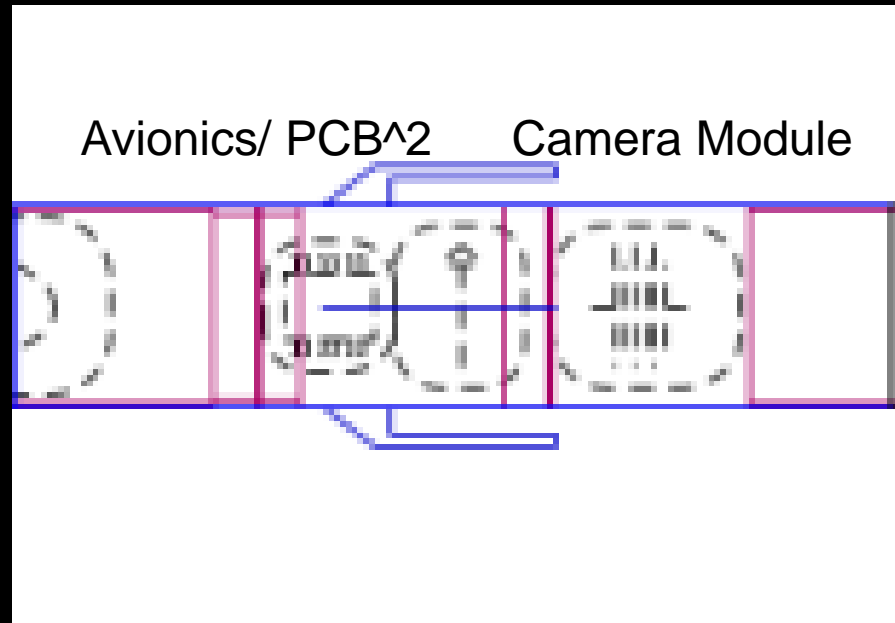
Payload Integration

- Payload will attach to main shock cord in upper body tube
- Payload will be situated above payload bulkhead and inside joint coupler of upper and mid tube

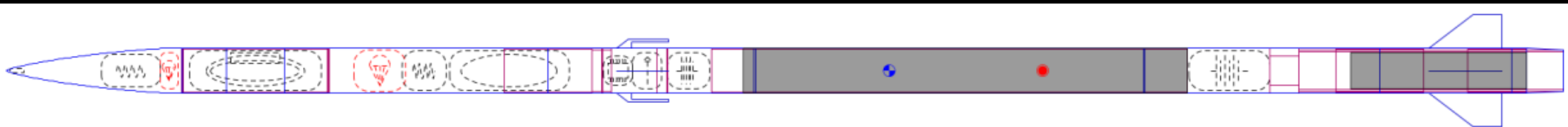
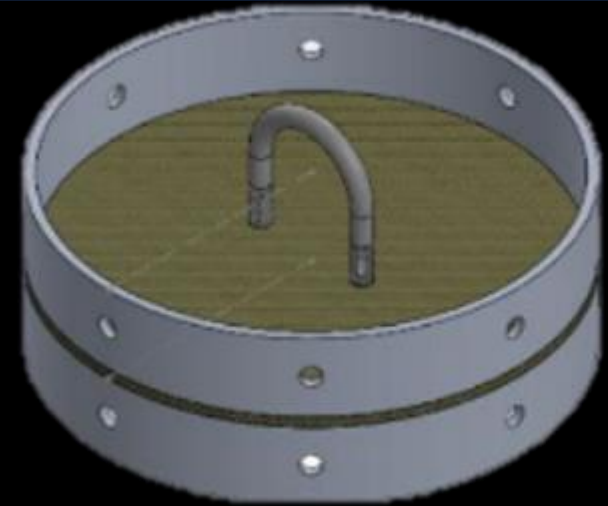
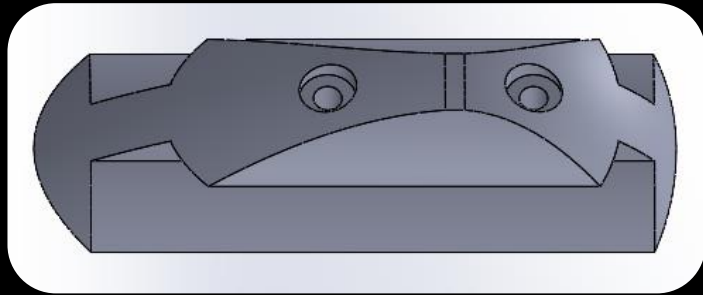


Avionics & Payloads Camera Module Integration

- Avionics Service Module and Camera Module to be located between recovery bulkhead and telemetry bulkhead
- Four equally spaced holes to account for cameras
- QD Camera faces outward away from rail between two camera module holes



Questions?

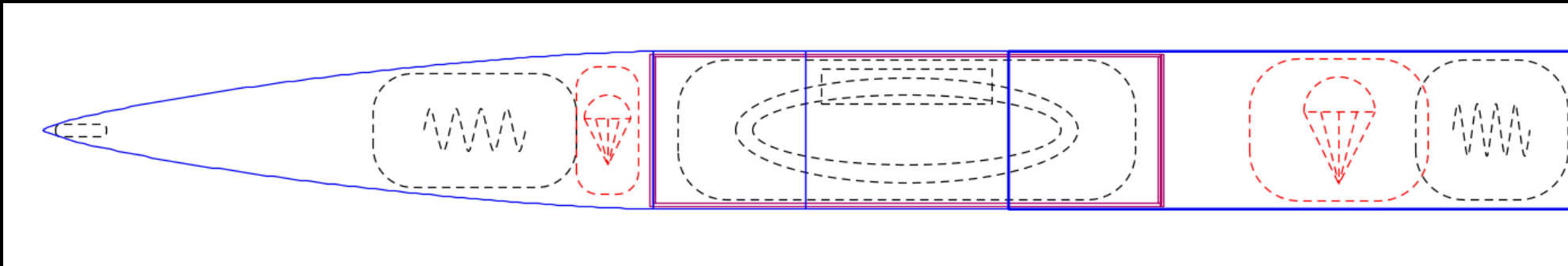


Risks & Mitigations

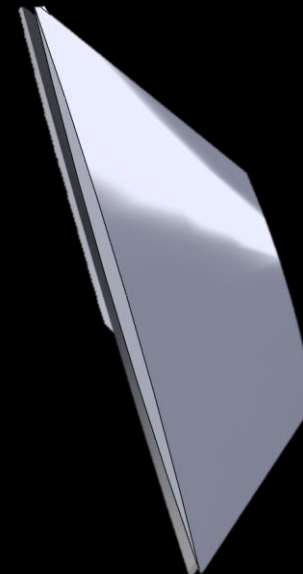
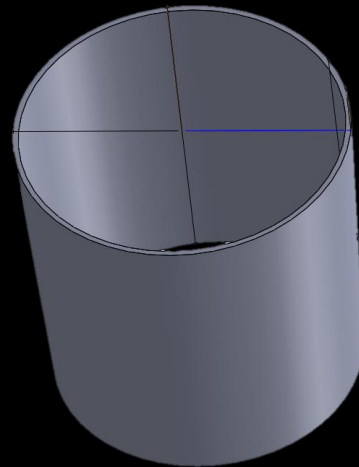
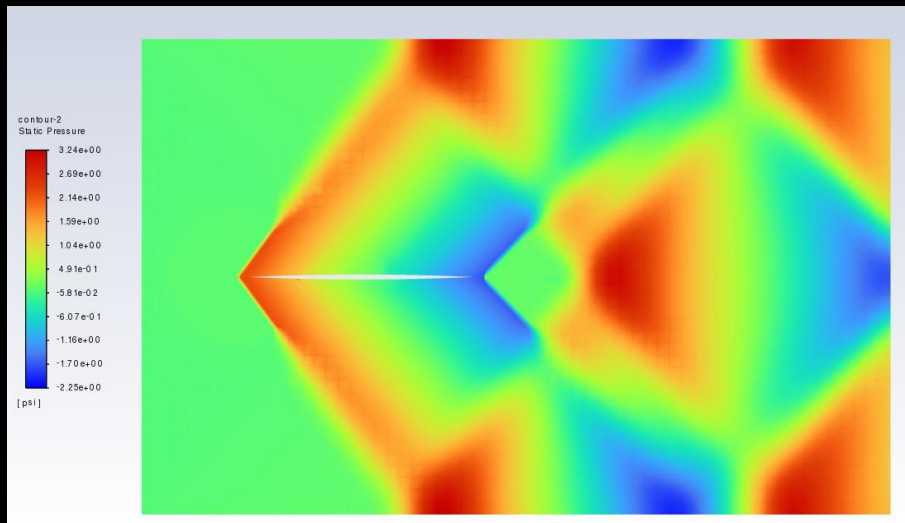
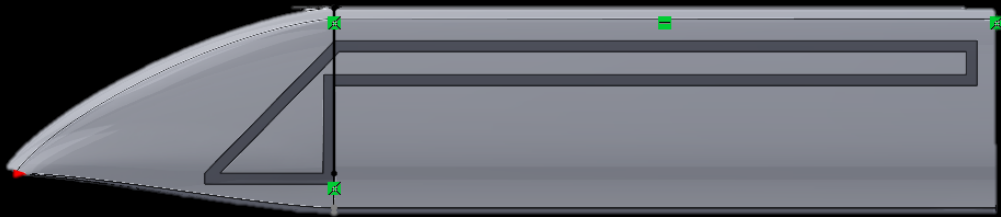
| Risk | Mitigation |
|---|---|
| External components shear off | Ensure proper attachment |
| Bulkhead tear out | Retaining rings are simulated to 2x safety factor |
| Bolt tear out | Bolts are rated to bearing stress below steel allowable stress |
| No signal from Avionics | Antennas are mounted properly to gather data |
| Incorrect simulation give faulty values | Ensure hand calc methods and simulations are verified |
| Components don't fit together | Track components through ICDs, communicate with REs with design, and track said changes within change log |

Recovery Sub-System

The Recovery Subsystem is responsible for designing parachutes that can withstand high opening forces, a robust coupler that can handle all recovery flight loads and all necessary electronics that can operate under supersonic flight conditions to ensure a controlled descent and a safe recovery



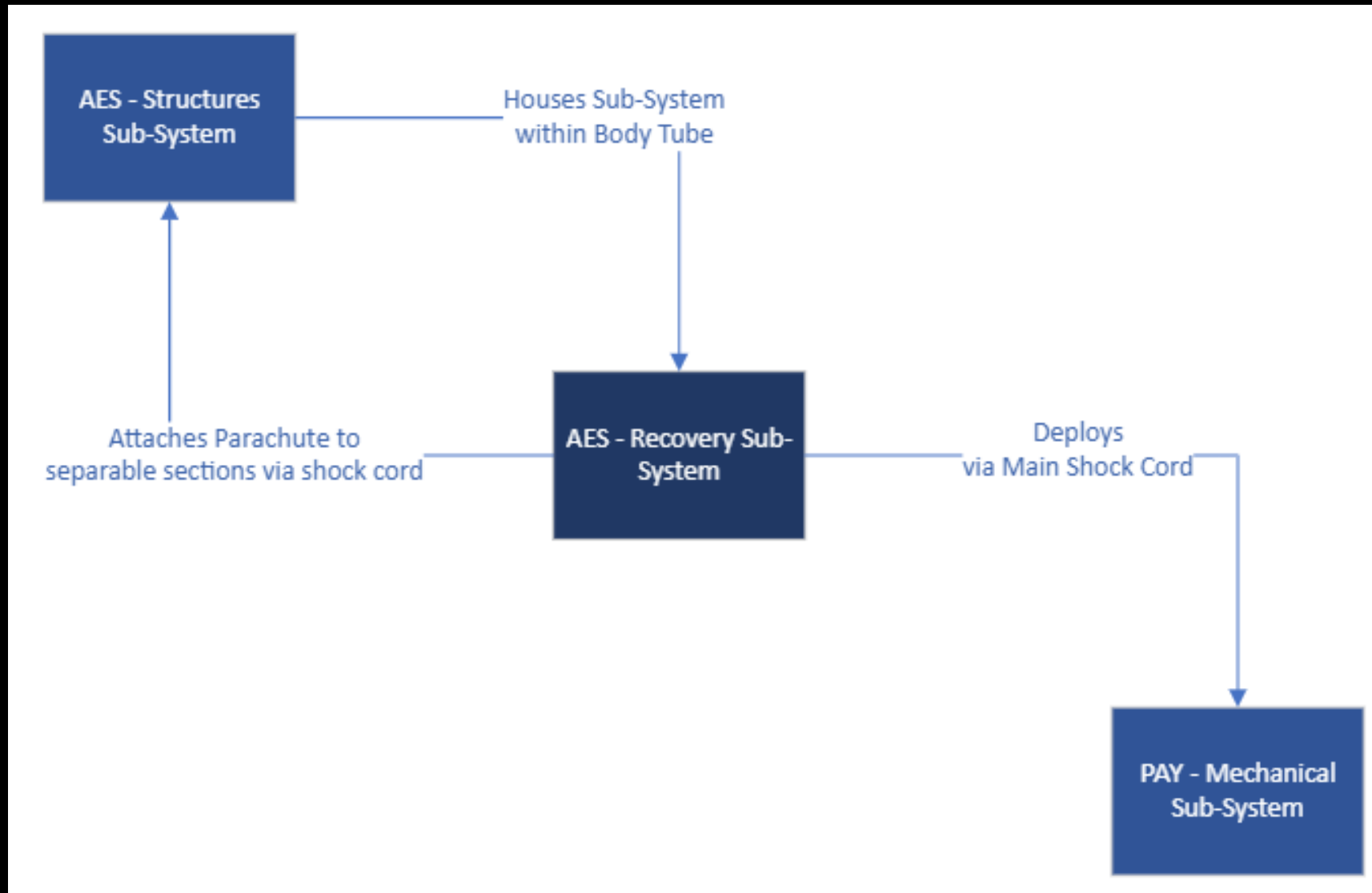
Questions?



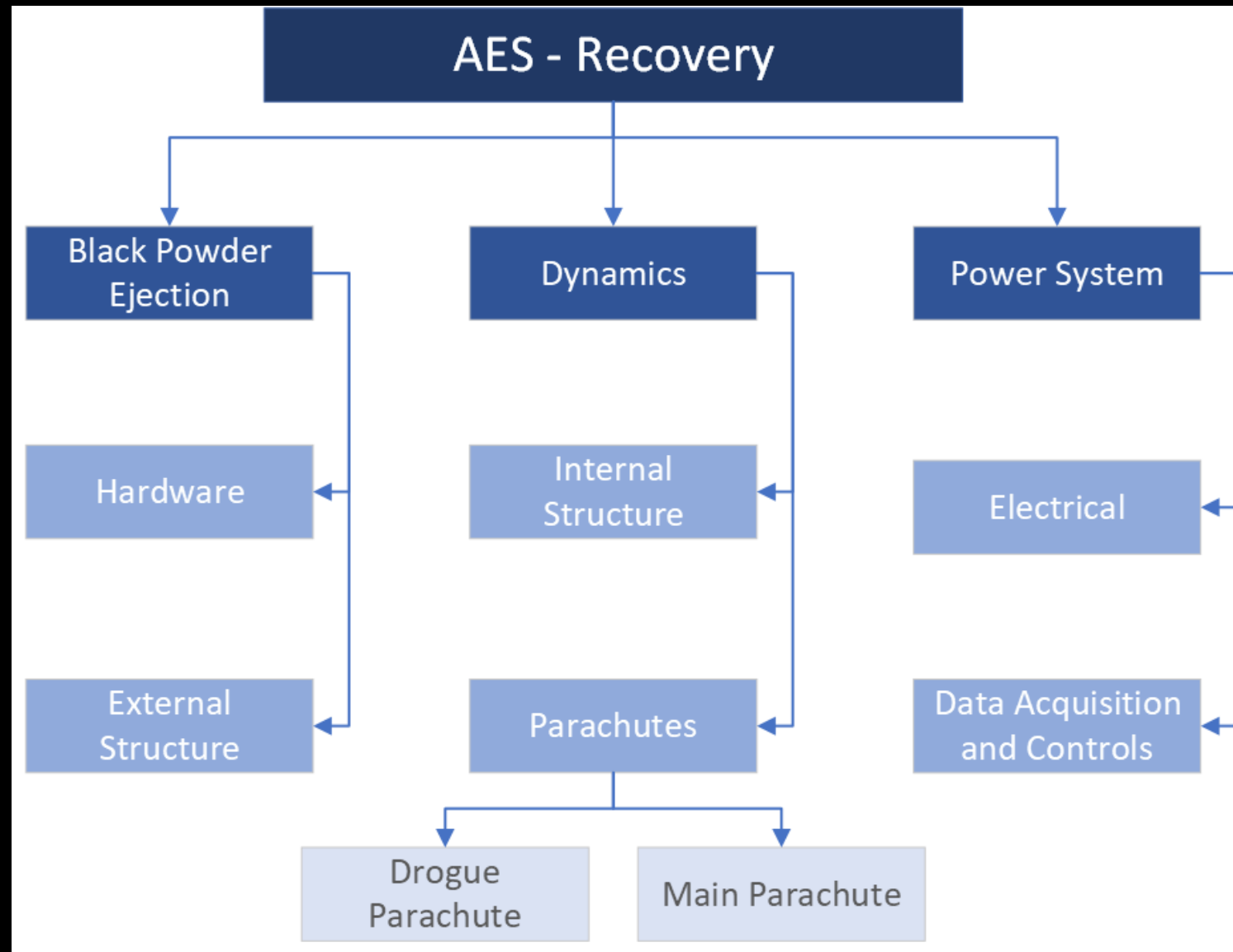
Recovery Requirements

| Requirement | Verification Method |
|---|---------------------|
| The Recovery Sub-System shall safely recover the rocket. | Demonstration |
| The Recovery Sub-System shall be reusable. | Demonstration |
| The Recovery Sub-System shall be recovered in reusable condition. | Demonstration |
| The Recovery Sub-System shall implement adequate protection. | Demonstration |
| The Recovery Sub-System shall fully ground and flight tested. | Demonstration |
| The Recovery Sub-System shall activate the Primary Main Deployment Charges at [1000 ft] AGL. | Test |
| The Recovery Sub-System shall activate Redundant Main Deployment Charges [2] seconds after Primary Main Charges are activated. | Test |
| The Recovery Sub-System shall activate Primary Drogue Charges at apogee. | Test |
| The Recovery Sub-System shall activate Redundant Drogue Deployment charges [2] seconds after Primary Drogue Charges are activated. | Test |
| The Recovery Sub-System shall successfully separate Body Tubes within [1] second of activation. | Test |

Recovery Interface Diagram



Recovery Component Breakdown

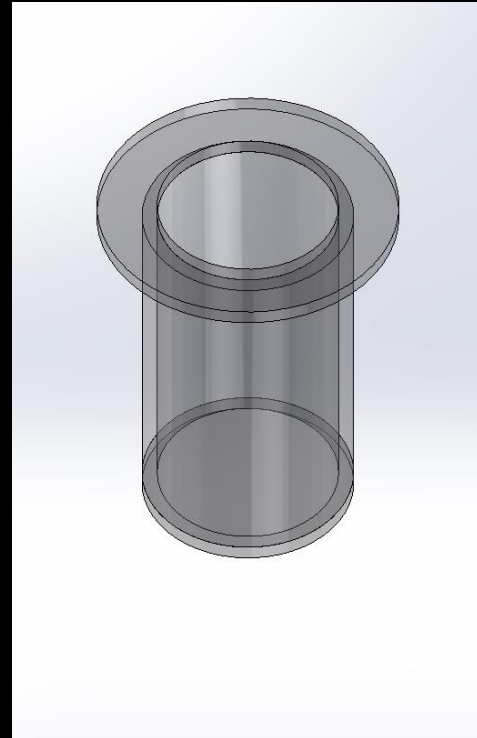


Black Powder Ejection

- Two Methods of Ejection:
 - Drogue – Charge Cannon
 - Main – Charge Well

Methodology of the Charge Cannon

- The Black Powder is ignited and then burns in the barrel. Due to the confined space in the barrel, it burns better without losing to much of its energy to generate the pressure required to shear the pins at high altitudes.
- Charge wells will be designed to fit 1.5x the amount of Black powder required to ensure there's space in the event more is needed.



Recovery TPMs

| Measure | TPM Value | Units | Verification Method |
|-------------------------------------|-----------|-----------------|---------------------|
| Maximum Compressive Force | [TBD] | lbs. | Analysis |
| Average Snatch Force (Main) | [928.4] | lbs. | Analysis/Calculator |
| Upper Bound Snatch Force (Main) | [1406] | lbs | Analysis/Calculator |
| Lower Bound Snatch Force (Main) | [492.3] | lbs | Analysis/Calculator |
| Volume of Recovery Chamber | [290.13] | in ³ | Inspection |
| Drogue Descent Rate (At deployment) | [92] | ft/s | Test |
| Main Descent Rate | [21] | ft/s | Test |
| Drogue Black Powder Mass | [20] | grams | Test |
| Main Black Powder Mass | [35] | grams | Test |
| Drogue Shear Pins | [10] | 6-32 | Inspection |
| Main Shear Pins | [8] | 10-32 | Inspection |

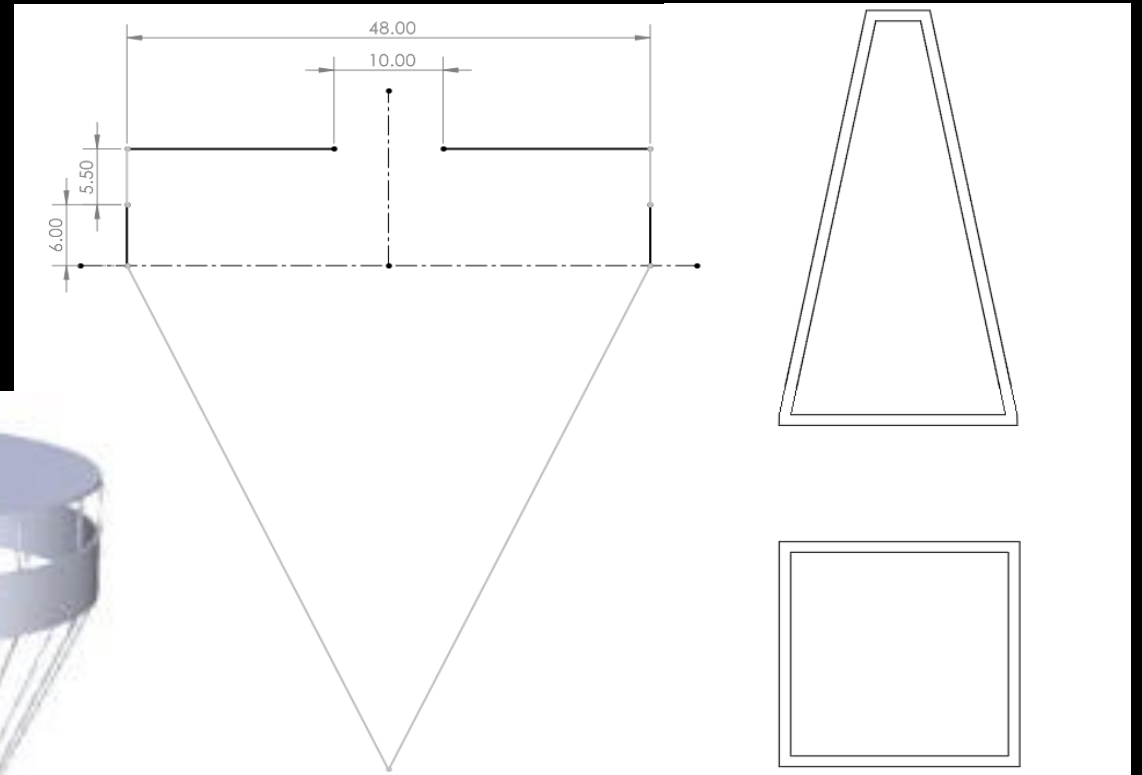
Black Powder Testing Plan

- Charge Cannons Testing Plan
 - High Performance Filament will be used
 - Use test results to fulfill our TPMs while keeping space to a minimal
- Charge wells test will occur during our scheduled ground test
 - The data acquired will be used to update the Black Powder amount needed
- Vacuum test will be used to test if altimeters are triggering at their respective altitudes
 - This will be conducted by removing the air inside the rocket. A barometer will be inside the rocket to verify results



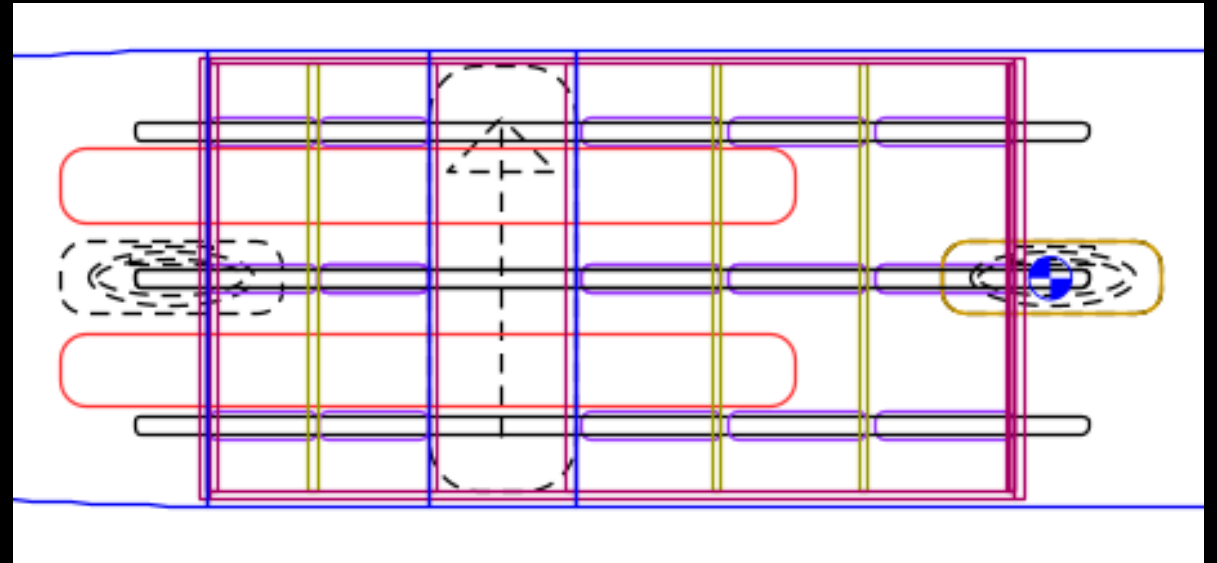
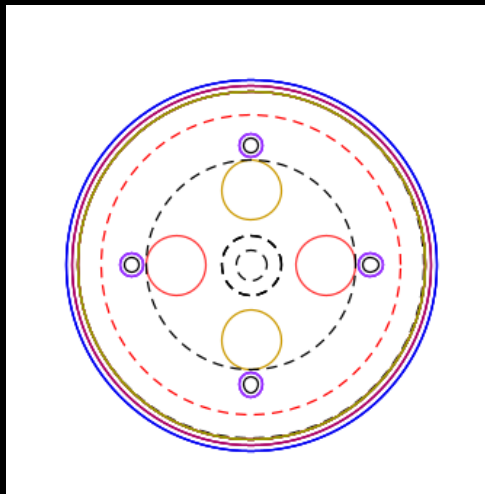
Drogue Parachute

- Slows the rocket to 115ft/s
- Generates a snatch force of 135lb at deployment
- 48" Outer Diameter
- 10" Vent Hole
- 55" Shroud Line
- Drag Coefficient – 0.6



Recovery Coupler Diagram

- Charge Cannons are pictured in red and are the longest cylinders to the left
- All internal components are kept in place via PVC pipes. They are pictured in purple
- Main Charge wells are pictured to the right in orange. To make room, they extend outwards

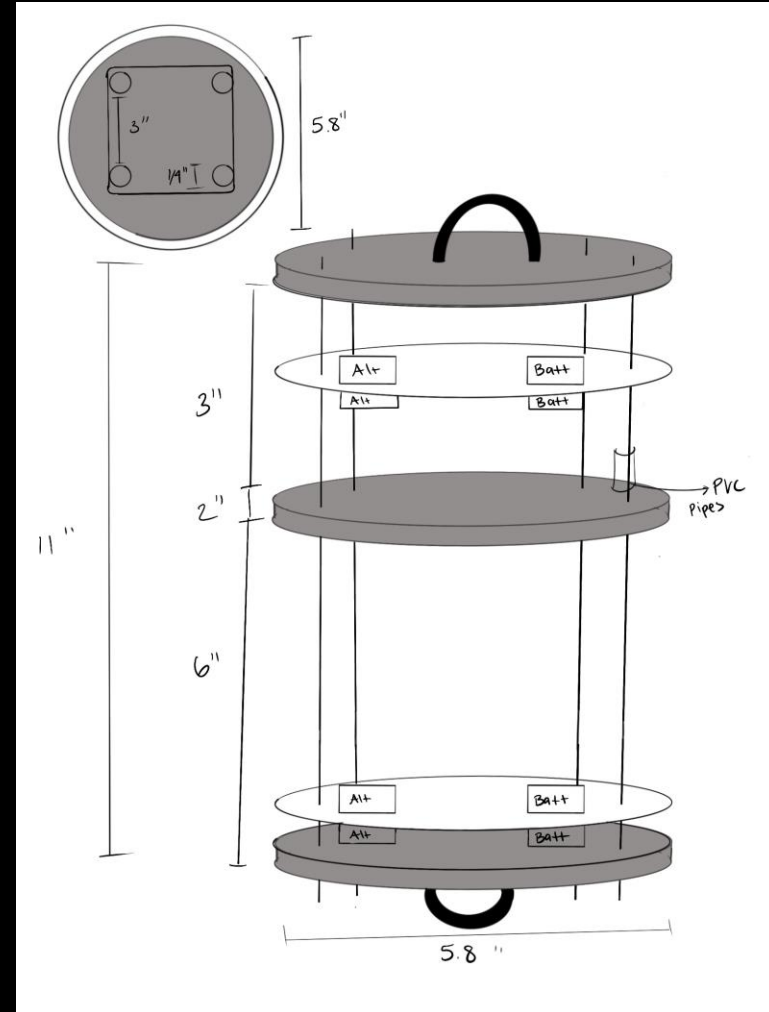
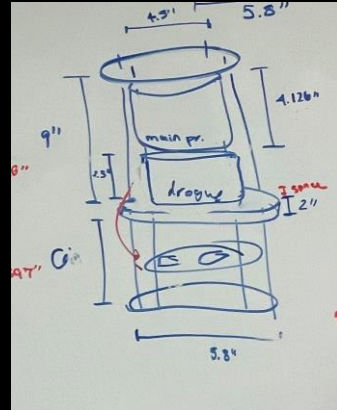
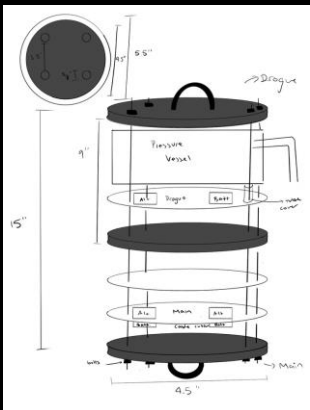


Internal Structures

Multiple designs were considered, but ultimately, we chose this one for space reasons, after deciding to go with black powder. -->

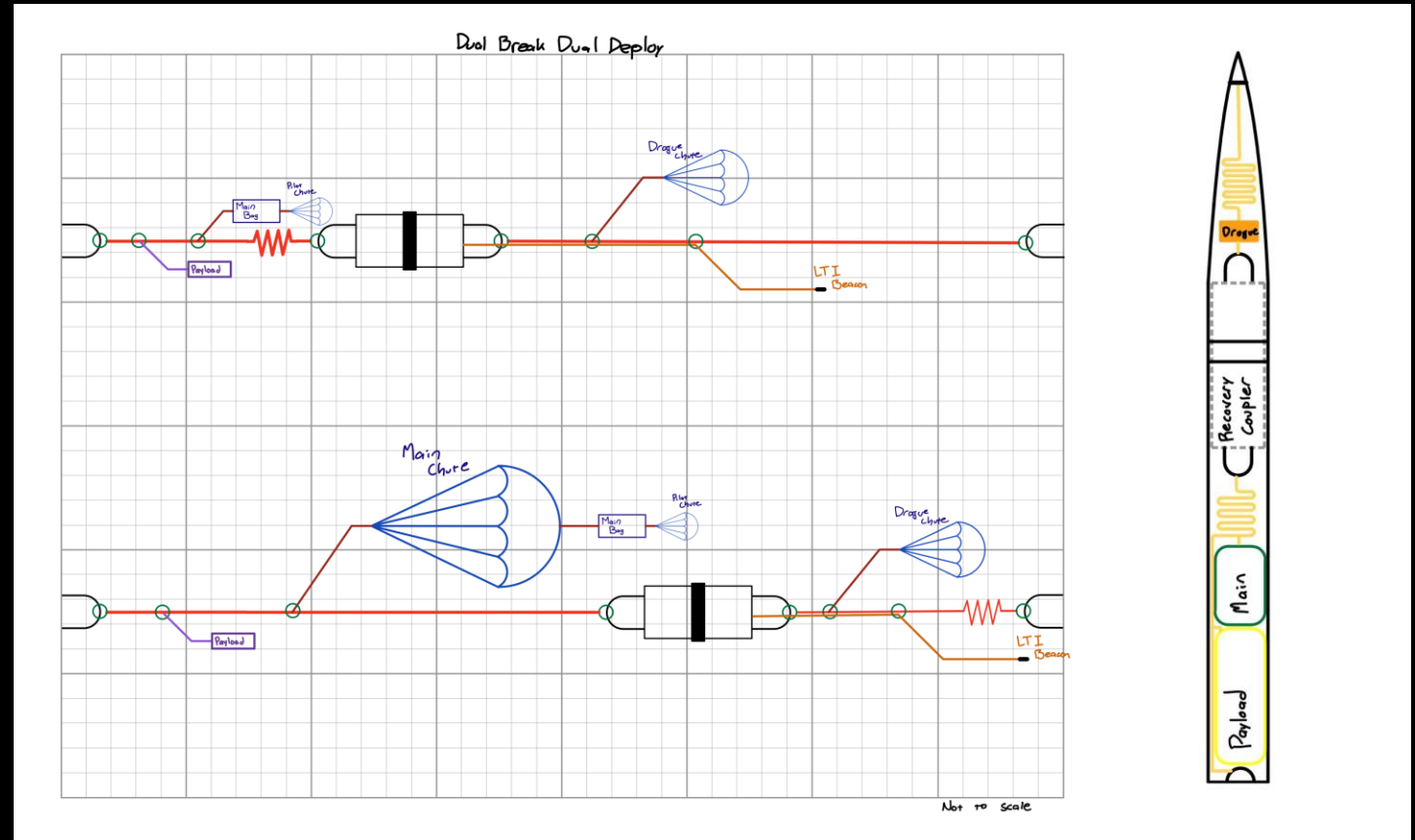
Previous Designs:

When we were doing co2 ejections these were the designs considered:



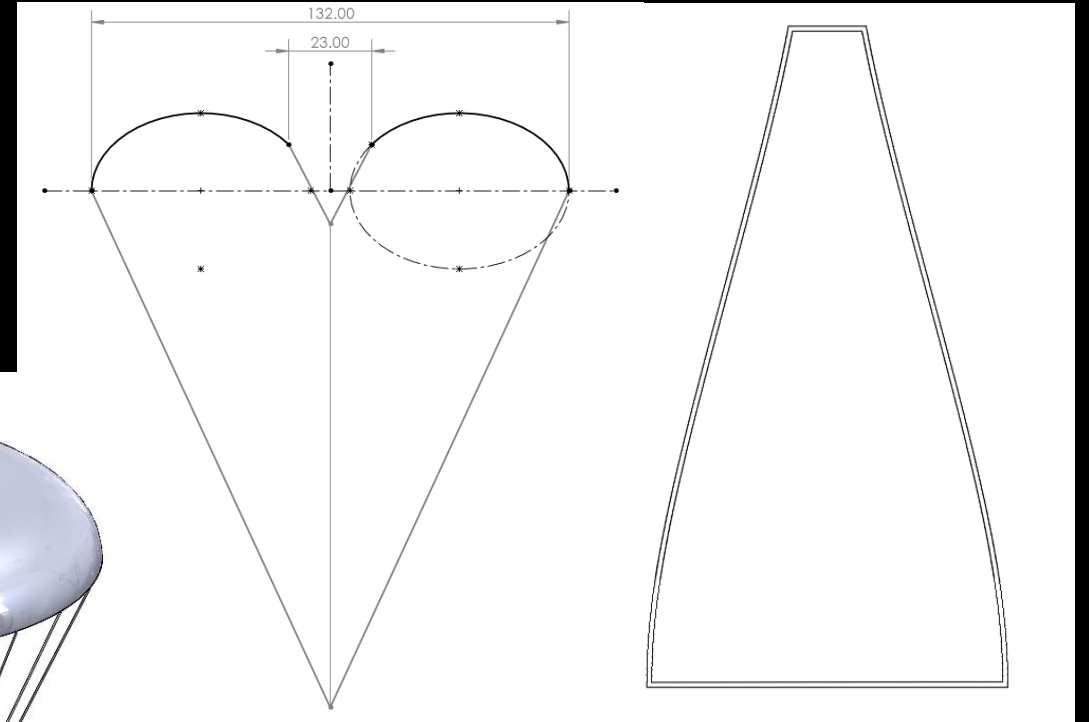
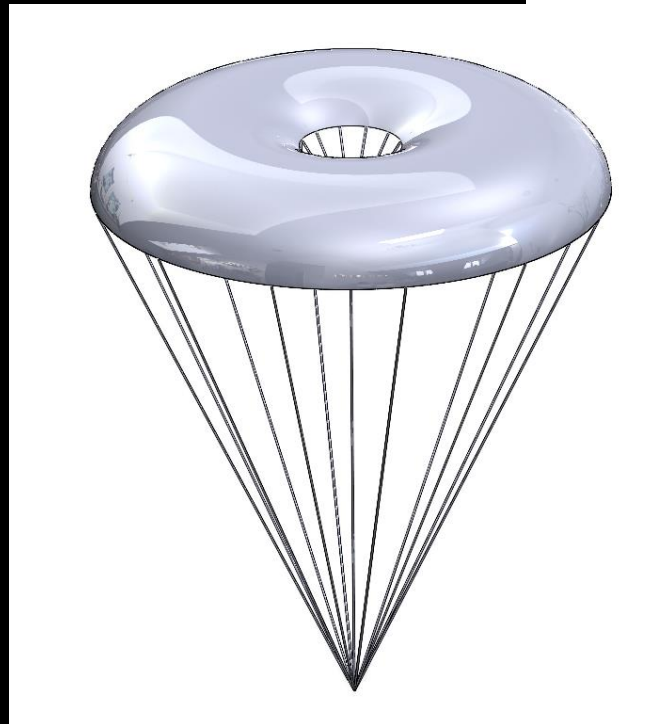
Parachutes Overview

- Drogue parachute deploys from nosecone at 30,000 ft
- Main parachute deploys from body tube at 1,500 ft with payload
- SRAD 48" Disk-Gap-Band Drogue Parachute
- SRAD 132" Toroidal Main Parachute
 - Custom deployment bag and pilot chute
- Kevlar Shock Cords



Main Parachute

- Slows the rocket to its touchdown velocity of 21ft/s
- Generates a snatch force of 1,400lb at deployment
- 132" Outer Diameter
- 27" Vent Hole
- 156" Shroud Line
- Drag Coefficient – 2.2



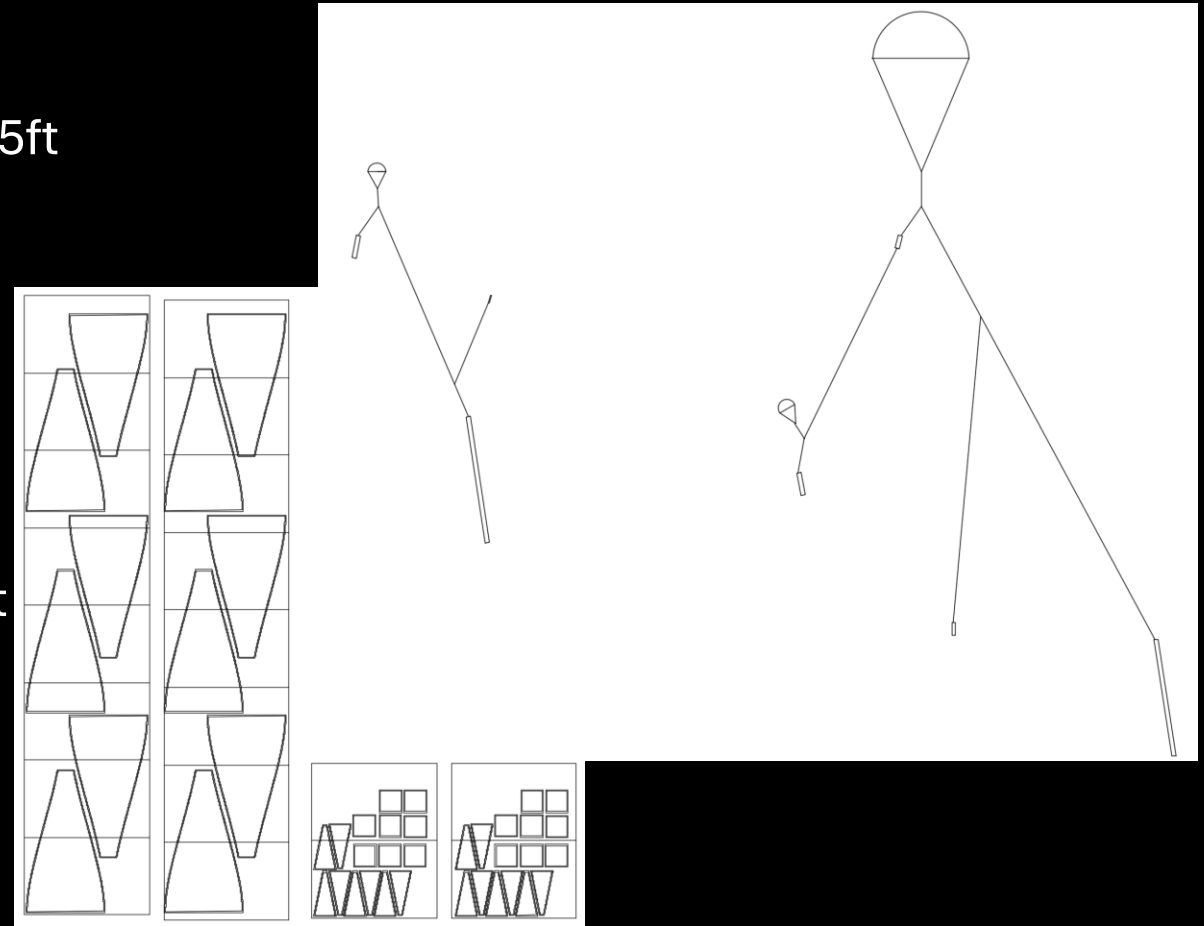
Parachute Testing Plan

- Manufacture drogue parachute and subscale main parachute
- Test parachutes to validate drag coefficient values (C_d)
 - Either drop test and measure descent rate, or wind tunnel test and measure drag force
- Use test results to update main parachute geometry



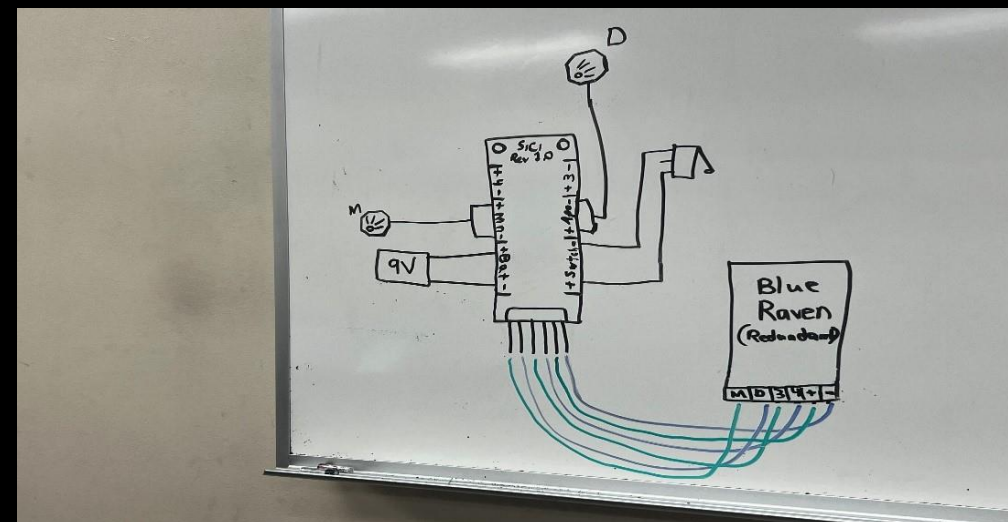
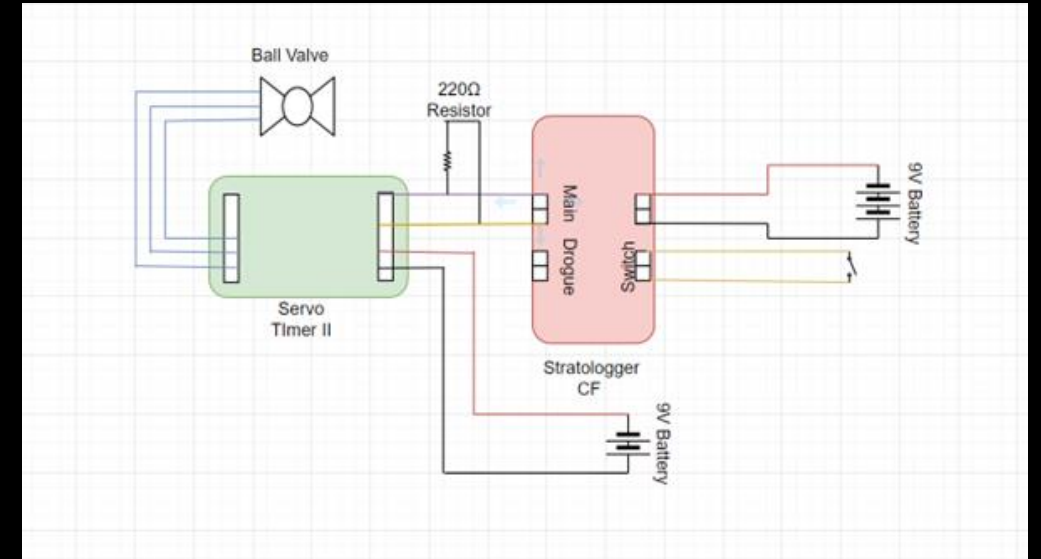
Parachute Materials

- Shock Cord
 - Drogue Chute Line – 3/8" tubular Kevlar, 45ft
 - Main Chute Line – 1/2" tubular Kevlar, 85ft
- Canopy
 - Drogue, Main, and Pilot Canopy – 1.1oz Ripstop Nylon, 24yd
- Shroud Line
 - Drogue Line – 275lb Nylon Paracord, 200ft
 - Main Line – 750lb Spectra, 180ft



Electrical and Controls

- Primary Altimeter: Stratologger CF
- Redundant Altimeter is the Blue Raven by Featherweight
- For the primary altimeter, the servo board is no longer needed as the e-matches will be wired directly to the altimeter.
- Redundant altimeter does require an interface board due to its complex wiring interface
- Both altimeters utilizes a 9V battery for power and a limit switch to arm and disarm



Battery Life Calculations

- Blue Raven Altimeter:
 - Power draw will be tested via a drain test
- Stratologger CF Altimeter:
 - 565 mAh / 1.5 mA = 376.6 hours
- Featherweight GPS:
 - Average consumption 400 mAh / 60 ma = 6.66 hours

$$\text{Battery Life} = \frac{\text{Battery Capacity in Milli amps per hour}}{\text{Load Current in Milli Amps per hour}}$$

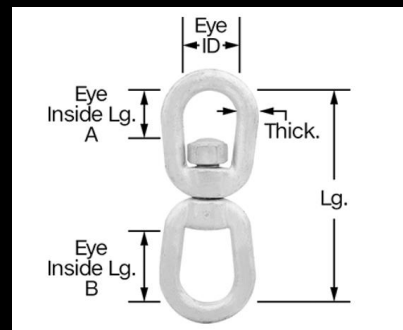
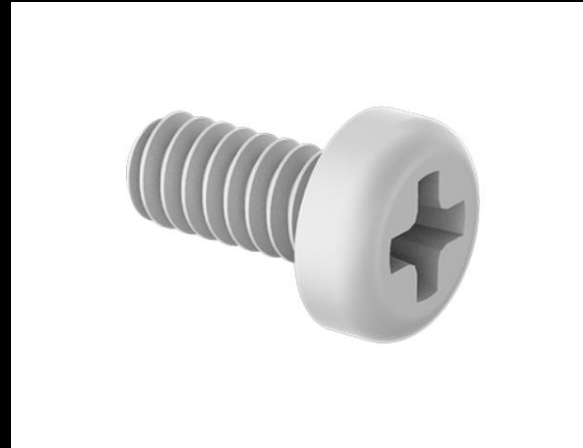
Data Acquisition

- Run-cam 2 is used to collect high-definition videos of our parachute deployment so that we can compare our test opening characteristic and our actual
 - It also enables us to get some cool shots of our deployment for promo videos
- Featherweight GPS will be our primary tracker on locating the rocket. We will have 2 SRAD GPSs (Beacon and ASM)
- Blue Raven will be acquiring the following data
 - Horizontal velocity at apogee
 - Tilt losses during motor burn
 - Max drag acceleration during coast
 - Roll rate and angle



Recovery Hardware

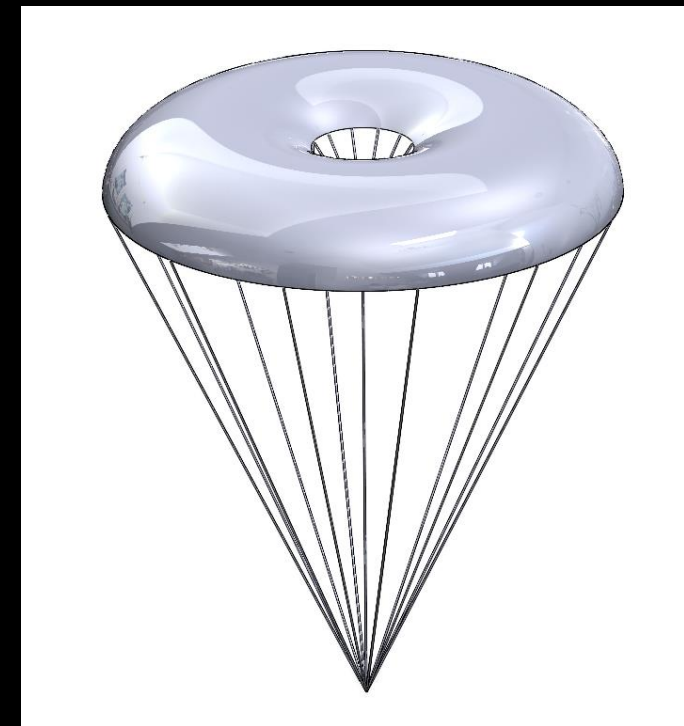
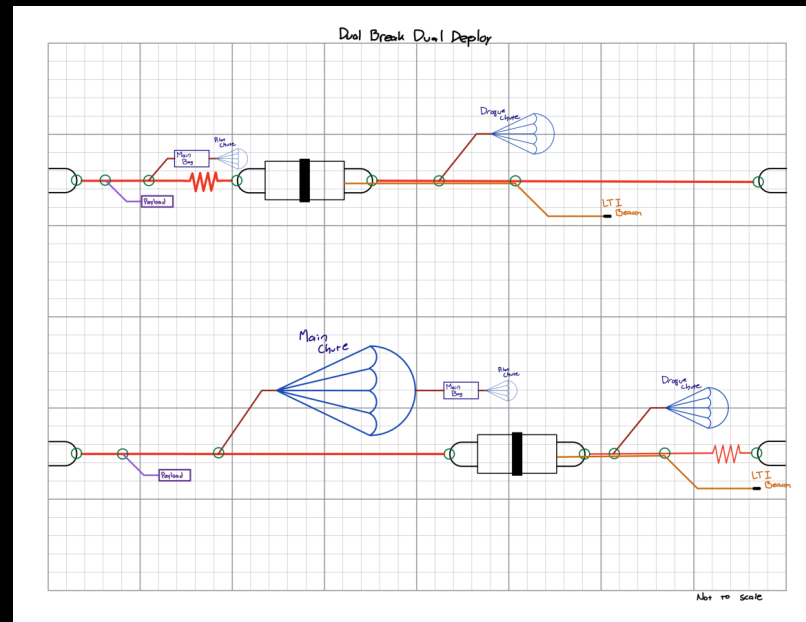
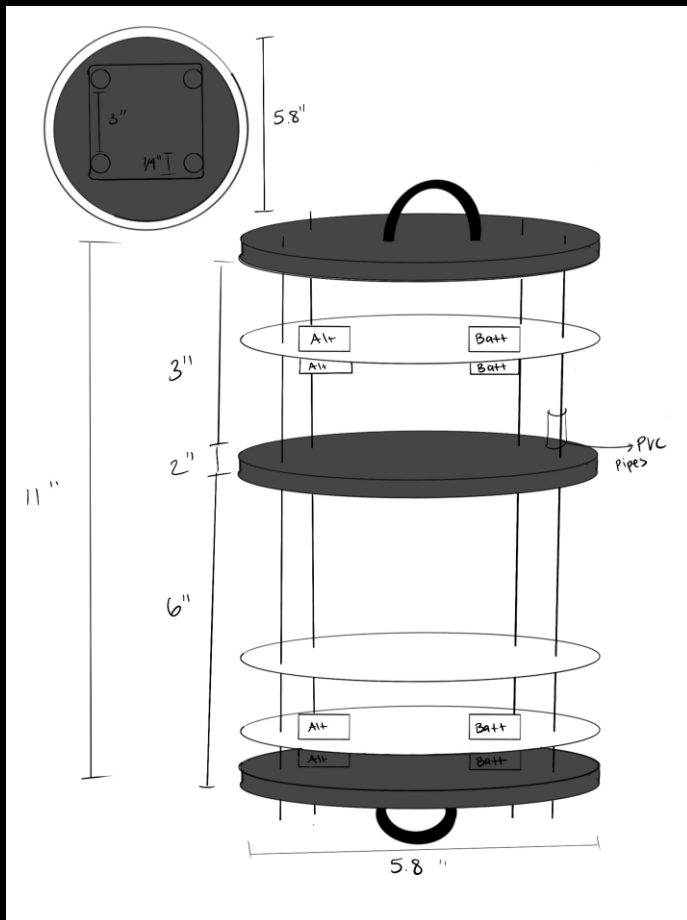
- Main Chute Eyebolt is rated for 3000lbs
- Drogue Eyebolt is rated for 1300lbs
- Drogue shear pins: 6-32 pins (10)
- Main shear pins: 10-32 pins (8)
- Drogue Swivel link rated for 600lbs
- Main Swivel link rated for 1900lbs
- Quick links are rated for 900 lbs
- All hardware are to spec with a margin of safety



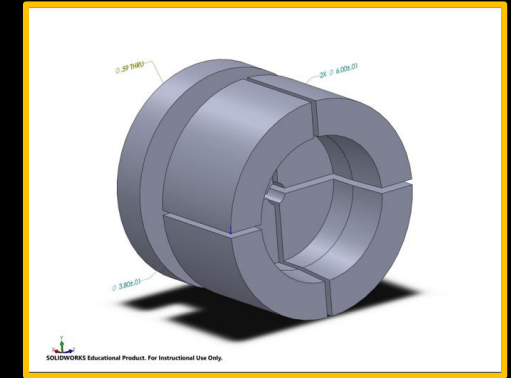
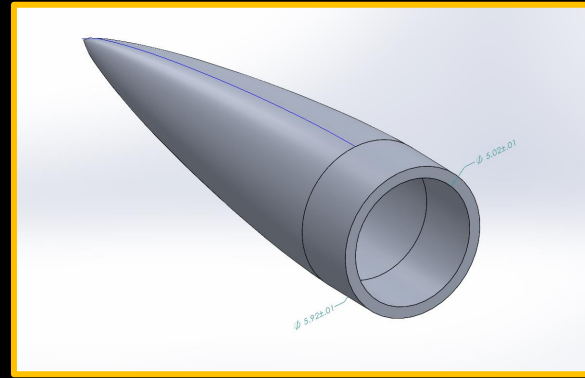
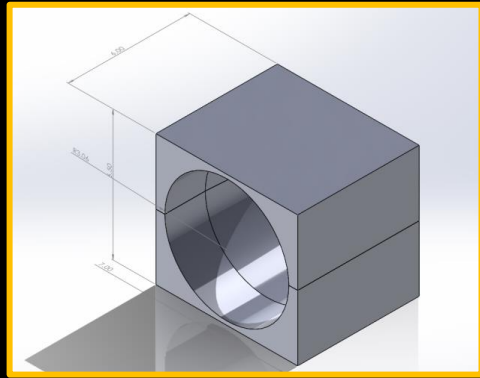
Risks & Mitigations

| Risk | Mitigation |
|---|--|
| Black Powder falling out of the charge well | Ensure that the lid is completely sealed with a cap and tape |
| Arming sequence | Utilize a Stethoscope to listen to the beeps |
| Altimeter doesn't ignite the black powder | A redundant altimeter will be installed in the coupler to fire off the redundant charges |
| Charges go off after going sonic | All altimeters must have a mach lockout |
| Parachutes getting burned | All parachutes will be surrounded by Nomex fire resistant material to ensure the hot gases doesn't burn it |
| Components don't fit together | Track components through ICDs, communicate with REs with design, and track said changes within change log |

Questions?

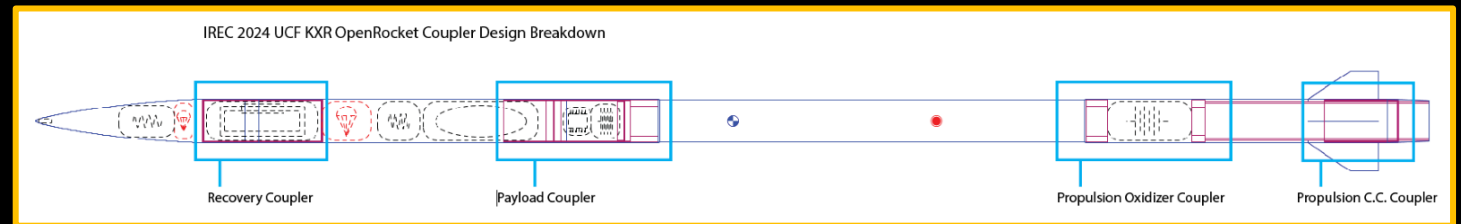
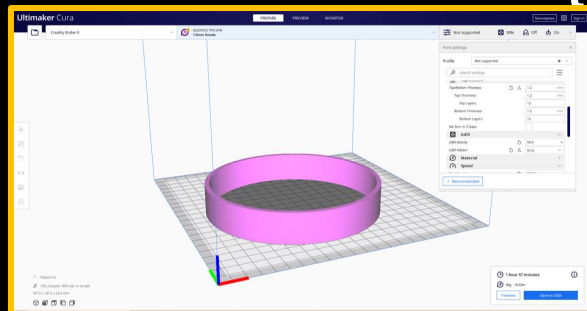


Manufacturing Team



The manufacturing team is responsible for all steps of the engineering process as it relates component fabrication, mold design, material selection, and physical airframe architecture.

We take into consideration the machinability, compatibility, cost, and scheduling when making manufacturing decisions. Our components have been separated amongst designated REs (Responsible Engineers) who are responsible for overseeing and managing the part throughout the manufacturing process.



Manufacturing Requirements

| Requirement | Verification Method |
|--|---------------------|
| The Manufacturing Team shall produce external structure components with minimal surface roughness. | Demonstration |
| The Manufacturing Team shall produce tubular components that are sufficiently round within [0.005] inches. | Inspection |
| The Manufacturing Team shall create manufacturing procedures and verify said procedures prior to manufacturing flight articles. | Inspection |
| Requirement | Verification Method |
| All Molds shall be able to withstand a vacuum pressure of [1] ATM. | Demonstration |
| All Molds shall be able to withstand the [300 Degrees Fahrenheit] temperature of the autoclave. | Demonstration |
| All Molds should be non-reactive with acetone. | Demonstration |
| All Molds shall be chemically resistant to bonding with chosen matrix material. | Demonstration |
| All Molds shall be easily reproducible for means of reusability. | Demonstration |
| All Molds shall resist bonding with composite fabric. | Demonstration |
| The Mandrel Mount shall safely secure the mandrel during the layup process. | Demonstration |
| The Mandrel Mount shall provide a means to achieve a secure vacuum seal. | Demonstration |
| The Mandrel Mount shall be able to withstand the temperatures of the autoclave. | Demonstration |
| The SRAD Couplers shall have a length of at least [12] inches. | Inspection |
| The SRAD Couplers shall have a diameter of [TBD] inches. | Inspection |
| The SRAD Couplers shall be stiff when joined with Body Tubes. | Demonstration |
| The SRAD Couplers shall have a diametrical tolerance of [-0.003] inches. | Inspection |

Manufacturing Risk Assessment

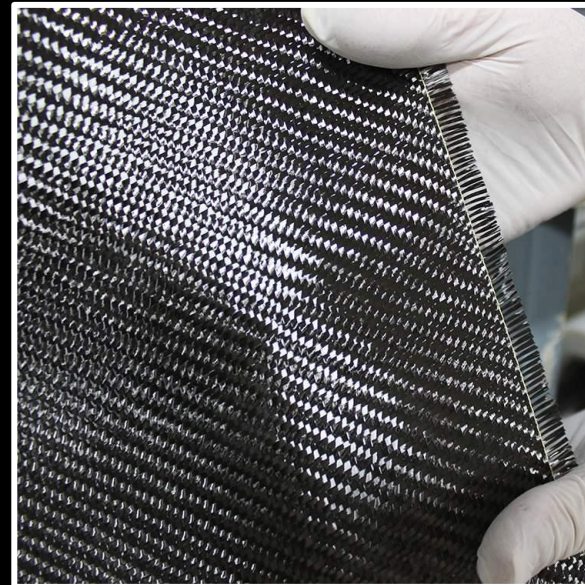
| <u>Risks</u> | <u>Mitigation Strategy</u> |
|---|---|
| Delay in arrival of manufacturing materials | Order materials with longer processing/shipping timeline ahead of schedule |
| Errors during the manufacturing process leading to a scrap of the entire part | Implement preliminary testing period with sub-scale models to ensure familiarity and proficiency with rocket layup technique/principles |
| Collapsing occurring during the 3D fabrication process | Perform excessive calibration of the 3D printer used as well as using practice prints to ensure the final product is optimized |
| Imperfections during the manufacturing process being extrapolated towards the final product | Combine all procedures and cautious practices when dealing with expensive components |
| Human errors in post-processing of components | Implement procedure to ensure that excessive care is utilized when dealing with layup product to prevent scrap |

Airframe Material

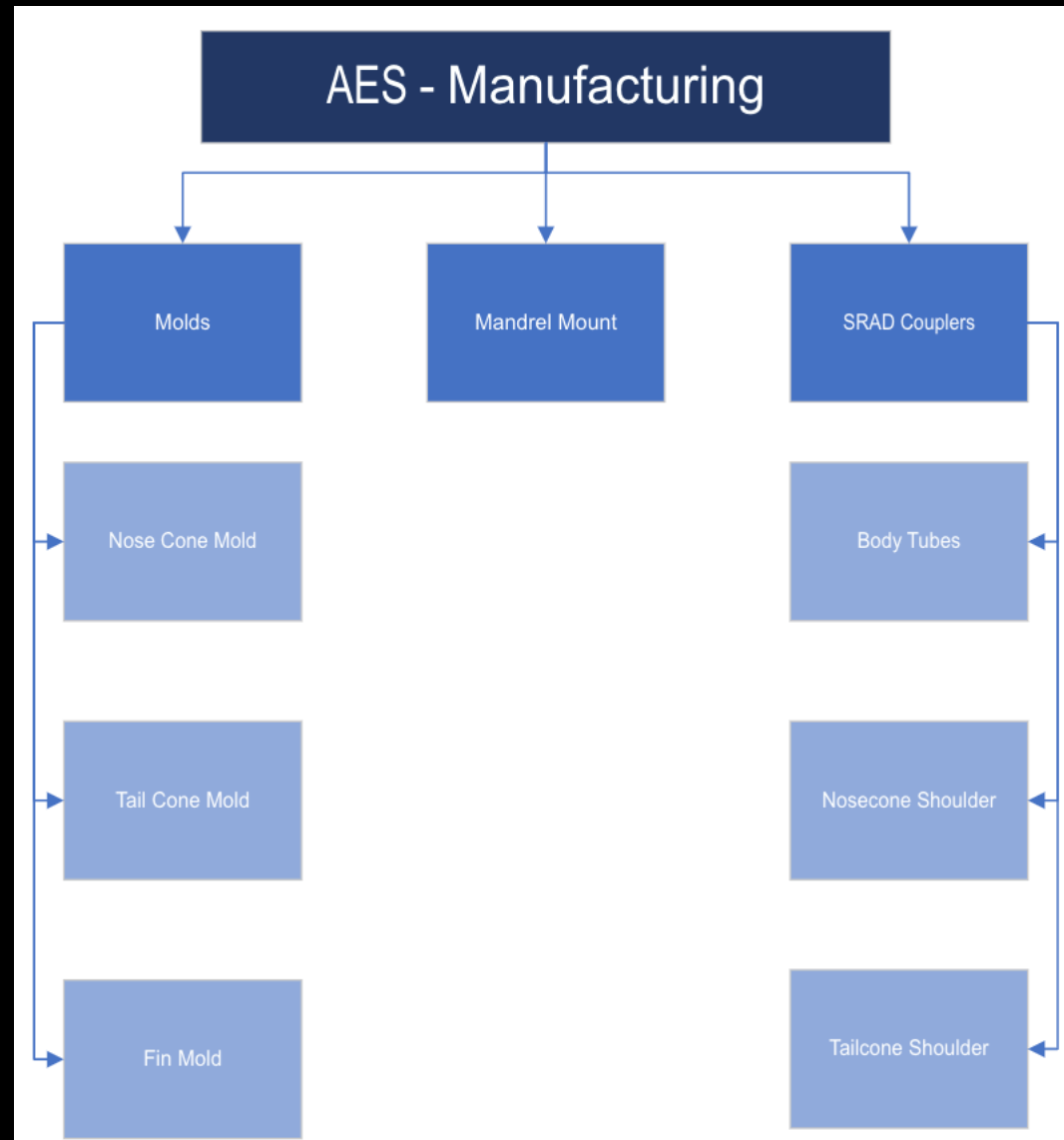


Using a pre-impregnated carbon fiber reinforcement will offer increased material strength properties while keeping the price within our budgetary constraints.

Material Decision: We are going to be utilizing a 3k 2x2 twill biaxial carbon fiber for our rocket's airframe. Initially, we considered using a hybrid composite carbon fiber/fiberglass, but ultimately decided that the increased costs and weight-to-thickness ratio were significant constraints that convinced us otherwise.



Manufacturing Component Breakdown



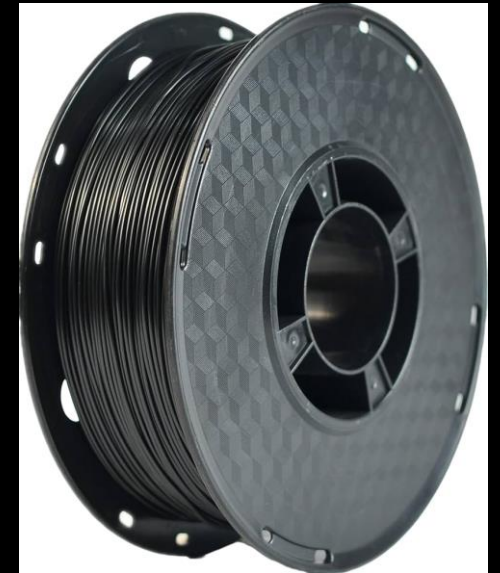
Mold Material

| | TPU | ABS |
|-------------------------------|------------|------------|
| Temperature Resistance | 250-300° F | 200-250° F |
| Price | \$20/Kg | \$30/Kg |

Material Decision: TPU due to its advantageous thermal capacity, ease of printing, and price difference.

| Female Mold | Male Mold |
|---|------------------------------------|
| <i>No experience/familiarity with process</i> | <i>More experience within KXR</i> |
| <i>Better surface finish before PP</i> | <i>Vacuum bag compatibility</i> |
| <i>High separation difficulty post-cure</i> | <i>Easier separation post-cure</i> |

Mold Decision: Male mold due to the team's experience and practicality. The surface finish benefit from a female mold does not warrant the extra difficulty in post processing.



Nose Cone Fabrication

Material: TPU (Thermoplastic polyurethane)

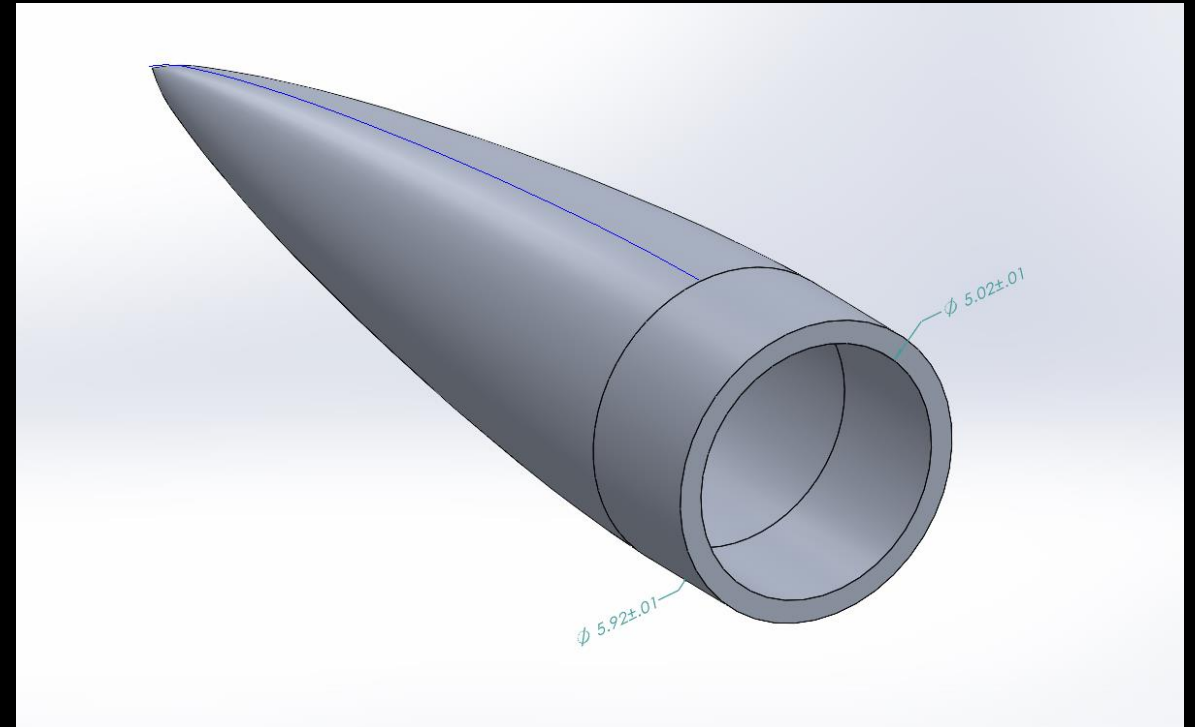
- Heat Resistance – 250-300F
- Pressure Resistance – 14.7 psi

Sub-scale Model

- Can be made simply by adjusting dimensions in CAD
- Will give new engineers experience for lay-up

Reasoning for male-mold selection

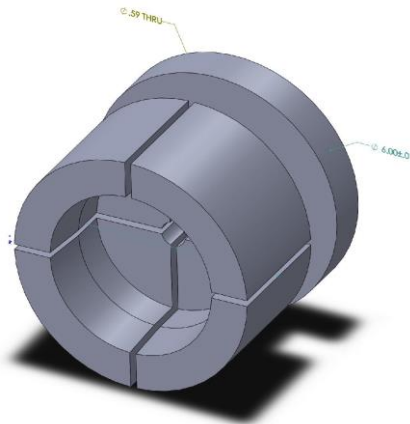
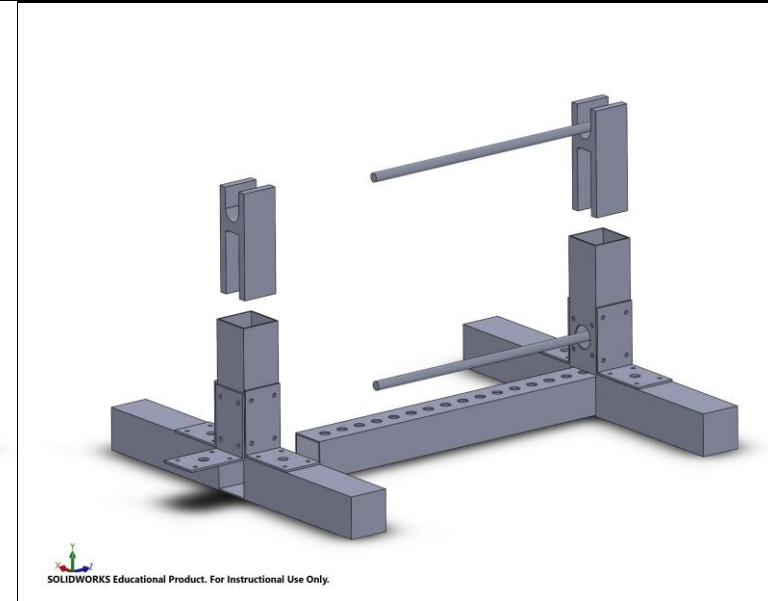
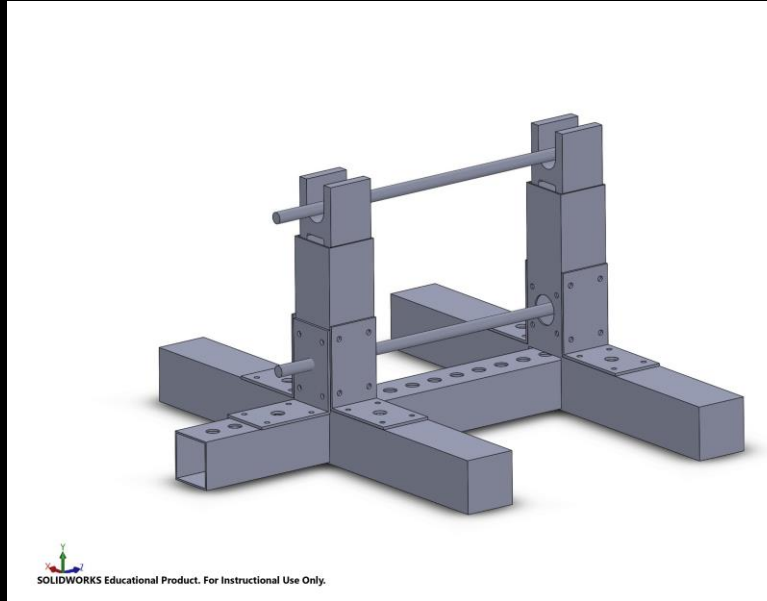
- Increased ease of removability of Nosecone after use.
- Cheaper to 3D-Print than a female mold due to a smaller amount of TPU filament being used.



Mandrel Mounting Method

Goals for the Mandrel Mounting Stand:

- Mandrel Size: 44in
- Responsible for both holding the mandrel up, and creating a smooth body tube product via the rolling pin style system
- This design is a modular system which will allow the team to disassemble the stand to fit within the space of an autoclave.



Expanding Mandrel Plug:

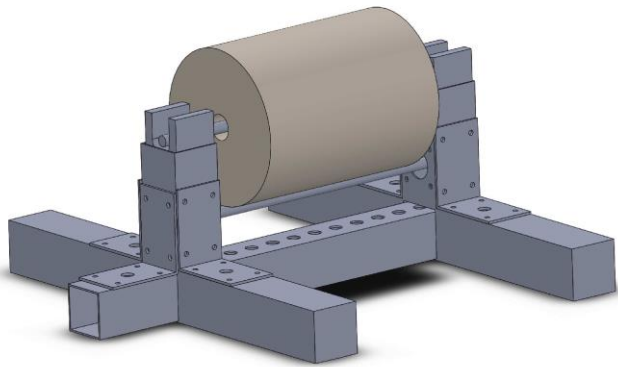
- This is a concept of an expanding mandrel plug; this is the female interface which would accept a wedge into the opening. Secured by a threaded fastener, when tightened, the wedge would cause the plug to expand.
- Material: Steel
- Reason of Refusal: High Cost (est. \$800/ft per 6" steel rod)

-Material:
Steel/

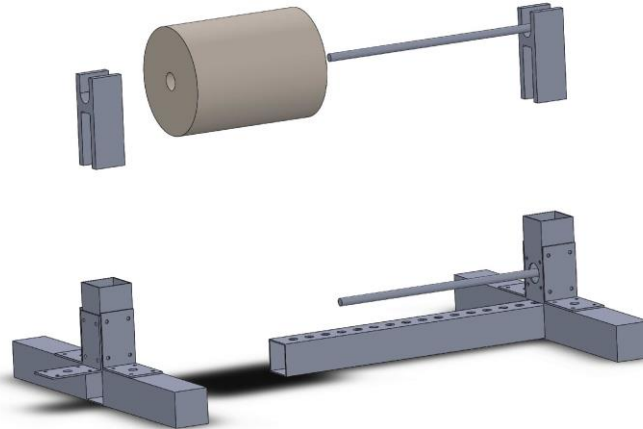
Aluminum

**-Temp
Resistance:**
932-1275 F

Mandrel Mounting Method



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Cost:

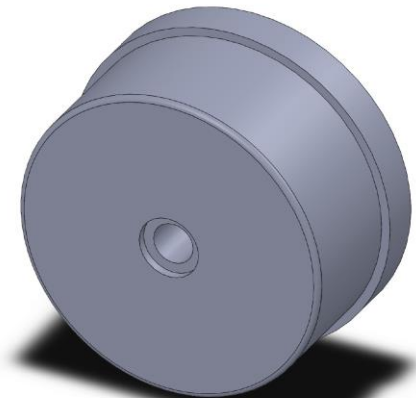
- 2" Square tubing \approx \$8.00/ft
- 1/2" Steel rod \approx \$3.00/ft
- 90° Brackets \approx \$2.00/bracket
- Ultem Filament \approx To be Determined

Summarized Body Tube Layup Procedure

- The Mandrel will be set down into place, then be secured via nuts and washers on either side of the assembly.
- Carbon Fiber Prepreg will then be rolled onto the mandrel.
- Entire assembly will be put into autoclave to be hardened.

Updated Mandrel Plug

- Press fit design.
- Accepts bearings for smooth operation of rolling procedure.
- 3D Printable Design.



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Coupler Fabrication

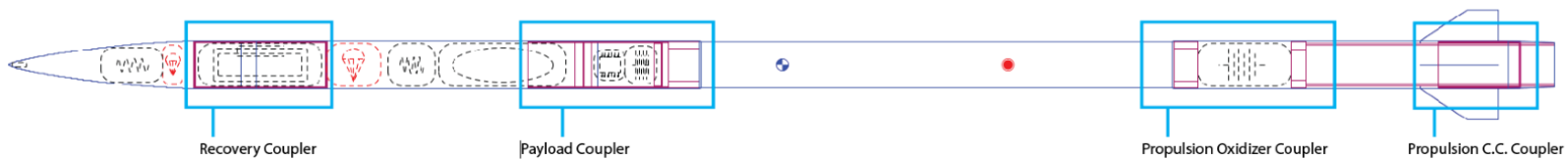
Design Consideration

Material of choice: *Carbon Fiber (same as airframe)*

Fabrication Method: *Pre-impregnation with 24" Steel Mandrel*

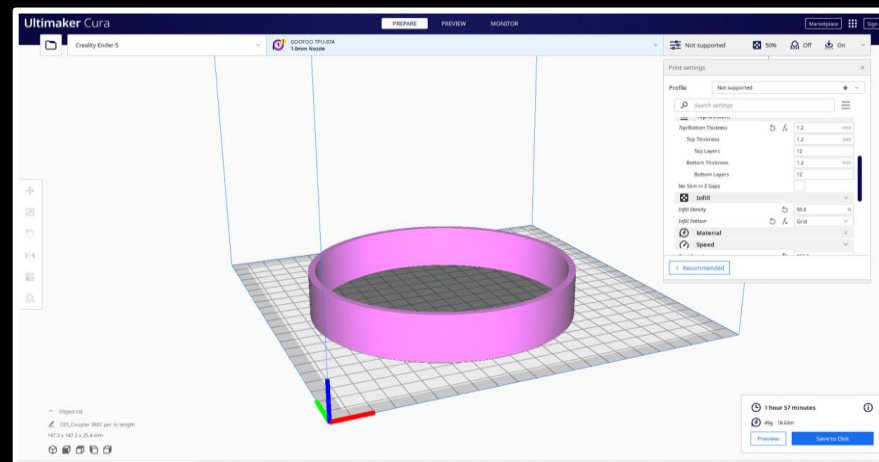
3 – 4 couplers for rocket body

IREC 2024 UCF KXR OpenRocket Coupler Design Breakdown



| Measure | TPM Value | Units | Verification Method |
|-----------------------------------|-----------|-------------|---------------------------------|
| Axial Compressive load | - | Lbs | Numeric calc. / Ansys |
| Bending moment load | - | Lbs | Numeric calc. / Ansys |
| Load Safety Factor | - | F.S./Lbs | Numeric calc. |
| Bending Moment Load Safety Factor | - | F.S./Lbs | Numeric Calc. |
| Parachute shock load | - | Lbs / in/in | Numeric Calc. / OSCALC software |
| Bolt holes shear value | - | Lbs | Numeric Calc. |
| Bolt requirements F.S. base | - | Lbs | Numeric Calc. |

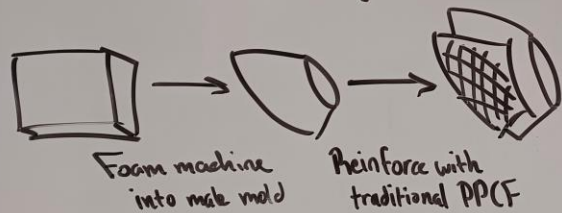
| Requirement | Verification Method |
|--|----------------------------|
| The couplers shall be 1.5 cal into each side of the airframe | Design specification |
| The couplers shall be structurally stable | Ansys/Numeric Calculations |
| The couplers shall include specific subsystem requirements (i.e. access holes) | Design specifications |
| The couplers shall include specific length tolerancing is accounted for from the propulsion-payloads –recovery section | Design Specifications |



Fin Fabrication

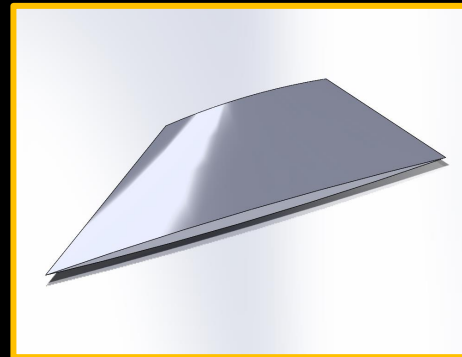
FOAM CORE FINs

- Foam make mold
- easy to machine into airfoil
- familiar layup process
- avoids release film
- easy to bond with CF
- CF provides structural integrity



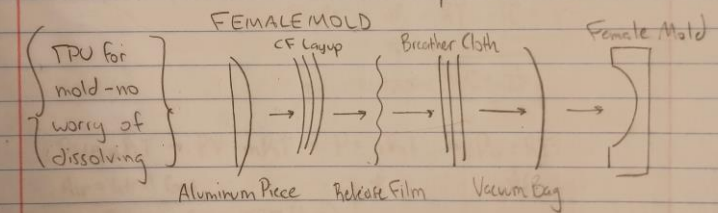
Same material as body tube
~ 3k 2k biaxial ~

Using a foam core fin structure to combine the machinability properties of foam to generate a male mold and the structural integrity provided by the 3k carbon fiber reinforcement.

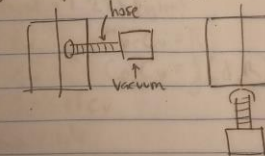


FIN FABRICATION

- AIRFOIL FIN
-
- Aluminum "Skeleton" - structural integrity
 - machined into airfoil
 - brackets?
 -
 - CF Layup - prepreg Traditional CF
 - layed over aluminum
 - autoclave / vacuum seal



System for maintaining seal with female mold



The layup process will be optimized as the foam core is not intended to be separated, meaning the separation agent present in most mold layups will be abandoned to create a bond with the carbon fiber shell. Instead, a 3M High Bond Adhesive will be used to encourage binding.

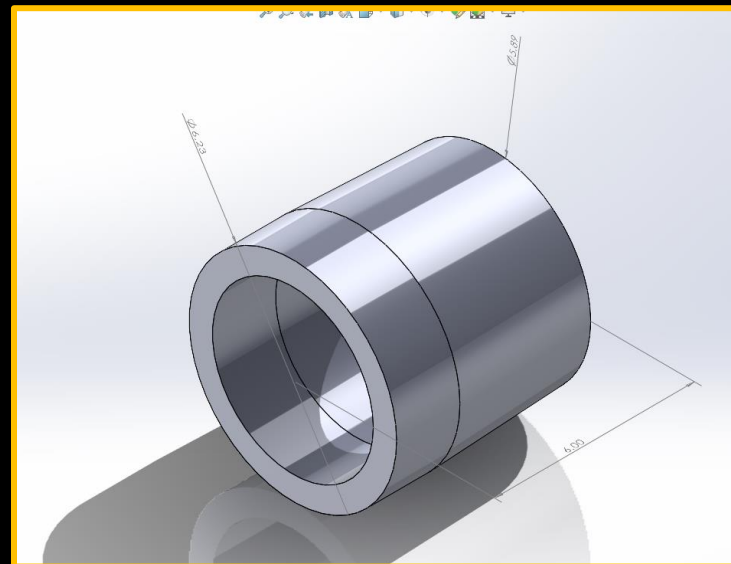
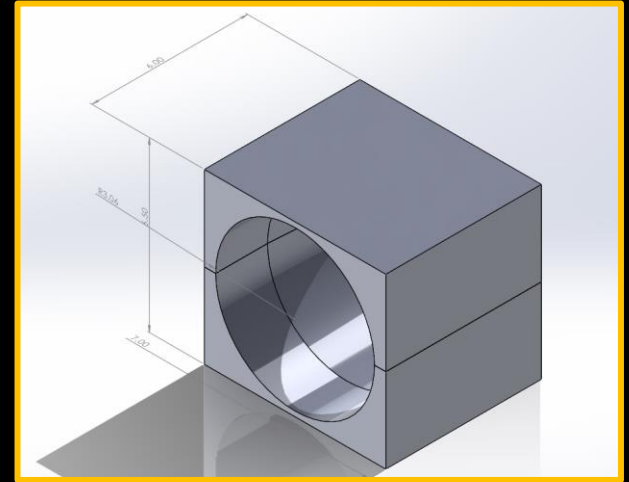
Tail Cone Fabrication

Mold Requirements:

- Withstand curing temperature of 250° Fahrenheit and a pressure of 14.7 Psi (1 atm) inside of the autoclave
- Easily reusable with minimal processing
- Doesn't bond to the composite

Design Considerations:

- Material choice for the mold
- Type of mold (Female/Male)
- Sizing and Structuring



Schedule

Material Order Date – Order materials

Estimated Lead time – 1-2 weeks

Date – Materials arrive

[2 days] - 3d printer subscale printing

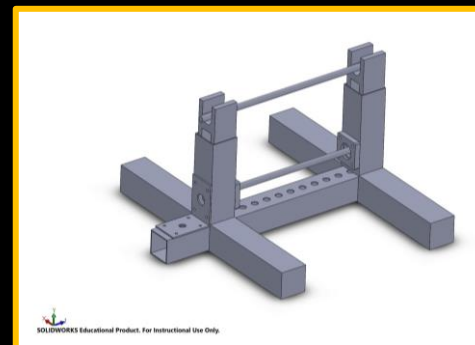
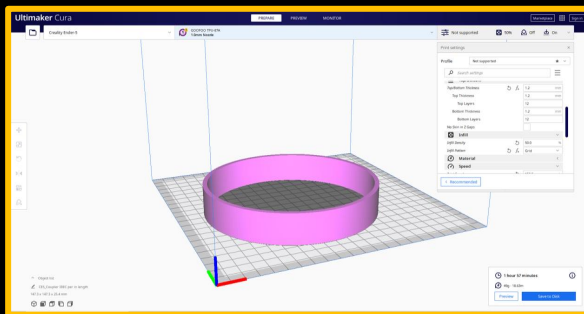
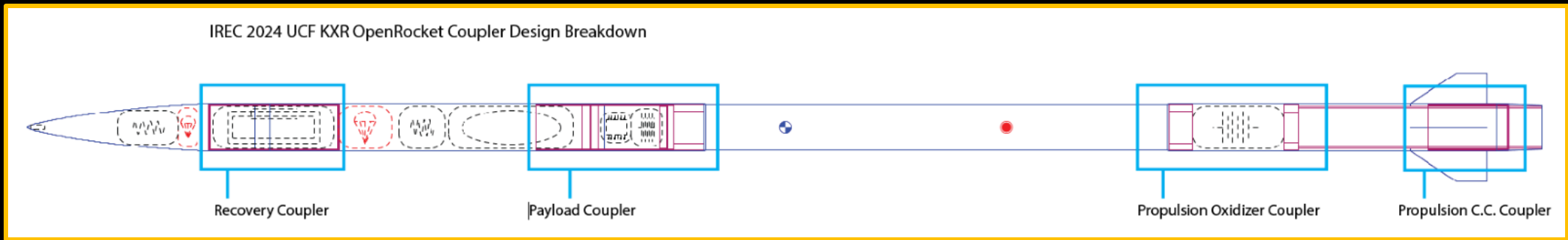
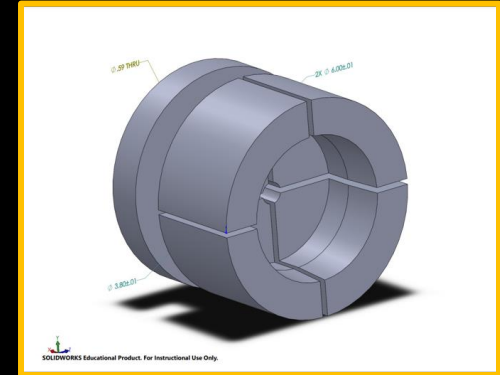
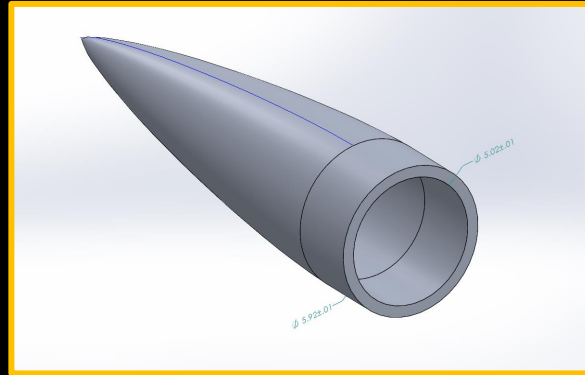
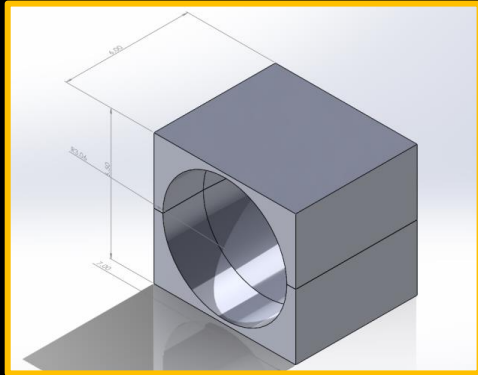
[2-3 days] - full scale prototype printing

[2-3 days] - autoclave testing/adjusting thickness

[3 days] - flight layup + curing process

[1 day] - post processing and flight ready product

Questions?





Propulsion System Preliminary Design Review

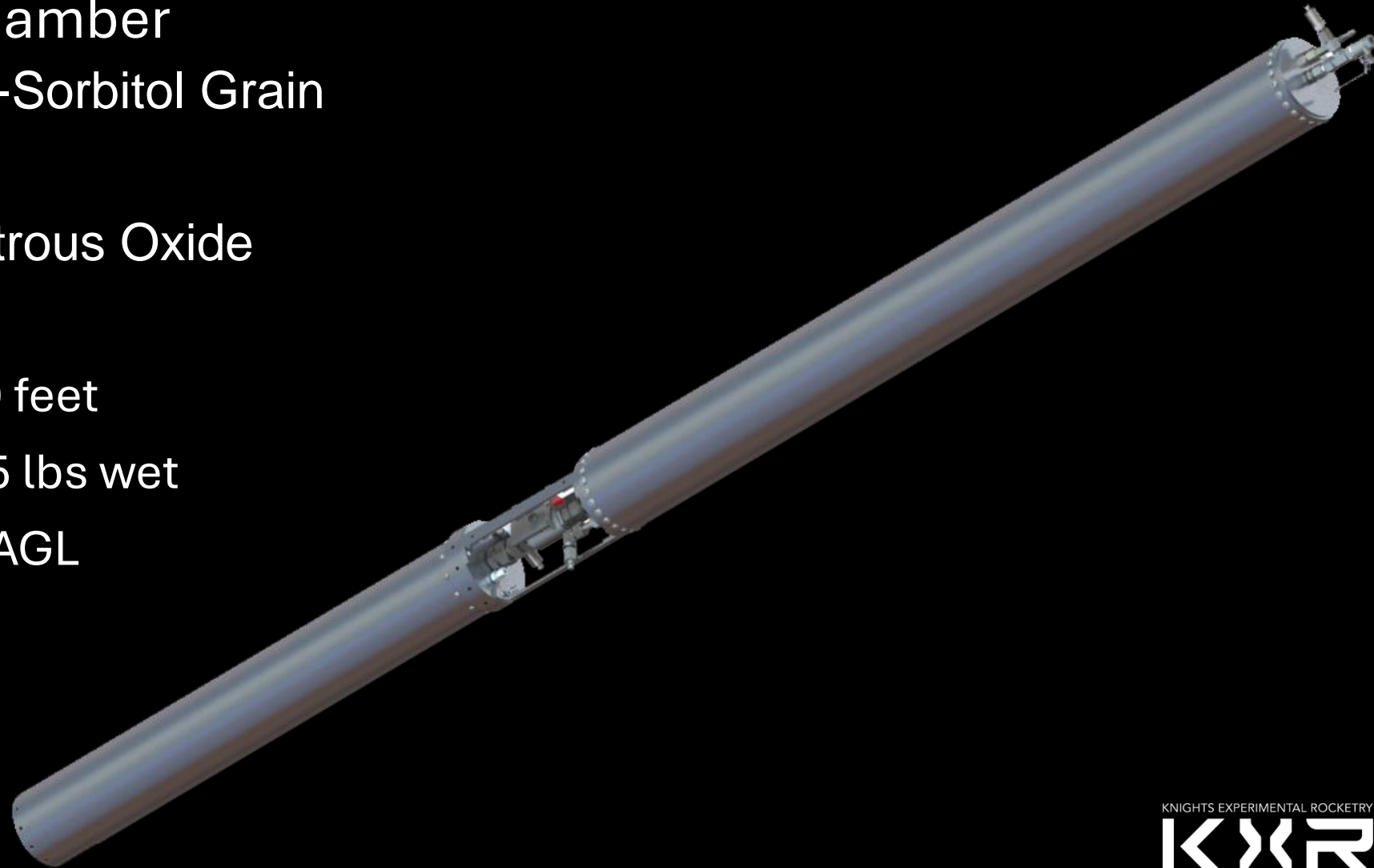
Spaceport America Cup 2024

IREC 30k SRAD Hybrid Engine, Experimental Payloads, and Airframe

10/30/23

Propulsion System Architecture

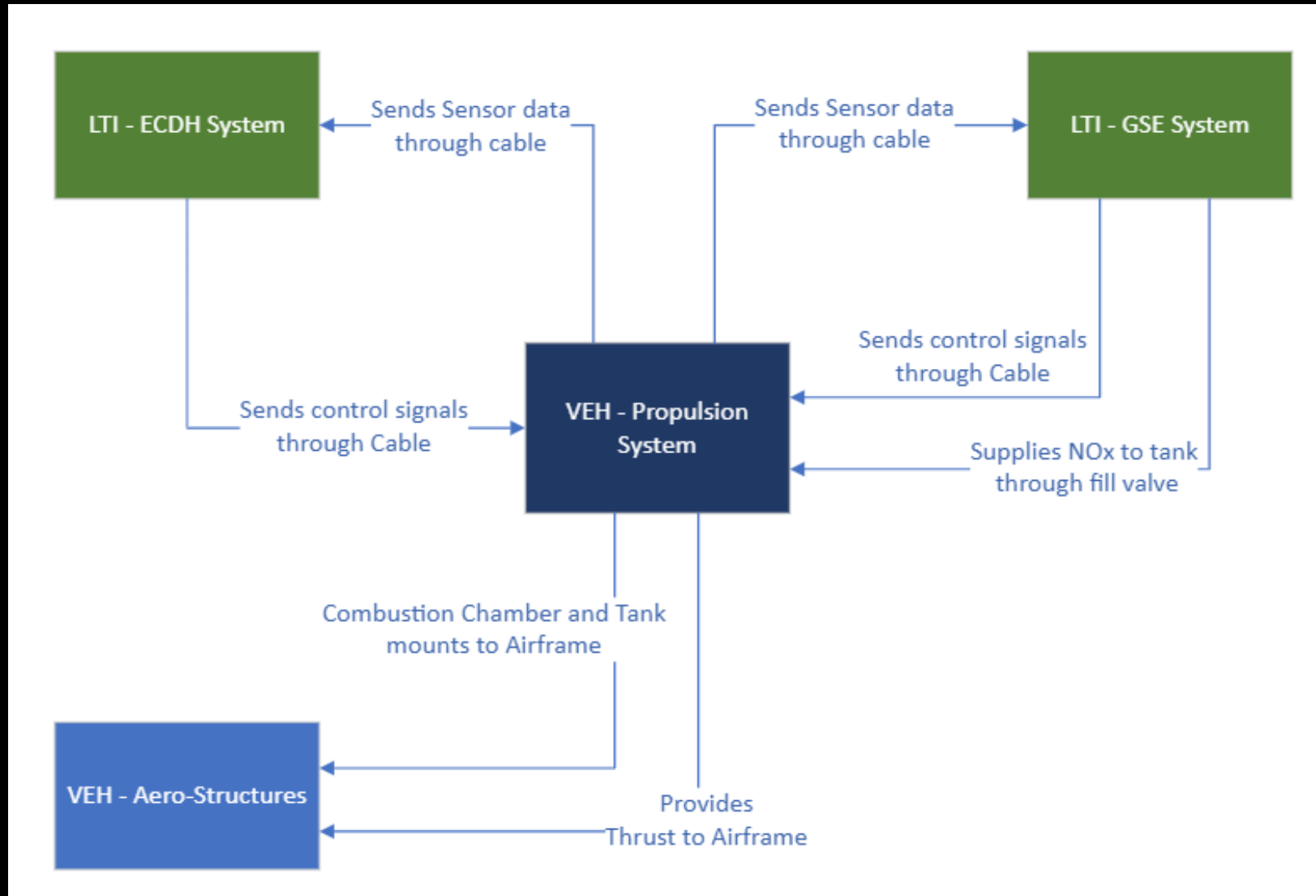
- 40 in Combustion Chamber
 - Fuel: Paraffin Wax-Sorbitol Grain
- 68in Oxidizer tank
 - Oxidizer: Liquid Nitrous Oxide
- Total System Height: ~10 feet
- Total System Weight: ~75 lbs wet
- Target Apogee: 30,000ft AGL
- Target Thrust:
 - 1,500lbf peak
 - 8,992lbf-s impulse (40,000Ns)



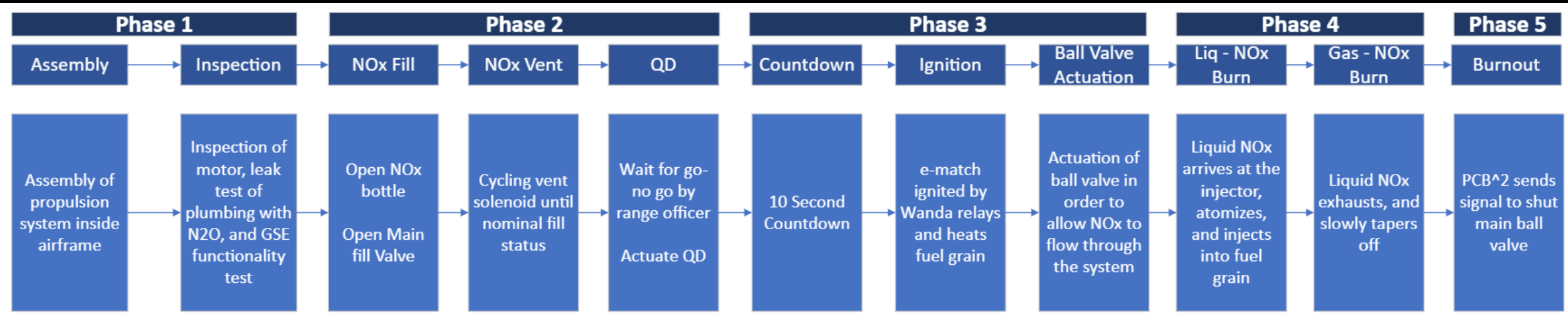
Propulsion System Requirements

| Requirement | Verification Method |
|--|---------------------|
| The Propulsion System shall have a wet mass of [80] lbs. | Inspection |
| The Propulsion System shall have an impulse of [36,948] Ns | Test |
| The Propulsion System shall have a dry mass of [55] lbs. | Inspection |
| The Propulsion System shall easily fit into the inner diameter of the airframe. | Demonstration |
| The Propulsion System shall have a peak thrust of [1,837] lbs. | Test |
| The Propulsion System shall implement Liquid Nitrous Oxide as an oxidizer. | Demonstration |
| The Propulsion System shall passively vent under [30] minutes. | Demonstration |
| The Propulsion System shall provide thrust to the Vehicle. | Demonstration |
| The Propulsion System shall withstand aerodynamic forces. | Analysis |
| The Propulsion System shall withstand its own produced forces. | Demonstration |
| The Propulsion System shall recover safely and without damage. | Demonstration |
| The Propulsion System shall be reusable. | Demonstration |
| The Propulsion System shall fill in under [30] minutes. | Demonstration |

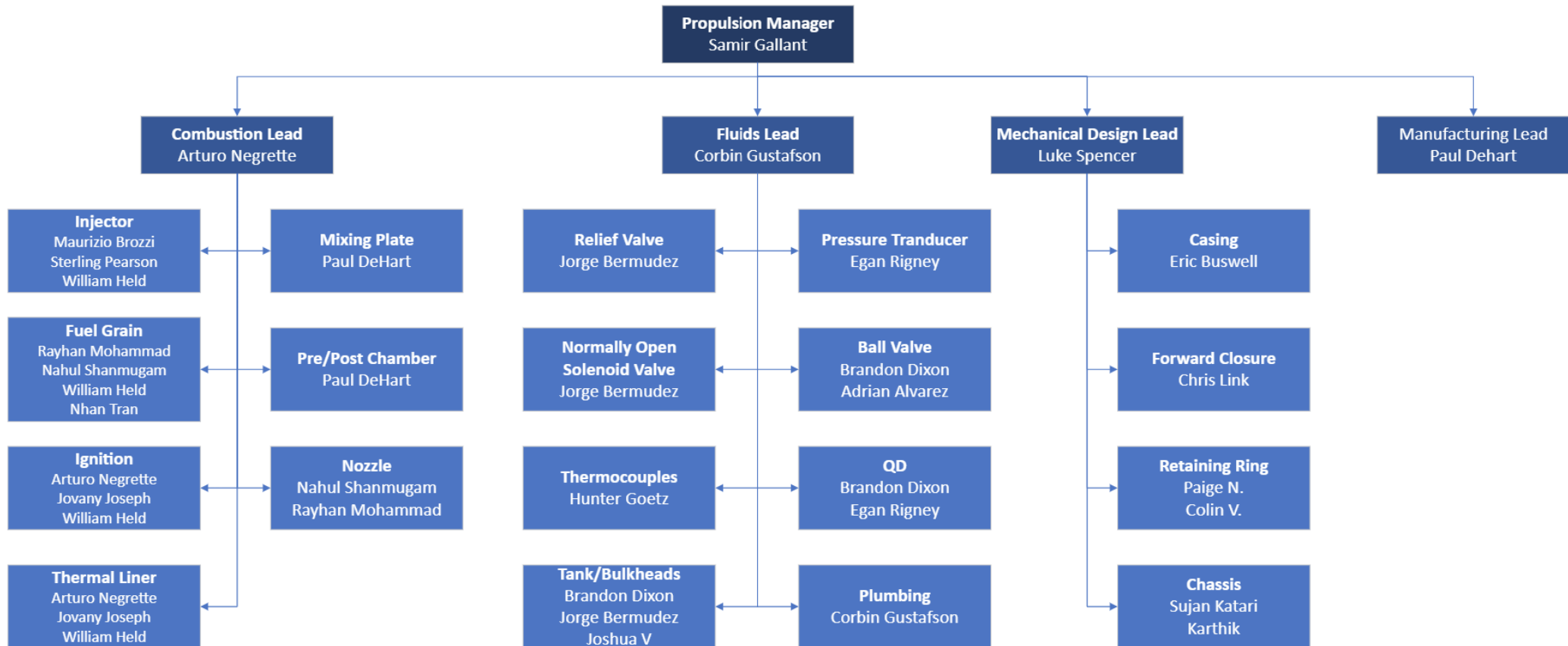
Propulsion System Interface Diagram



Propulsion System CONOPS



Propulsion Organization Chart



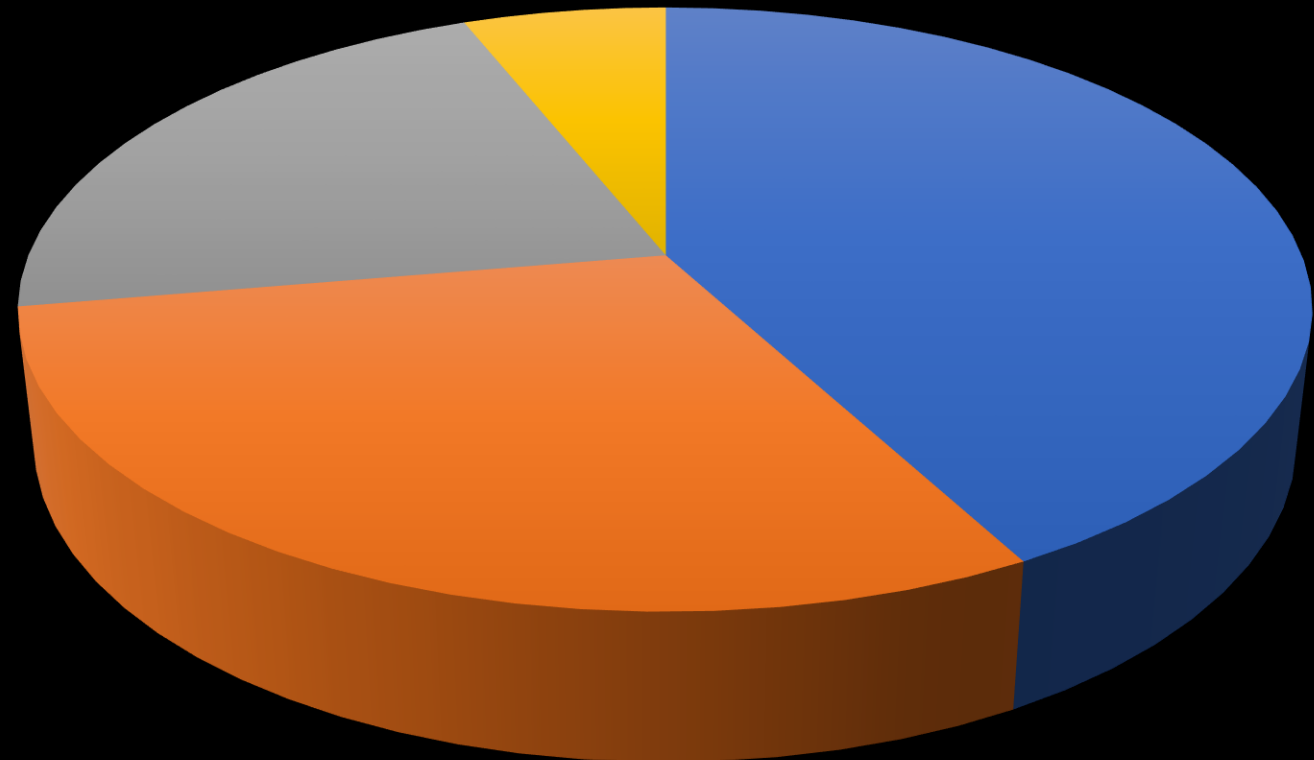
Propulsion System TPMs

| Measure | TPM Value | Units | Verification Method |
|-----------------|-----------|-------|---------------------|
| Total Length | [120] | in | Inspection |
| Max OD | [6] | in | Inspection |
| Peak Thrust | [1837] | lbf | Testing |
| Impulse | [36,948] | Ns | Testing |
| Total Burn Time | [10-12] | s | Testing |
| Dry Weight | [25] | lbs | Inspection |
| Wet Weight | [80] | lbs | Inspection |

Propulsion System Cost

- Fluids Cost ~ \$3500
- Combustion Cost ~ \$2500
- Mechanical Cost ~ \$1800
- Buffer ~ \$500
- **Total ~ \$8300**

Cost Breakdown



■ Fluid Systems ■ Combustion ■ Mechanical Design ■ Surplus \$\$

Propulsion System Risks

Risk

Mitigation

NOx Leak

Properly installing all fittings, as well as choosing Yor/Swage/Hy-Lok fittings.

Ignition Failure

Using two E-matches for redundancy, as well as including a mold in the fuel grain that will retain the igniter at the top and in the port of the grain.

Incomplete Ignition

Using a mold in the fuel grain to hold the igniter against it.

NO Solenoid Burnout

Designing a system that automatically closes the Solenoid if it's duty cycle is a starting to get reached, as well as monitoring the time it's open.

Burn through

Oversizing and properly casting the fuel grain. Also thickening exposed parts of the liner.

Incomplete Fill

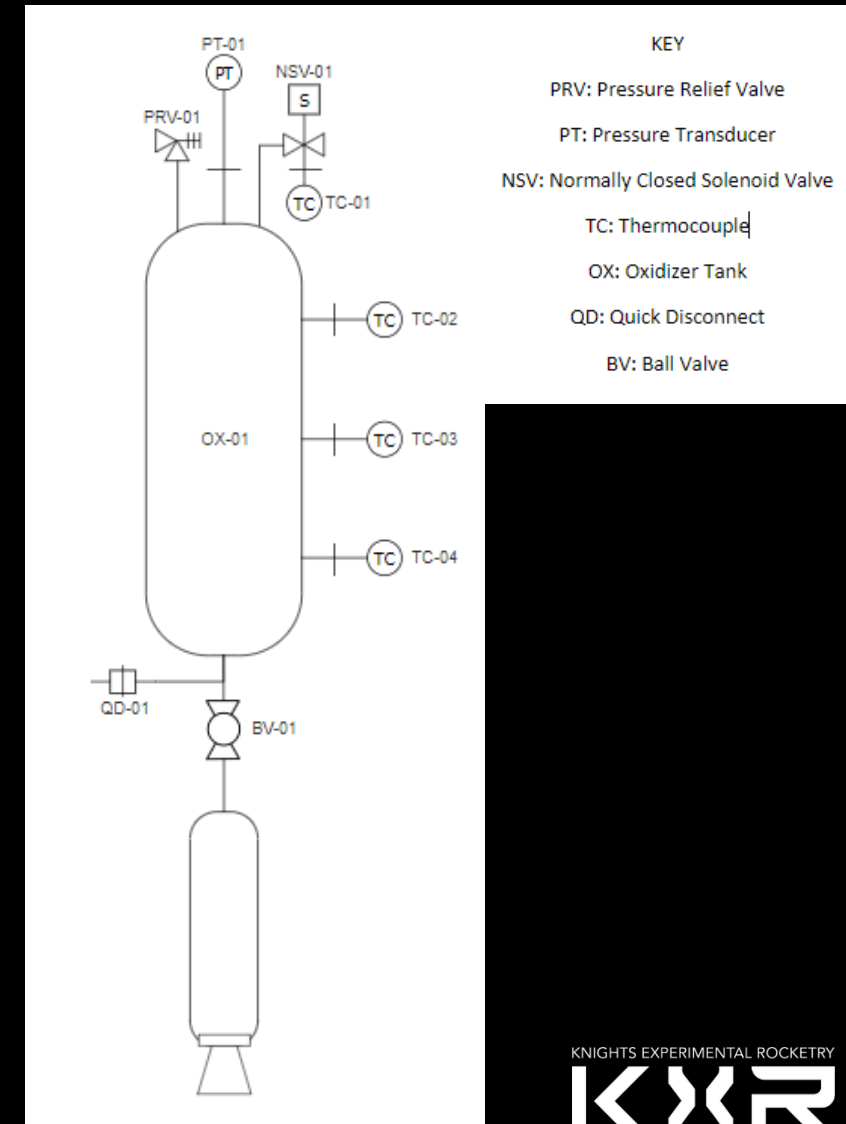
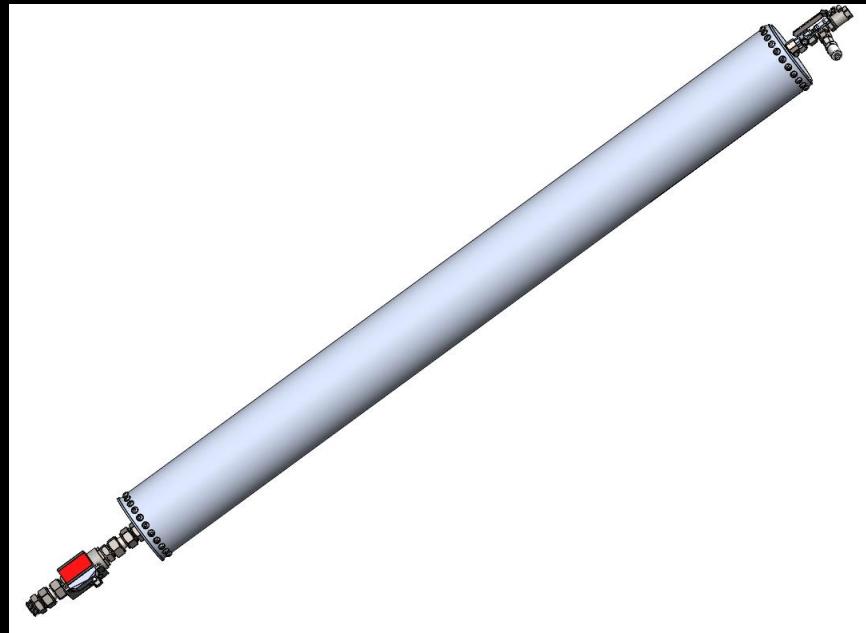
Adding redundant sensors as well as coordinating with LTI to ensure tha there's enough NOx for each fire/launch attempt.

Propulsion System Verification Plans

1. FEA on mechanical design and fluids sub-systems components
2. CFD on nozzle and injector
3. Inspection of COTS and machined components
4. Dry fit of propulsion system
5. Valve and electronic component testing
6. Hydrostatic Test
7. Water Flow Injector Test
8. Cold Flow Test
9. Static Fire Test
10. Launch

Fluids Sub-System

- Store oxidizer
- Provide mass flow to combustion chamber
- Fill through GSE
- Measure tank temperature and pressure
- Regulate pressure
- Integrate with airframe



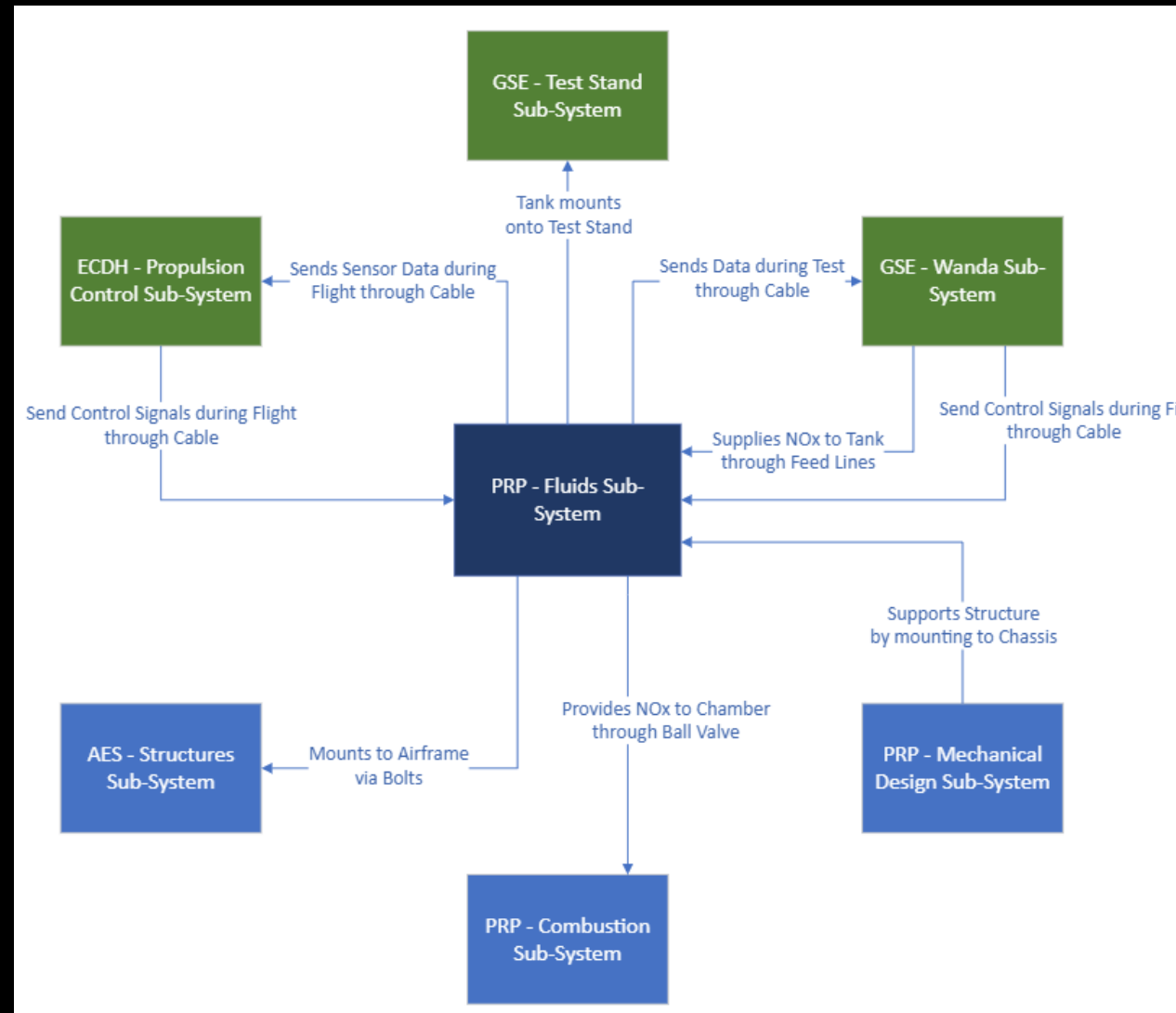
Fluids Sub-System Requirements

| Requirement | Verification Method |
|---|---------------------|
| The Fluids Sub-System shall have a Dry Weight of [TBD] lbs. | Inspection |
| The Fluids Sub-System shall have a Total Length of [TBD] inches. | Inspection |
| The Fluids Sub-System shall withstand a minimum temperature of [223.15] Kelvin. | Analysis |
| The Fluids Sub-System shall withstand a Maximum Expected Operating Pressure of [1500] psi. | Test |
| The Fluids Sub-System shall be reusable. | Demonstration |
| The Fluids Sub-System shall provide liquid Nitrous Oxide to the Combustion Sub-System. | Demonstration |
| The Fluids Sub-System shall provide sensor data to the Propulsion Control Board. | Demonstration |
| The Fluids Sub-System shall provide an initial Oxidizer Mass Flow Rate of [4.8 lb/s] to the Injector. | Test |
| The Fluids Sub-System shall contain all liquid Oxidizer without unintentional flow or leaking. | Test |
| The Fluids Sub-System shall vent when overpressurized. | Demonstration |

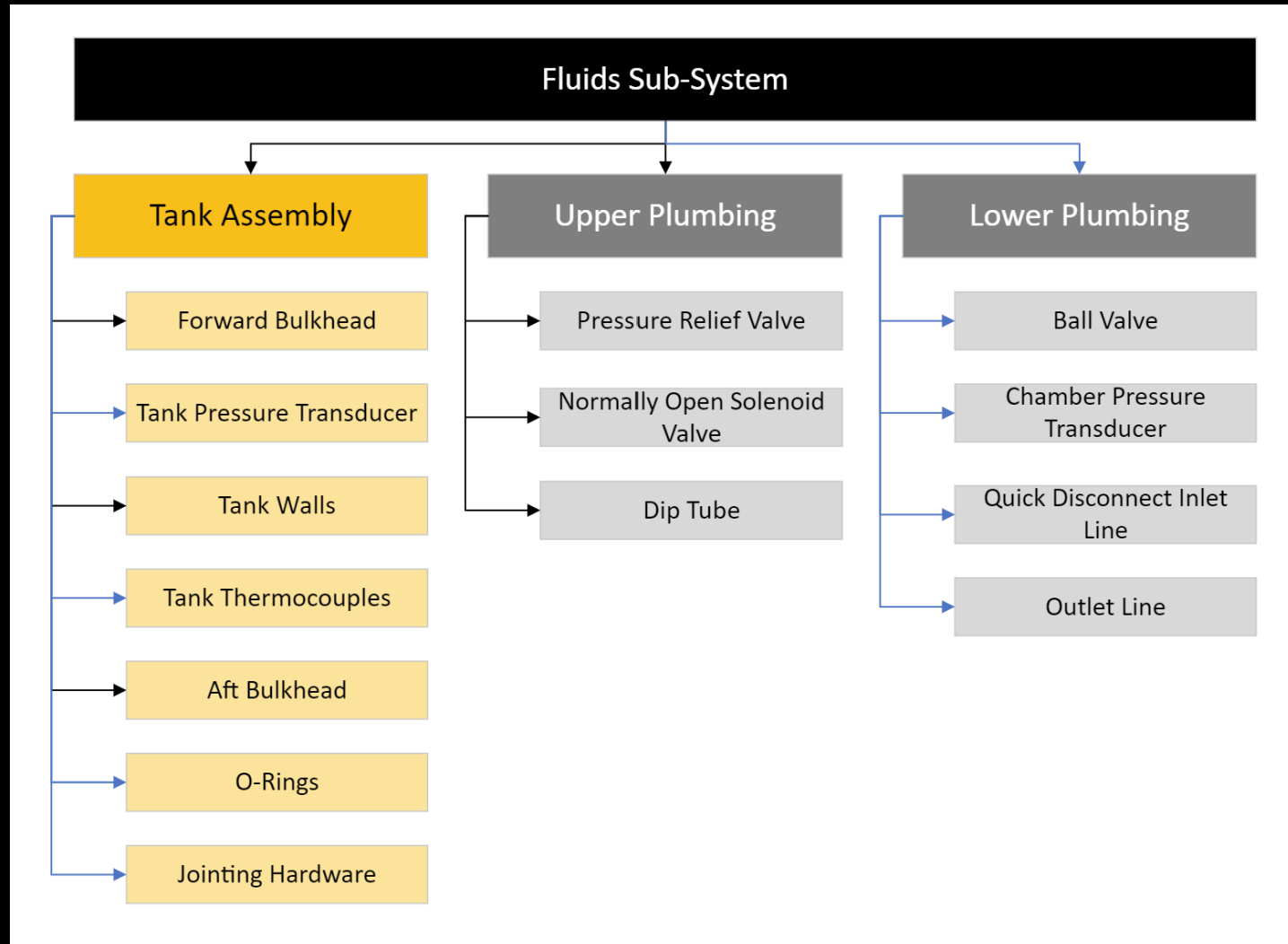
Fluids Sub-System TPMs

| Measure | TPM Value | Units | Verification Method |
|-------------------------------------|-----------|--------|---------------------|
| Total Length | [80] | inches | Inspection |
| Total Oxidizer Weight | [45] | lbs | Inspection |
| Maximum Expected Operating Pressure | [1000] | psi | Test |
| Delivered Mass Flow | [7.97] | lb/s | Test |

Fluids Sub-System Interface Diagram



Fluids Sub-System Component Breakdown



Oxidizer Tank Casing

Thin Wall Approximation Validity:

$$31 = \frac{D}{t} > 20$$

Barlow's Formula:

$$P = \frac{2 TS}{D_o}$$

Hoop Stress:

$$Q_H = \frac{PD_I}{2 T} = 22,500 \text{ psi}$$

Axial Stress:

$$Q_A = \frac{PD_I}{4 T} = 11,250 \text{ psi}$$

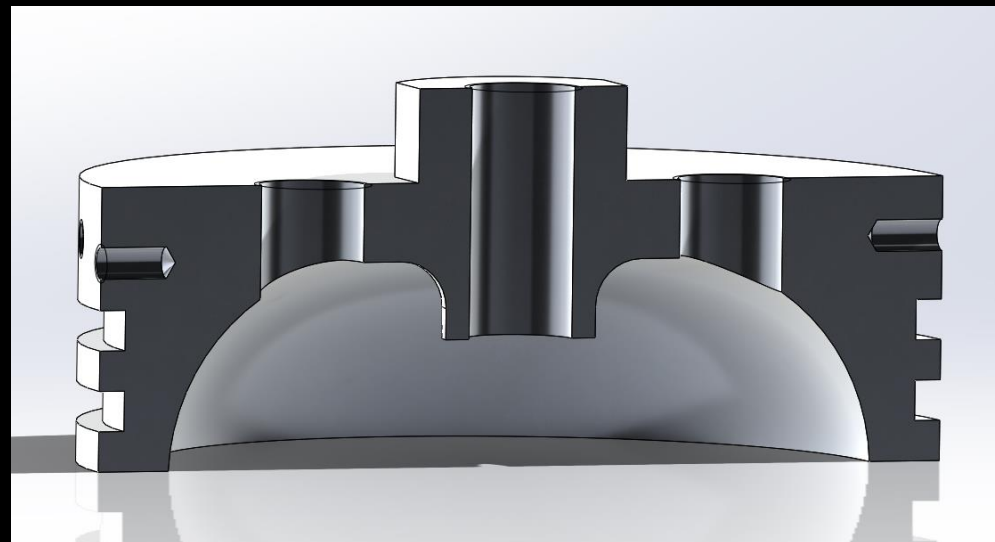
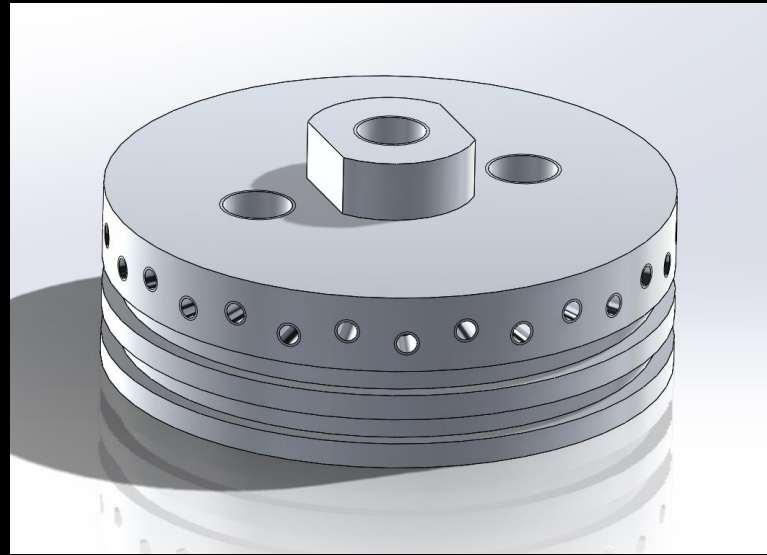
| Measure | TPM Value | Units |
|---------------------|-----------|-------|
| Total Length | [68] | in |
| Outer Diameter | [6] | in |
| Dry Weight | [23.42] | lbs |
| Volume | [27] | L |
| Tank Wall Thickness | 3/16 | in |



Machined from
Aluminum
Cylinder: \$138.23

Oxidizer Tank: Forward Bulkhead

- Aluminum 6061 T6
- 1/4-20 bolts X30
- 2 radial bolt patterns
- 3/4-16 threads X3
- 2-355 O-rings
- Machined in house



Force on Bulkheads

Surface Area of 1/2 a 5.625" diameter sphere.

$$A = 2\pi r^2 = 2\pi \left(\frac{5.625 \text{ in}}{2}\right)^2 = 49.7 \text{ in}^2$$

$$F_{\text{on each bulkhead}} = (49.7 \text{ in}^2)(1000 \text{ psi})(1.15) = 74550 \text{ lbs}$$

Number of bolts = $\frac{F_{\text{on bulkhead}}}{A_{\text{bolt}} \cdot \sigma_{\text{shear bolt}}}$ $d_{\text{shear bolt}} = 0.196"$ $\sigma_{\text{shear}} = 40,000 \text{ psi}$

$$N = \frac{74550 \text{ lbs}}{\frac{\pi}{4}(0.196 \text{ in})^2(40,000 \text{ psi})} = 2745 \text{ bolts}$$

N = 30 bolts

Two rings of 15 bolts. $t = 3/16"$

$$F_{\text{bolt}} = \frac{74550 \text{ lbs}}{30 \text{ bolts}} = 2485 \frac{\text{lbs}}{\text{bolt}}$$

$$\sigma_{\text{shear-out}} = \frac{F_{\text{bolt}}}{(E_{\text{min}})(2t)} = 30 \text{ ksi}$$

shear bolt $t_b = 30 \text{ ksi}$ strength

$$E_{\text{min}} = \frac{2485 \text{ lbs}(1.5)}{\left(\frac{\pi}{16} \text{ in}\right)(20,000 \text{ psi})} = 0.3314 \text{ in}$$

For two rows of bolts:

$$E_{\text{min}} = \frac{E_{\text{min}1} + E_{\text{min}2}}{2}$$

$E_1 = E_{\text{min}1} \left(4 \left(\frac{d_{\text{shear bolt}}}{2}\right)\right) = 1.5 d_{\text{bolt major}}$

$$E_{\text{min}1} = 0.277 \text{ in}$$

$$E_{\text{min}2} = 2E_{\text{min}} - E_{\text{min}1} = 2(0.3314 \text{ in}) - 0.277 \text{ in}$$

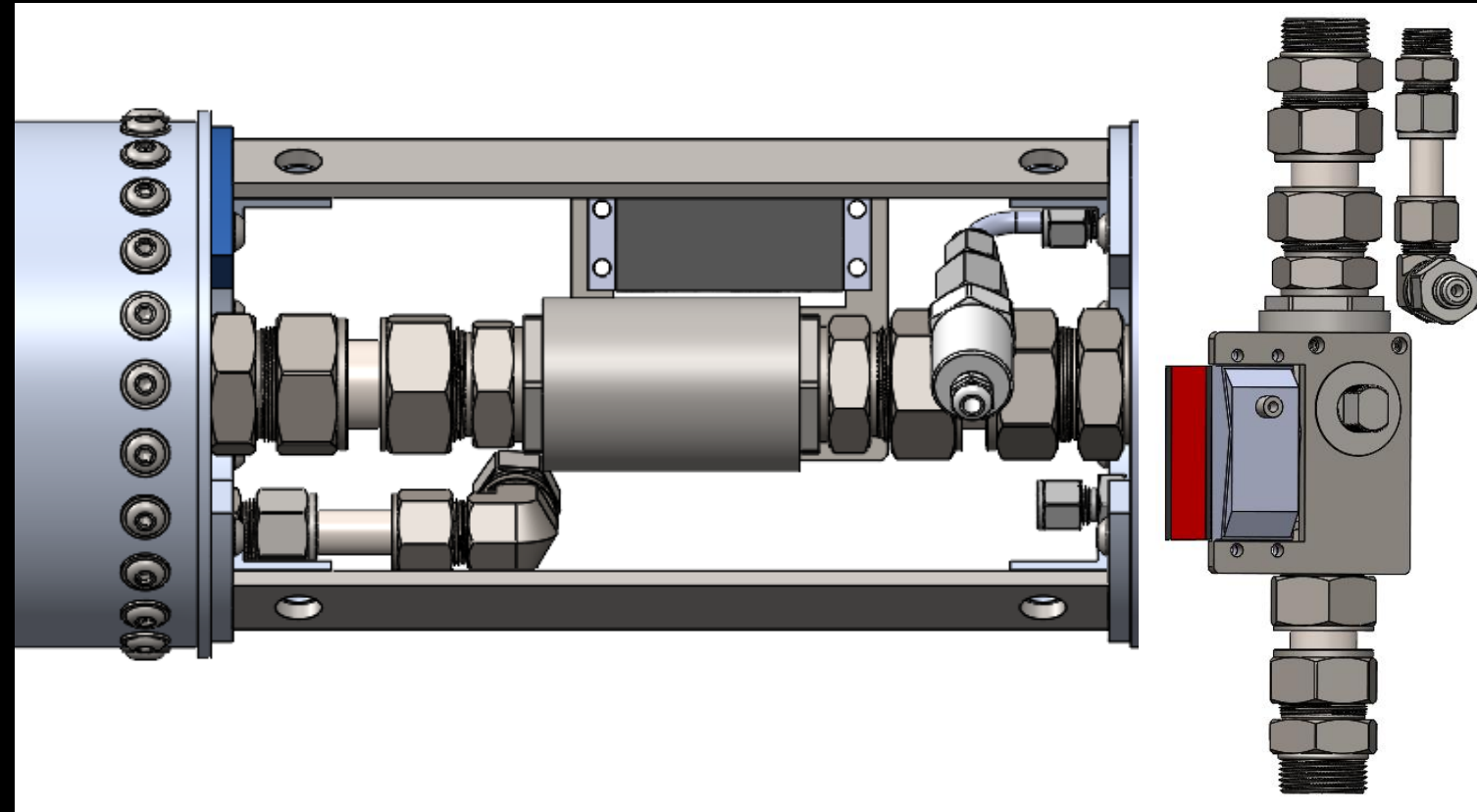
$$E_{\text{min}2} = .3858 \text{ in}$$

$$E_1 = 0.375 \text{ in}$$

$$E_2 = 0.4838 \text{ in}$$

Main Plumbing Overview

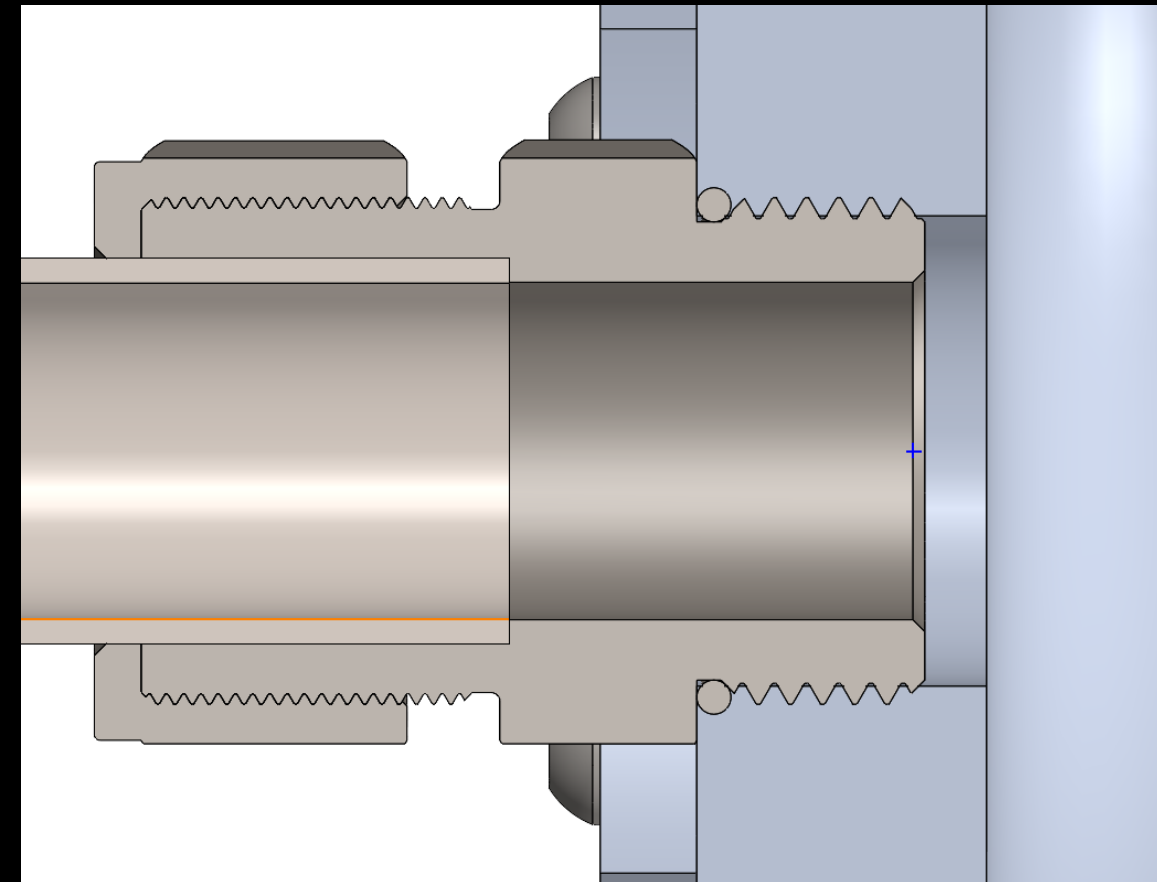
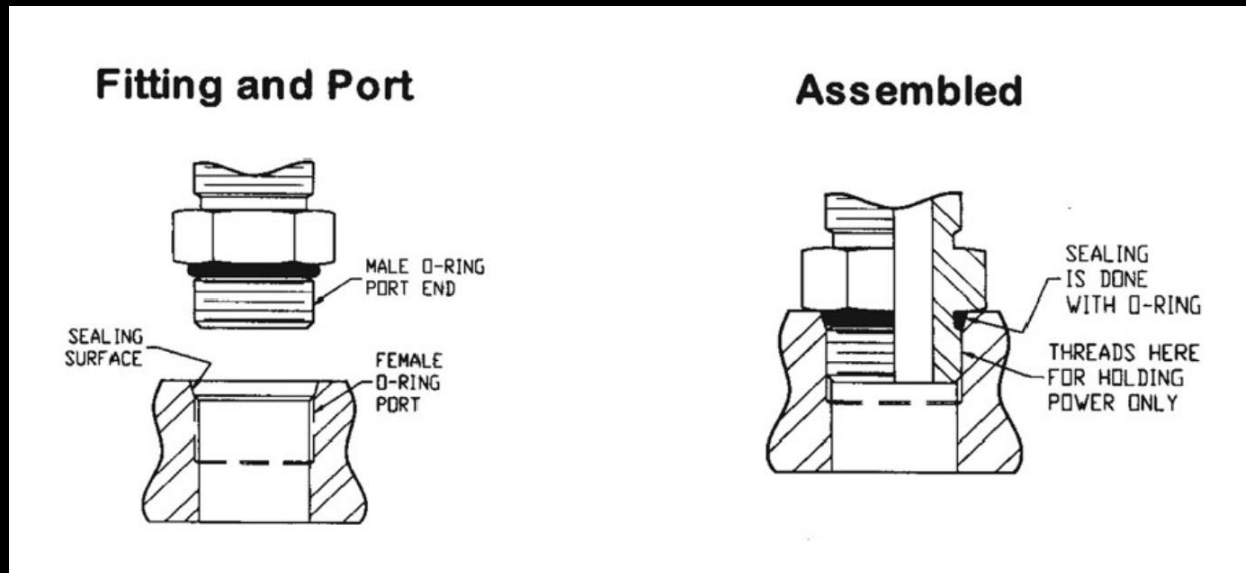
- 1" OD Stainless Steel Lines
- Servo Ball Valve Control
- Standardized torque specifications
- ~10" overall length
- Constrained by the tank-chamber chassis



Fitting Selection

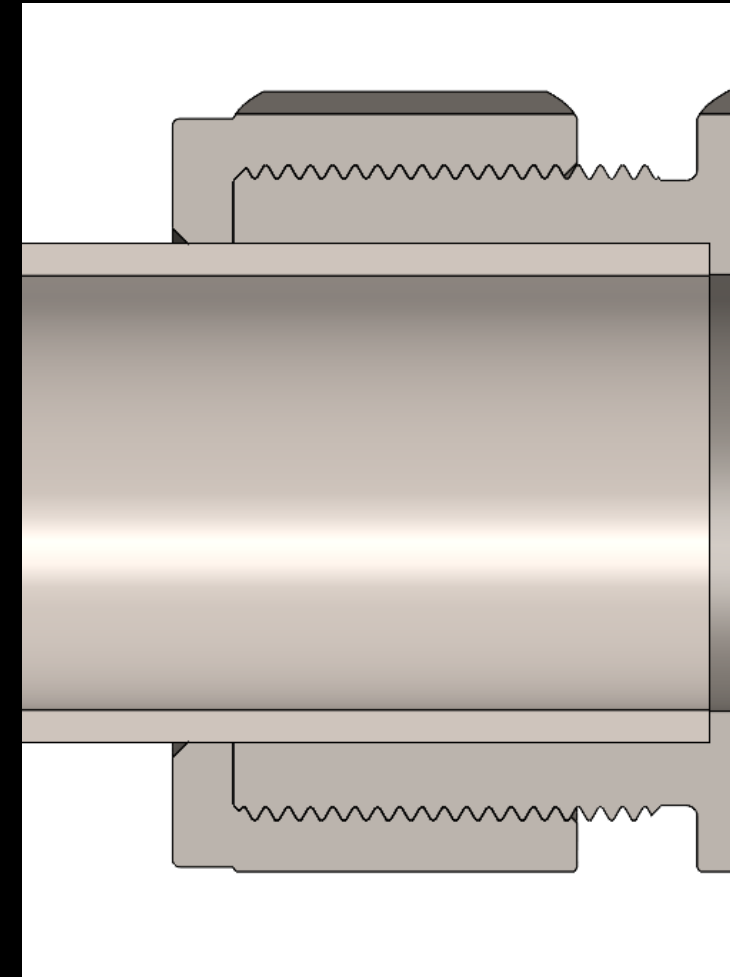
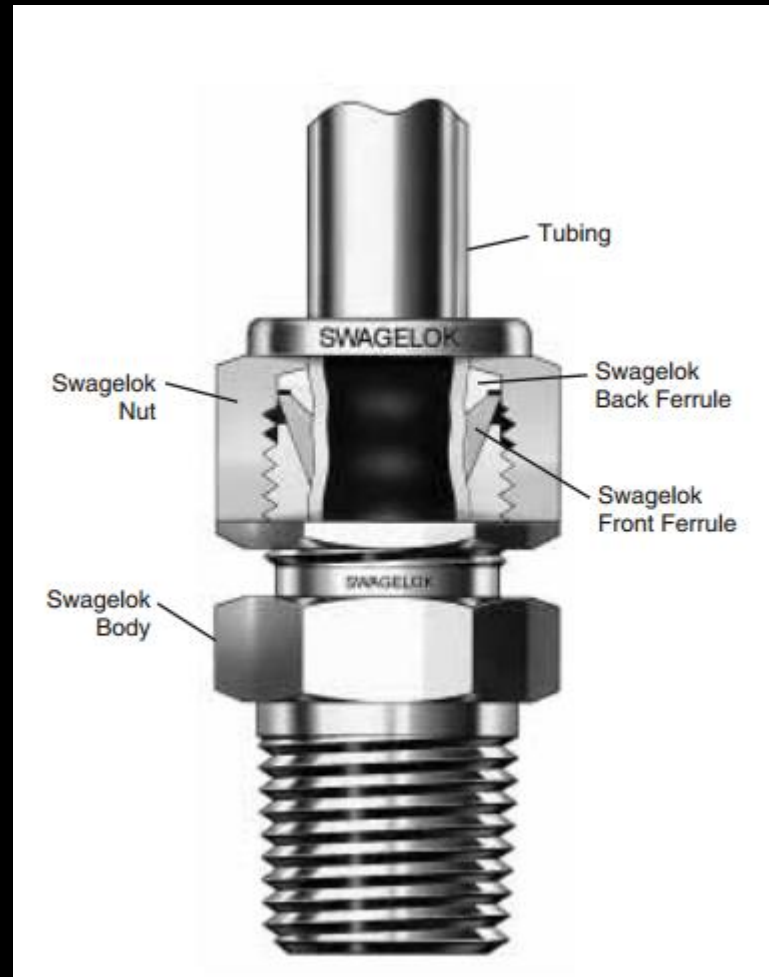


- SAE Straight Thread Fittings
 - Designed following SAE J1926
 - No need for consumable sealing



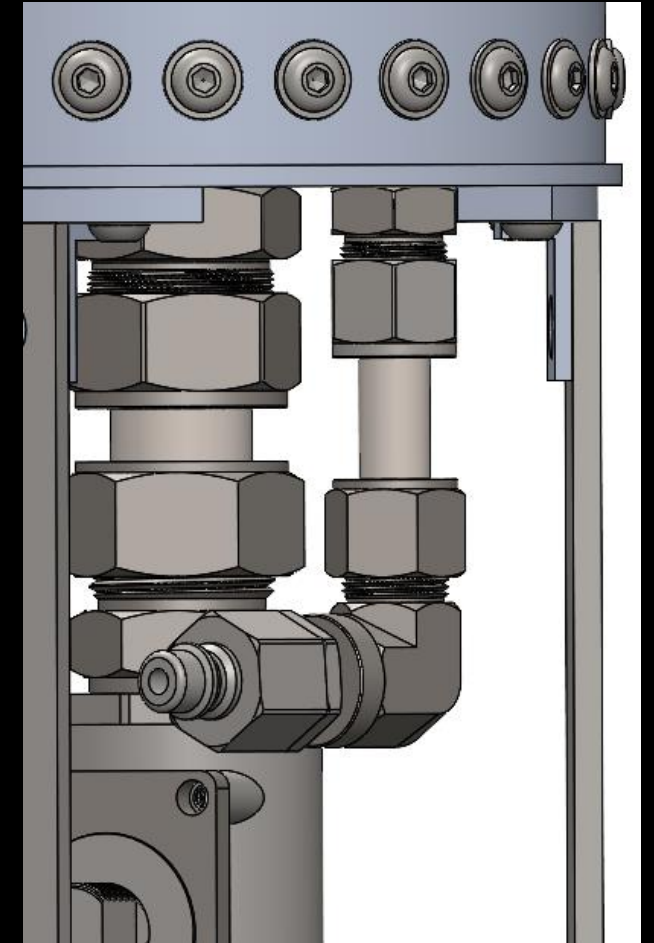
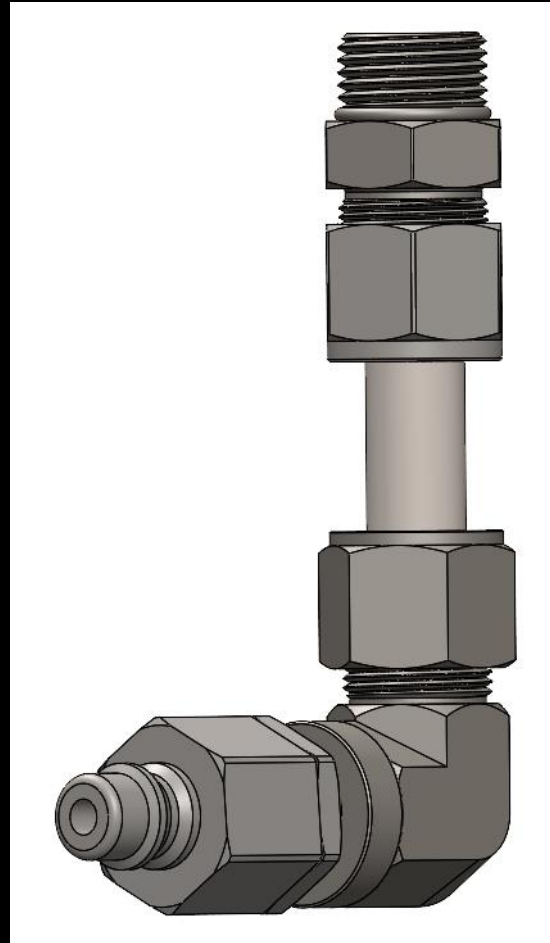
Fitting Selection

- Swagelok-type Tubing
 - Aerospace standard, widely available at any price range
 - No flaring = no need for special tooling
 - Pre-swaging is recommended for hard tubing but not required



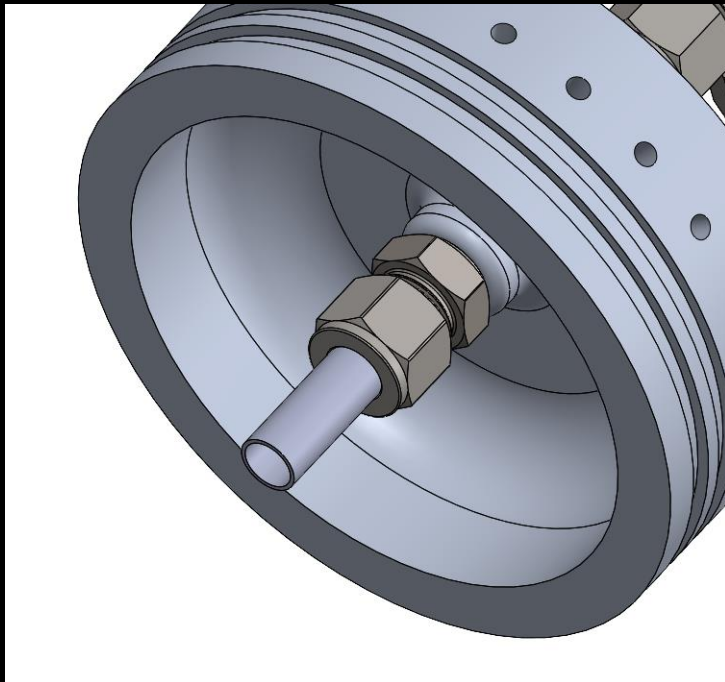
Fill Line Overview

- 1/2" OD Stainless Steel Lines
- Automatic shut-off post umbilical retract

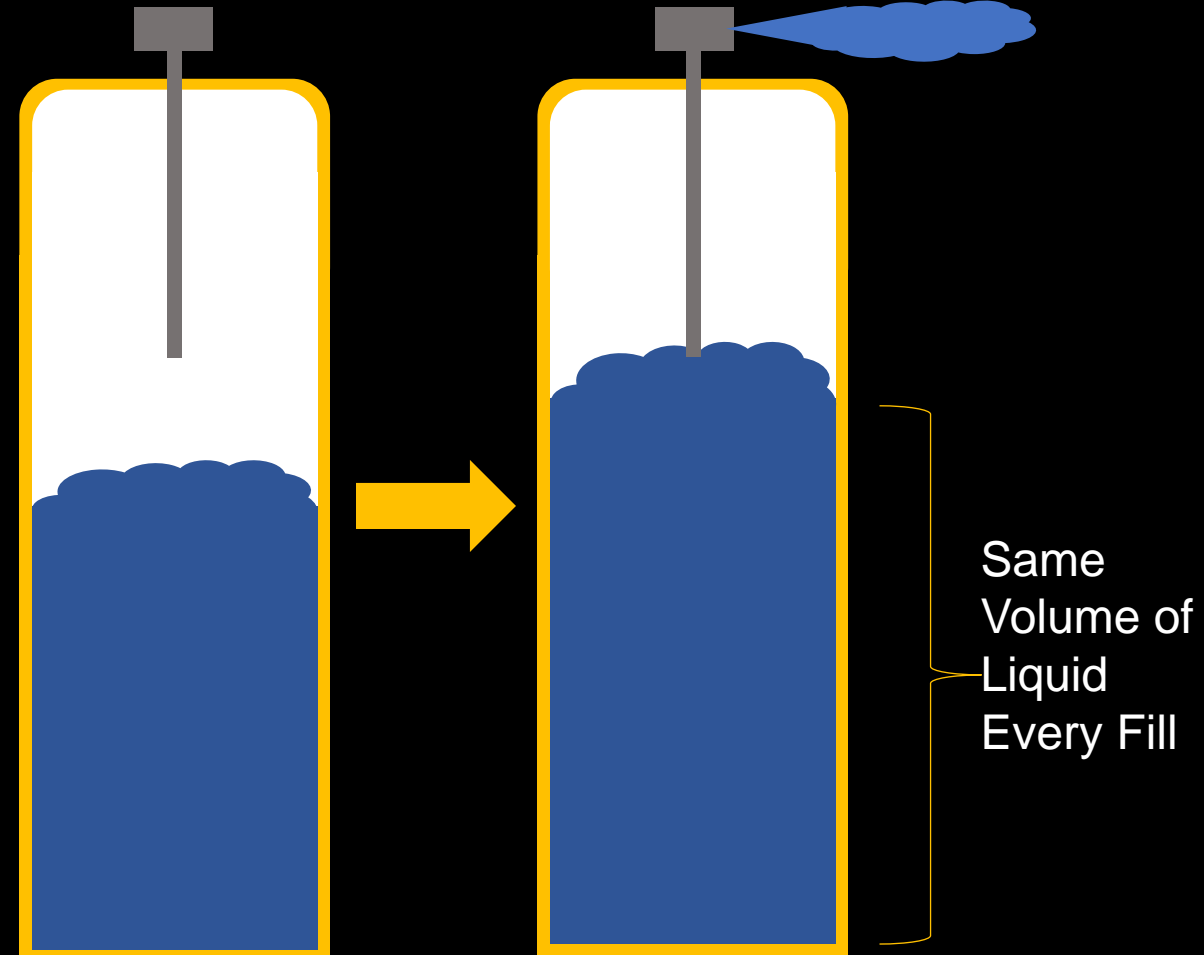


Dip Tube

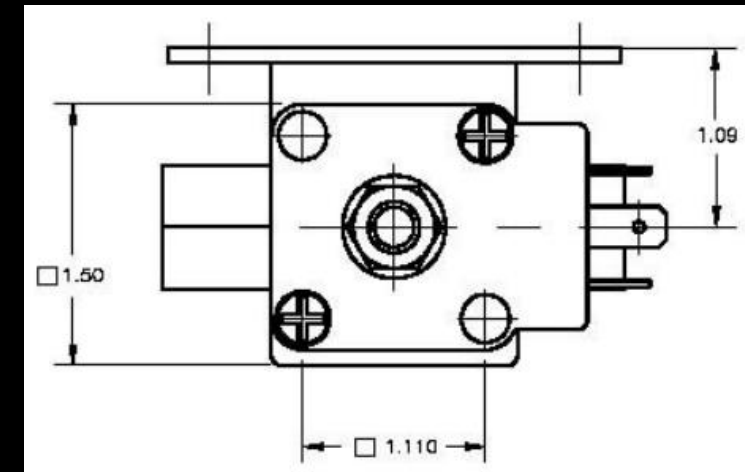
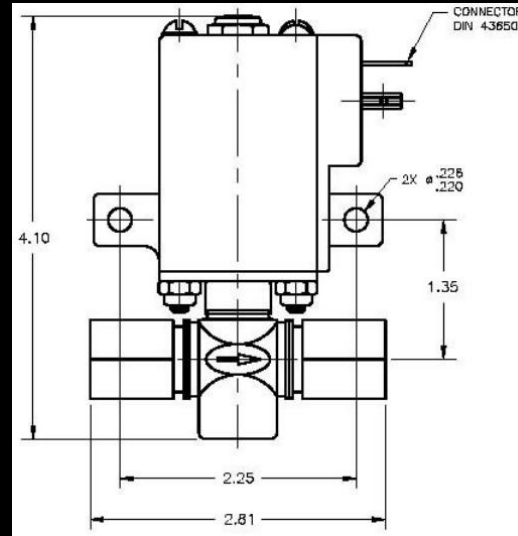
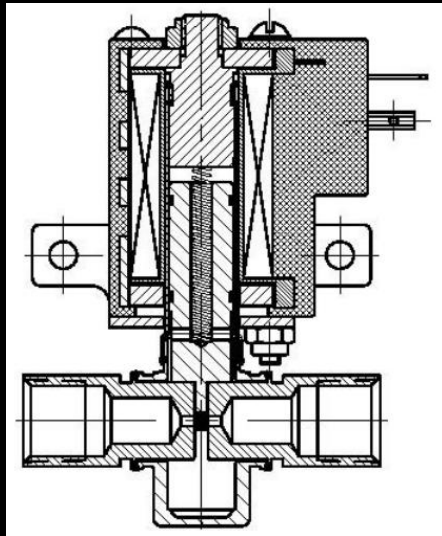
- Adjustable length
- Ullage is guaranteed through vapor lock



Placeholder
Dip Tube As
Integrated
with
Bulkhead



Normally Open Solenoid Valve



| Valve Parts in Contact With Fluid | |
|-----------------------------------|-----------|
| Body | SS |
| Seal Disc | Polyimide |
| Plunger Tube | 303 SS |
| Plunger | 430F SS |
| Plunger Stop | 430F SS |
| Spring | 303 SS |
| Rider Rings | PTFE |

| Port Size (NPT) | Orifice Size (ins.) | Cv | Max. Pressure (PSI) | | | | Model Number | | | | Wattage | |
|---------------------------------------|---------------------|------|---------------------|--------|-----|--------|-----------------|------------|---------------|------------|---------------|----|
| | | | AC | | DC | | Normally Closed | | Normally Open | | 115/ 24/ 60Hz | DC |
| | | | Gas | Liquid | Gas | Liquid | AC | DC | AC | DC | | |
| 2-Way Normally Closed & Normally Open | | | | | | | | | | | | |
| 3/8 | 3/32 | 0.20 | 1200 | 800 | 400 | 125 | SV91D28C1C | SV91D24C1C | SV91D28O1C | SV91D24O1C | 14 | 21 |

Normally Open Solenoid Valve

| Measure | TPM Value | Units | Verification Method |
|-----------------------|------------------------|----------|-------------------------|
| Discharge Coefficient | 0.89 | Cd | Calculation and Testing |
| Orifice Size | 2.38 | mm | Inspection |
| Dimensions | 73.66 x 46.74 x 104.14 | mm | Inspection |
| Temperature Range | -452°F - 200°F | °F | Inspection |
| Pressure Range | ≤ 1200 | psi | Inspection |
| Voltage | 21 | W @ 24DC | Inspection |
| Continuous Lifecycle | 5 | minutes | Demonstration |

Coefficient of Discharge:

$$C_d = \frac{\dot{m}}{\rho \dot{V}} = \frac{\dot{m}}{\rho A u} = \frac{\dot{m}}{\rho A \sqrt{\frac{2\Delta P}{\rho}}} = \frac{\dot{m}}{A \sqrt{2\rho\Delta P}}$$

Mass Flow:

$$\dot{m} = \frac{dm}{dt}$$

Schedule:

- 4-7 Weeks Estimated Shipping

Integration:

- NOSV is Attached to Forward Bulkhead of Tank

Design Considerations:

- Replace COTS NOSV with an SRAD normally open purge valve

Pressure Relief Valve

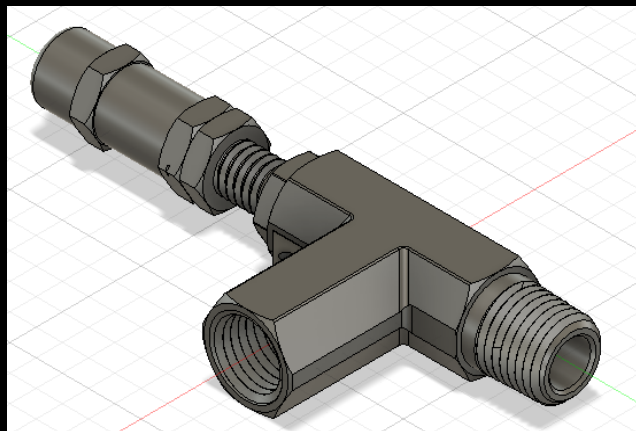
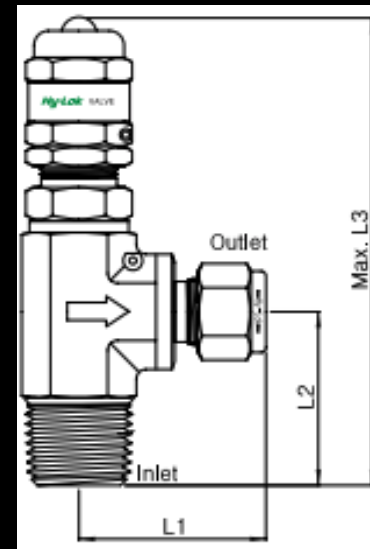
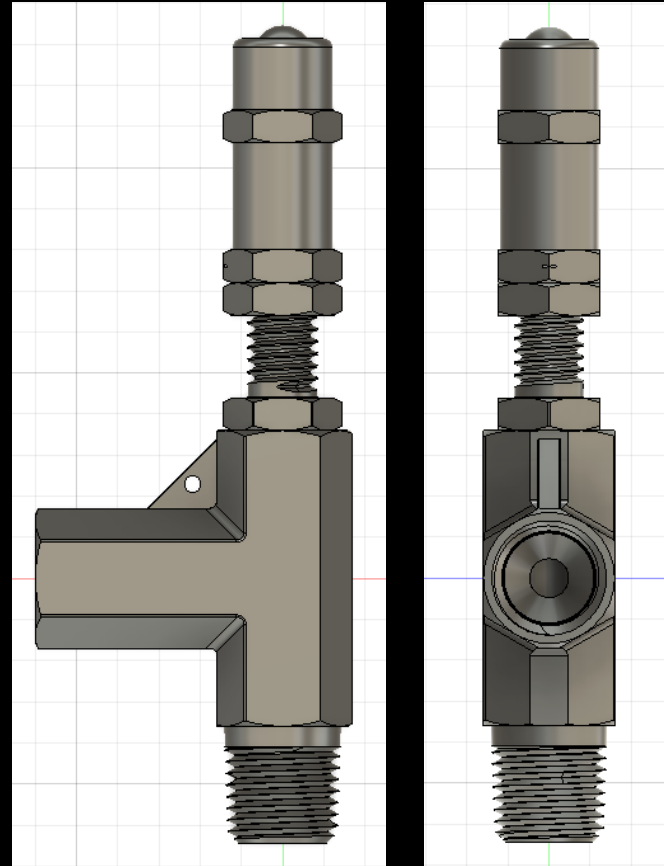
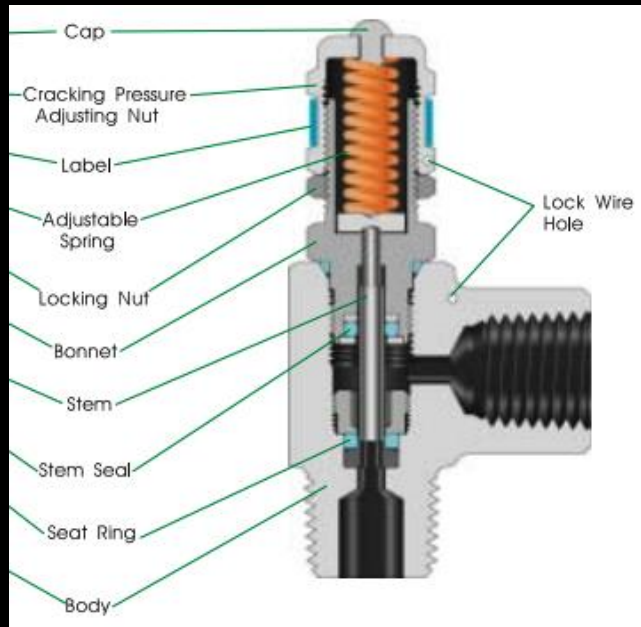


Table of Dimensions

| Basic Part No. | Orifice | End Connections | | Dimensions | | | |
|------------------|---------|-----------------|---------------|-----------------|------|------|-------|
| | | Inlet | Outlet | L1 | L2 | L3 | |
| RV1 or RV2 | 4.8 | H -4 T- | 1/4" Hy-Lok | 1/4" Hy-Lok | 38.7 | 37.3 | 104.6 |
| | | H -6M- | 6mm Hy-Lok | 6mm Hy-Lok | | | |
| | | H -8M- | 8mm Hy-Lok | 8mm Hy-Lok | | | |
| | | H -8 T- | 1/2" Hy-Lok | 1/2" Hy-Lok | 46.7 | 46.7 | 114.0 |
| | | H -12M- | 12mm Hy-Lok | 12mm Hy-Lok | | | |
| | | MH -8N8T- | 1/2" Male NPT | 1/2" Hy-Lok | 35.7 | 35.7 | 103.0 |
| | | MH -8N12M- | 1/2" Male NPT | 12mm Hy-Lok | | | |
| | | MF -4N- | 1/4" Male NPT | 1/4" Female NPT | 30.0 | 32.2 | 99.5 |
| | | MF -6N- | 3/8" Male NPT | 3/8" Female NPT | 34.5 | | |
| | | MF -8N- | 1/2" Male NPT | 1/2" Female NPT | 38.0 | 35.7 | 103.0 |

All dimensions are in millimeters.

Pressure Relief Valve

| Measure | TPM Value | Units | Verification Method |
|--------------|---|--------------------|---------------------|
| Orifice Size | 4.80 | mm | Inspection |
| Dimensions | 99.5 x 33.099 x 15.65 | mm | Inspection |
| Weight | 124.59 | g | Inspection |
| Pressures | ≤ 1500 | psi | Test |
| Temperatures | $-10^{\circ}\text{F} \leq x \leq 400^{\circ}\text{F}$ | $^{\circ}\text{F}$ | Test |

Discharge Coefficient:

$$C_d = \frac{\dot{m}}{\rho \dot{V}} = \frac{\dot{m}}{\rho A u} = \frac{\dot{m}}{\rho A \sqrt{\frac{2\Delta P}{\rho}}} = \frac{\dot{m}}{A \sqrt{2\rho\Delta P}}$$

Mass Flow:

$$\dot{m} = \frac{dm}{dt}$$

Schedule:

- 3-6 Weeks Estimated Shipping

Integration:

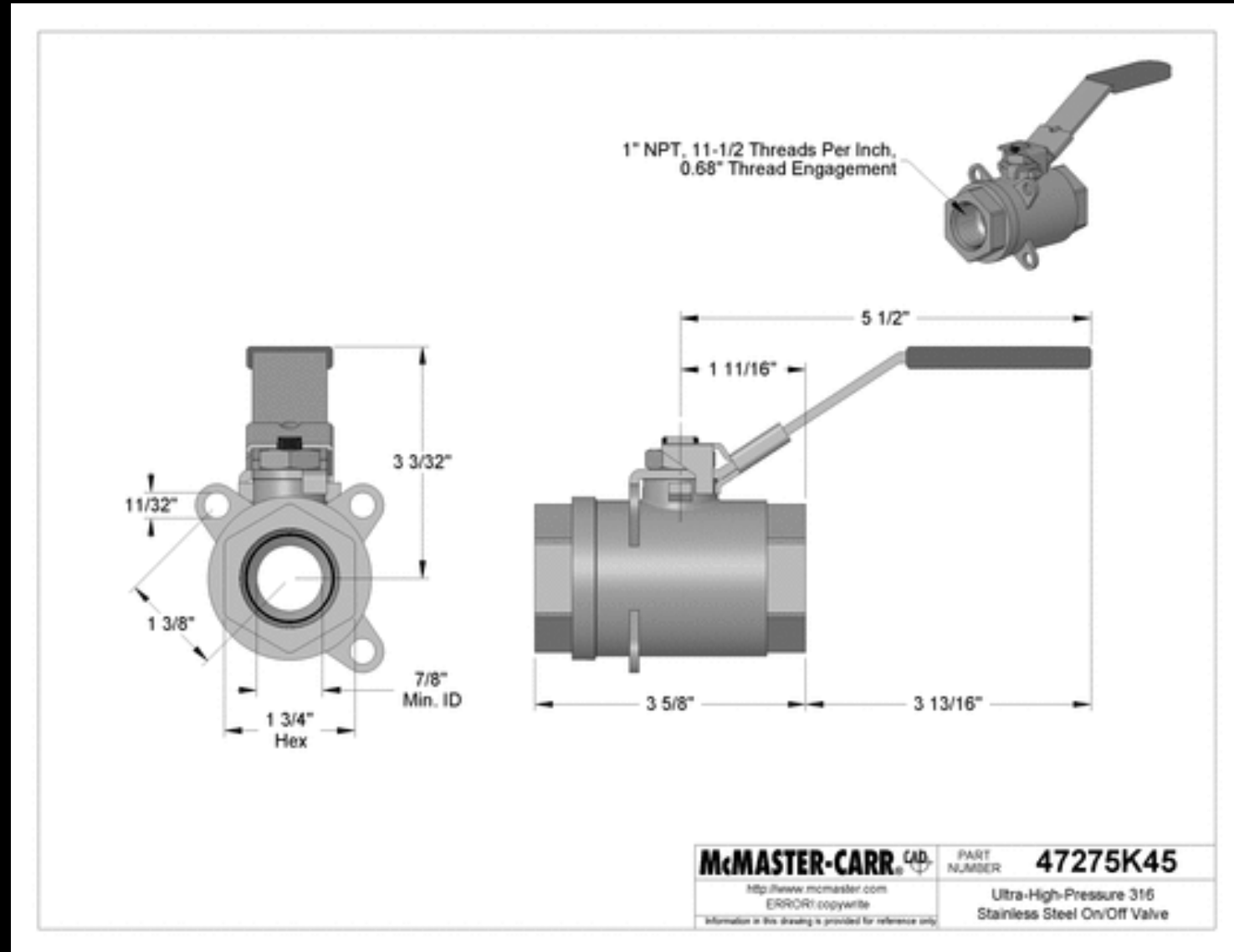
- PRV is Attached to Forward Bulkhead of Tank

Design Considerations:

- Search for PRV Smaller in Length
- Alternative with lower minimum working temperature

Servo Actuated Ball Valve

- COTS
- 7/8 inch internal diameter
- 4500 psi at 120° F
- -40° to 230° F
- 44 CV
- \$300
- 316 Stainless Steel

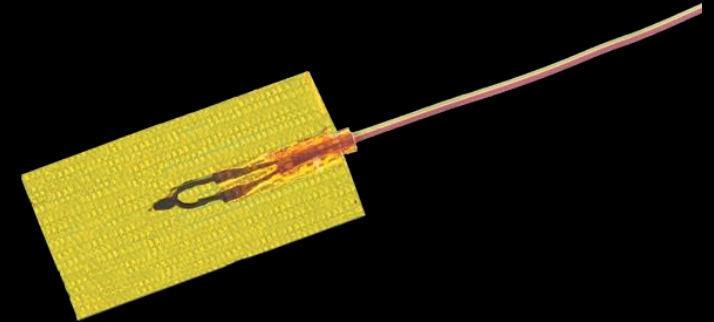
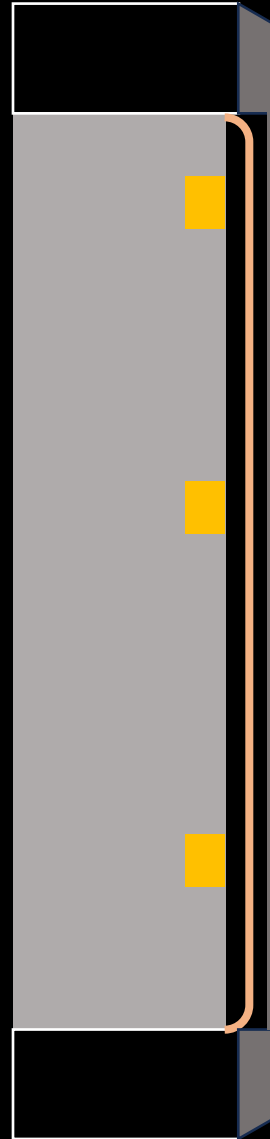


Thermocouples + Tank Raceway

Thermocouples bonded to surface of tank

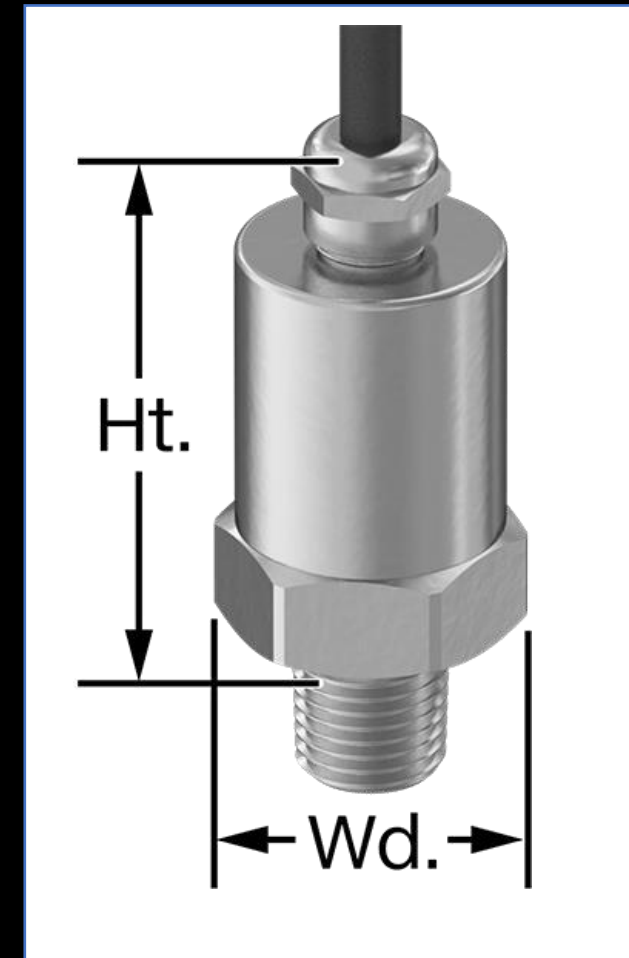
Ribbon cable

3D printed shroud, depends on integration with aerostructures



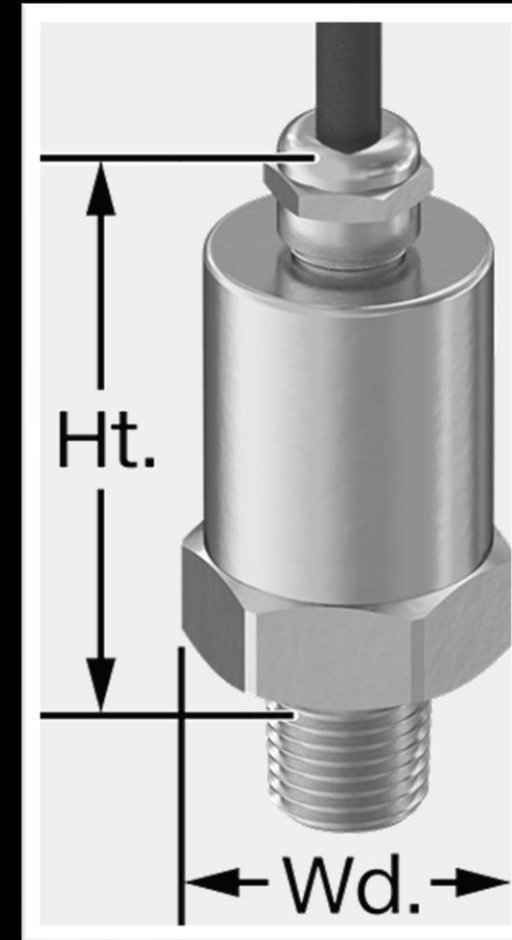
Top Plumbing Pressure Transducer: Important Details

- Given ratings by Sub-System level requirements:
 - $<0^{\circ}\text{C}$
 - $>1500\text{ psi}$
 - Corrosive oxidizer fluids
- Needs to be properly rated to avoid over pressurization and reliability.
- Only required material is the component itself.
- Made of 204 Stainless Steel, which is known to be corrosion resistant.
- \$107.56, ships from McMaster NJ warehouse



Top Plumbing Pressure Transducer: Item Details

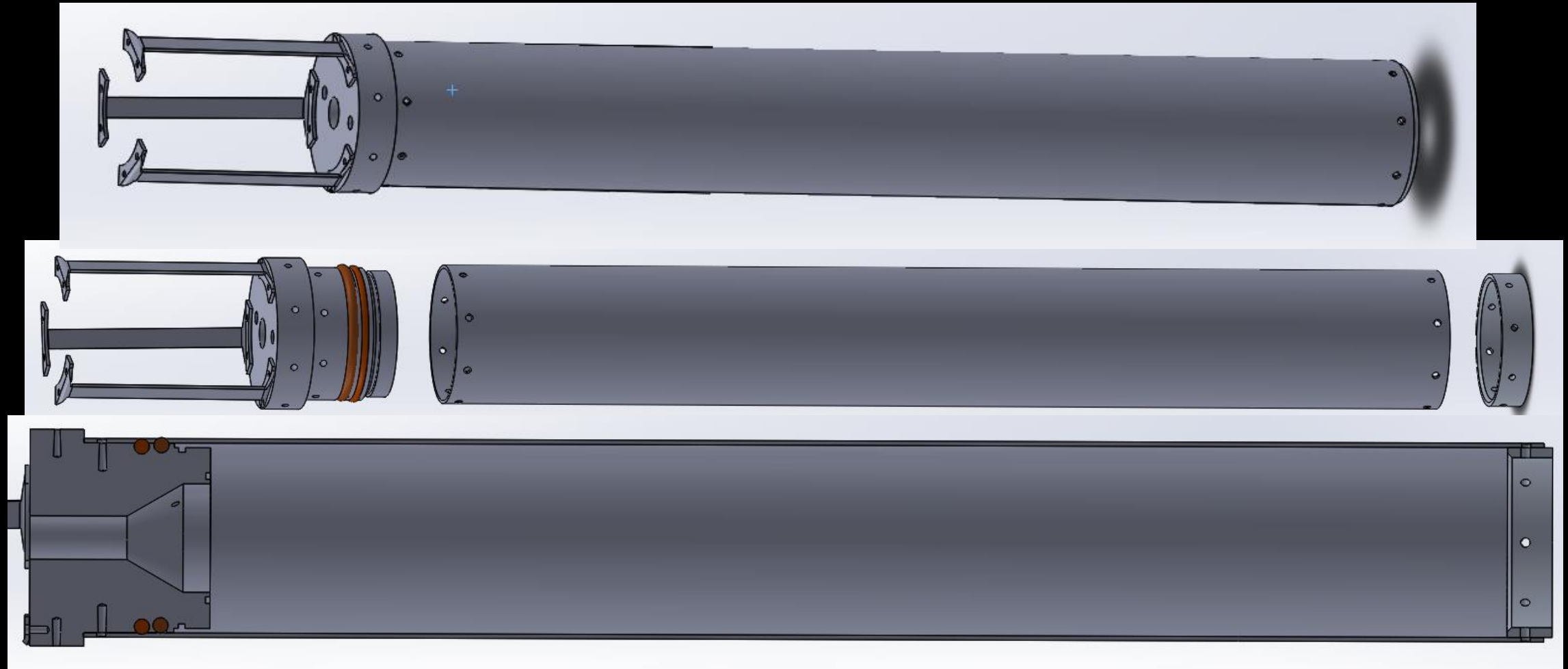
Pressure Range: 0-1500psi
Max Short-Term Pressure: 2250 psi
Output Signal: 4-20mA
Accuracy: +/- 1.5%
Response Time: 0.001 s
Connection: 1/4" NPT, Male
Temp Range: -20F – 220F
Wire #: 2
Wire Length: 18"
Material: 304 Stainless Steel
Height: 1 13/16"
Width: 15/16"



Future Plans

- NOS is changing to QD normally open relief
- Ball Valve Servo design
- Finalize NOX Tank – Airframe connection
- Finalize Thermocouple selection

Mechanical Design Sub-System



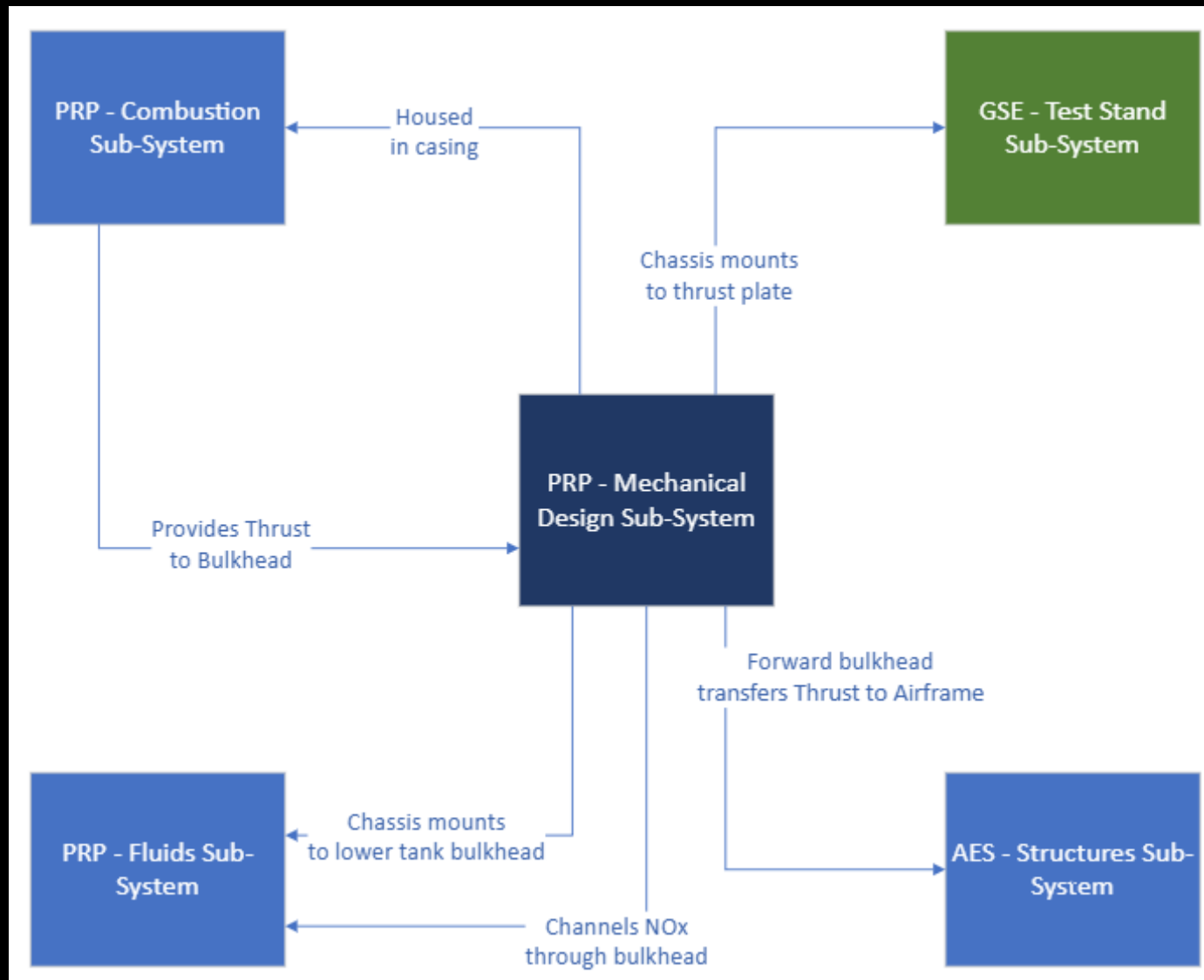
Mechanical Design Requirements

| Requirement | Verification Method |
|---|---------------------|
| The Mechanical Sub-System shall withstand maximum operating temperature of [2600] Kelvin | Test |
| The Mechanical Sub-System shall withstand maximum operating pressure of [1000] PSI. | Test |
| The Mechanical Sub-System shall withstand all flight loads. | Analysis |
| The Mechanical Sub-System shall withstand all engine loads with a safety factor of [2]. | Analysis |
| The Mechanical Sub-System shall be reusable. | Demonstration |
| The Mechanical Sub-System shall have a weight of [TBD] lbs. | Inspection |
| The Mechanical Sub-System shall have a length of [TBD] feet. | Inspection |

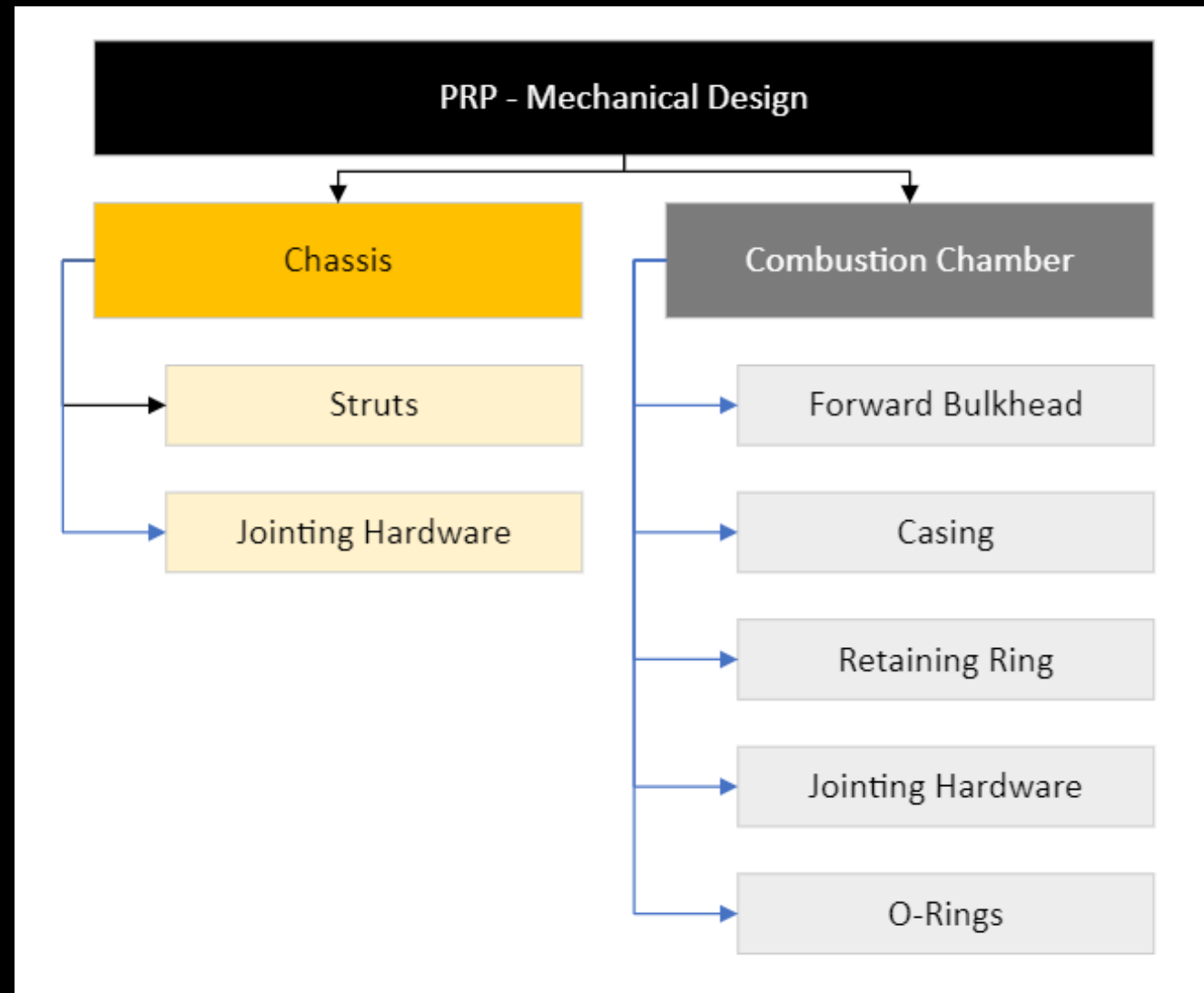
Mechanical Design TPMs

| Measure | TPM Value | Units | Verification Method |
|---------------------------|-----------|-------|---------------------|
| Weight | 18.305 | lbs | Inspection |
| Length | 40 | in | Inspection |
| MEOP (with Safety Factor) | [1000] | psi | Testing |
| MEOT | [2600] | K | Testing |
| Max Thrust to Withstand | [1837] | lbs. | Testing/Analysis |

Mechanical Design Interface Diagram



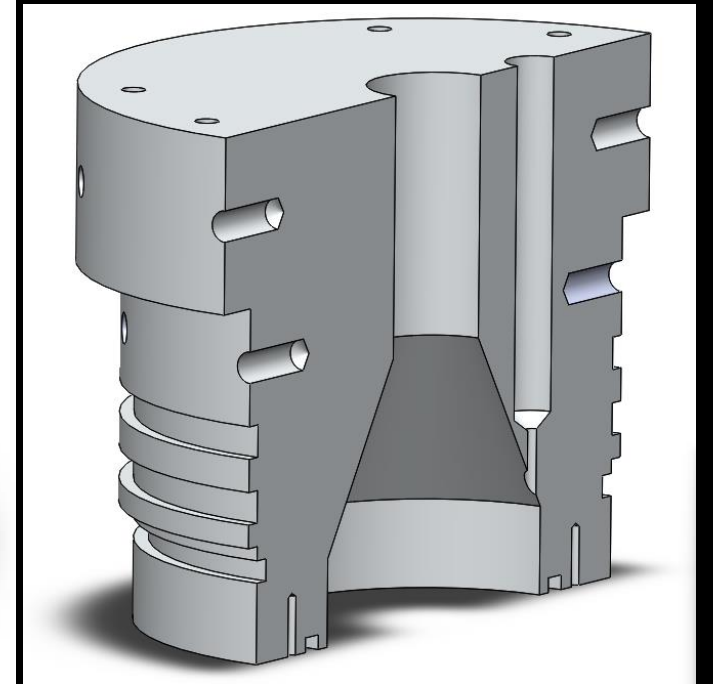
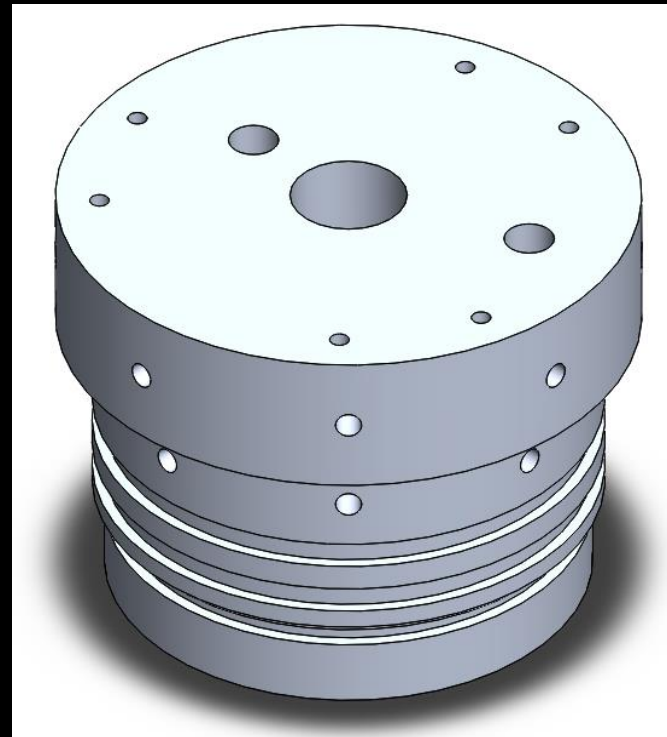
Mechanical Component Breakdown



Chamber Forward Closure/Bulkhead

- The forward closure is integrated with four components

1. Integrated with Airframe to transfer thrust to the rest of the rocket.
2. Integrated with the oxidizer tank to allow the flow of oxidizer to the combustion chamber.
3. Integrated with the chamber casing to maintain pressure inside the combustion chamber.
4. Integrated with the injector plate to allow even distribution of oxidizer inside the combustion chamber.



- Additional uses for the forward closure

- Housing for combustion chamber sensors.

Chamber Forward Closure/Bulkhead

- **Measurements**

- Upper diameter (6 in)
- Lower diameter (5 in)
- Length (4.50 in)

- **Calculations for the forward closure**

- # bolts for the thrust plate portion (4 bolts minimum given from aerostructures) using 8 for a safety factor of 2.
- Thrust plate portion 1 in tall with bolts .5 in from the edge (over safety factor of 2 for tear-out)
- # bolts for mounting casing to bulkhead calculated at 8 for safety factor of 2
- 3 Buna O rings

- **Risk**

- There are four main risk of failure
 1. Bolt shear
 2. Bolt tear-out
 3. Shear where the thrust plate and the casing meet
 4. O-ring failure

- **Schedule**

- 3 weeks to procure the raw metal.
- 2 weeks to fabricate the part.

- **Cost**

- \$150 for 6 in diameter, 4.5 in long Aluminum 6061 cylindrical rod.
- \$100 for the fabrication of the part.

Tensile to shear stress
 $\sigma_{yield(0.75)} = \tau_{yield}$

Factor of safety
 $FOS = 2$

Shear stress equation
 $\tau_{yield} = \frac{F_{thrust}}{A_{in\ shear}}$

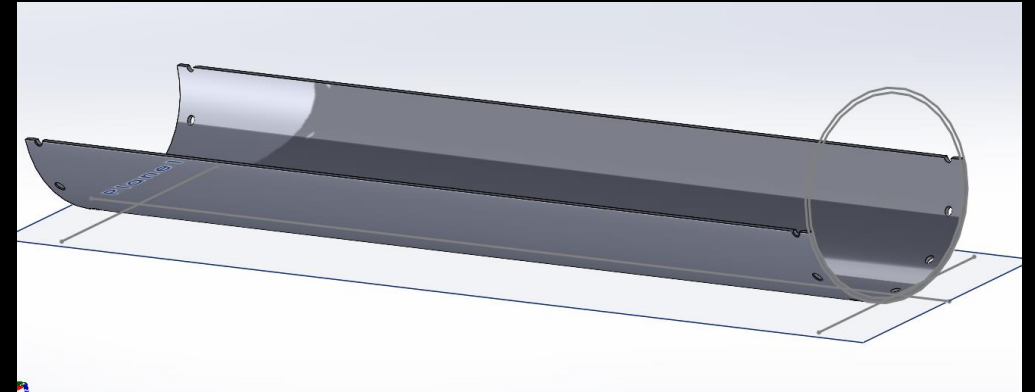
Bolt tear-out
 $distance_{edge\ to\ center\ of\ bolt} = 2(diameter_{bolt})$

Number of bolts for the thrust plate portion of the forward closure
 $\tau_{yield} = \frac{F_{thrust}}{A_{in\ shear}} = \frac{4F}{\pi d^2} = \frac{4(1500 \times FOS)}{\pi(0.196)^2} = 99430.41\ PSI$
 $\tau_{yield} = 85,000\ PSI(0.75) = 69,000\ PSI$
 $\tau_{yield} = 69,000 \times N_{bolts}$
 $N_{Bolts} = \frac{99430.41}{69000} = 1.44 \approx 2\ Bolts$
 For symmetry purposes 4 Bolts

Thickness of thrust plate portion of forward closure (bv bolt tear-out)
 $distance_{edge\ to\ center\ of\ bolt} = 2(diameter_{bolt})$
 $distance = 2\left(\frac{1}{4}\right) = 0.5\ in$
 $thickness = 2(distance) = 2(0.5) = 1\ in$

Chamber Casing

- Assumed safety factor = 2
- Machined in house (UCF Machine shop)
- Outer diameter is 5.5 inches and inner diameter is 5.25 inches
- Calculated force on each bolt is 3517N which using the safety factor of 2 it was found bolts should be .6 inches from the edge of casing to prevent bolt tear-out
- The calculated number of bolts on each side to prevent bolt shear with a safety factor of 2 is 8



$$F_{\text{bolt}} = \frac{\pi}{4} (D_{\text{casing}}^2) \times \text{MEOP}$$

$$F_{\text{bolt}} = \frac{\pi}{4} (5.25)^2 \times 1300 = 3517.724743 \text{ N}$$

$$E_{\text{min}} = \frac{F_{\text{bolt}}}{S \times t} = \frac{3517.15}{(30,000)(.125)} = .4690299 \text{ in}$$

$$E = E_{\text{min}} + .125 = .5940299 \text{ in}$$

MEOP = maximum expected operating pressure
 D_{casing} = inner casing diameter
 N = Number of Bolts
 $d_{\text{bolt, minor}}$ = minor diameter of the bolt

$\frac{1}{4}$ 20 steel bolts grade 4
 $\sigma_{\text{bolt shear}} = \frac{(\frac{\pi}{4} D_{\text{casing}}^2 \times \text{MEOP})}{(N \times \frac{\pi}{4} d_{\text{bolt, minor}}^2)}$

$N_{\text{bolts}} = \frac{[\frac{\pi}{4} (D_{\text{casing}})^2 \times \text{MEOP}]}{[\frac{\pi}{4} (d_{\text{bolt, minor}})^2 \times S_{\text{bolt shear}}]}$

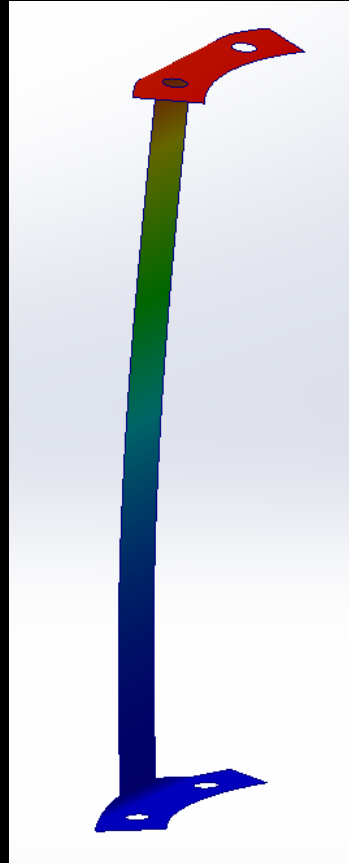
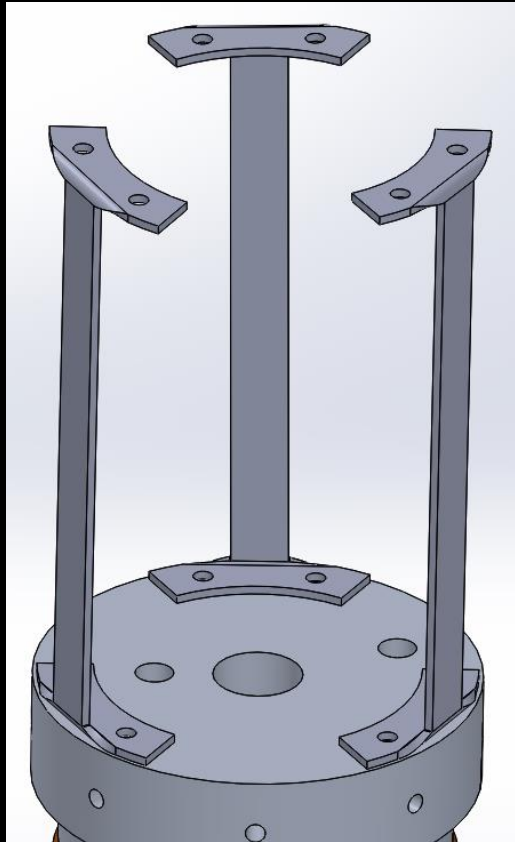
$D_{\text{casing}} = 5.25 \text{ in}$ $\text{MEOP} = 2 \left(\frac{1300}{2000} \right) = 1300 \text{ PSI}$
 $d_{\text{bolt, minor}} = .196 \text{ in}$ $S_{\text{bolt shear}} = 120,000 \text{ PSI}$

$N_{\text{bolts}} = \frac{[\frac{\pi}{4} (5.25)^2 \times 1300]}{[\frac{\pi}{4} (.196)^2 \times 120000]} = 8$

Gold T-6 Aluminum
 8 Bolts

$F_{\text{bolt}} = \frac{\pi D_{\text{casing}}^2 \times \text{MEOP}}{N} = \frac{\pi (5.25)^2 \times 1300}{8} = 3517.7247$

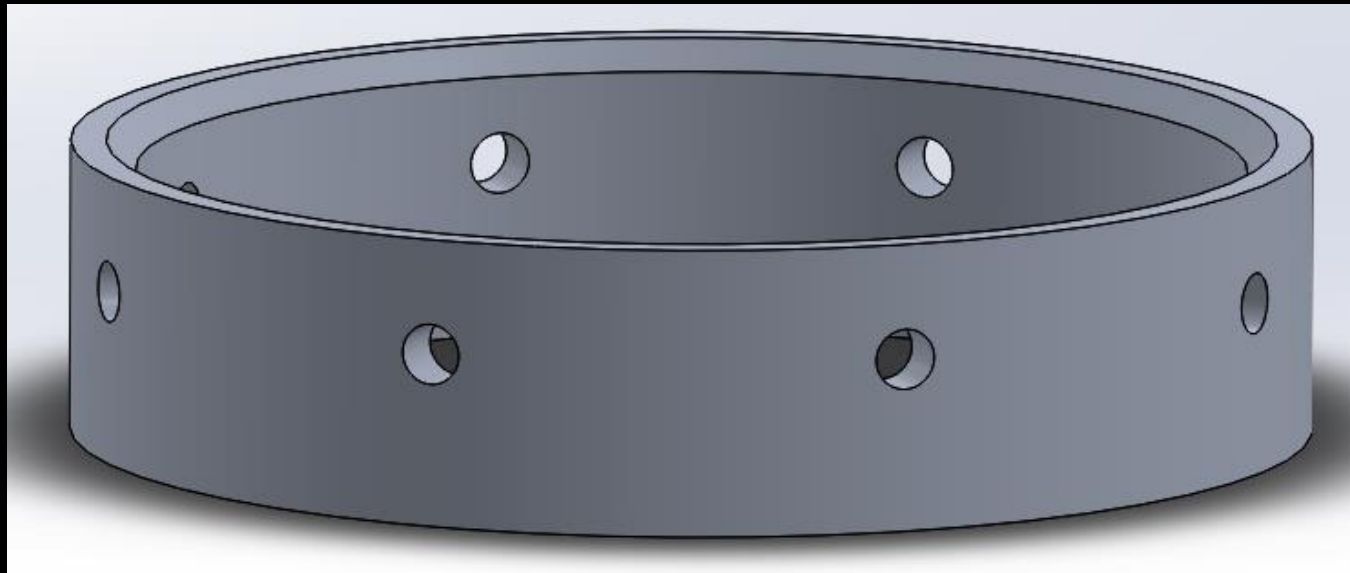
Tank-to-Chamber Chassis



- Purpose: to hold the oxidizer tank in place above the combustion chamber while also protecting the plumbing system from the force of the tank
- Implementation: We will install 3 struts made of cut and bent sheet metal (probably aluminum) that run from the bottom tank bulkhead to the forward closure (bolted directly downward)
- The sheet metal gauge is still being determined
- The chassis will only need to withstand the weight of the tank during the test fire

Nozzle Retaining Ring

- Outer Diameter: 5.25in
- Inner Diameter: 4.7in
- Calculated number of Bolts: 8 (safety factor of 2)
- Holes are .6in from edge (safety factor of 2)
- Alloy: Aluminum 6061-T6
- There will be slant on top of the ring to alleviate stress



Tensile to shear stress.

$$\sigma_{yield}(0.75) = \tau_{yield}$$

Factor of safety

$$FOS = 2$$

Shear stress equation

$$\tau_{yield} = \frac{F_{thrust}}{A_{in\ shear}}$$

Bolt tear-out

$$distance_{edge\ to\ center\ of\ bolt} = 2(diameter_{bolt})$$

Number of bolts for the thrust plate portion of the forward closure

$$\tau_{yield} = \frac{F_{thrust}}{A_{in\ shear}} = \frac{4F}{\pi d^2} = \frac{4(1500 \times FOS)}{\pi(0.196)^2} = 99430.41\ PSI$$

$$\tau_{yield} = 85,000\ PSI(0.75) = 69,000\ PSI$$

$$\tau_{yield} = 69,000 \times N_{bolts}$$

$$N_{Bolts} = \frac{99430.41}{69000} = 1.44 \approx 2\ Bolts$$

For symmetry purposes 4 Bolts

Thickness of thrust plate portion of forward closure (by bolt tear-out)

$$distance_{edge\ to\ center\ of\ bolt} = 2(diameter_{bolt})$$

$$distance = 2\left(\frac{1}{4}\right) = 0.5\ in$$

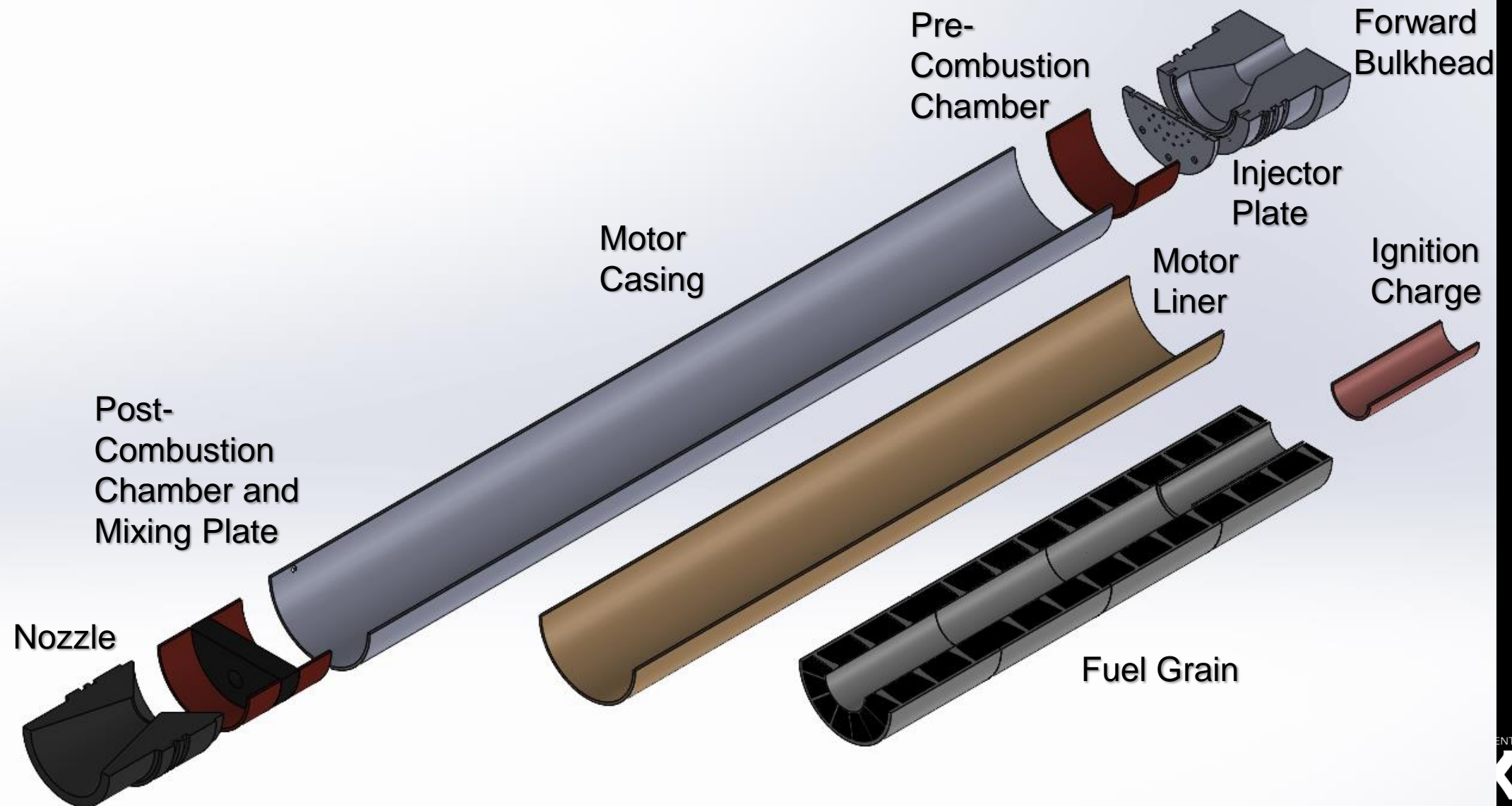
$$thickness = 2(distance) = 2(0.5) = 1\ in$$

Future Plans

- Finalize chassis calculations
- Finalize O-Ring Calculations

Questions?

Combustion Sub-System



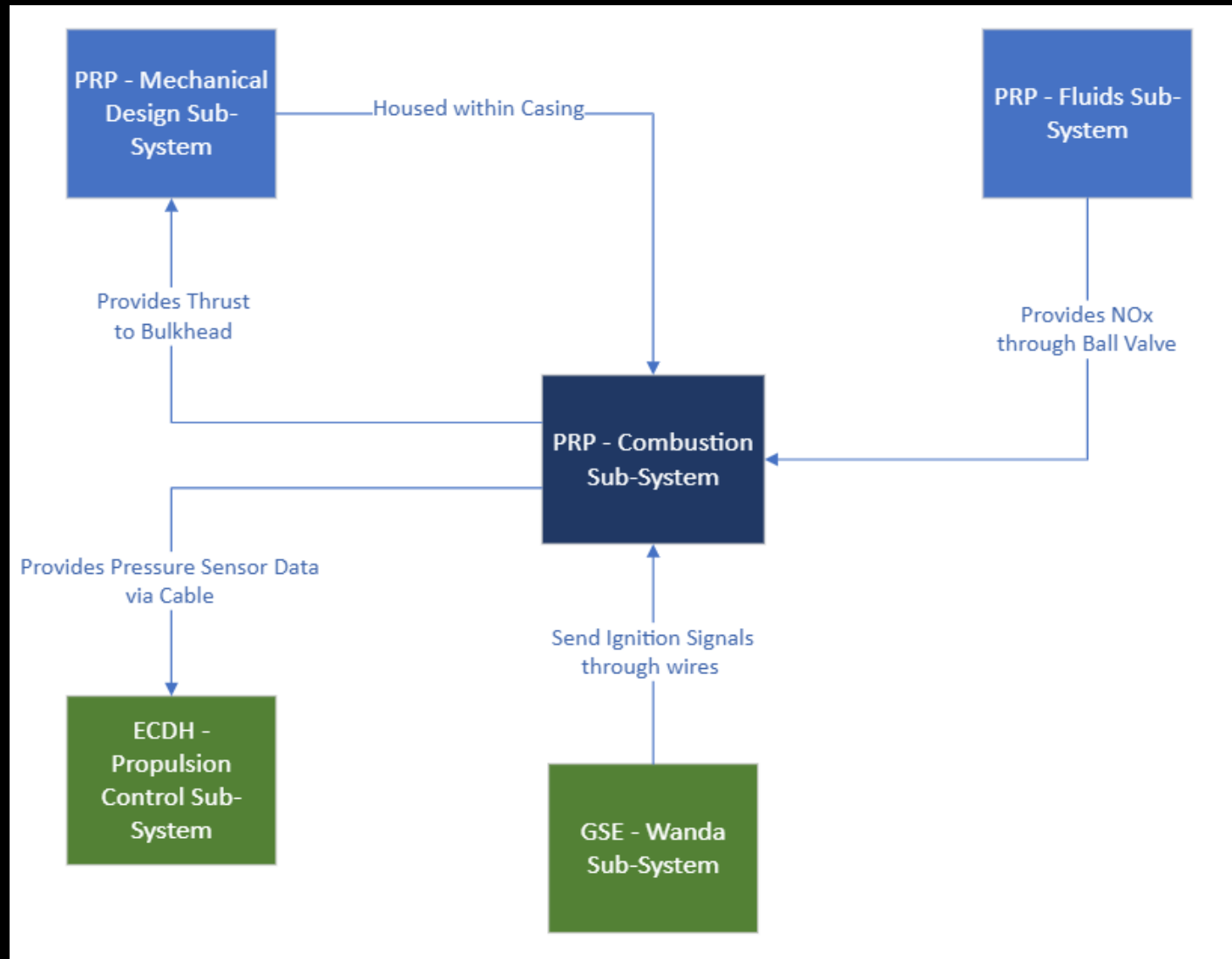
Combustion TPMs

| Measure | TPM Value | Units | Verification Method |
|-------------------------------------|-----------|---------|---------------------|
| Thrust (peak) | [1837] | lbs | Testing |
| Maximum Expected Operating Pressure | [500] | psi | Testing |
| Burn Time | [10-12] | seconds | Testing |
| C* Efficiency | [>93%] | N/A | Testing |
| Impulse | [36,948] | N-s | Testing |

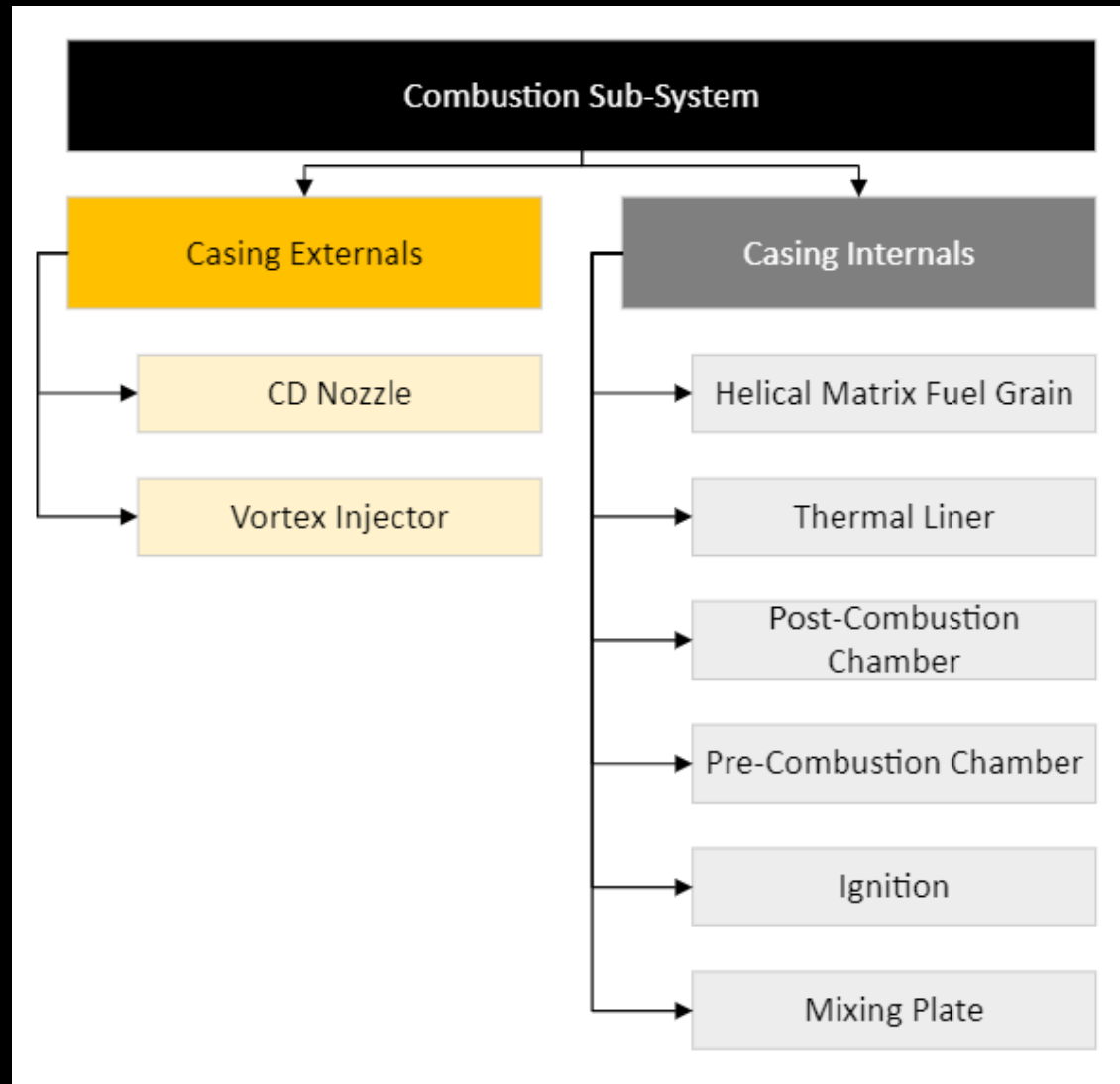
Combustion Sub-System Requirements

| Requirement | Verification Method |
|--|---------------------|
| The Combustion Sub-System shall produce a C* Efficiency of [95%]. | Test |
| The Combustion Sub-System shall adequately mix propellants during combustion. | Demonstration |
| The Combustion Sub-System shall produce a maximum chamber pressure of [500] psi. | Test |
| The Combustion Sub-System shall be housed within the Combustion Chamber Casing. | Inspection |
| The Combustion Sub-System shall have a Burn Time [10] s. | Test |
| The Combustion Sub-System shall provide a stable burn with minimal pressure instabilities. | Test |
| The Combustion Sub-System shall emit non-toxic exhaust. | Inspection |

Combustion Interface Diagram

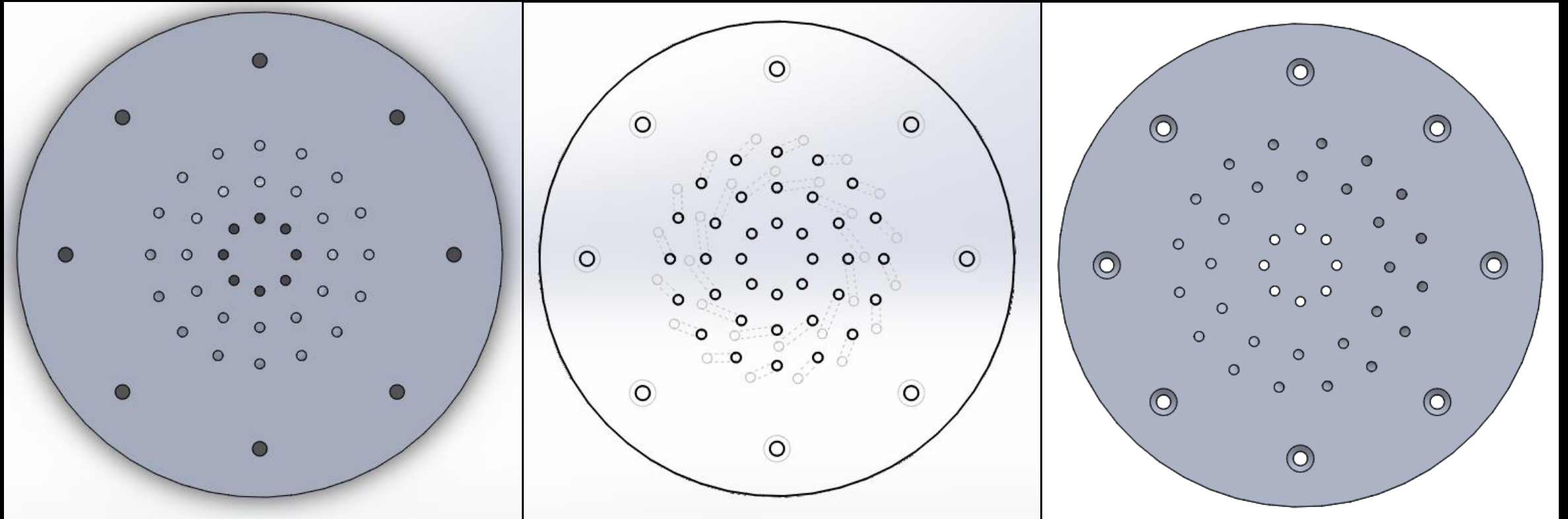


Combustion Component Breakdown



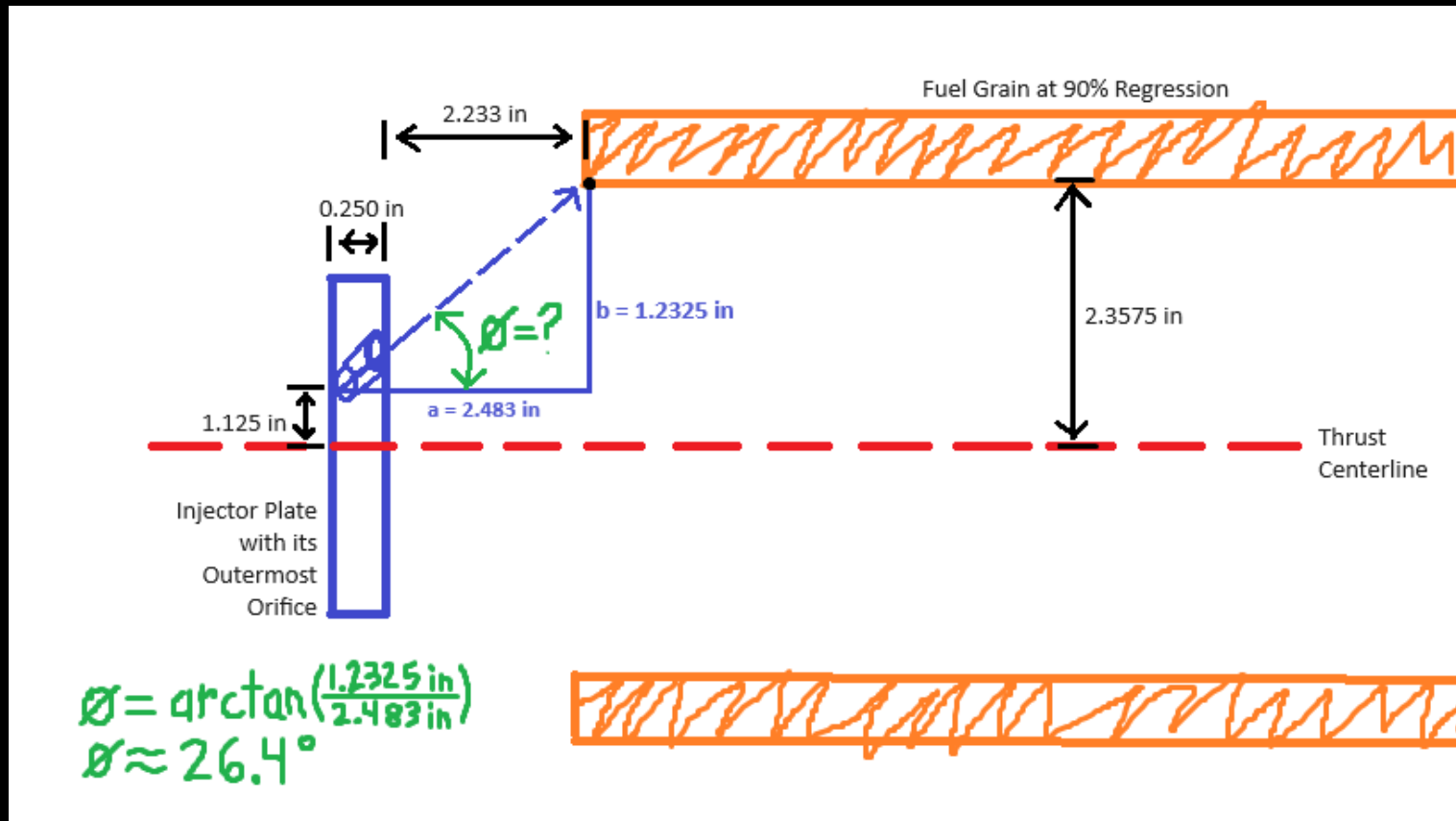
Injector Plate

Visual Representations



Injector Plate

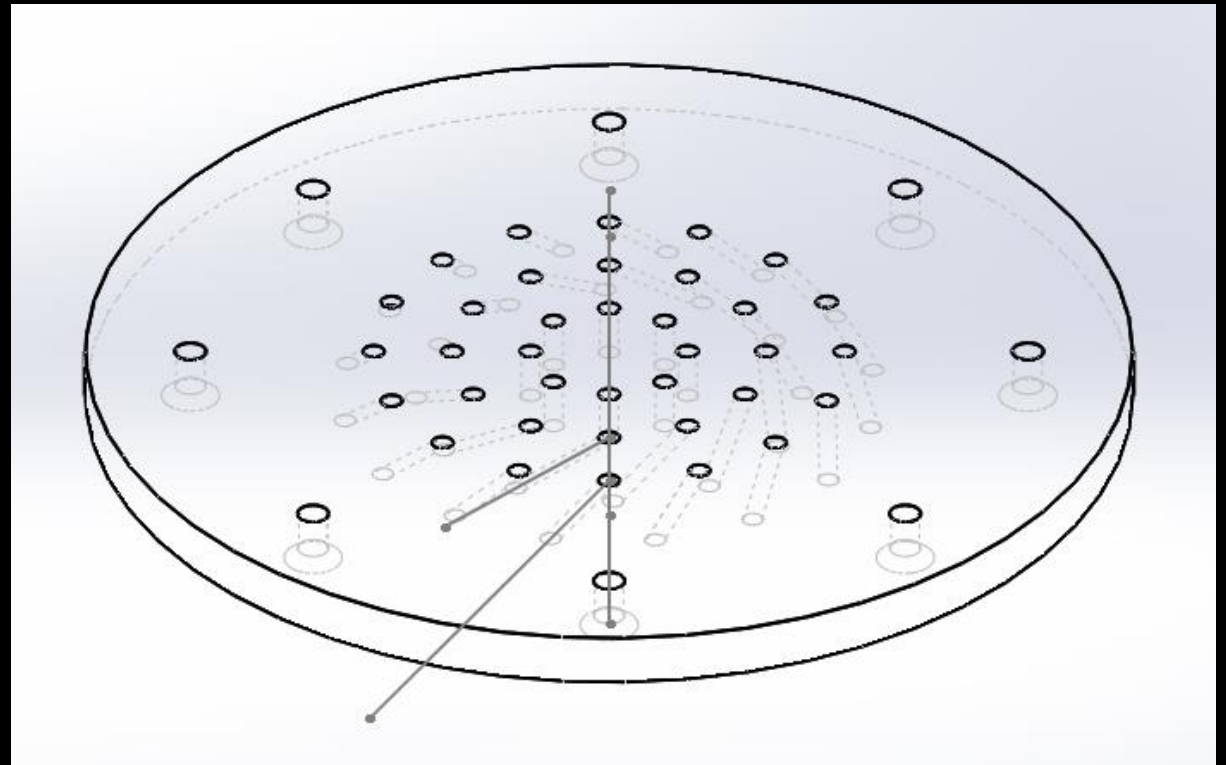
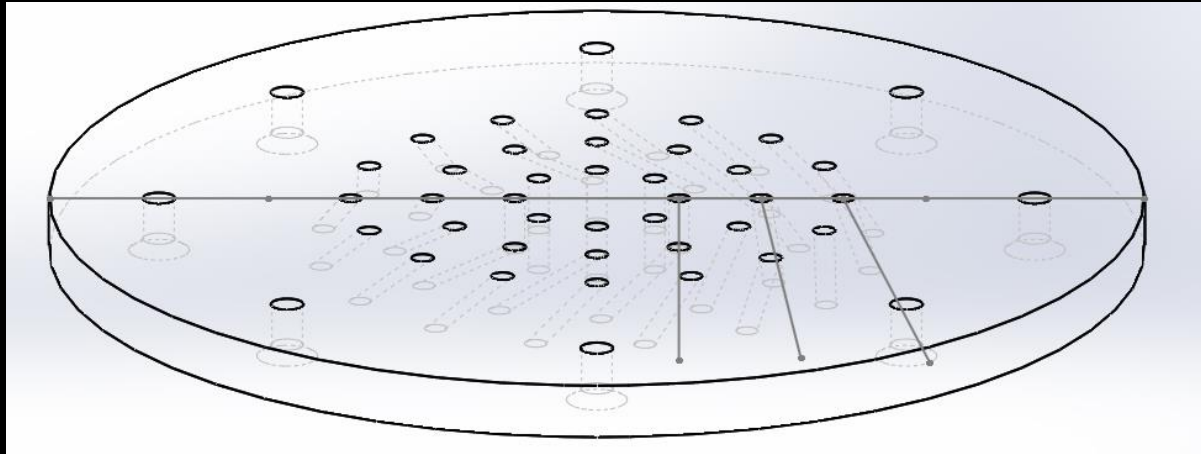
Design Sheets – Orifice’s Angles



| Injector Angle Relative to Thrust Centerline | Angle (degrees) |
|---|---|
| Conical Angle (causes diverging flow) | 26.4 for outer, 13.2 for mid, 0 for inner |
| Vortex Angle (causes vortex flow) | 45 for outer, 60 for mid, 0 for inner |

Injector Plate

Visual Representations (cont.)



Injector Plate

Design Sheets - MathCAD

Chamber Pressure: $P_c := 500 \text{ psi}$

Oxidizer Tank Pressure: $P_{tank} := 800 \text{ psi}$

$$\rho := 589.4 \frac{\text{kg}}{\text{m}^3}$$

Head-Loss Coefficient: $K := 1.7$

Pressure Drop: $\Delta P := P_c \cdot 20\%$

Hole Count: $N := 36$

O/F Ratio: $OF := 5$

Mass Flow Rate:

$$m_{dot} := \frac{F_{thrust}}{I_{sp} \cdot g} = 3.402 \frac{\text{kg}}{\text{s}}$$

Oxidizer Mass Flow Rate:

$$m_{dot_{ox}} := m_{dot} \cdot \left(\frac{OF}{OF + 1} \right) = 6.25 \frac{\text{lb}}{\text{s}}$$

Injector Area:

$$A_{inj} := m_{dot_{ox}} \cdot \sqrt{\frac{2.238 K}{\rho \cdot \Delta P}} = 0.425 \text{ in}^2$$

$$D_{inj} := \sqrt[2]{A_{inj} \cdot \frac{4}{\pi}} = 0.736 \text{ in}$$

Orifice Diameter:

$$d_{orif} := \sqrt{\frac{4 \cdot A_{inj}}{\pi \cdot N}} = 0.12263 \text{ in}$$

Injector Plate

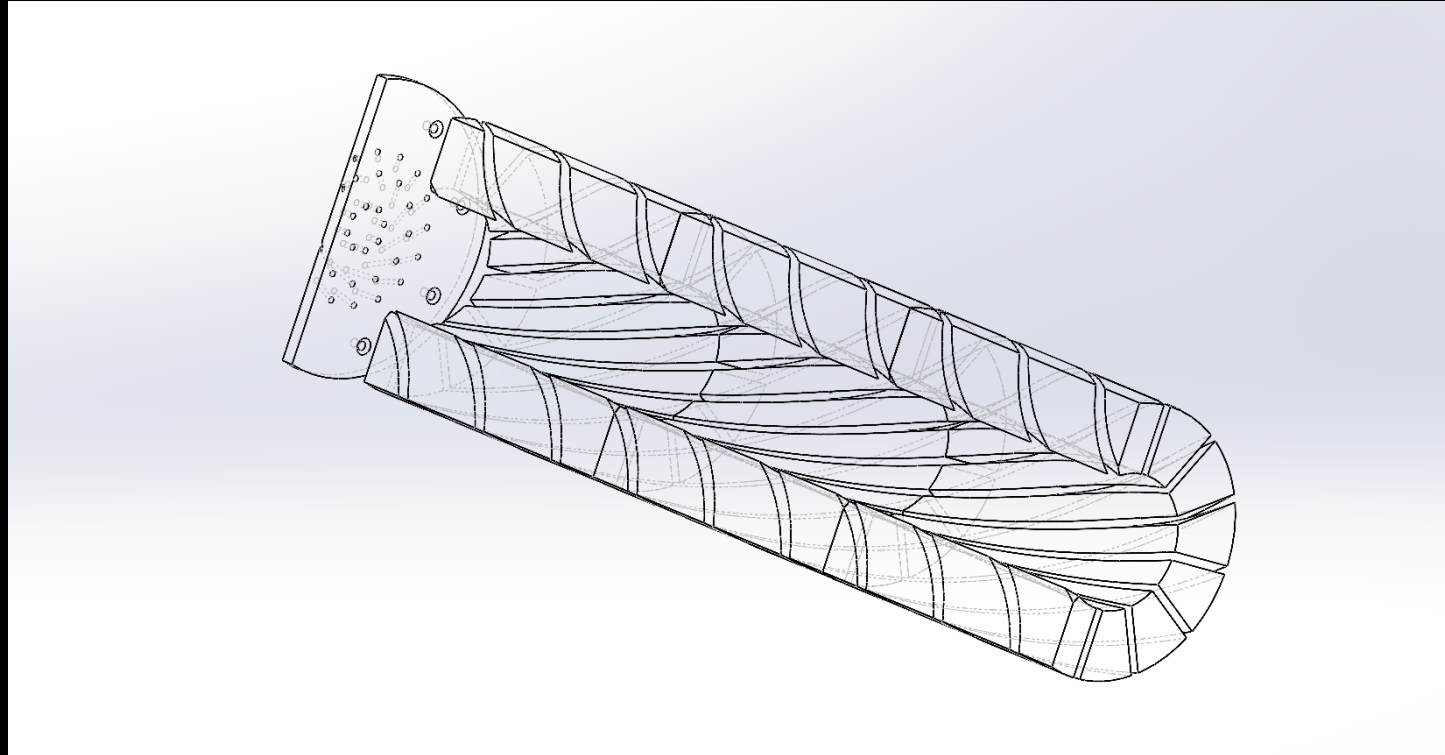
Technical Preliminary Measures

| Measure | TPM Value | Units | Verification Method |
|---------------------|-----------|-------|---------------------|
| Component Weight | [0.467] | lbs | Inspection |
| Mass Flow | [6] | lbs/s | Test |
| Number of Orifices | [36] | N/A | Inspection |
| Diameter of Orifice | [0.122] | in. | Inspection |

Injector Plate

Failure Modes

- Bad alignment with fuel grain geometry.



Injector Plate

Implementation Plans

- Student Researched And Developed Component.
- Vortex injector plate with angled orifices of increased steepness as we get closer to the center of the plate.

Estimated Cost

- Injector Plate - \$ 120

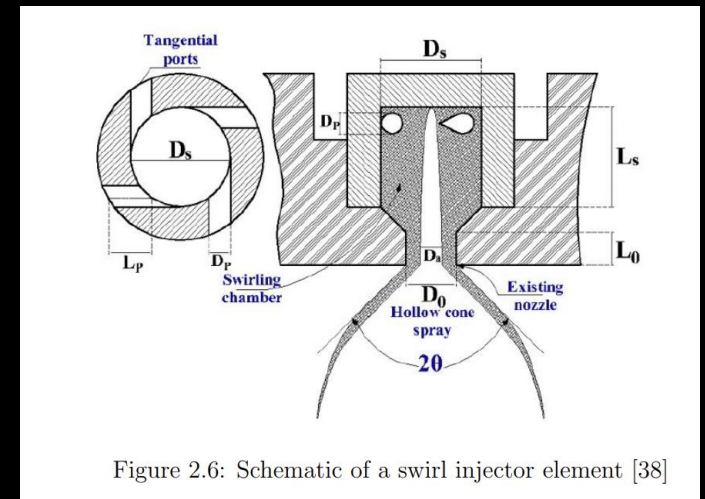
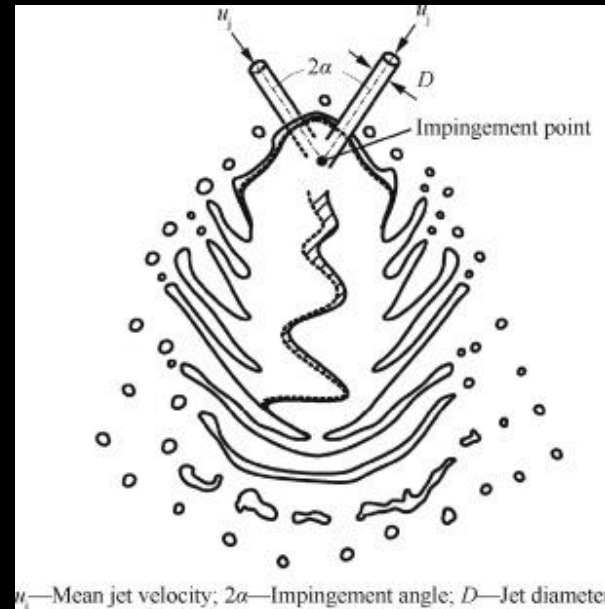
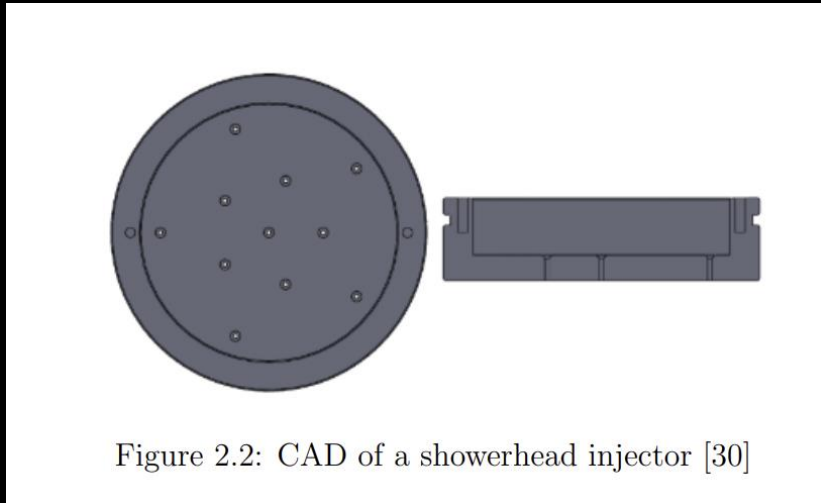
Schedule

- About a week to manufacture, contingent on stock lead times.

Injector Plate

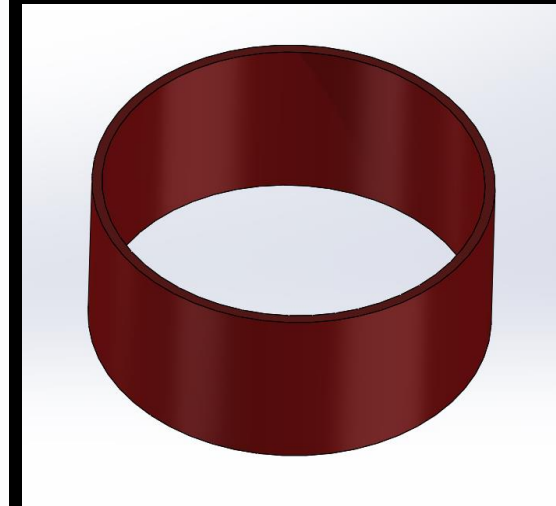
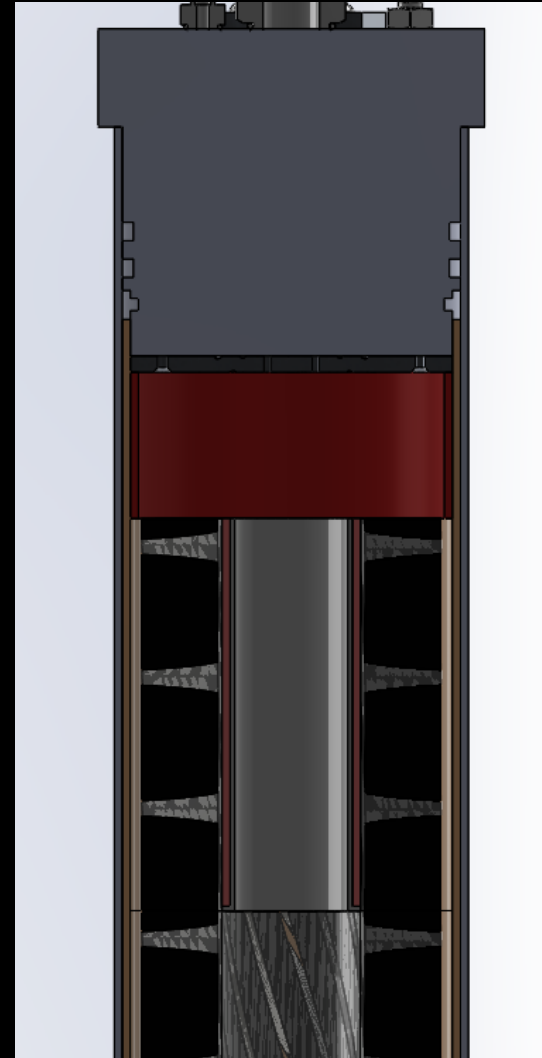
Other Options Considered

- Before deciding on the vortex injector plate, we considered a showerhead injector, an impinging injector, and a swirl injector as well.



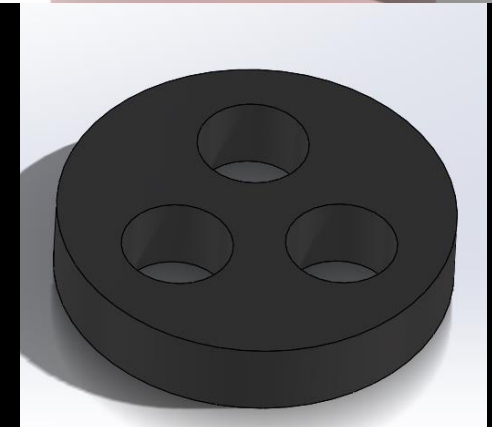
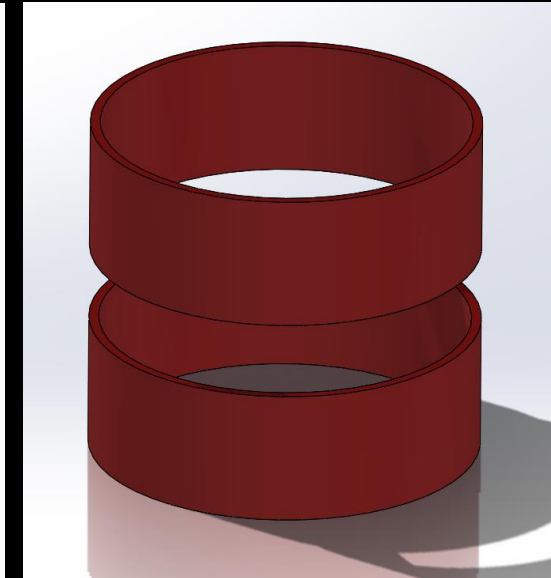
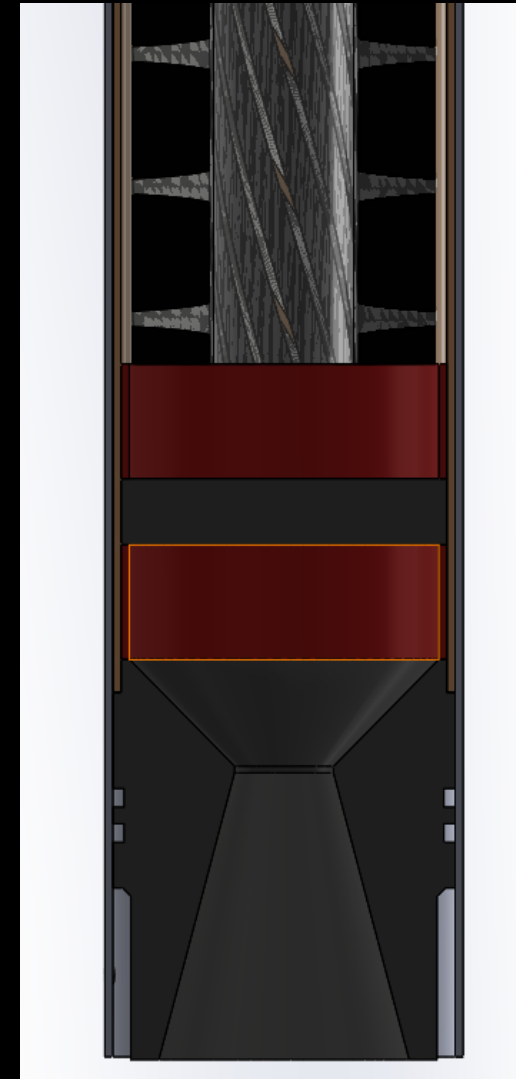
Pre-Combustion Chamber

- Increased NOx residency time
- Pre-heating of NOx
- Ideal length-to-diameter ratio of 0.5
- Actual length 2.233"
- Extra layers of thermal liner
- TPMs
 - Pre-combustion chamber pressure and temperature
 - Pre-combustion chamber dimensions and thickness



Post-Combustion Chamber and Mixing Plate







- Propellants allowed to fully mix
- Ideal length-to-diameter ratio of 1
- Ideal post-pre chamber length ratio 2
- Actual length 4.467" including 1" mixing plate
- Additional thermal liner layers
- Graphite mixing plate further mixes propellants
- TPMs
 - Post-combustion chamber pressure and temperature
 - Post-combustion chamber dimensions and thickness



Pre- and Post- Combustion Chambers

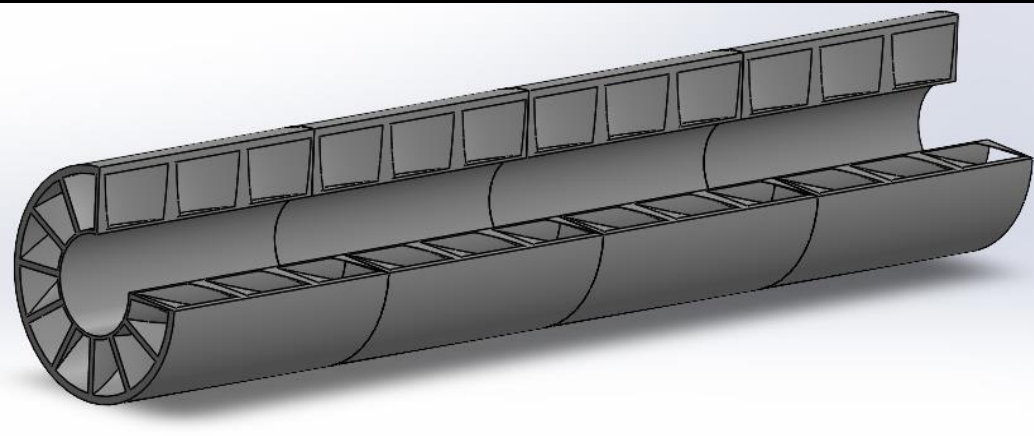
- Considered a design without these features
 - Decided implementation would increase efficiency
- Considered different mixing plate geometries
 - Current geometry suggests highest combustion efficiency
- Failure modes
 - Burn through liners
 - Introduction or strengthening of pressure oscillations

Table 1: All simulated post-combustion chamber devices with resulting c^* efficiency (ideal $c^* = 1520$ m/s)

| ID | Shape | η_{comb} achieved |
|----|---|------------------------|
| 1 | no mixer | 87.2 % |
| 2 |  | 79.1 % |
| 3 |  | 84.0 % |
| 4 |  | 87.3 % |
| 5 |  | 87.6 % |
| 6 |  | 91.4 % |
| 7 |  | 92.4 % |

Fuel Grain Geometry

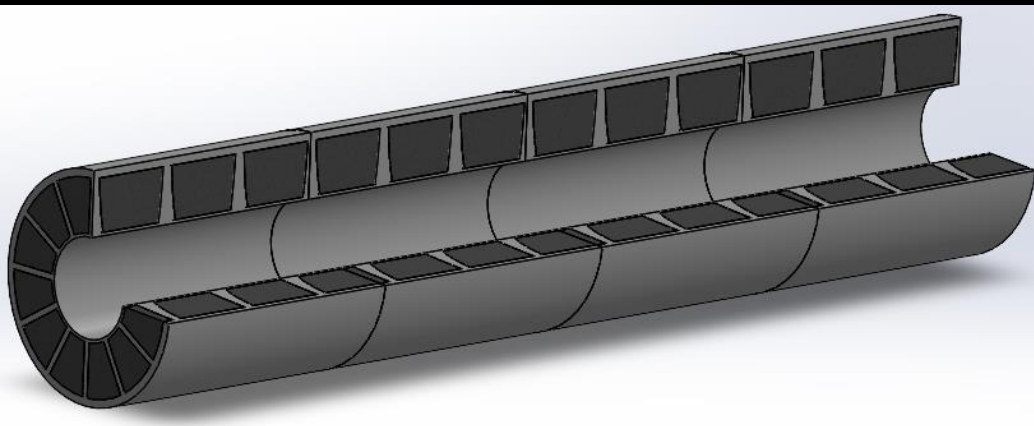
- Nested helical port geometry



Helical
ABS
matrix

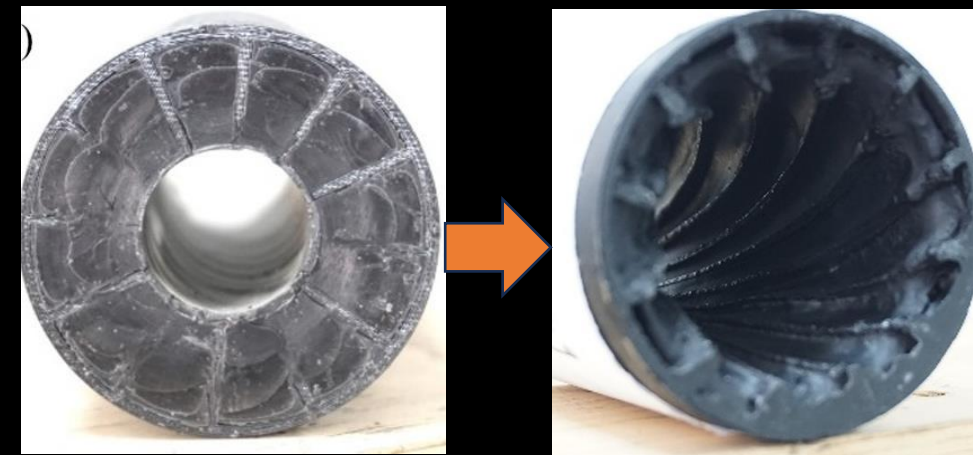
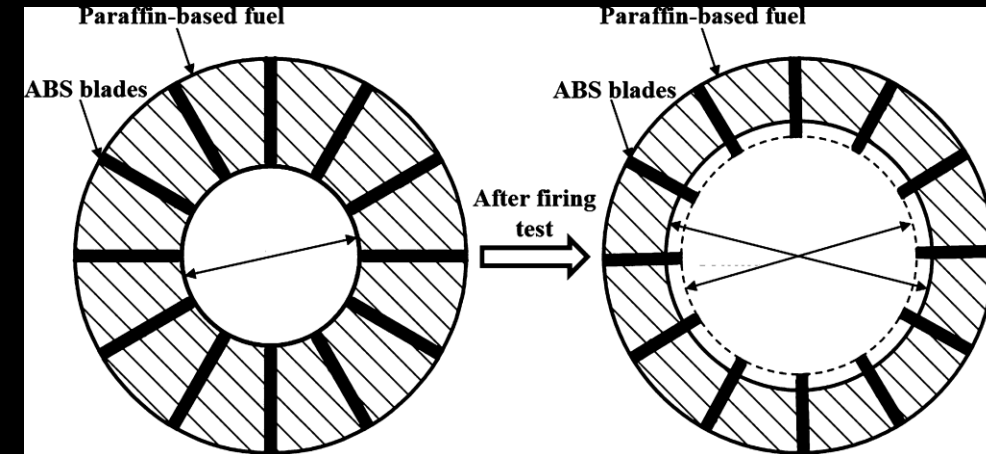
Grain Specs:

- OD: 5 in
- ID: 2.15 in
- Length: 24 in



ABS
matrix +
paraffin
based
fuel

Regression behavior of nested helical grain



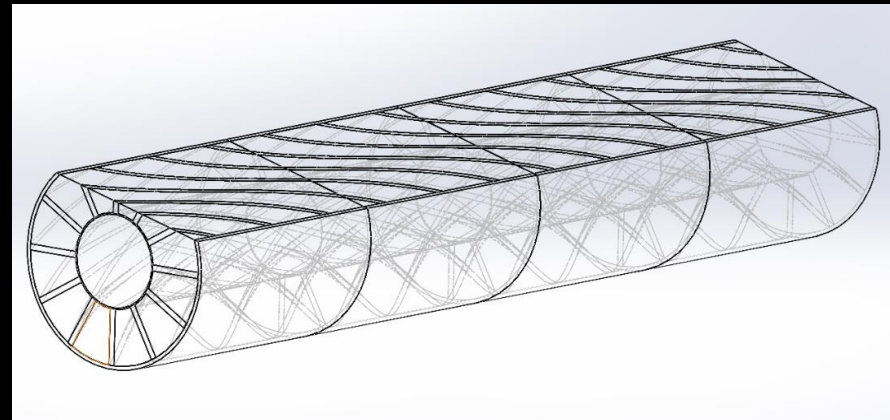
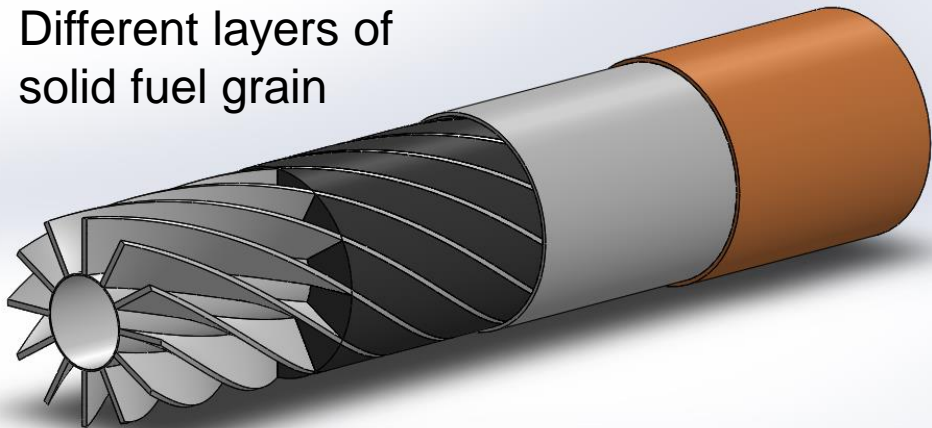
Test-firing results of
nested helical grain

Fuel Grain Geometry

Benefits of nested helical geometry

- Increase in structural integrity of the grain
- Turbulence due to blade vortices formed in the recirculation zone
- Maintains angular momentum of NO_x along the grain
- Longer NO_x residence time in the chamber
- Higher instantaneous burning surface area
- Ideal web fraction & volumetric loading efficiency
- Benefits increase over time due to difference in regression rates between ABS and paraffin

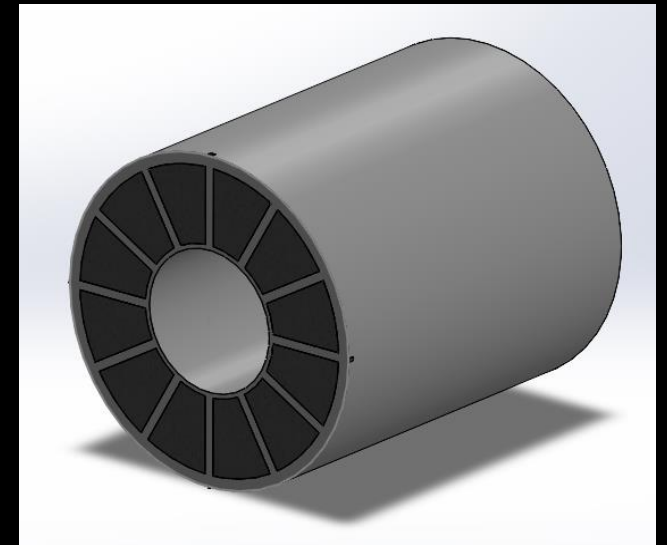
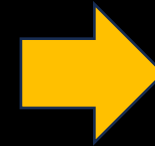
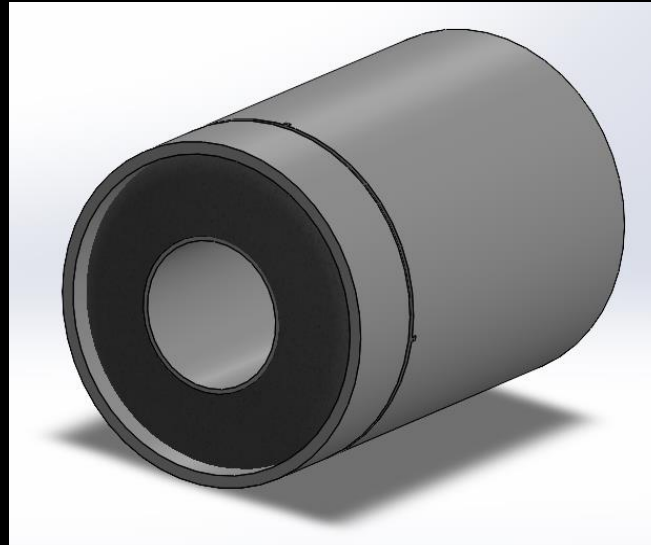
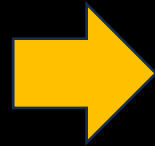
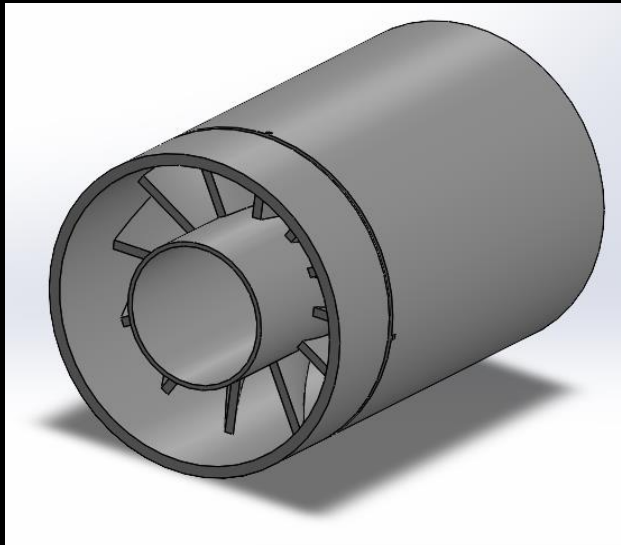
Different layers of solid fuel grain



Transparent cross-sectional view of solid fuel grain

Fuel Grain Geometry

- ABS matrix manufacturing technique
 - Matrix made using additive manufacturing (3D printing) w/ thermoplastic ABS
 - Matrix will serve as mold for paraffin-based fuel which will be poured into negative space between adjacent fins
 - After paraffin wax has cooled, grain will be cut down to 6 inches along the pictured groove to create a flush surface



Fuel Grain Geometry

- General specs chart (see right)
- Modes of failure
 - Grain loses structural integrity and breaks up
 - Grain does not ignite
 - Grain ignites poorly and causes sputtering start
 - Grain causes uneven burn (burn-through occurs)
 - Grain structure impedes NOx flow

| Fuel Grain Dimension | Expected value |
|----------------------------------|--------------------------------------|
| Initial inner port diameter (ID) | 2.15 in |
| Outer grain diameter (OD) | 5 in |
| ABS Fin Width | 0.143 in |
| Outer ABS Layer Width | 0.179 in |
| Inner ABS Layer Width | 0.0717 in |
| Fuel Grain Length | 24 in (four 6-in segments) |
| Pitch of Helical Fins | 24 in (one full 360-degree rotation) |
| Total Initial Grain Thickness | 1.425 in |
| Total Grain Volume | 384.08 in ³ |
| Total Grain Weight | 15.8 pounds |

Fuel Grain Composition

- Previously consisted of sorbitol and paraffin; with a mixture ratio of ~4:1
- With the novel fuel grain geometry, this year's fuel grain mixture ratio will consist of a ~3:2 ratio due to the nested helical providing more structural support to the fuel grain.

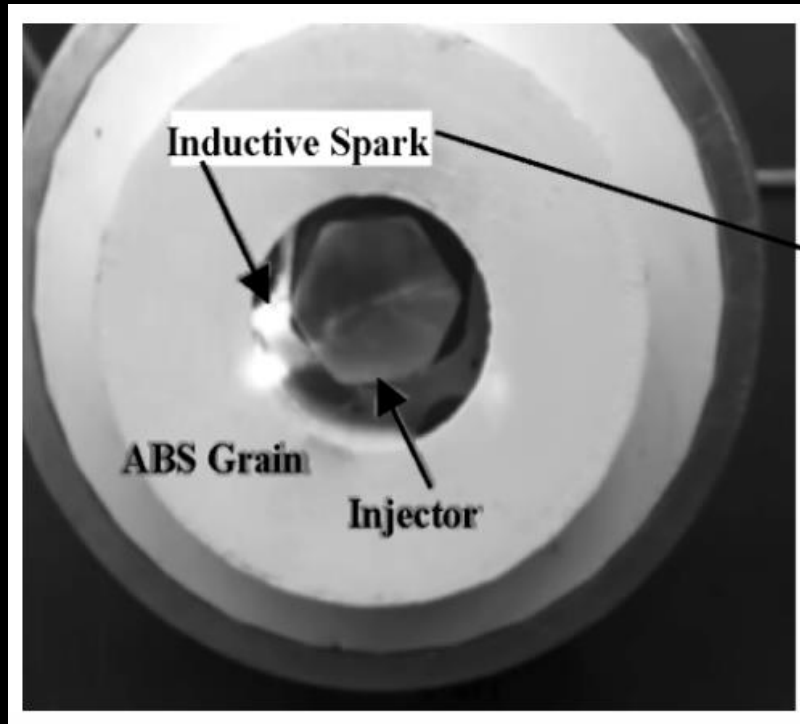


Fuel Grain Composition

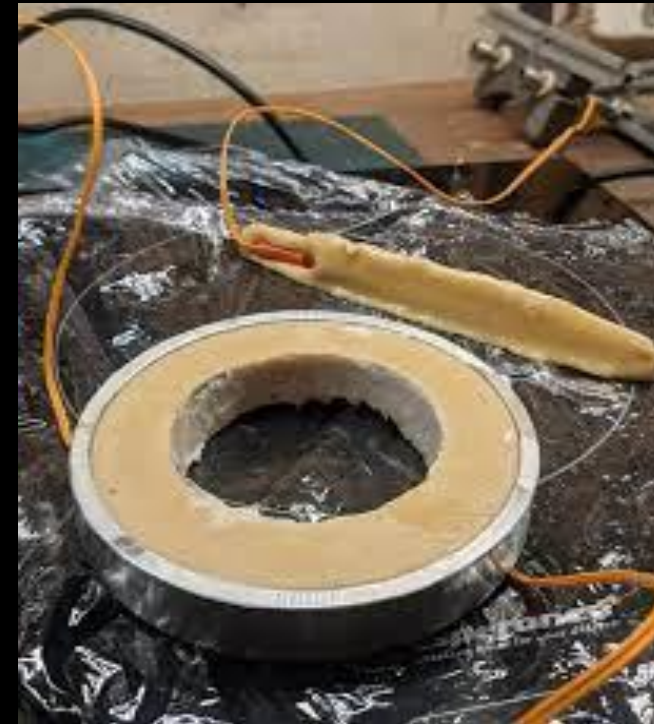
| Chemical | %weight |
|-----------------|---------|
| Sorbitol | 55 |
| Paraffin | 35 |
| Aluminum Powder | 10 |

- This year we will be looking into potential additives to create better combustion characteristics.
- Some potential additives include: Metallic Boride Powders, Carbon Powders, and other metal powders.

Ignition Mechanism

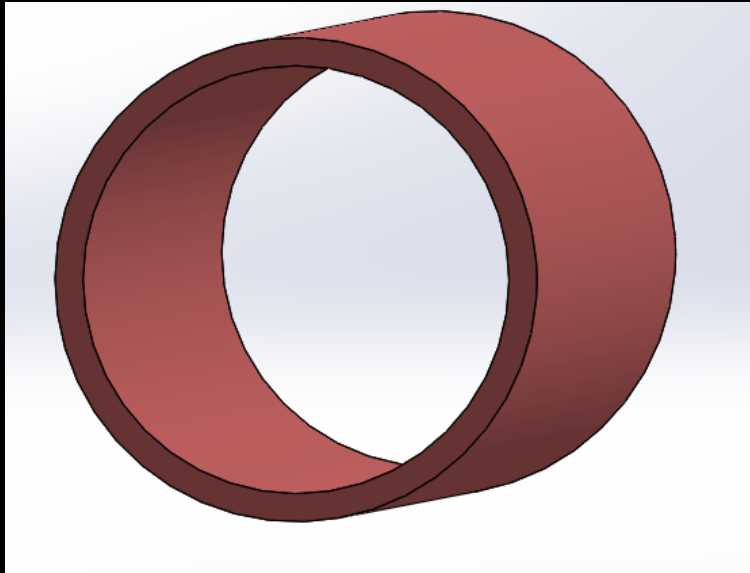


The ignition mechanism is positioned within the ABS Matrix for efficient fuel heating – this image shows interaction of the spark with the grain.



The design of the ignition mechanism is a "puck" shape, modified to assimilate with the fuel grain geometry.

Ignition Mechanism



Structural representation of igniter

Material Test Cases

| Fuel | Oxidizer | Additive |
|--------------------|-----------------------|-------------------|
| Sucrose | Potassium Nitrate | Epoxy |
| Thermite (Al & Fe) | Potassium Perchlorate | Iron Oxide (rust) |

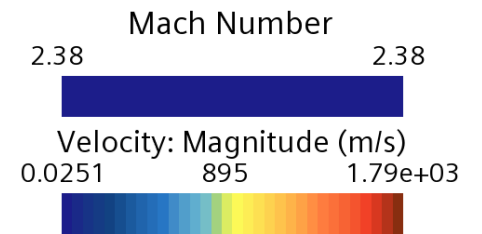
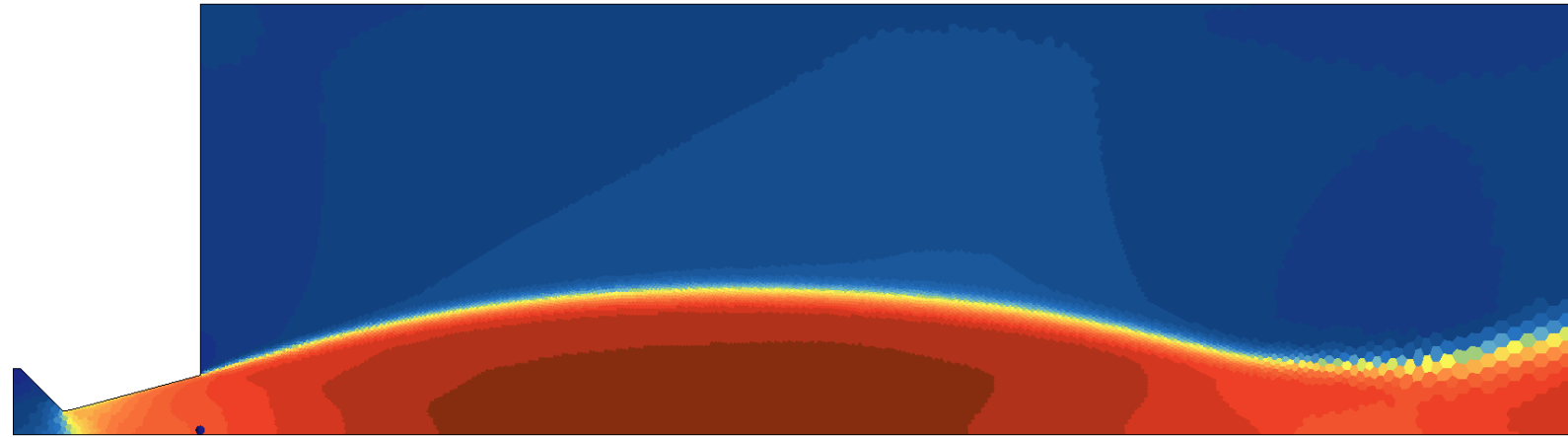
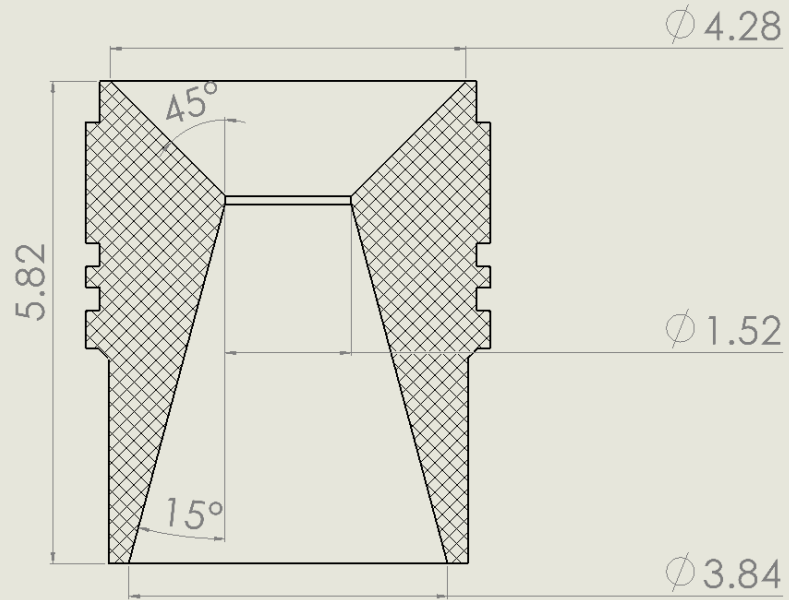
Nozzle

Design Sheet - MathCAD

| | | | | | |
|--------------------------------|--|-----------------------------|--|---|---|
| Chamber Pressure: | $P_c := 500 \text{ psi}$ | Throat Temperature: | $T_T := \frac{T_c}{\left(1 + \frac{k-1}{2}\right)} = (2.014 \cdot 10^3) \text{ K}$ | Thrust: | $F_{thrust} := TWR \cdot M_{total} \cdot g = (6.672 \cdot 10^3) \text{ N}$ |
| Oxidizer Tank Pressure: | $P_{tank} := 800 \text{ psi}$ | Throat Area: | $A_T := \frac{m_{dot}}{P_T} \left(\sqrt{\frac{R \cdot T_T}{M_{mol} \cdot k}} \right) = 1.808 \text{ in}^2$ | Mass Flow Rate: | $m_{dot} := \frac{F_{thrust}}{I_{sp} \cdot g} = 3.402 \frac{\text{kg}}{\text{s}}$ |
| | $\rho := 589.4 \frac{\text{kg}}{\text{m}^3}$ | Mach Number at Exit: | $M_{exit} := \sqrt{\left(\frac{2}{k-1}\right) \cdot \left(\left(\frac{P_c}{P_a}\right)^{\frac{k-1}{k}} - 1\right)} = 2.901$ | Oxidizer Mass Flow Rate: | $m_{dot_{ox}} := m_{dot} \cdot \left(\frac{OF}{OF+1}\right) = 6.25 \frac{\text{lb}}{\text{s}}$ |
| Head-Loss Coefficient: | $K := 1.7$ | Throat Radius: | $r_T := \sqrt{\frac{A_T}{\pi}} = 0.759 \text{ in}$ | Gas Constant: | $R := 8.31 \frac{\text{J}}{\text{mol} \cdot \text{K}}$ |
| Pressure Drop: | $\Delta P := P_c \cdot 20\%$ | Exit Area: | $A_e := \left(\frac{A_T}{M_{exit}}\right) \cdot \left(\frac{1 + \left(\frac{k-1}{2}\right) \cdot (M_{exit})^2}{\frac{k+1}{2}}\right)^{\frac{k+1}{2k-2}} = 11.586 \text{ in}^2$ | Chamber Temperature: | $T_c := 2184.73 \text{ K}$ |
| Hole Count: | $N := 35$ | Area Ratio: | $\varepsilon := \frac{A_e}{A_T} = 6.41$ | Atmospheric Pressure: | $P_a := 12.2 \text{ psi}$ |
| O/F Ratio: | $OF := 5$ | Exit Radius: | $r_e := \sqrt{\frac{A_e}{\pi}} = 1.92 \text{ in}$ | Specific Heat Ratio (for exhaust): | $k := 1.17$ |
| Specific Impulse: | $I_{sp} := 200 \text{ s}$ | | | Throat Pressure: | $P_T := P_c \cdot \left(1 + \frac{k-1}{2}\right)^{\frac{k}{k-1}} = (1.966 \cdot 10^6) \text{ Pa}$ |

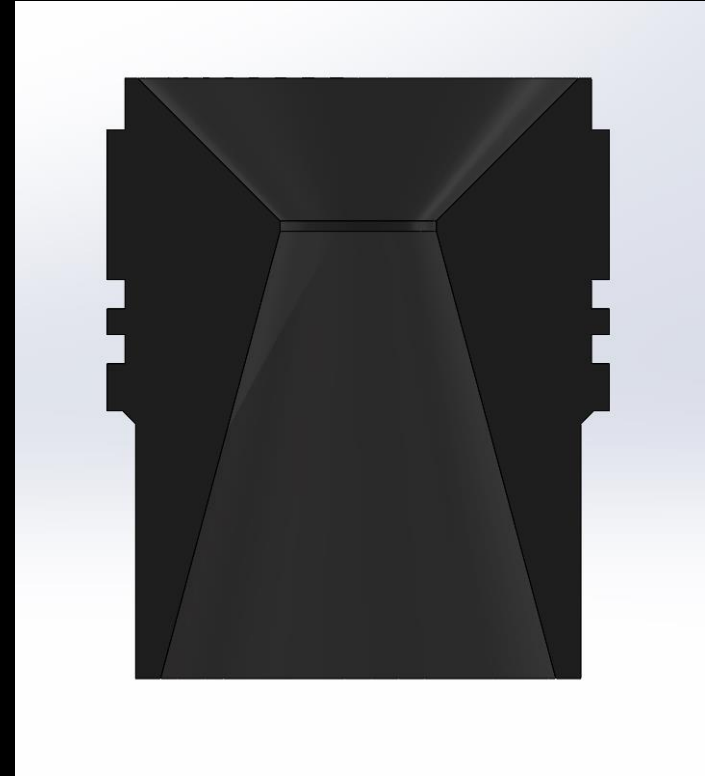
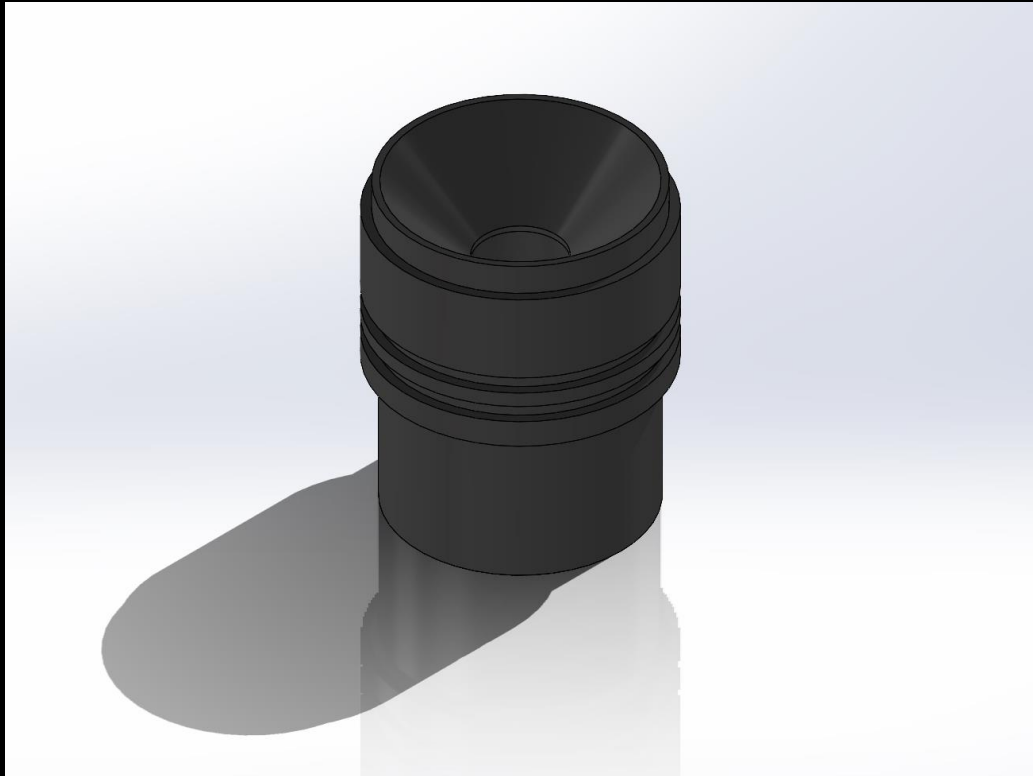
Nozzle

ANSYS Fluent Verification



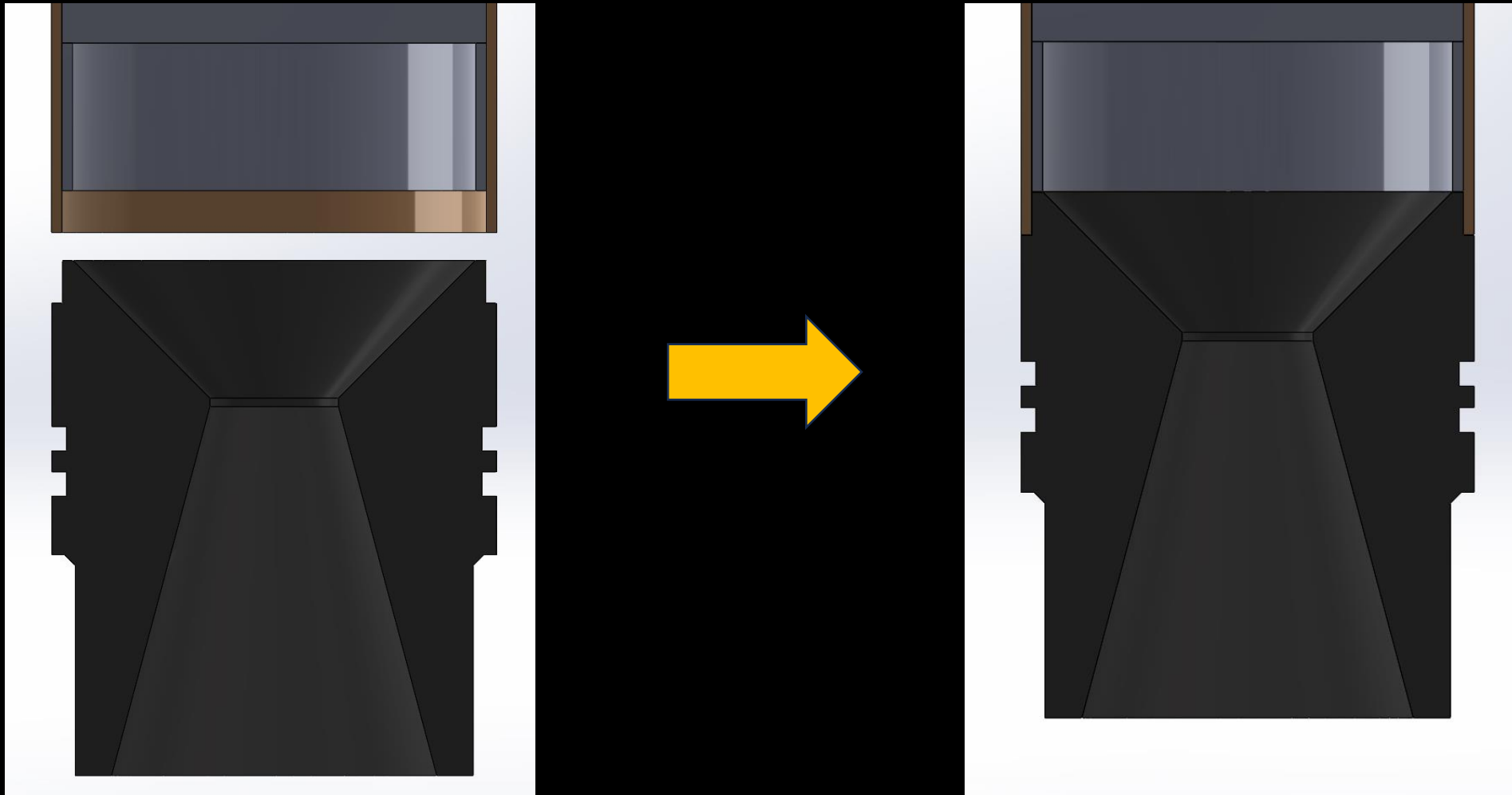
Nozzle

Visual Representations



Nozzle

Interfacing



Nozzle

Implementation Plans

- COTS
- Graphite Nozzle

Estimated Cost

- 2 Nozzles - \$ 350 each

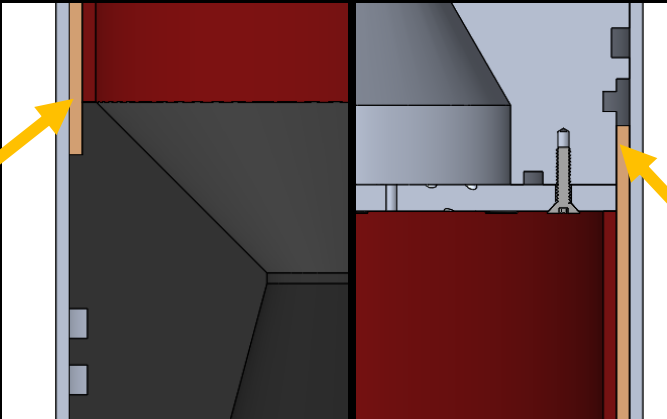
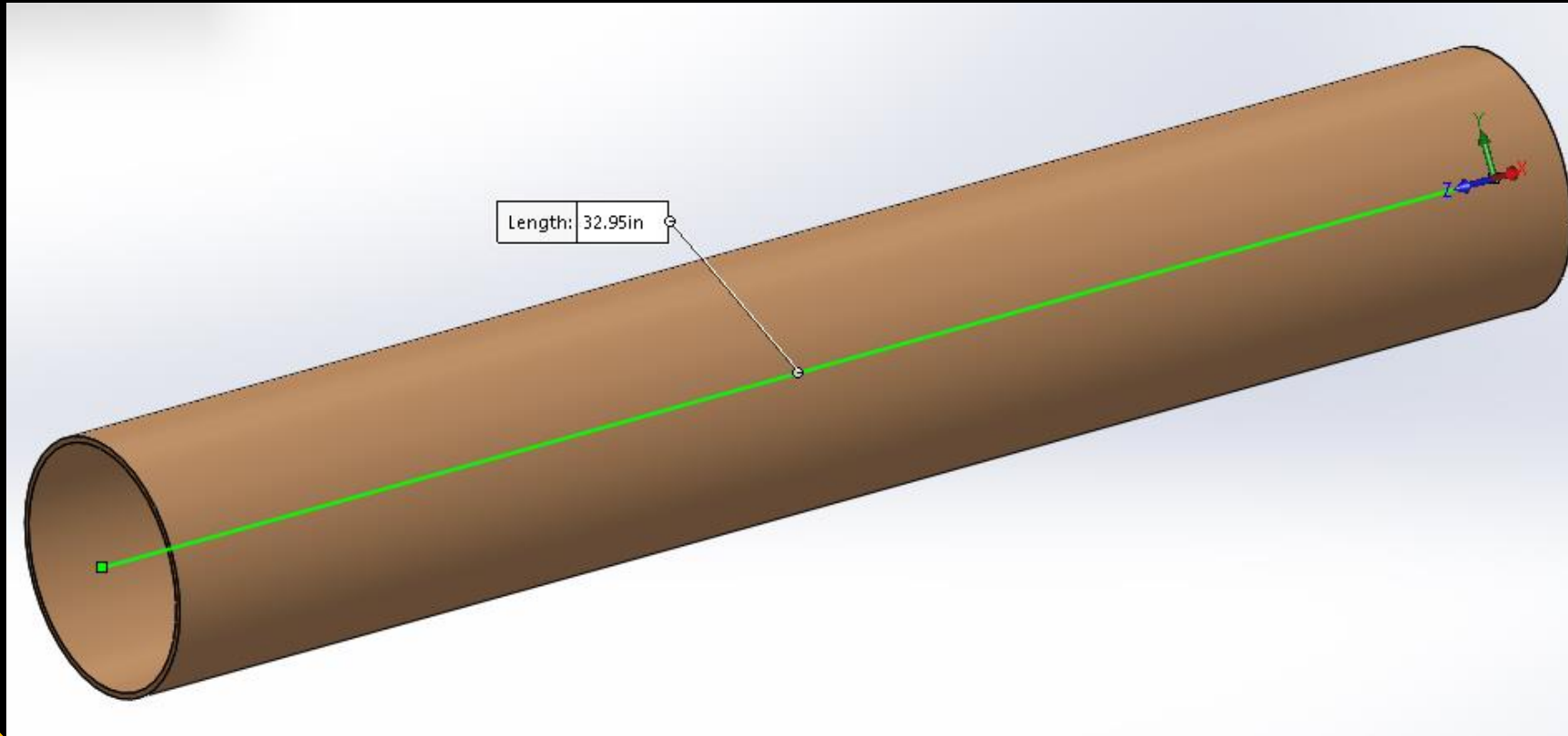
Schedule

- Two+ weeks from order to delivery, material lead time may extend timeline

Ablative Thermal Liner

Approach

- SRAD Phenolic Fiberglass
- ~33 inches long, subject to change
- ~0.125" thick
- Integrates with nozzle and injector to contain combustion



Ablative Thermal Liner

Options Considered

- G-10
- Fibrous Refractory Composite Insulation (FRCI)
- Lamitex XX
- Lamitex CE

Notes

- G-10, XX, and CE share same manufacturer and 94HB Flammability Rating
- Lamitex prices through quotes

Option Information

| Product | Type of Material | Tensile Strength | Highest Rated Temp. |
|--|---------------------------|----------------------|---------------------|
| G-10 | Fiberglass | 38000-65000 | 140C (284F) |
| Fibrous Refractory Composite Insulation (FRCI) | Ceramic Composite | 876 PSI (Flexural) | 1540C (2804F) |
| Lamitex XX | Phenolic Paper Composite | 18850 PSI (Flexural) | 140C (284F) |
| Lamitex C | Cotton Phenolic Composite | 13500 PSI (Flexural) | 125C (257F) |

Ablative Thermal Liner

Strength Information for 0.125" Sheets

| Product | Flexural | Tensile | Compressive |
|------------|---|---------------------------------------|----------------------------|
| G-10 | LW: 55 kpsi CW: 45 kpsi | LW: 40 kpsi CW: 32 kpsi | FW: 68 kpsi EW: 35 kpsi |
| Lamitex XX | LW: 15 kpsi CW: 14 kpsi | LW: 16 kpsi CW: 13 kpsi | FW: 34 kpsi |
| Lamitex CE | (.062") LW: 17.5 kpsi CW: 15.5 kpsi | (0.125") LW: 11 kpsi CW: 9 kpsi | (.5") FW: 34 kpsi |

LW: Lengthwise, CW: Crosswise, FW: Flatwise, EW: Edgewise

Questions?

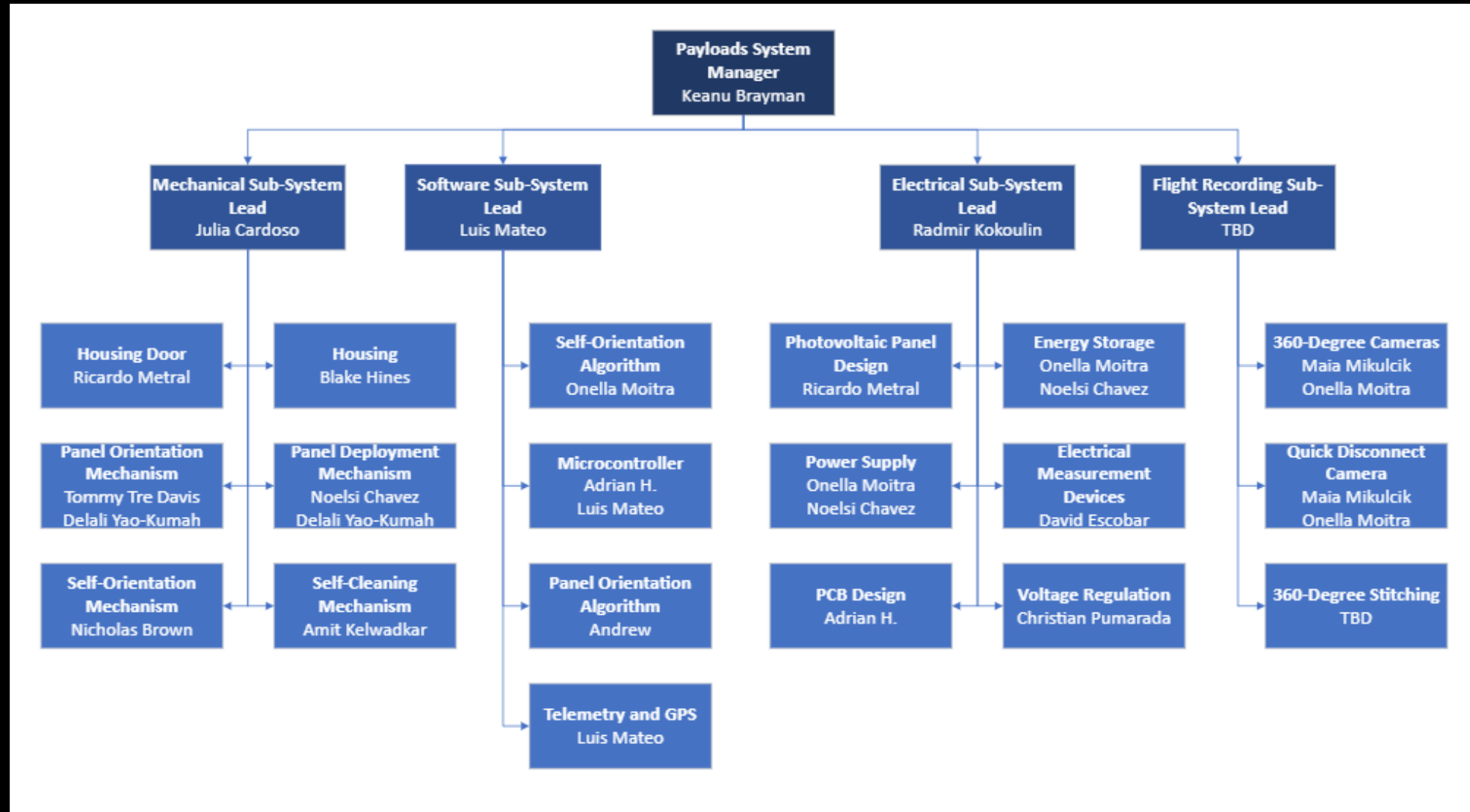
Payloads System Concept Review

Spaceport America Cup 2024

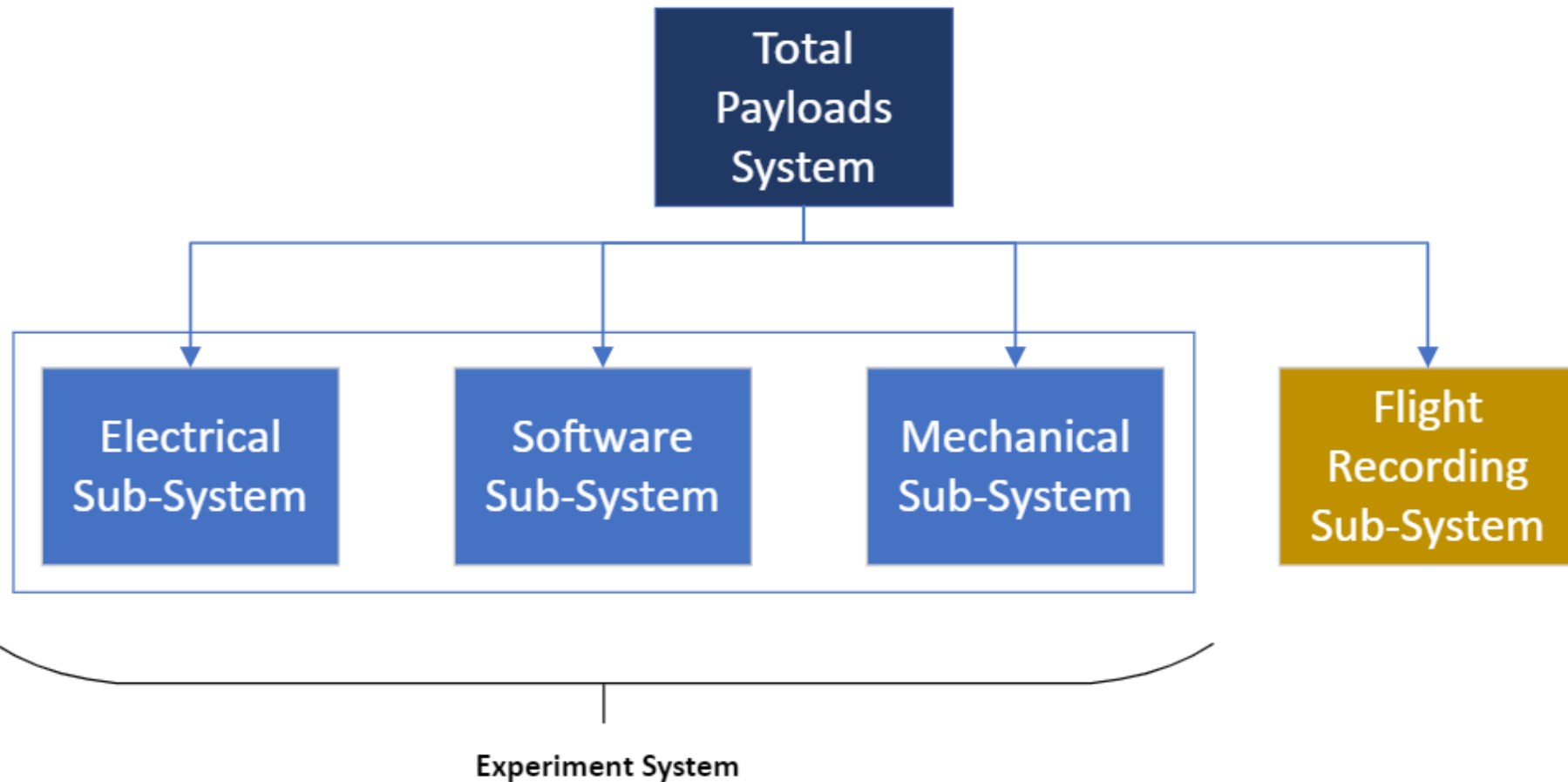
IREC 30k SRAD Hybrid Engine, Experimental Payloads, and Airframe

10/30/23

Payloads System Organization Chart



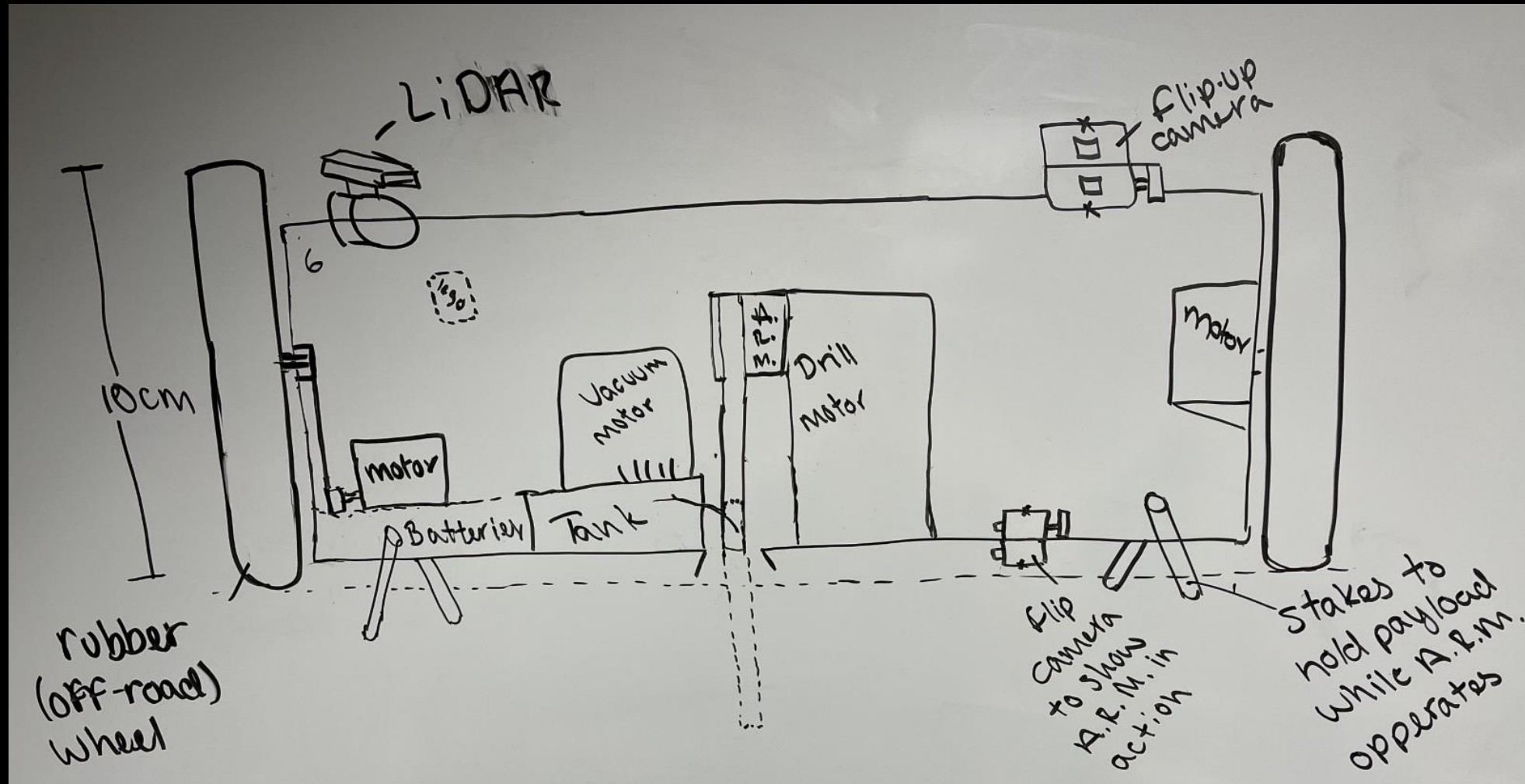
Payloads System Breakdown



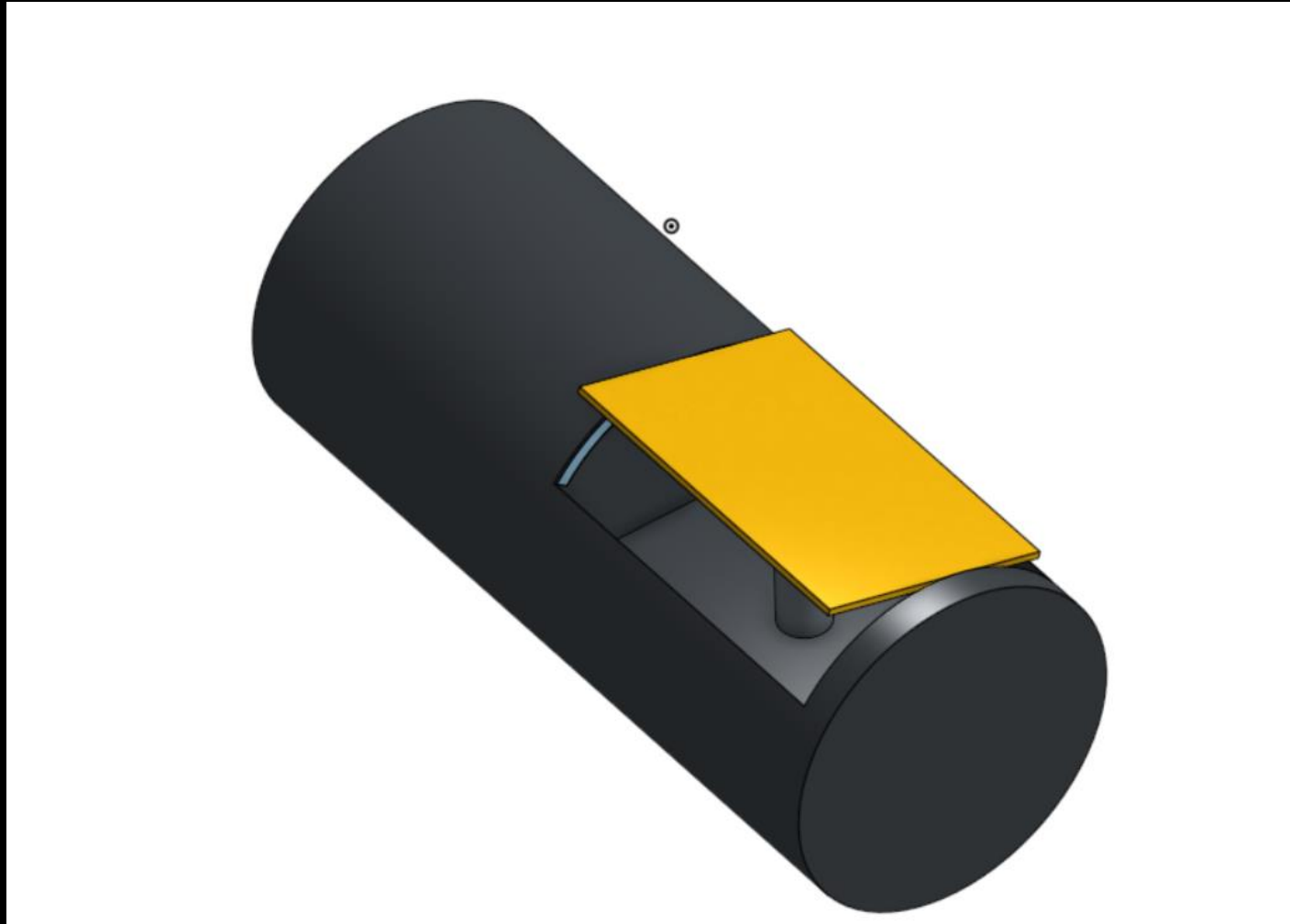
Proposal Solicitation

| | |
|-----------------------------|---|
| Background | 3 |
| Objective | 3 |
| Proposal Preparation | 3 |
| Format | 3 |
| Cover Sheet (1 page) | 4 |
| Main Body (1-3 pages) | 4 |
| Appendix (1 page) | 5 |
| Submission | 5 |
| Evaluation | 5 |

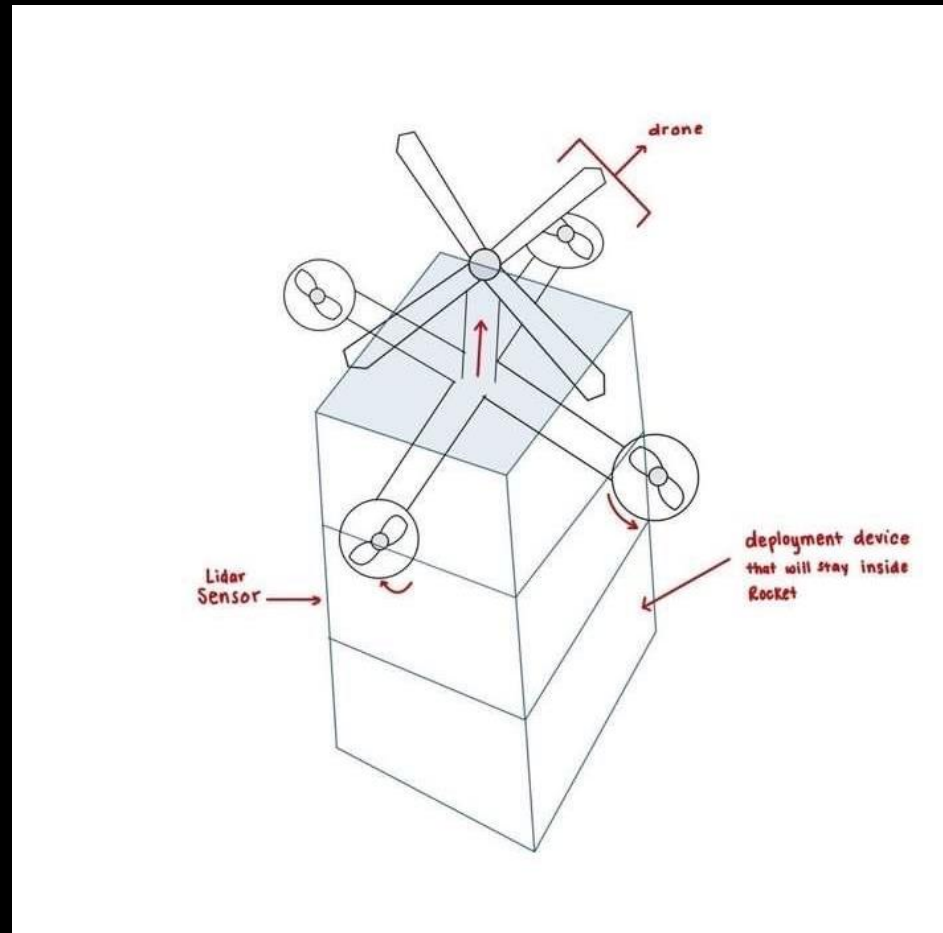
Concept #1: Ground Sampling Rover



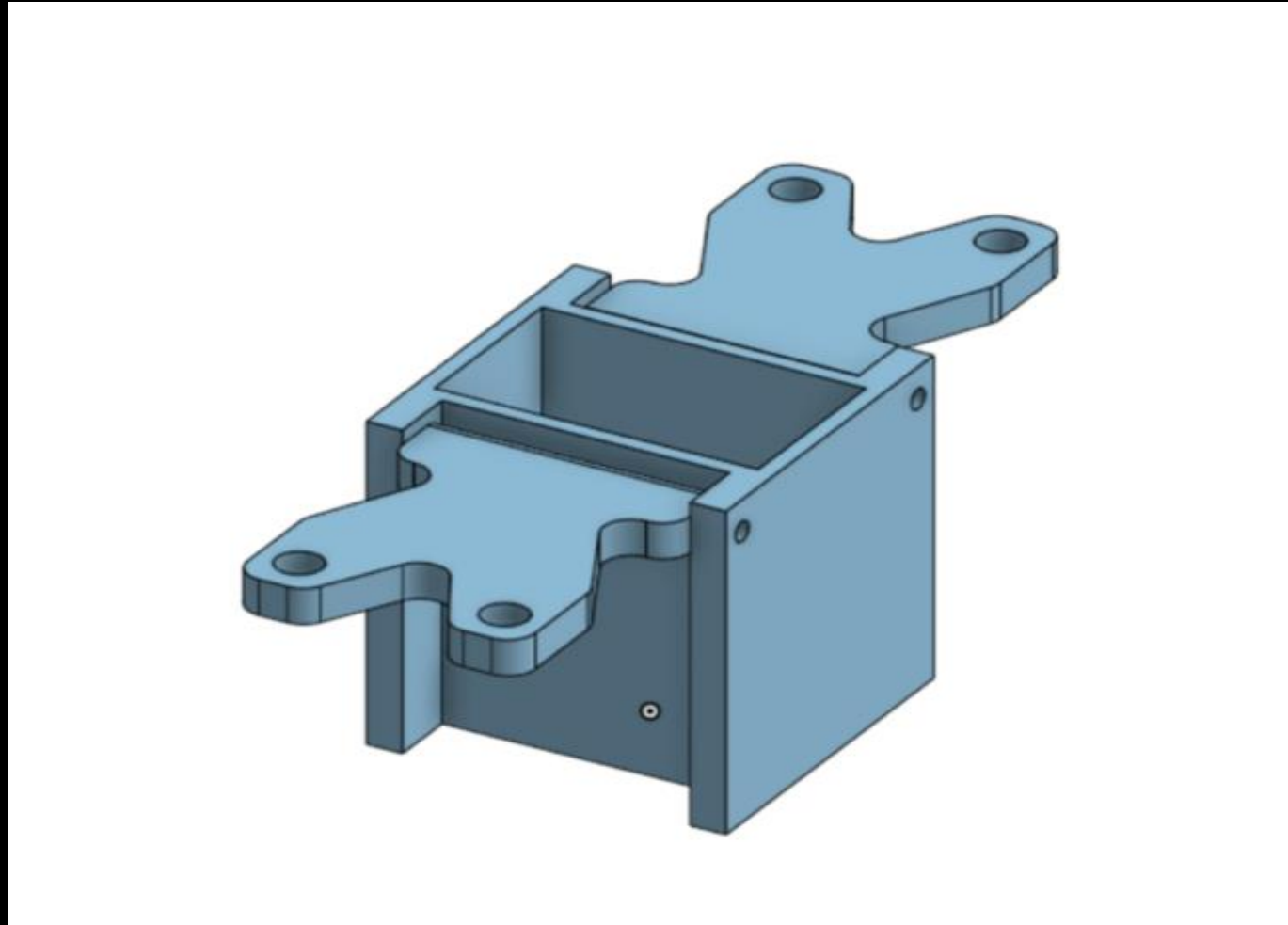
Concept #2: Deployable Solar Panel



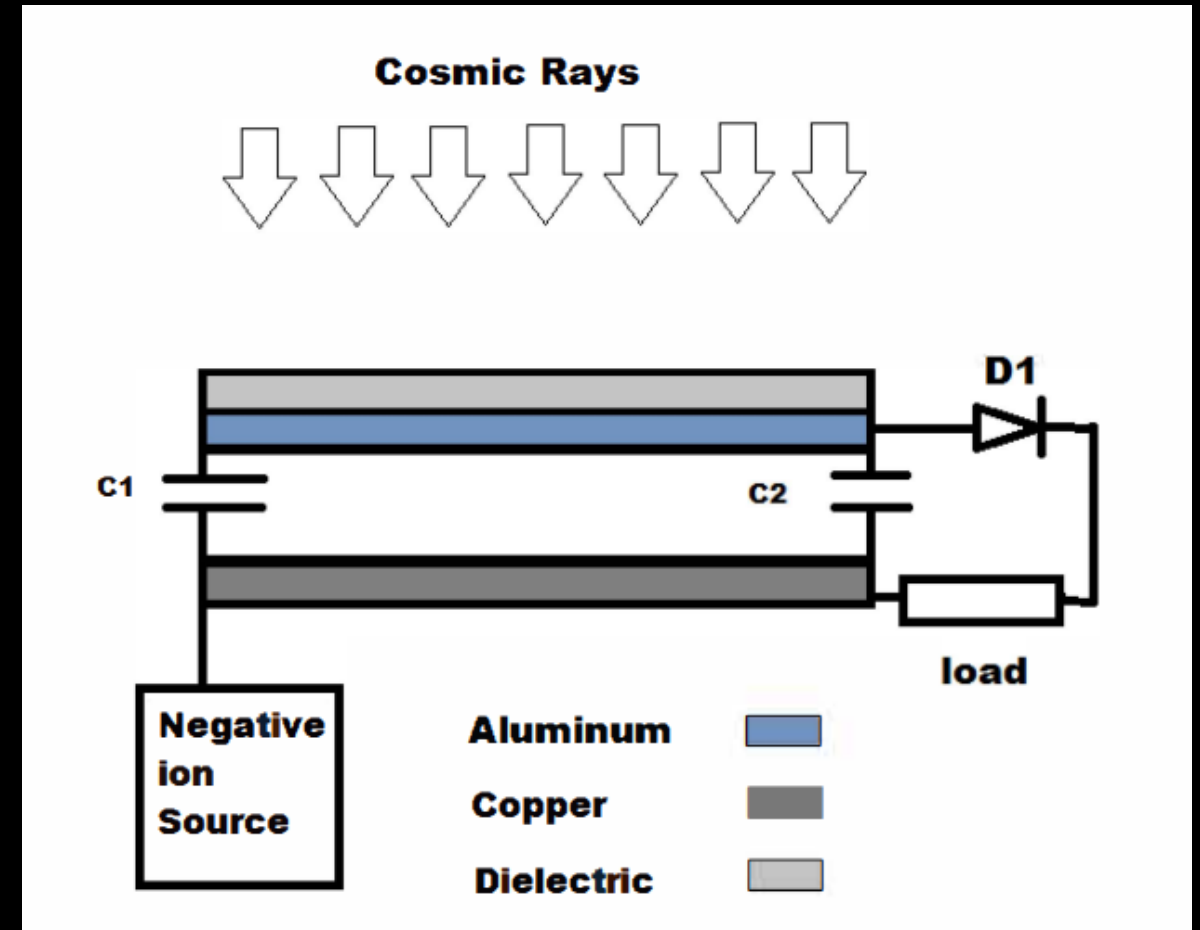
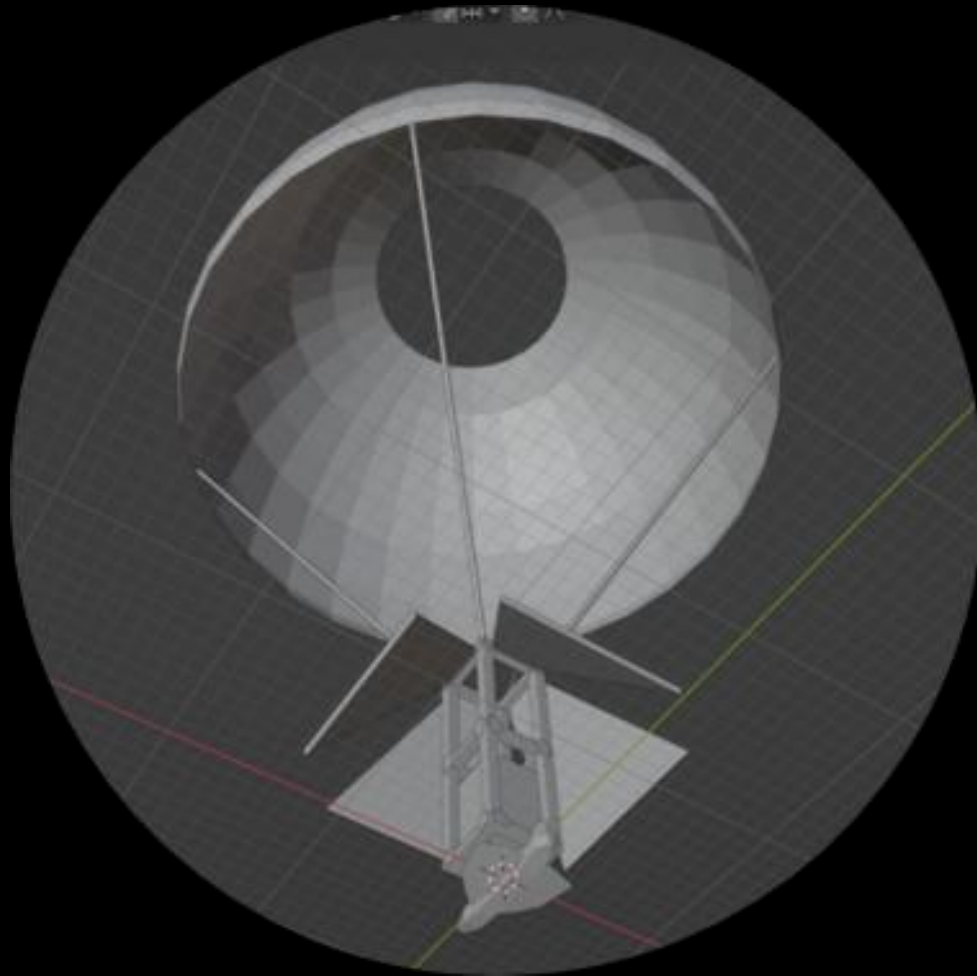
Concept #3: Topographic Mapping Drone



Concept #4: Advanced Research and Exploration Systems



Concept #5: Plasmon Resonance Powered Drone



Evaluation Criteria

The following criteria were used to evaluate each proposal concept

Each criterion was assigned a 0–5-point value

- Difficulty (15%)
- Cost (25%)
- Presentability (20%)
- Functionality (25%)
- Schedule (15%)
- Regulations (Pass/Fail)

| Point Value |
|-------------|
| 5 |
| 3 |
| 1 |
| 0 |

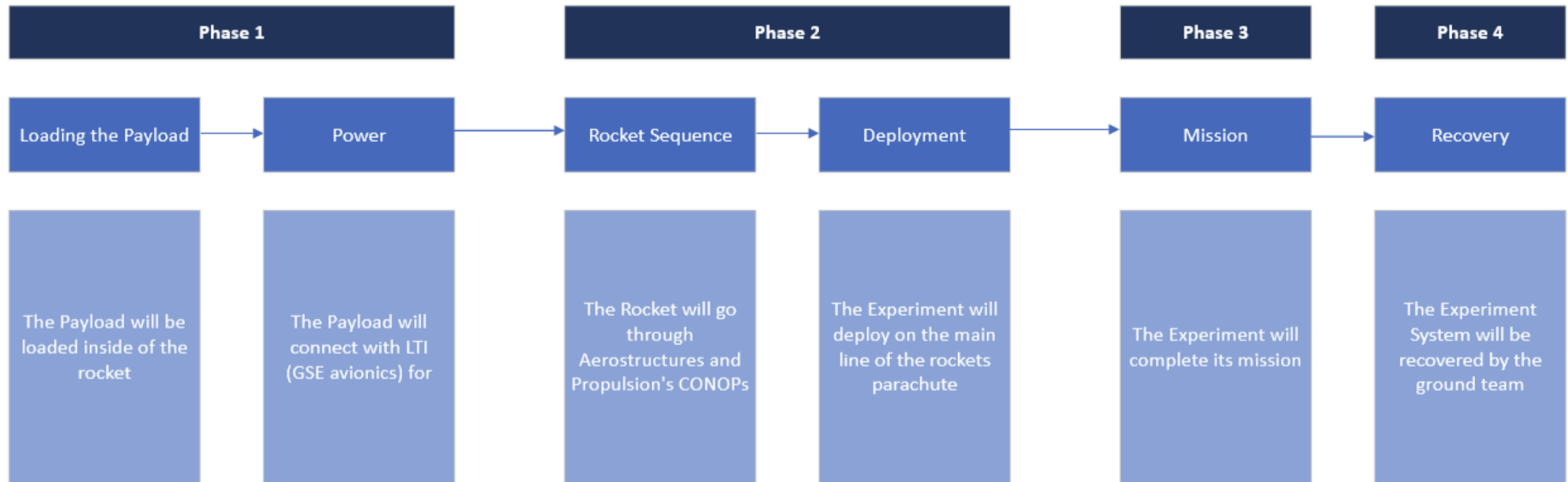
Proposal Selection Decision Matrix

| Criteria | Weight | Team 1 | Team 2 | Team 3 | Team 4 | Team 5 |
|----------------|-----------|--------|--------|--------|--------|--------|
| Difficulty | 0.15 | 1 | 3 | 3 | 1 | 2 |
| Cost | 0.25 | 1 | 4 | 2 | 0.5 | 5 |
| Presentability | 0.2 | 4 | 3 | 3 | 5 | 5 |
| Functionality | 0.25 | 3 | 3 | 1.5 | 1 | 3 |
| Schedule | 0.15 | 1 | 5 | 3 | 0 | 1 |
| Regulations | Pass/Fail | Pass | Pass | Pass? | Pass? | Pass |
| Weighted Score | | 2.1 | 3.55 | 2.375 | 1.525 | 3.45 |

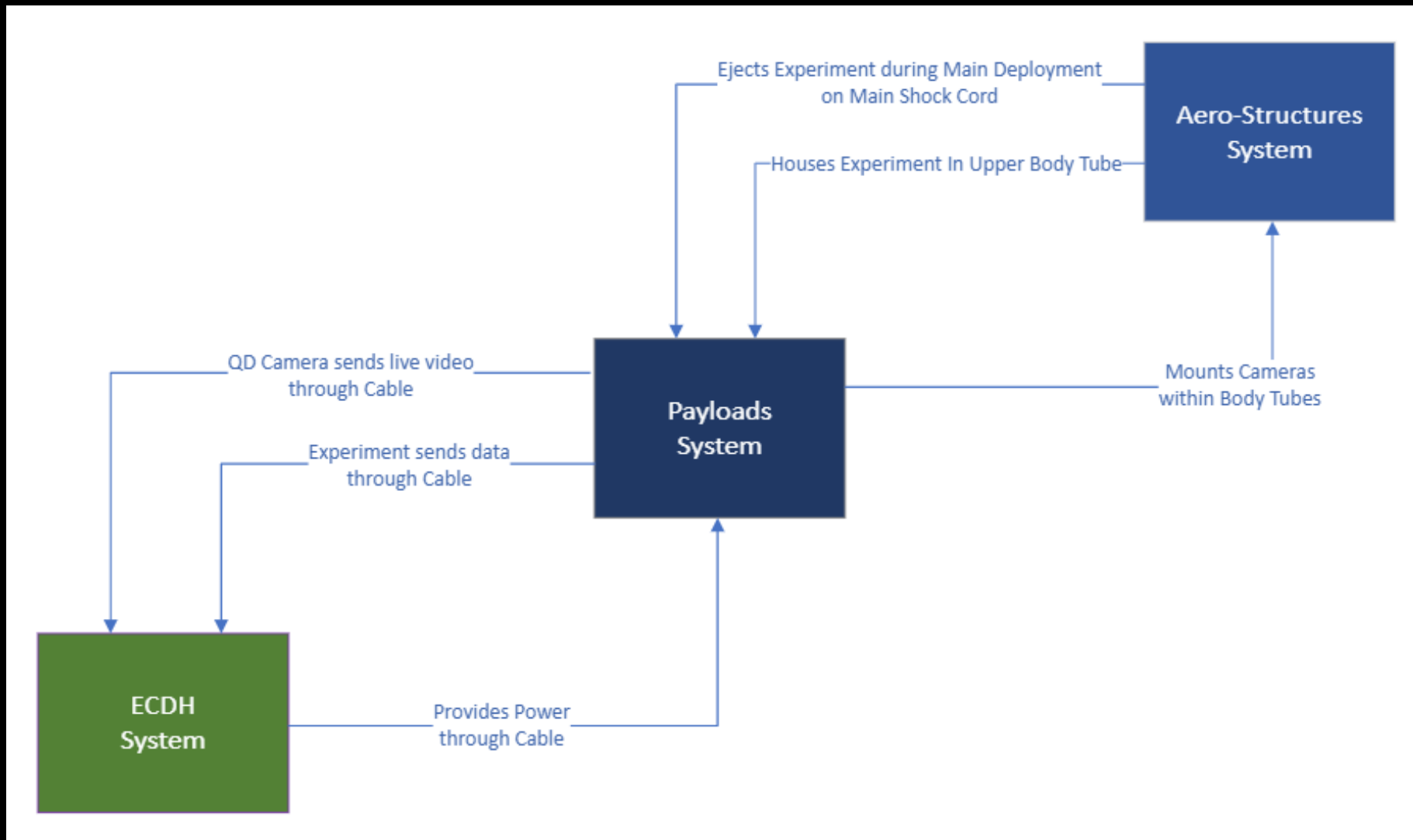
Proposal #2 was selected as the IREC 2024 payload

Payloads System CONOPS

Payloads Experiment System CONOPS



Payloads System Interface Diagram



Total Payloads System Requirements

| Requirement | Verification Method |
|--|---------------------|
| The Total Payloads System shall consist of an Experiment System and a separate Flight Recording Sub-System. | Inspection |
| The Total Payloads System shall fully fit inside the Aero-Structures System. | Inspection |
| The Total Payloads System shall be fully designed by the first week of February 2024. | Inspection |
| The Total Payloads System shall be entirely procured by the last week of February 2024. | Inspection |
| The Total Payloads System shall have been fully verified through testing by the last week of April 2024. | Inspection |
| The Total Payloads System shall be designed, manufactured, tested, and validated within a budget of [\$1,600.] | Inspection |
| The Total Payloads System shall reserve [\$400] for overhead and emergency purchases. | Inspection |
| The Total Payloads System shall be safely operable. | Demonstration |
| The Total Payloads System shall be designed and integrated such that it does not jeopardize the overall Vehicle's safety. | Demonstration |
| The Total Payloads System shall operate in temperatures above [-47.2 °C] and below [110 °C]. | Demonstration |
| The Total Payloads System shall survive [11 Gs] of acceleration. | Demonstration |

Experiment System Requirements

| Requirement | Verification Method |
|--|---------------------|
| The Experiment System shall consist of Mechanical, Electrical, and Software Sub-Systems. | Inspection |
| The Experiment System shall have a weight of [9] lbs. | Inspection |
| The Experiment System shall have a CubeSat factor of [4U]. | Inspection |
| The Experiment System shall have dimensions of [10 cm] diameter by [41.32 cm]. | Inspection |
| The Experiment System shall remain independently powered for [1 hour]. | Test |
| The Experiment System shall interface with the Aero-Structures system. | Inspection |
| The Experiment System shall withstand impact with the ground at [30] ft/s. | Test |
| The Experiment System shall be easily removeable from Aero-Structures System for inspection by ESRA judges. | Demonstration |
| The Experiment System shall fit inside [16.3] inches of the Airframe Upper Body Tube. | inspection |
| The Experiment System shall be deployed adjacent on the Main Parachute's Shock Cord. | Demonstration |
| The Experiment System shall be powered via cable connection by the Avionics Service System until Main Parachute Deployment event. | Test |
| The Experiment System shall send data via cable connection to the Avionics Service System until Main Parachute Deployment event. | Test |
| The Experiment System shall receive data via cable connection from the Avionics Service System until Main Parachute Deployment event. | Test |
| The Experiment System shall separate from the harnessing of the Avionics Service System upon Main Parachute Deployment event. | Demonstration |
| The Experiment System shall collect and store solar energy in a separate dedicated battery bank. | Demonstration |
| The Experiment System shall collect and store a solar energy amount of [TBD Watts]. | Test |
| The Experiment System shall be designed, manufactured, tested, and validated within a budget of [\$1,200]. | Inspection |
| The Experiment System shall orient the Solar Panel within [5] degrees towards the Sun's direction. | Test |
| The Experiment System shall be deployed in such a way that it does not put spectators in direct harm. | Demonstration |

Payloads Systems Technical Performance Measures

| Measure | TPM Value | Units | Verification Method |
|-------------------|-------------|-----------------|---------------------|
| Experiment Weight | [9 lbs] | Pounds | Inspection |
| Volume | [4 U] | CubeSat Units | Inspection |
| Length | [16.3 in] | Inches | Inspection |
| Min Temp | [-47.2 °C] | Fahrenheit | Demonstration |
| Max Temp | [110 °C] | Fahrenheit | Demonstration |
| Max Impact Speed | [30 ft/sec] | Feet per Second | Demonstration |

Payloads Systems Cost

| Subsystem | Estimated Cost |
|------------------|----------------|
| Mechanical | \$600 |
| Electrical | \$300 |
| Software | \$300 |
| Flight Recording | \$400 |
| Overhead | \$400 |
| Total | \$2000 |

Payloads Systems Verification Plans

Inspection on
Machined Parts

FEA on Payload
Housing

Battery
Longevity

Solar Panel
power
generation test

Software-In-
The-Loop for all
Systems

Camera
Stitching occurs
AFTER launch

Hardware-In-
The-Loop for all
Systems (with
Avionics)

MQD testing

Drop Test

Snatch Force
Test

Payload Dry Fit

Weight
Inspection

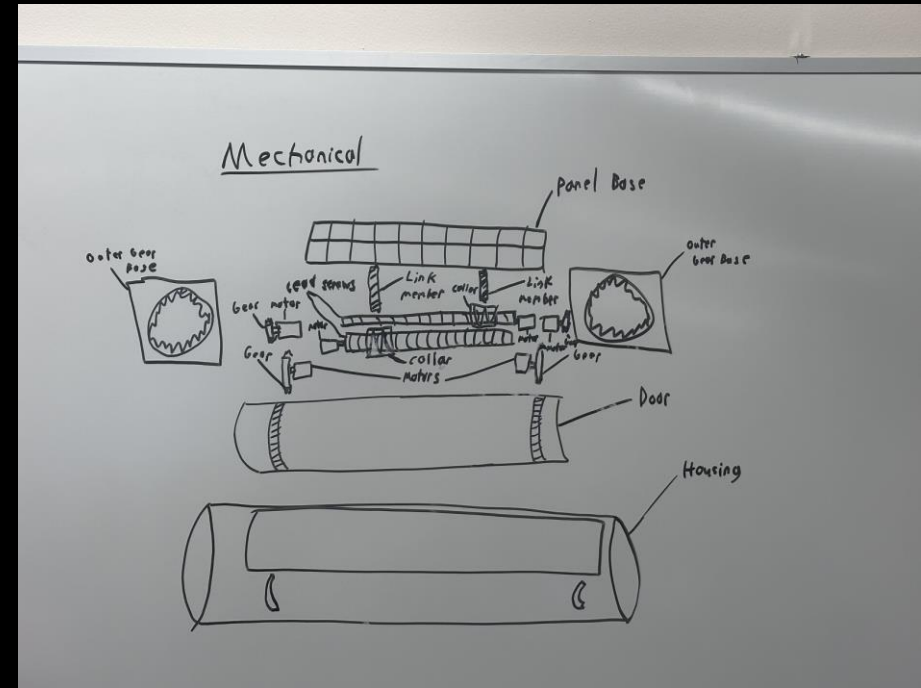
Shock and
Vibration
Testing*

Temperature
Test

Mechanical Sub-System



A prototype of the outer housing



Exploded view drawing of the entire mechanical system

Mechanical Requirements

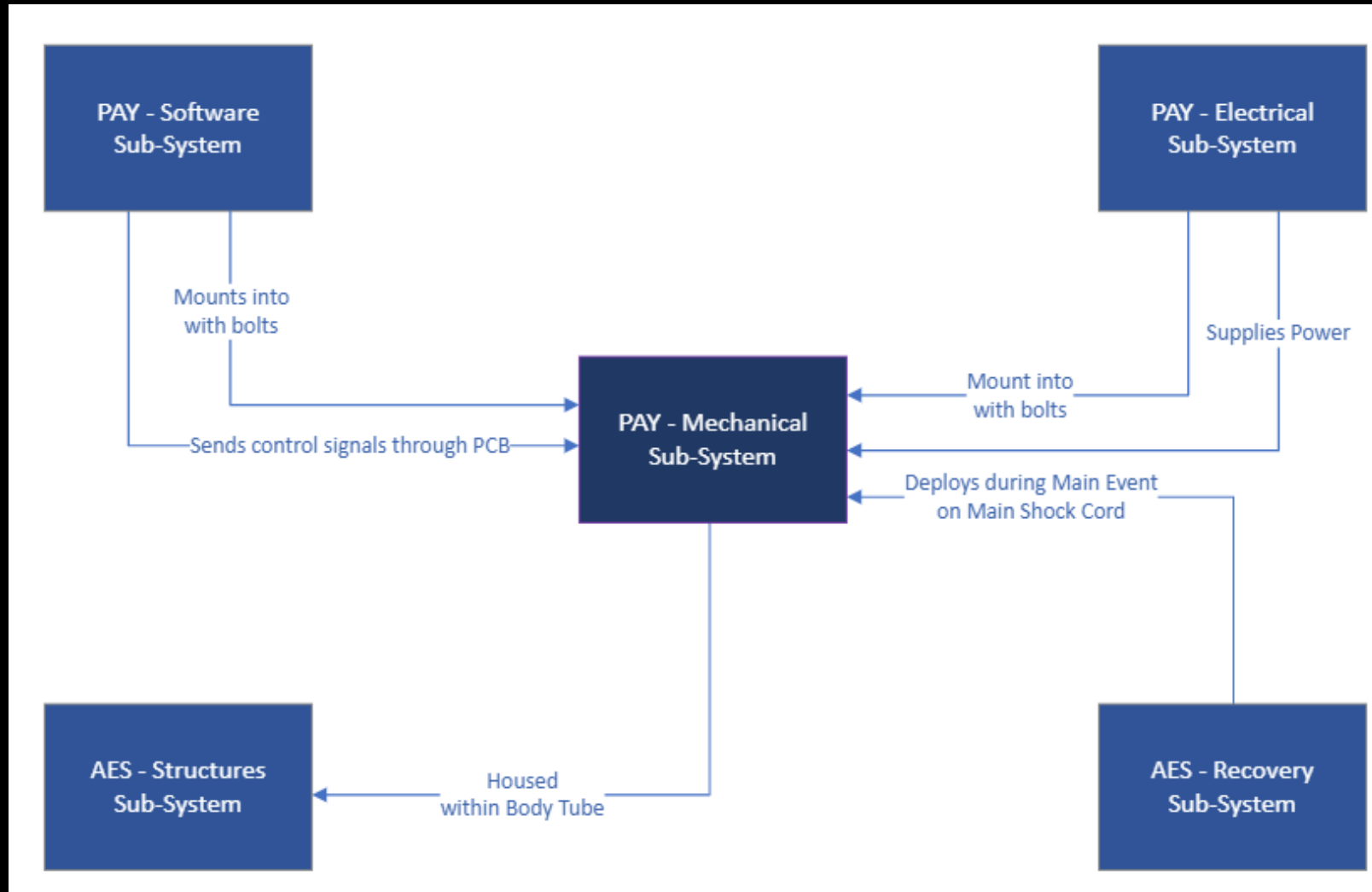
| Requirement | Verification Method |
|--|---------------------|
| The mechanical subsystem shall weigh [8.8 lbs] | Inspection |
| The mechanical subsystem shall have a [sliding door] to protect the solar panel during launch and during adverse weather | Demonstration |
| The mechanical subsystem shall have a self-orientation system with square ends and a rotating middle section in order to stay stabilized upon landing | Inspection |
| The mechanical subsystem shall have an offset weight to initially orient the housing | Inspection |

| Requirement | Verification Method |
|---|---------------------|
| The mechanical subsystem shall have a [system of lead screws] to get the solar panel outside of the main body after landing as well as orient solar panel for tracking the sun | Test |
| The mechanical subsystem shall have a motor driven orientation system that rotates the inner cylinder to ensure correct positioning for operation | Test |
| The mechanical subsystem shall have a self-cleaning system to clean off the solar panel | Demonstration |

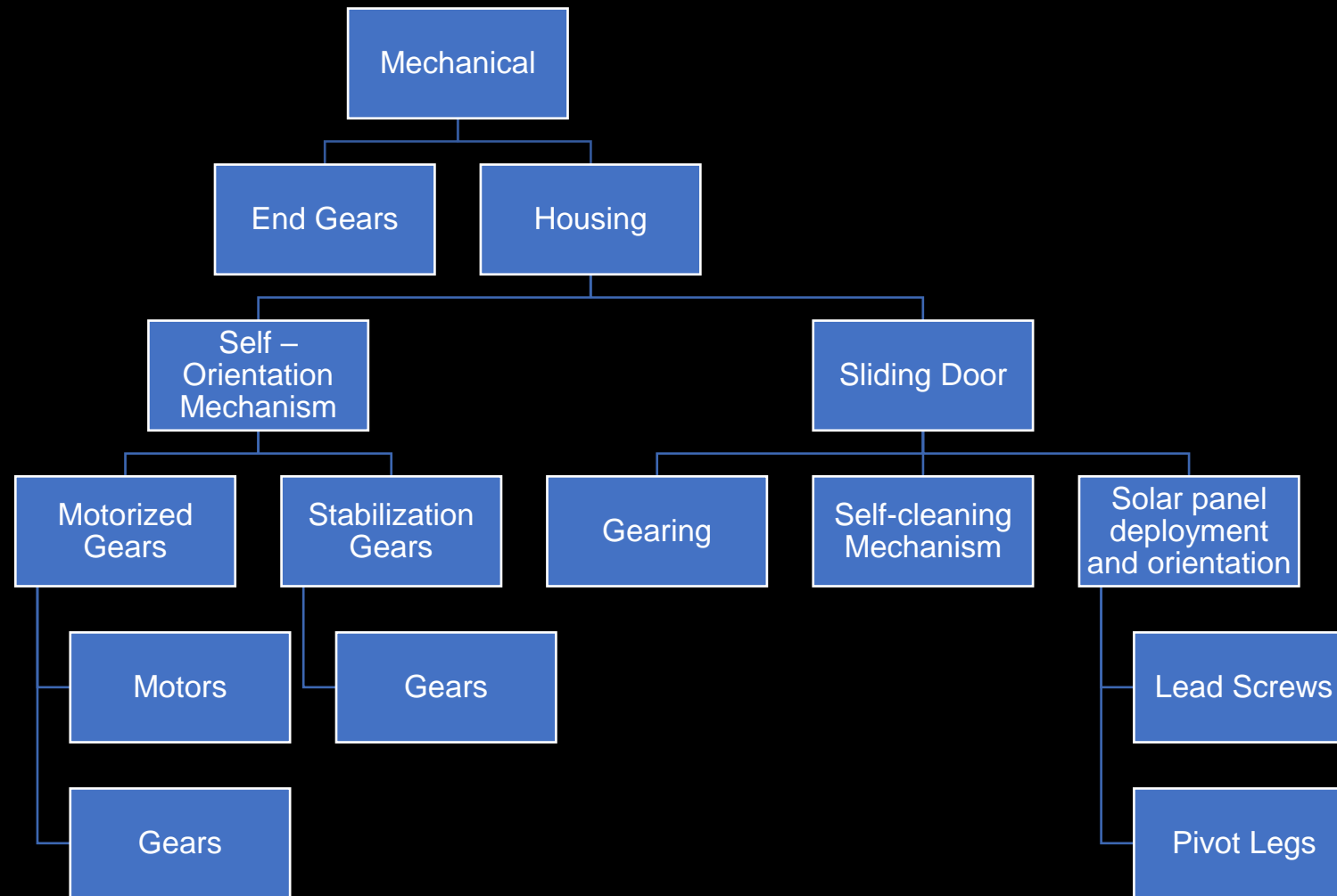
Mechanical TPMs

| Measure | TPM Value | Units | Verification Method |
|------------|-----------|-----------------|---------------------|
| Mass | [TBD] | lbs | Inspection |
| Length | [40] | cm | Inspection |
| Volume | [4000] | cm ³ | Inspection |
| Power Draw | [12] | Volts | Testing |

Mechanical Interface Diagram



Mechanical Component Breakdown

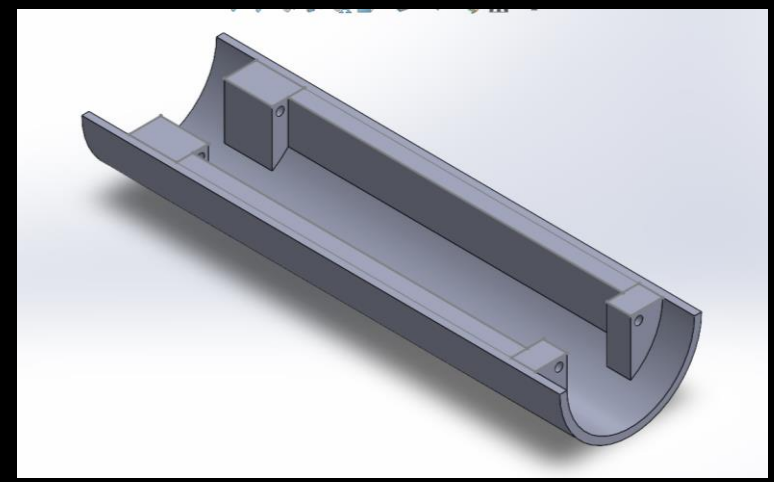
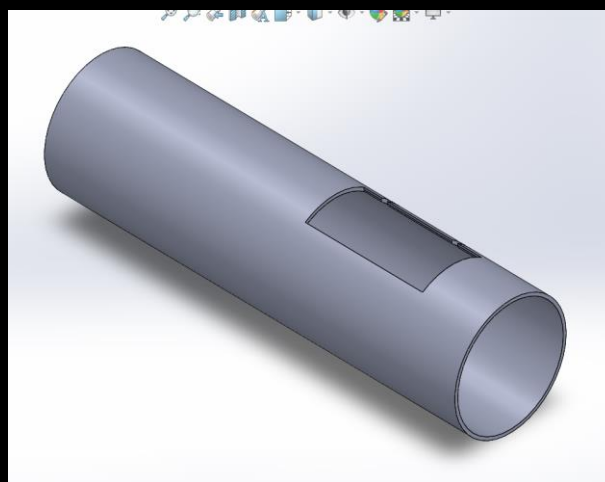
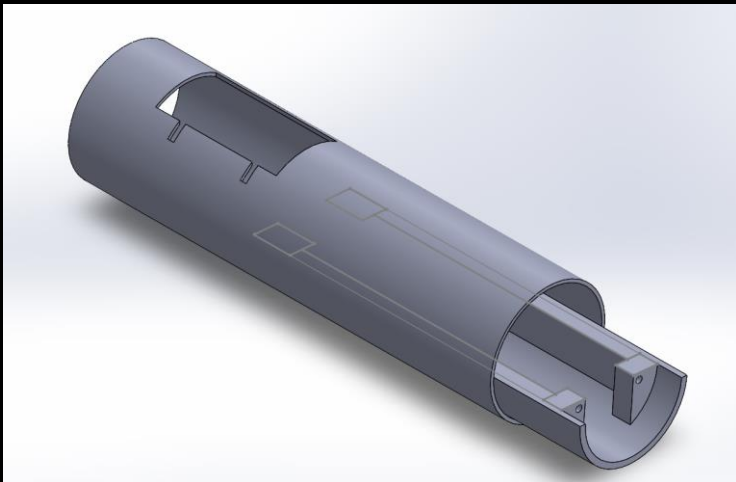


Housing

Exterior shell for the experiment which shall house all components

Possible Points of Failure

- Housing breaks upon impact
- Housing breaks upon receiving force from shock cord connection
- Housing breaks mid-flight



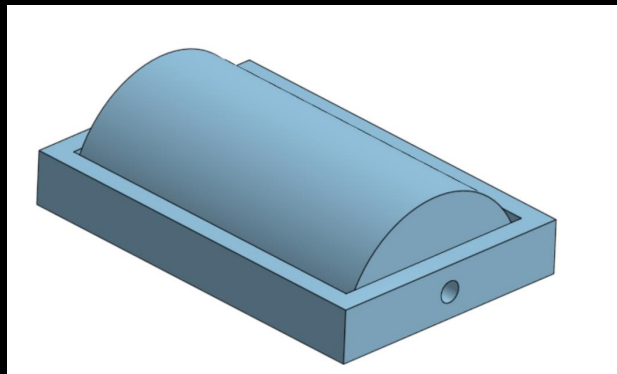
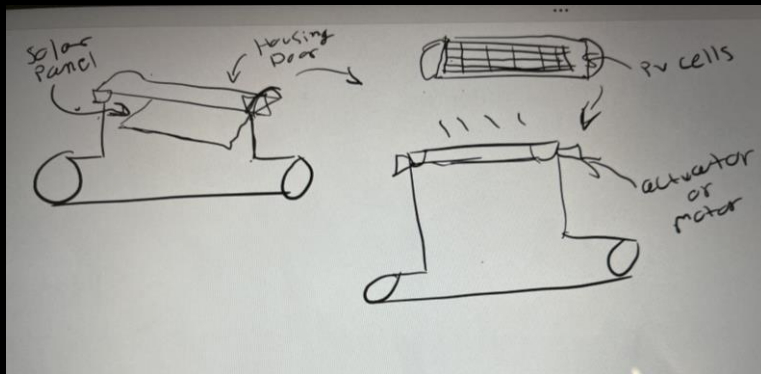
Housing

| Requirement | Verification Method |
|---|---------------------|
| The body shall be able to endure [TBD lbs] of impact | Test |
| The housing shall be assembled out of [TBD material] | Inspection |
| The housing shall have a footprint of [9 cm] diameter by [30 cm] | Inspection |

| Measure | TPM Value | Units | Verification Method |
|----------|-----------|-------|---------------------|
| length | ! | cm | Measurement |
| diameter | [9] | cm | Measurement |

Housing Door

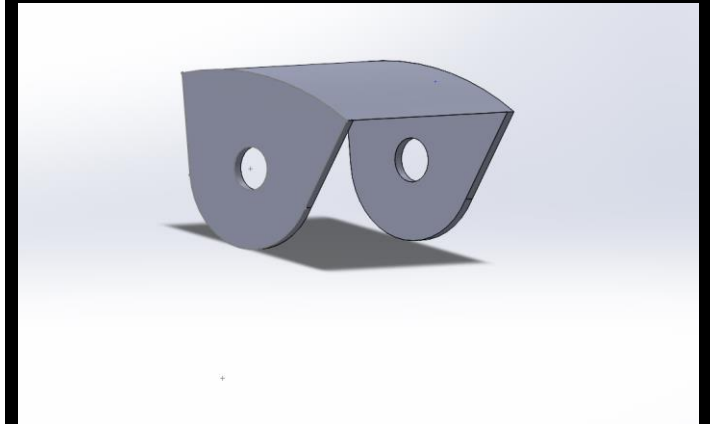
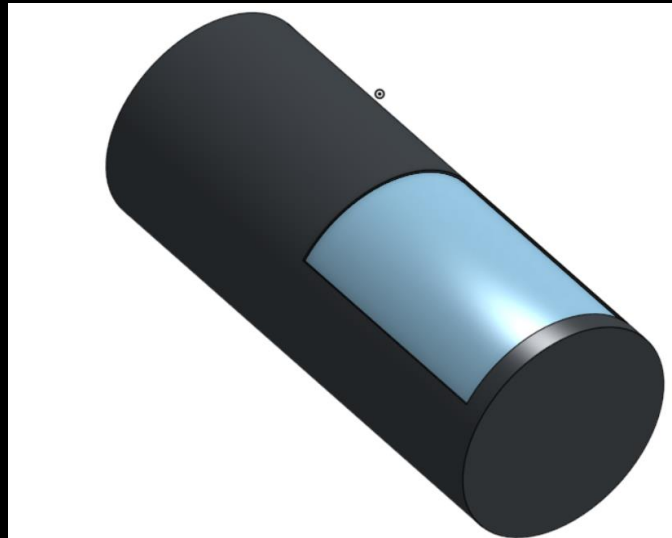
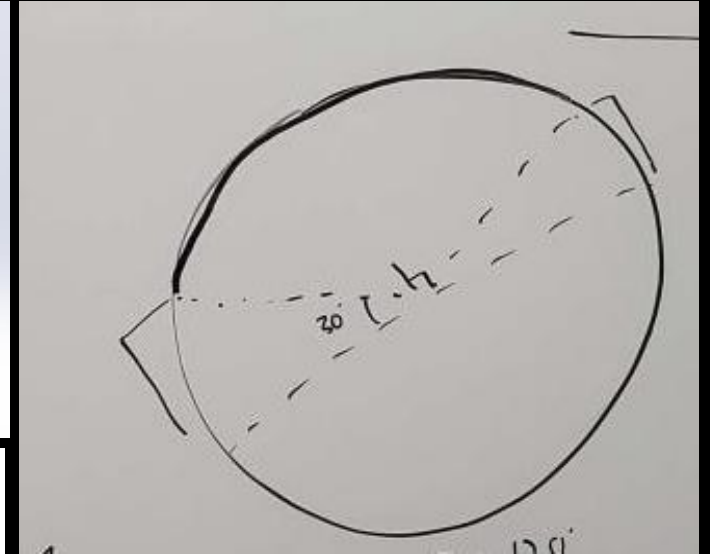
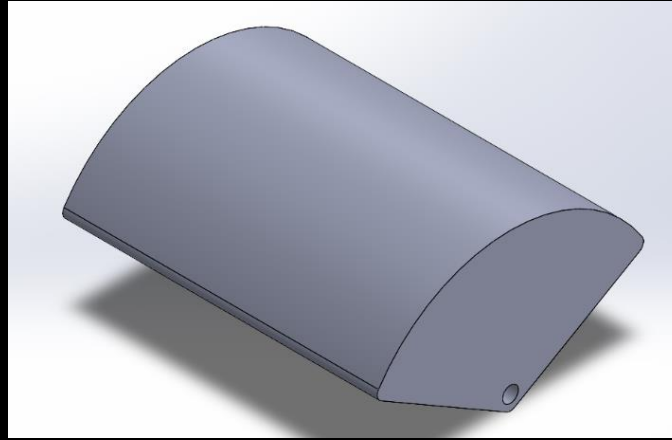
- Materials currently being considered for the door are:
 - Polyamide
 - Acrylonitrile Butadiene Styrene(ABS)
 - Polyethylene terephthalate glycol(PET-G)
 - Aluminum



| Requirements |
|---|
| The housing door shall be able to protect the payload components during landing |
| The housing door shall be able to open and close easily to protect the components during adverse weather |
| The housing door shall minimize debris entering the payload |
| The housing door shall be able to provide clearance for the panel to deploy once landed |
| The housing door shall stay open while panel is deployed |
| The housing door shall be [40 cm] long and have an arc length of [4 cm] |
| The housing door shall weigh [1 lb] |

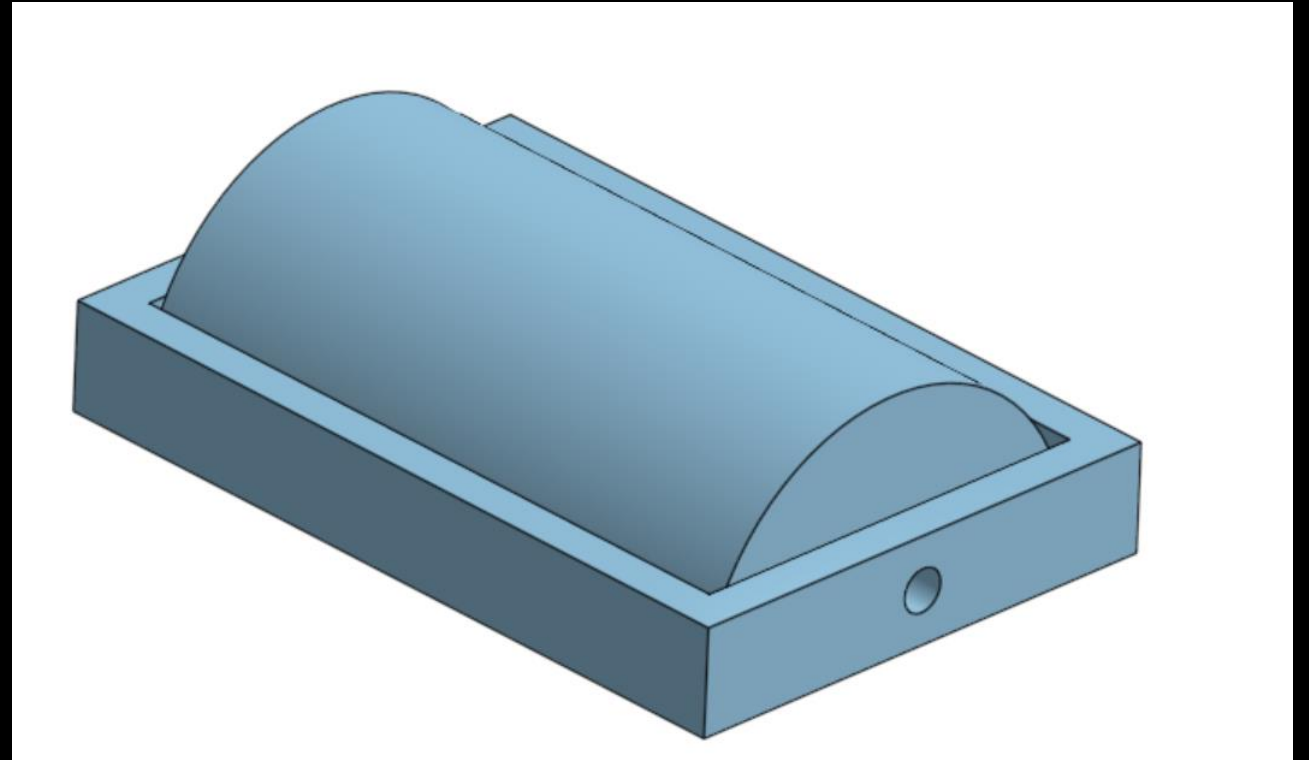
Housing Door

Before our final design these were some options considered:



Housing Door

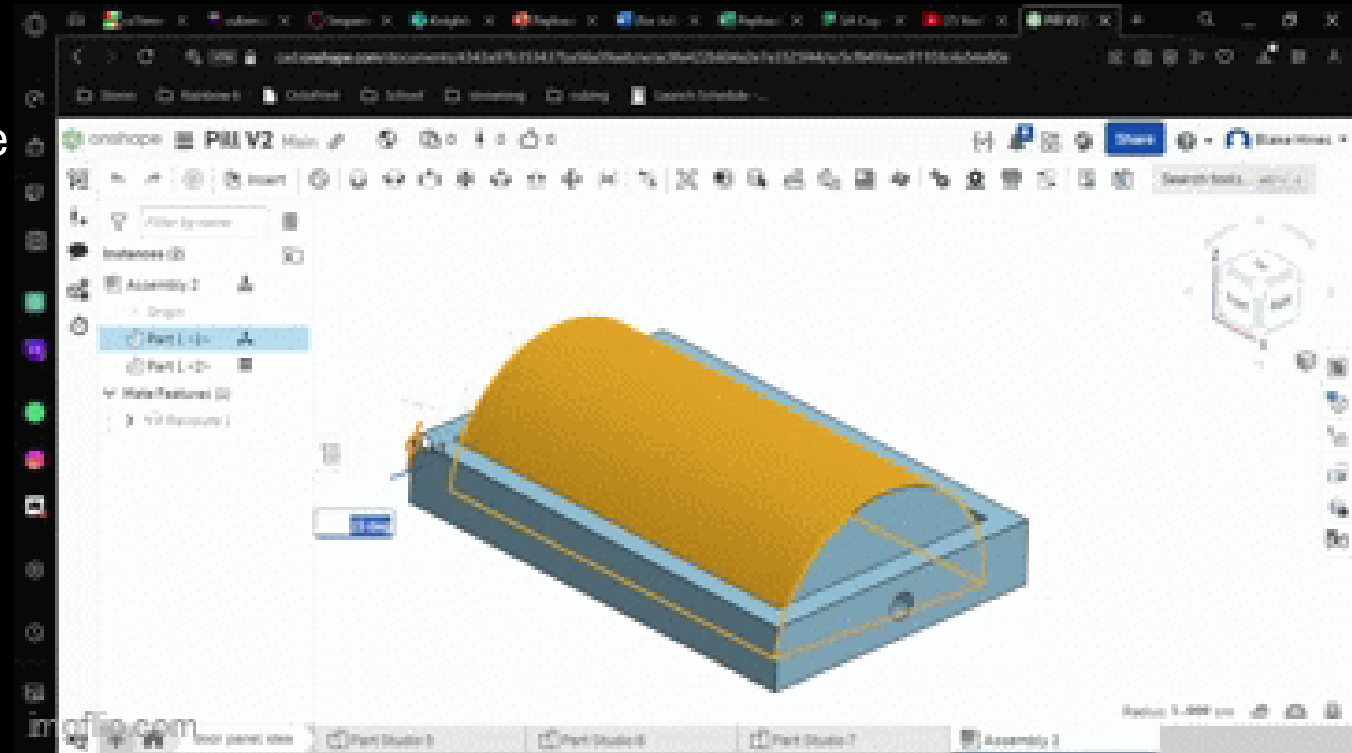
Design we went with adds another degree of freedom upon the payload's rotation by putting the panel on the flat side of the door



Housing Door

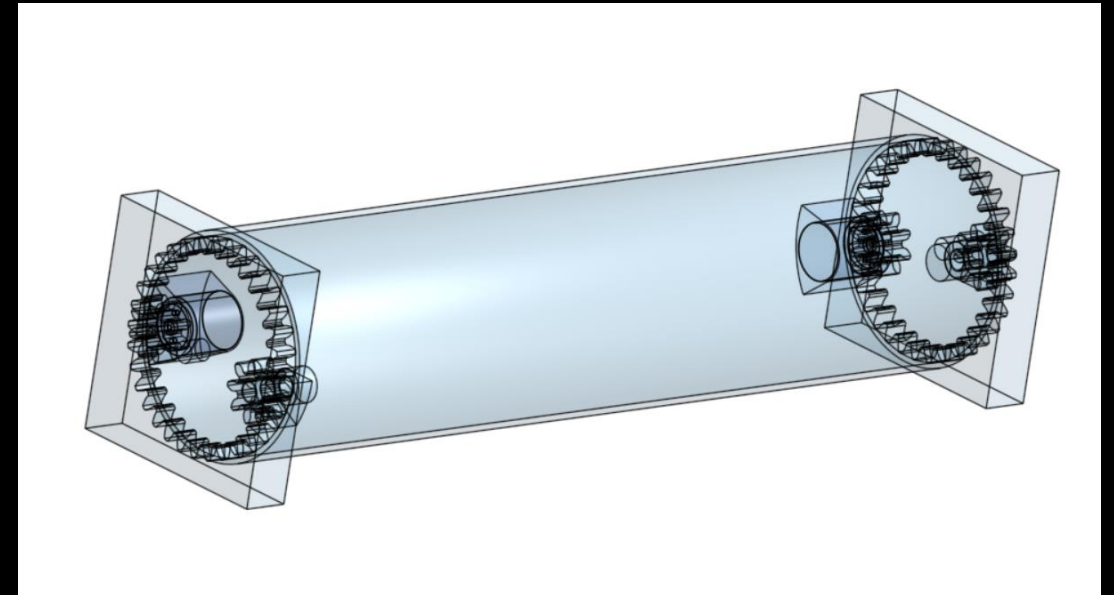
Possible Points of Failure:

- Damage upon landing impact
- Failure in motor to provide enough torque to rotate door and expose panel
- Weight of the top side of the door throws off motor calculations and doesn't align towards the sun.



Self-Orientation

| Requirement | Verification Method |
|--|---------------------|
| The self-orientation mechanism shall have a stationary outside while rotating the inner components with a [type of motor] | Inspection |
| The self-orientation mechanism will be able to track the sun along a rotating axis to aid with the solar orientation | Test |
| The self-orientation mechanism shall be driven by [TBD] motors | Inspection |



Self-Orientation

Possible Points of Failure:

- Misalignment of internal gear system upon landing
- Failure of motor to apply enough torque to rotate payload
- Self-orientation subassembly causes Panel Orientation subassembly to fail because of unaccounted for rotation.

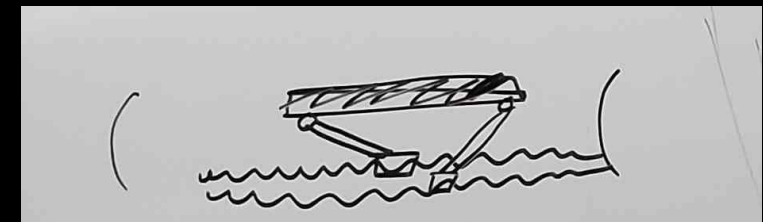
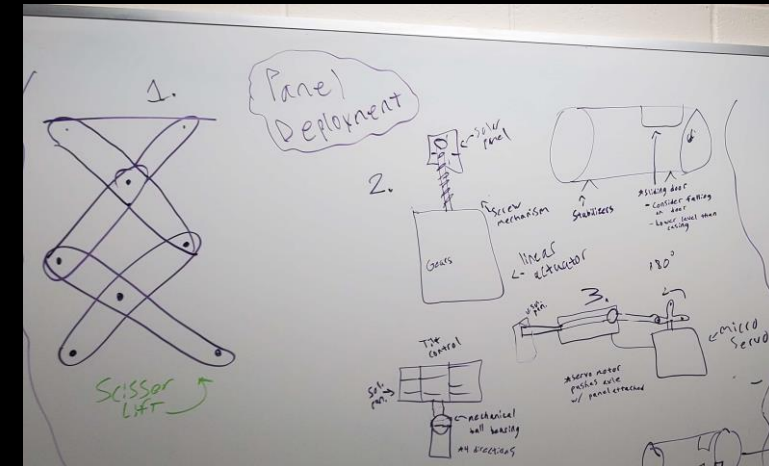
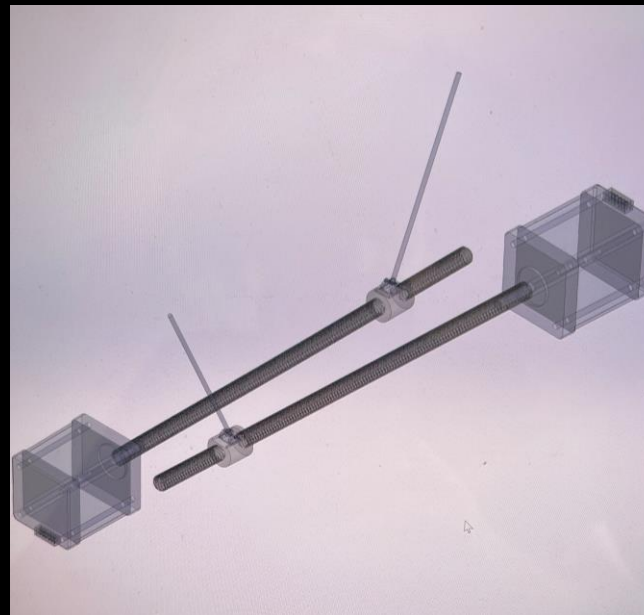
| Measure | TPM Value | Units | Verification Method |
|-------------------|-----------|---------|---------------------|
| Torque | 77.571 | lb-ft | Analysis |
| Angle of Rotation | 90 | degrees | Demonstration |

Solar Panel Deployment and Orientation

We considered a few designs such as

- Scissor Lift
- Linear Actuator
- Mechanical Arm with Ball Joint

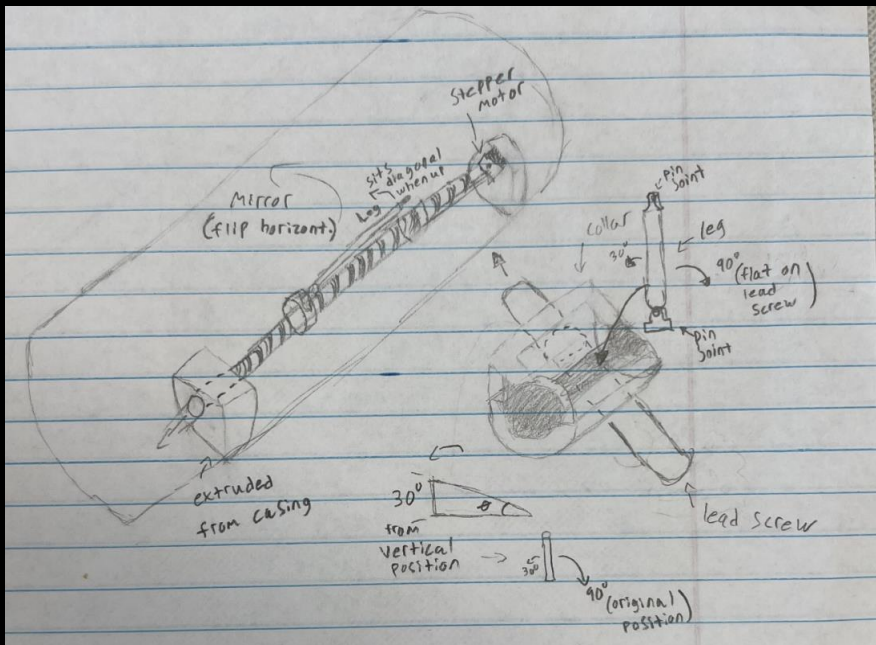
We decided to use a lead screw system that would double as a deployment mechanism and an orientation mechanism through pivoting



Solar Panel Deployment and Orientation

Possible points of failures:

- Not fully extending upon deployment
- Rotation in the pivot legs
- Incorrect movement in the stepper motor
- Not orienting correctly



| Requirements | Verification Method |
|---|---------------------|
| The solar orientation assembly shall be able to move the panel toward any given direction | Inspection |
| The deployment mechanism shall have a [TBD] base that accounts for the solar panel | Inspection |
| The deployment mechanism shall have 2 leadscrews running in parallel to allow the panel to lift and tilt along an axis | Inspection |
| The solar panel deployment and orientation mechanism will drive lead screws with [2 Stepper] motors | Inspection |

Solar Deployment and Orientation

| Measure | TPM Value | Units | Verification Method |
|------------------------------------|-----------|--------|---------------------|
| Length of connecting rod (l) | [11.811] | in | Inspection |
| Length of solar panel (l_{sp}) | [7.874] | in | Inspection |
| Screw lead (k) | [0.495] | in/rad | Inspection |
| Mass | [TBD] | lbs | Analysis |
| Torque | [9.63] | lbs/in | Analysis |

$$y = \sqrt{l^2 + (k\theta)^2}$$

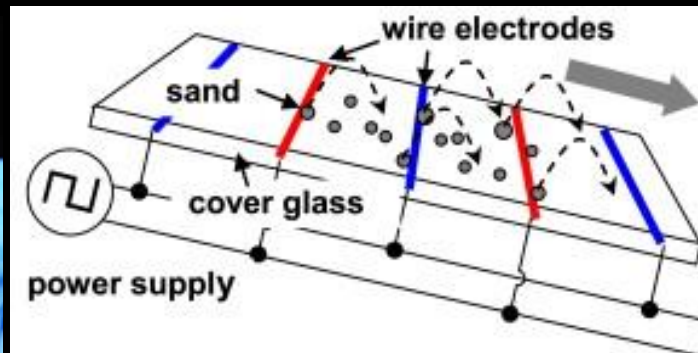
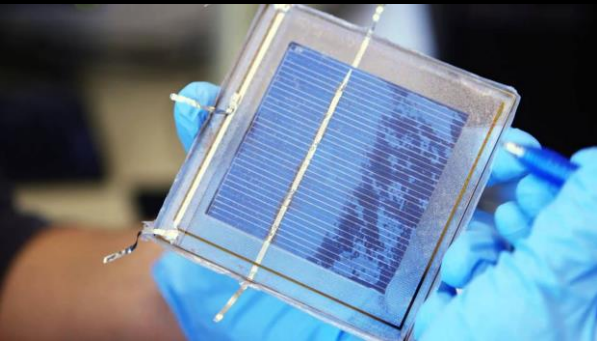
$$\theta_n = \tan^{-1} \left(\frac{l_{sp}}{|y_2 - y_1|} \right)$$

$$\theta_n = \tan^{-1} \left(\frac{l_{sp}}{\sqrt{l^2 + (k\theta_2)^2} - \sqrt{l^2 + (k\theta_1)^2}} \right)$$

$$l_{window} = l_{sp} \cdot \sin(\theta_n)$$

Self-Cleaning Mechanism

- The solar panel will have some form of cleaning itself to theoretically survive long missions without maintenance
- The most prospective manner to do so currently, is through electrostatic discharge

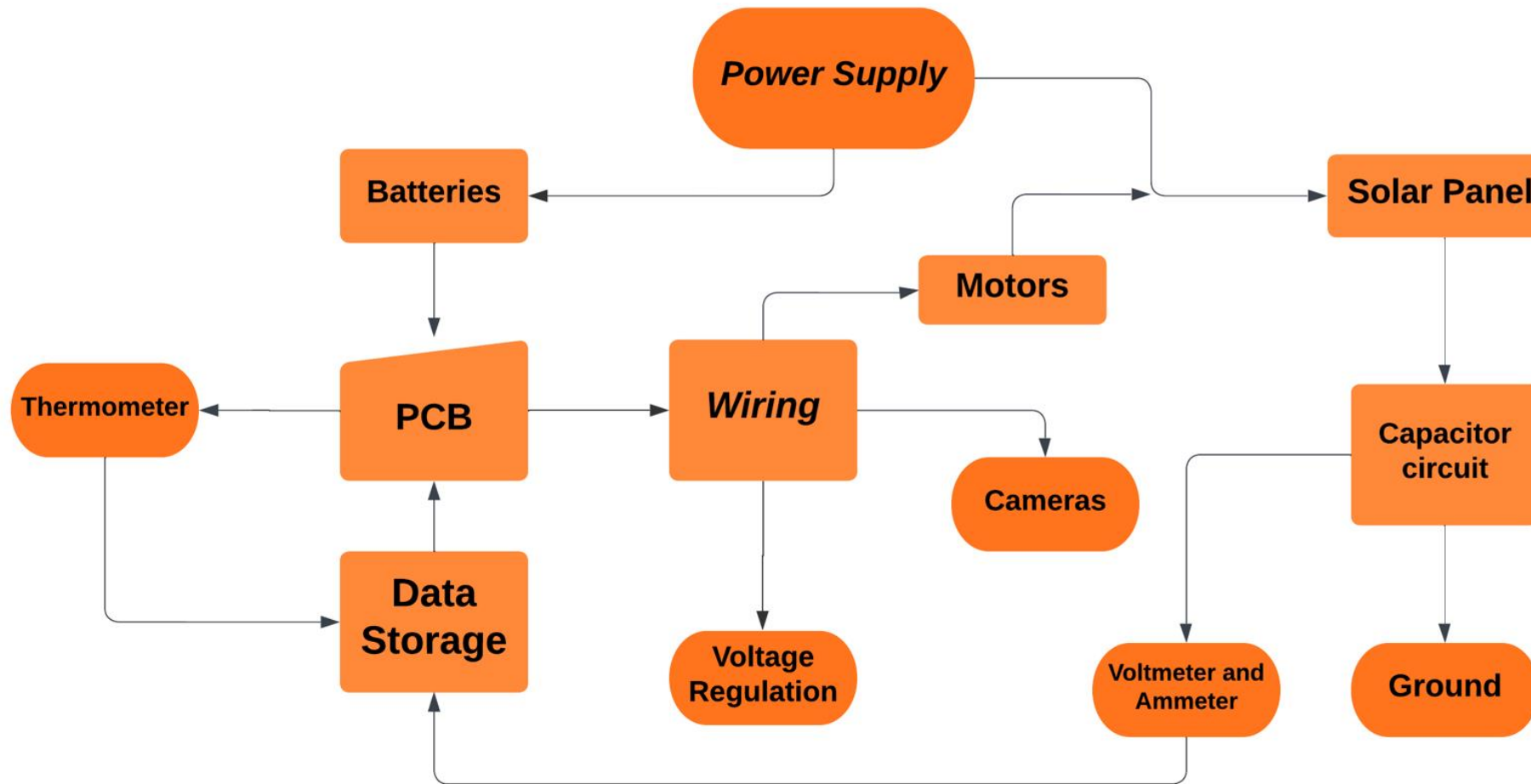


| Requirements |
|--|
| The self-cleaning mechanism shall be able to remove debris from the solar panel using [electrostatic discharge] |
| The self-cleaning mechanism shall use three different, repeating electrodes to remove debris |
| The self-cleaning mechanism shall function at any given time |
| The self-cleaning mechanism should not decrease panel energy generation |
| The self-cleaning mechanism shall use [dielectric film coating] to protect the electrodes |
| The self-cleaning mechanism shall minimize power usage |
| The self-cleaning mechanism should not require maintenance to retain function |

Mechanical Path Forward

- ***Making the full CAD***
- ***Test further prototypes***
- ***Finalize the designs***

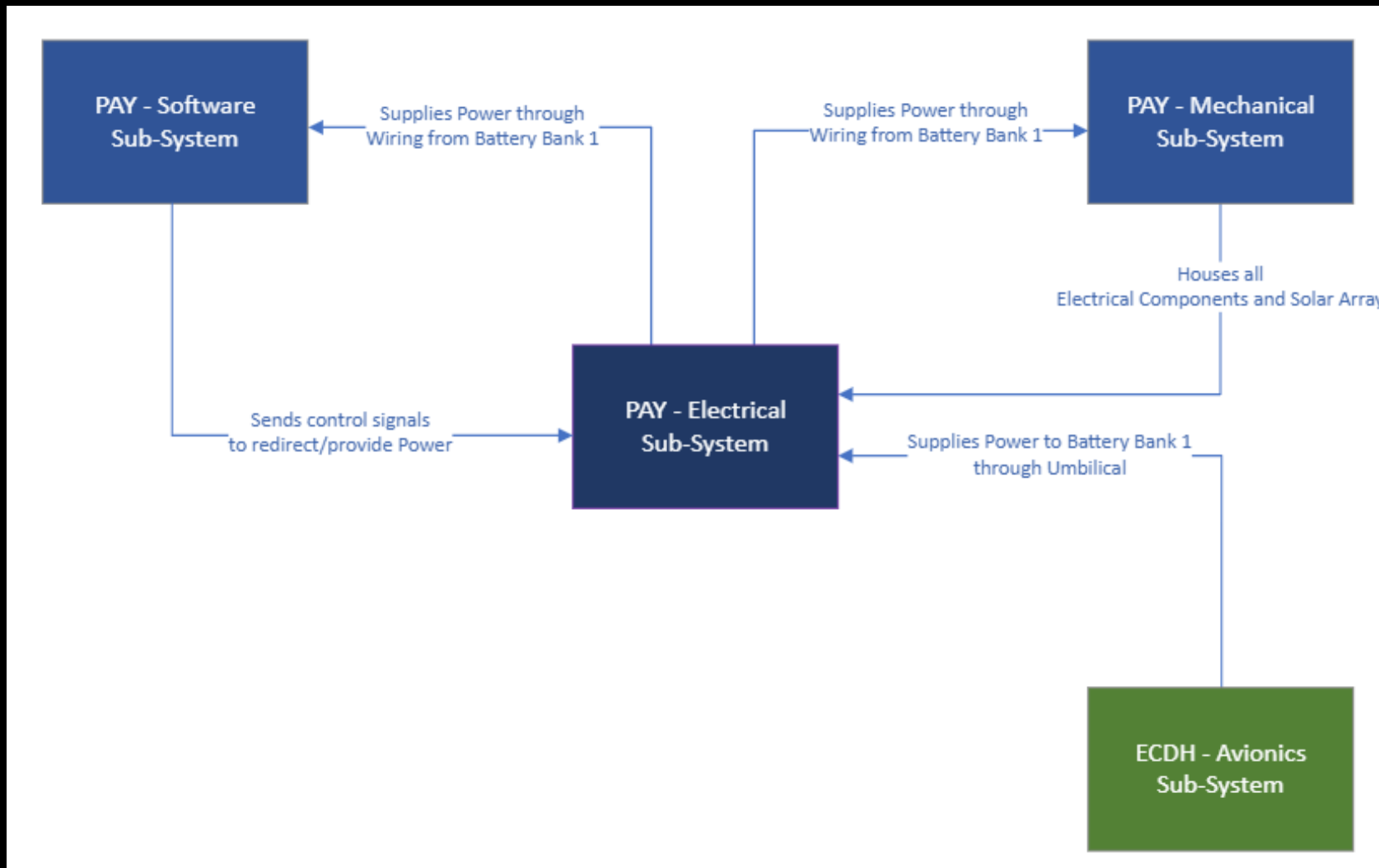
Electrical Sub-System



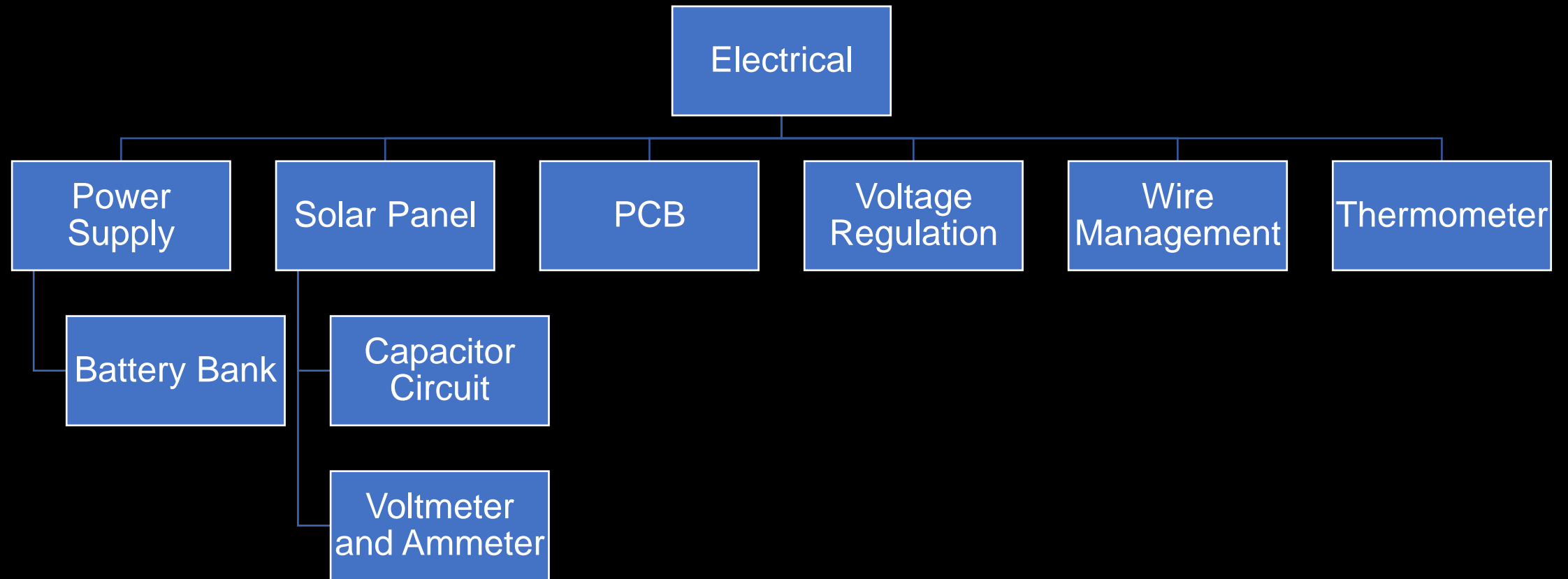
Electrical Functional Requirements

| Requirement | Verification Method |
|---|---------------------|
| The electrical subsystem shall have a power source independent of the photovoltaic panel and battery experiments that will supply power to the payload | Demonstration |
| The electrical subsystem shall include a circuit connected to the photovoltaic panel with a voltmeter, ammeter, and an empty dischargeable capacitor that will act as energy storage | Demonstration |
| The electrical subsystem shall facilitate the exchange of data between all components of the payload | Demonstration |
| The electrical subsystem shall have a system of voltage regulation | Test |

Electrical Interface Diagram



Electrical Component Breakdown



Electrical Component Requirements

| Solar Panel Requirements | Verification Method |
|---|---------------------|
| The solar panel shall be stored and released from within the payload | Demonstration |
| The solar panel shall produce [TBD] amount of energy | Demonstration |
| The solar panel shall cost a maximum of [TBD] dollars | Inspection |
| The solar panel shall follow the sun's path throughout the day | Test |
| The solar panel should have a self-cleaning function | Test |
| Battery Requirements | Verification Method |
| The batteries shall be stored within the payload | Demonstration |
| The power supply shall produce [TBD] amount of energy | Test |
| The power supply shall cost a maximum of [TBD] dollars | Inspection |

Solar Panel Trade Study

| <i>Criteria</i> | <i>Weight</i> | <i>Monocryst. Silicon</i> | <i>Polycryst. Silicon</i> | <i>Amorphous Silicon</i> | <i>CdTe</i> | <i>CIGs</i> |
|------------------------|---------------|---------------------------|---------------------------|--------------------------|-------------|-------------|
| <i>Cost</i> | <i>.25</i> | <i>2</i> | <i>3</i> | <i>5</i> | <i>5</i> | <i>4</i> |
| <i>Schedule</i> | <i>.25</i> | <i>4</i> | <i>4</i> | <i>4</i> | <i>4</i> | <i>4</i> |
| <i>Risk</i> | <i>.25</i> | <i>4</i> | <i>4</i> | <i>2</i> | <i>1</i> | <i>1</i> |
| <i>Performance</i> | <i>.25</i> | <i>5</i> | <i>4</i> | <i>1</i> | <i>3</i> | <i>2</i> |
| <i>Weighted Scores</i> | | <i>3.75</i> | <i>3.5</i> | <i>3</i> | <i>3.25</i> | <i>2.75</i> |

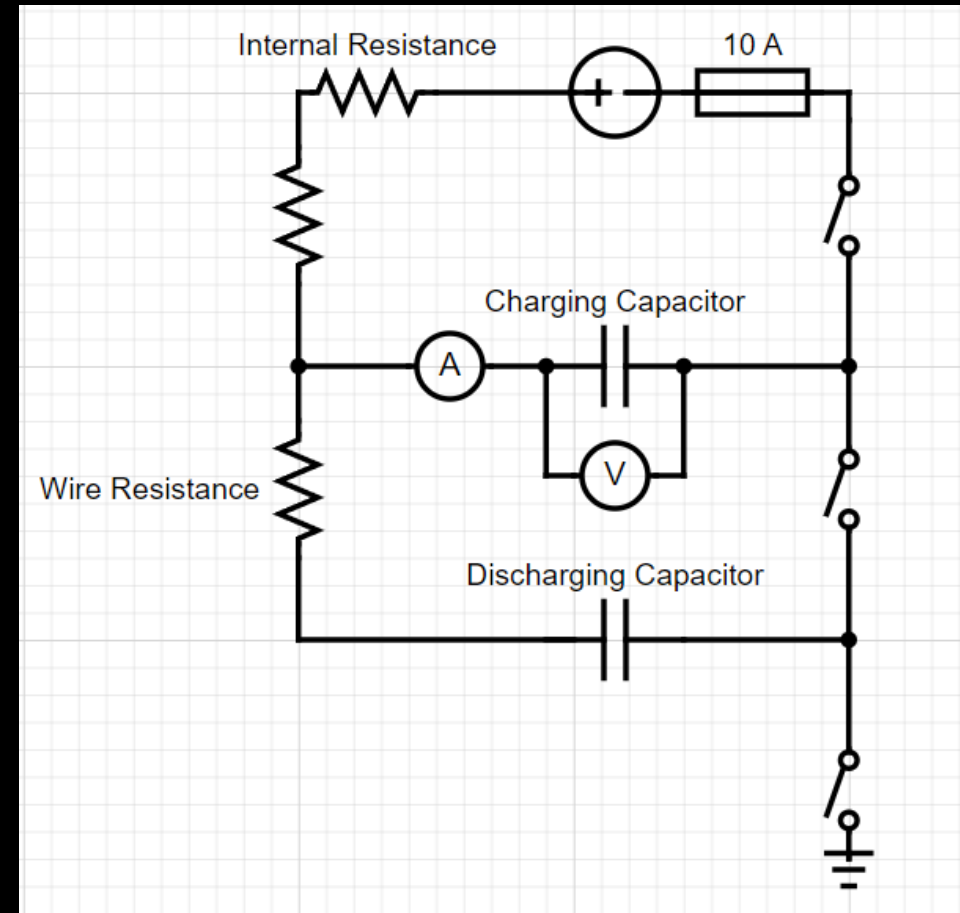
- Researched Monocrystalline and Polycrystalline Panels, Cadmium Telluride, Amorphous silicon, and Copper Indium Gallium Selenide panels
- Our final choice was to use Monocrystalline Panels

Battery Trade Study Decision Matrix

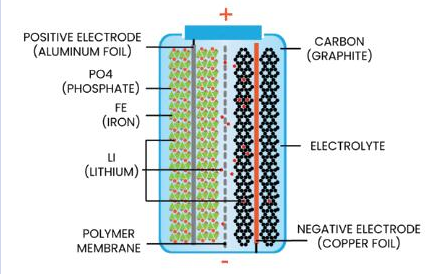
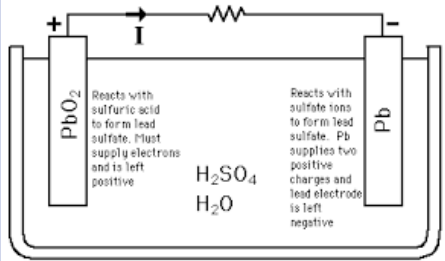
| Criteria and Weights | | Options and Scores | | | |
|------------------------|--------|--------------------|------------|-----------|----------|
| Criteria | Weight | LFP | NiMH | Lead Acid | Alkaline |
| Cost | 0.25 | 3 | 5 | 5 | 5 |
| Schedule | 0.25 | 5 | 3 | 5 | 5 |
| Risk | 0.25 | 3 | 3 | 3 | 1 |
| Performance | 0.25 | 5 | 3 | 3 | 1 |
| Weighted Scores | | 4 | 3.5 | 4 | 3 |

Top two choices from decision matrix were the Lithium-ion Phosphate Batteries (LiFePO₄ or LFP) and the Lead Acid Batteries. Lead Acid Batteries were forbidden by ESRA, so LiFePO₄ Batteries were the final choice.

Preliminary Solar Panel Circuit Design



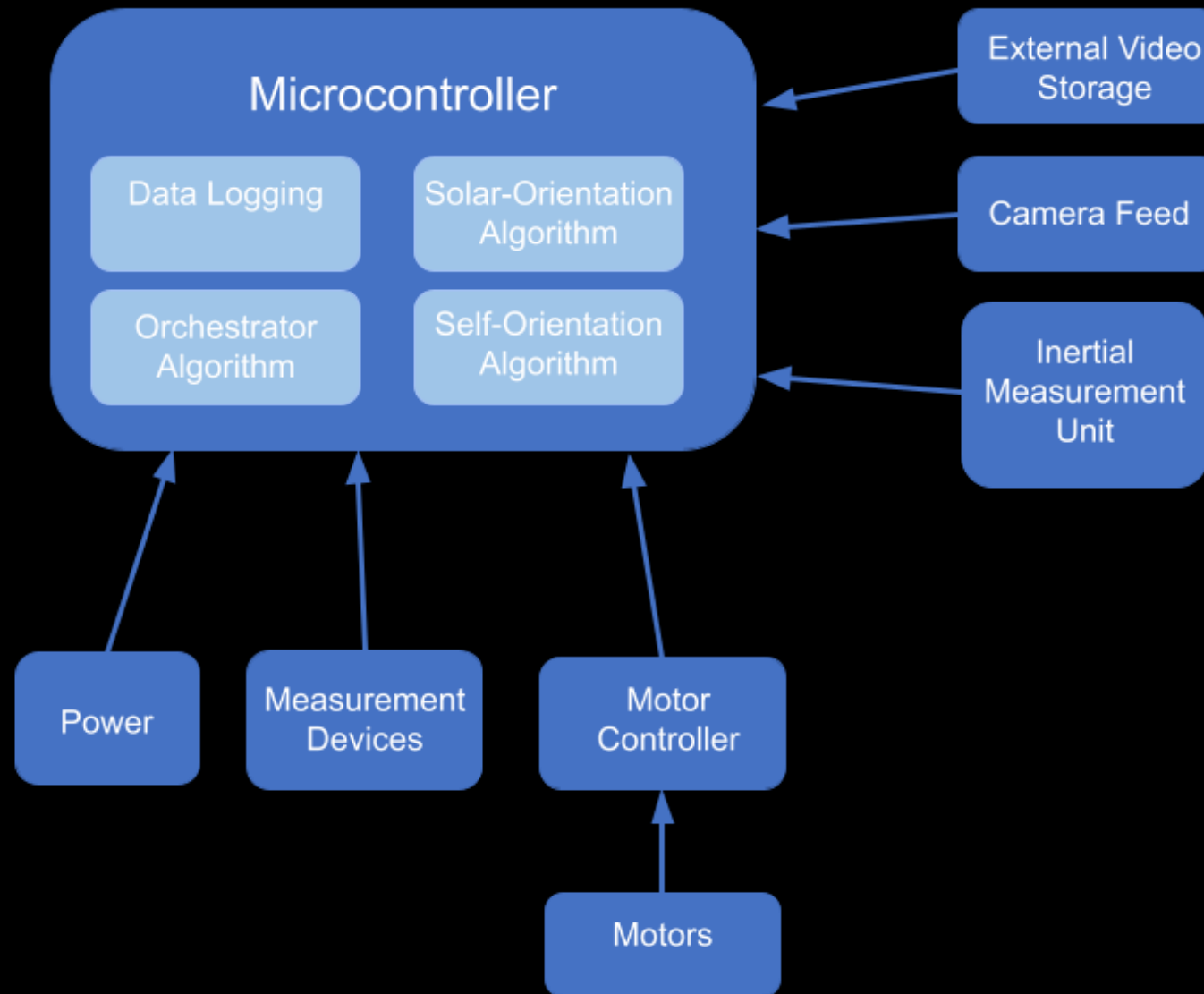
Lithium-Ion Phosphate Battery

| Type of Battery | Pros | Cons | Pictures |
|-----------------|---|---|--|
| LiFePO4 | <ul style="list-style-type: none"> • Expected cycle life of 3000 – 10,000 cycles • 98% efficient (when you put 100 AH into an LFP battery, you get about 98 Ah back out) • Short absorb time • Operates between 32°F and 120°F, with little degradation • Very lightweight | <ul style="list-style-type: none"> • Will cost about twice as much as an equivalent high quality AGM battery • Will have a very small reserve capacity (about 20%) designed into the bank • Subject to damage if over or under charged |   |

Electrical Path Forward

- ***Determine exact voltage and current specifications for the power supply***
- ***Create a detailed circuit schematic for the entire payload***
- ***Decide on specific components to purchase***

Software Sub-System



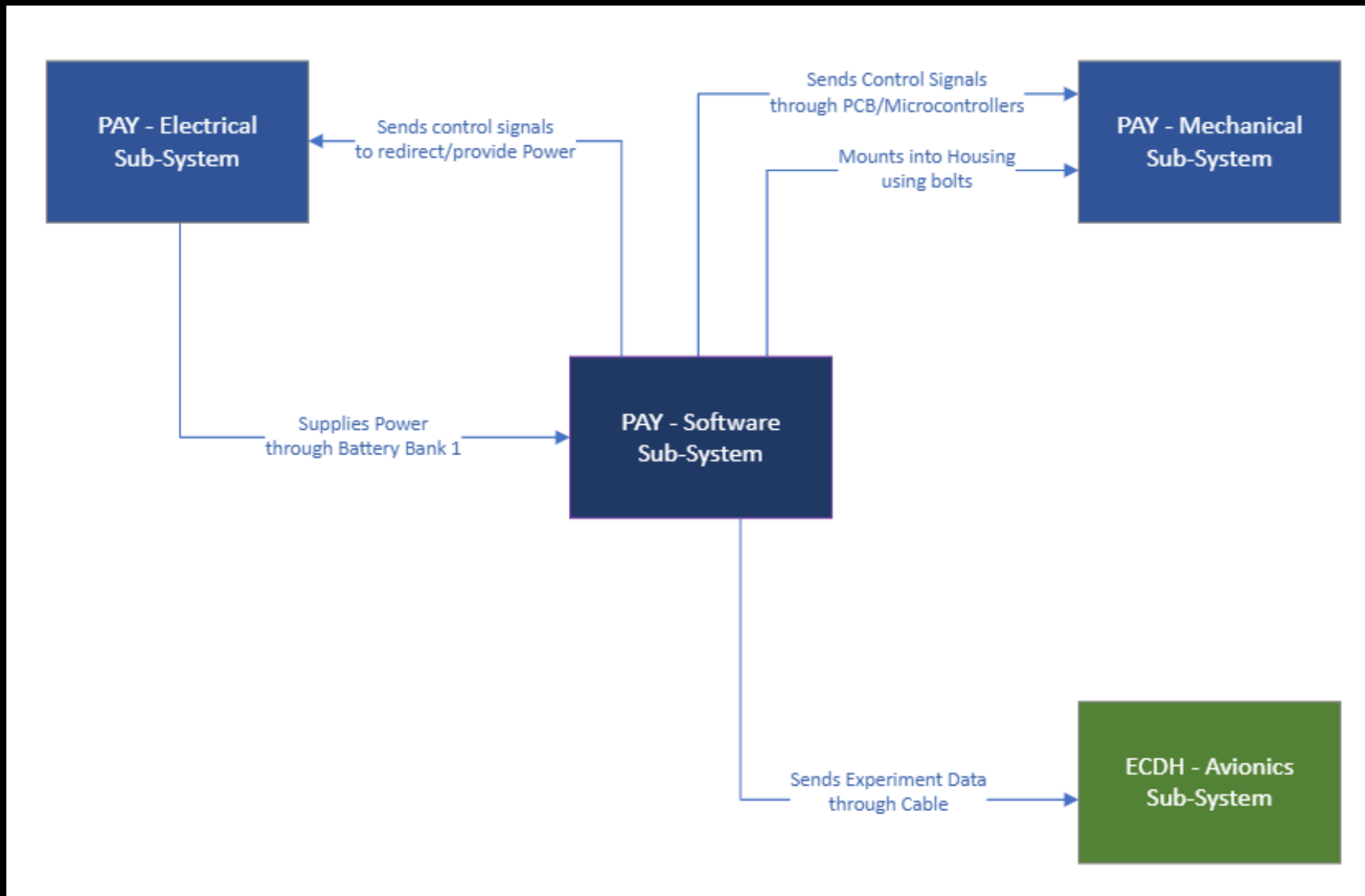
Software Requirements

| Requirement | Verification Method |
|---|---------------------|
| The software subsystem shall interface correctly with other subsystems | Inspection |
| The software subsystem shall orchestrate the mission to completion | Test |
| The software subsystem shall track the sun in the sky | Test |
| The software subsystem shall orient the payload upright | Test |
| The software subsystem shall record experimental data starting from launch | Test |
| The software subsystem shall track the payload after launch time | Test |

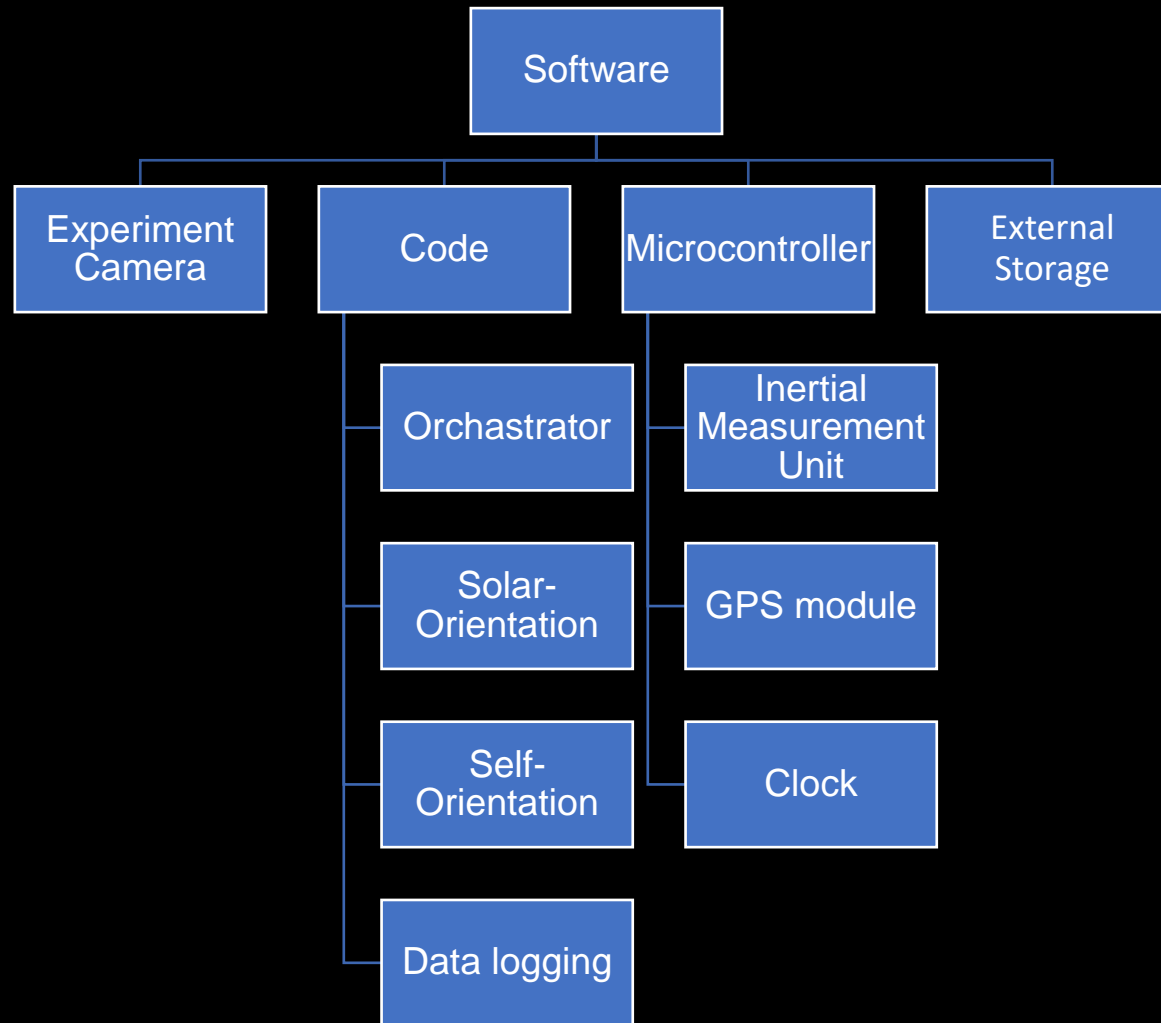
Software TPMs

| Measure | TPM Value | Units | Verification Method |
|-------------------------|-----------|---------|---------------------|
| Solar panel orientation | [5°] | Degrees | Test |
| GPS position accuracy | [5%] | Percent | Test |
| IMU accuracy | [5%] | Percent | Test |
| Execution speed | [240MHz] | MHz | Inspection |

Software Interface Diagram



Software Component Breakdown

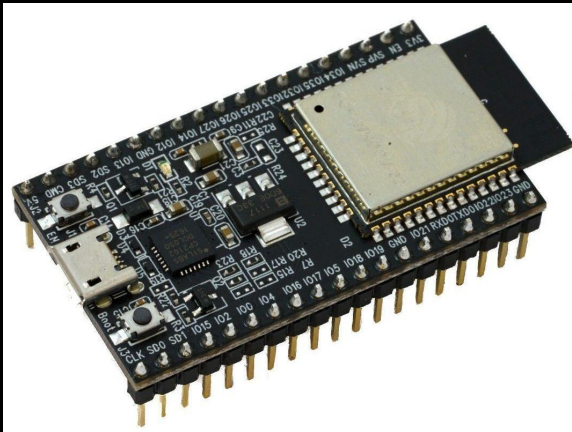


Microcontrollers Trade Study

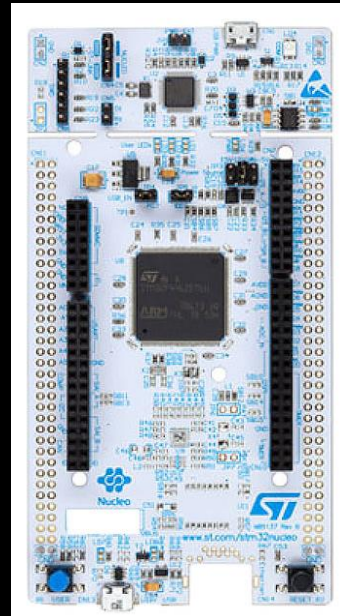
A study on which microcontroller family was to be used for the software system paired with an MPU9250 inertial measurement unit and possible long-range transmission.

Options considered:

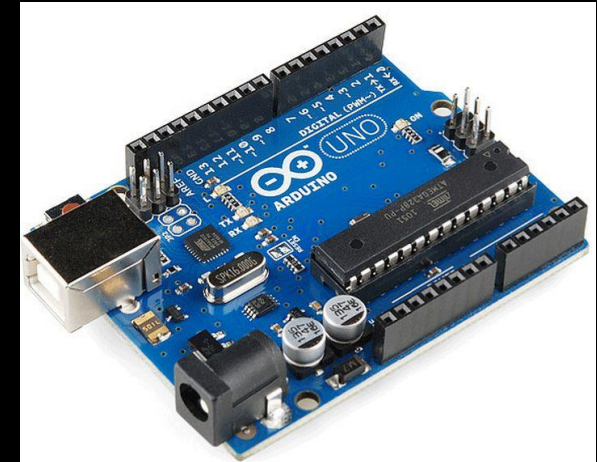
ESP32



STM32



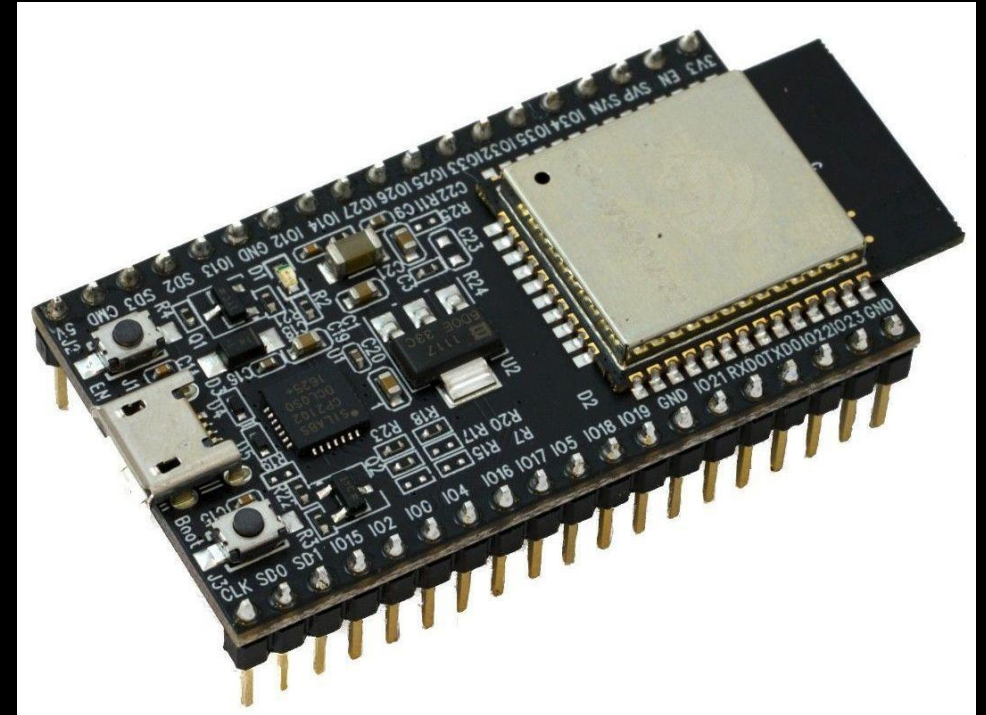
Arduino



Microcontrollers Trade Study

Option 1: ESP32

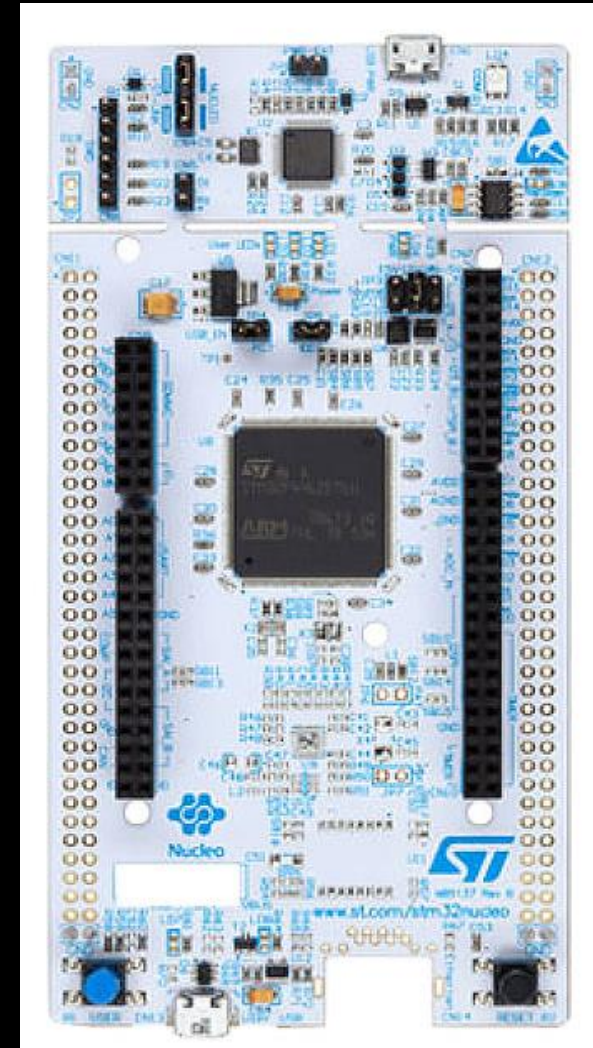
| Pros | Cons |
|---|------|
| Wireless | |
| Can act as a server | |
| Extendable internet connection (1km unobstructed through 802.11LR mode) | |
| Not IDE specific | |
| Widely Used | |
| Bluetooth | |



Microcontrollers Trade Study

Option 2: STM32

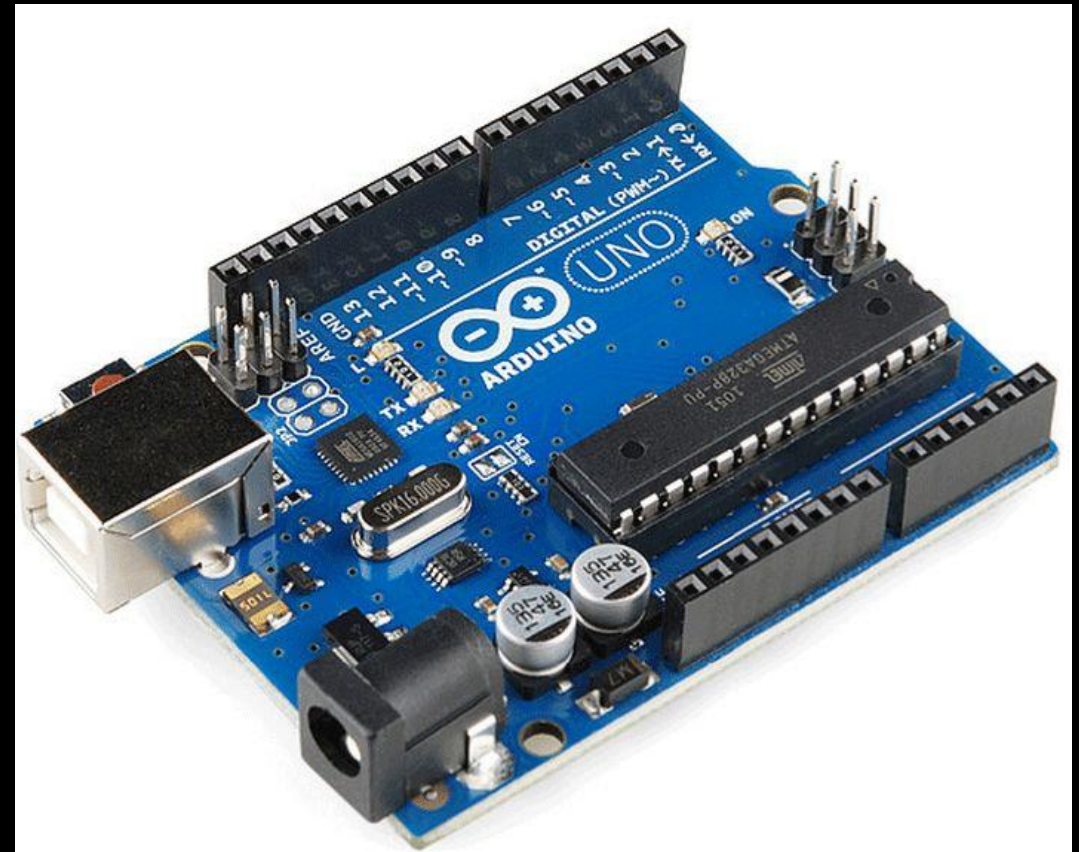
| Pros | Cons |
|------------------|-----------------------------------|
| Wireless support | No Long range wireless standard |
| Not IDE specific | Not compatible with HC-12 module |
| Bluetooth | Very limited long-range ecosystem |
| Widely used | Not widely used for long range |



Microcontrollers Trade Study

Option 3: Arduino

| Pros | Cons |
|-----------------------|-------------------------------------|
| Widely used | Arduino IDE required |
| RF module gives 1.8km | Hard wire connection to upload code |
| | Requires Long range RF module |
| | Questionable Antenna geometry |



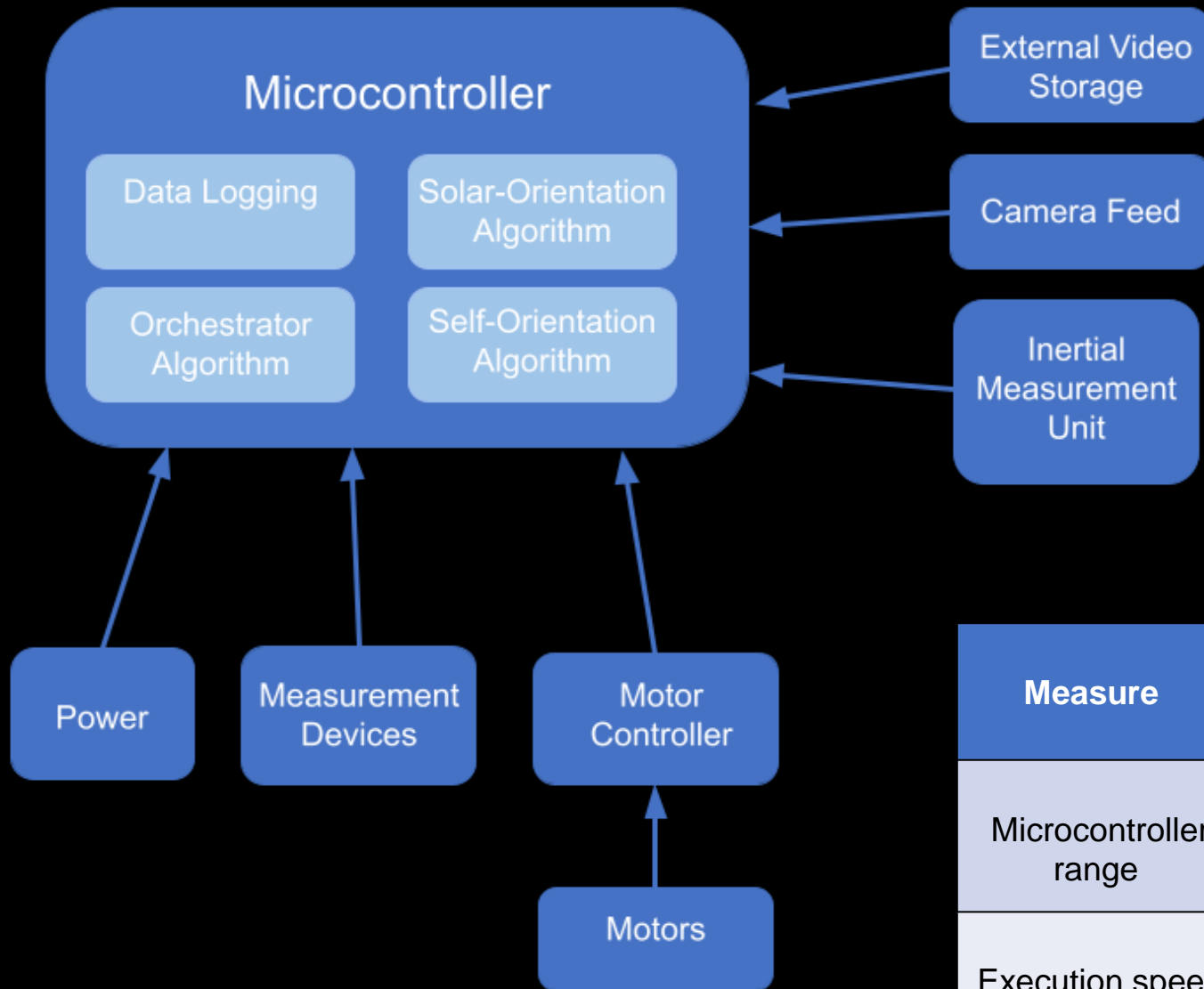
Microcontrollers Trade Study

Decision matrix

| Criteria and Weights | | Options and Scores | | | |
|------------------------|--------|--------------------|----------|----------|--|
| Criteria | Weight | STM32 | Arduino | ESP32 | |
| Cost | 0.25 | 1 | 3 | 5 | |
| Schedule | 0.25 | 5 | 5 | 5 | |
| Risk | 0.25 | 5 | 5 | 5 | |
| Performance | 0.25 | 1 | 3 | 5 | |
| Weighted Scores | | 3 | 4 | 5 | |

The ESP32 paired with the MPU9250 Inertial Measurement Unit, and possible NEO-8M GPS module was decided.

Microcontroller Component PDR



| Requirement | Verification Method |
|---|---------------------|
| The payload shall have an IMU & GPS | Inspection |
| The microcontroller shall be able to transmit its status | Test |
| The microcontroller shall have a range of >600m | Test |

| Measure | TPM Value | Units | Verification Method |
|-----------------------|-----------|--------|---------------------|
| Microcontroller range | 600m | meters | Test |
| Execution speed | [240MHz] | MHz | Inspection |

Microcontroller Component PDR

Microcontroller Failure Modes:

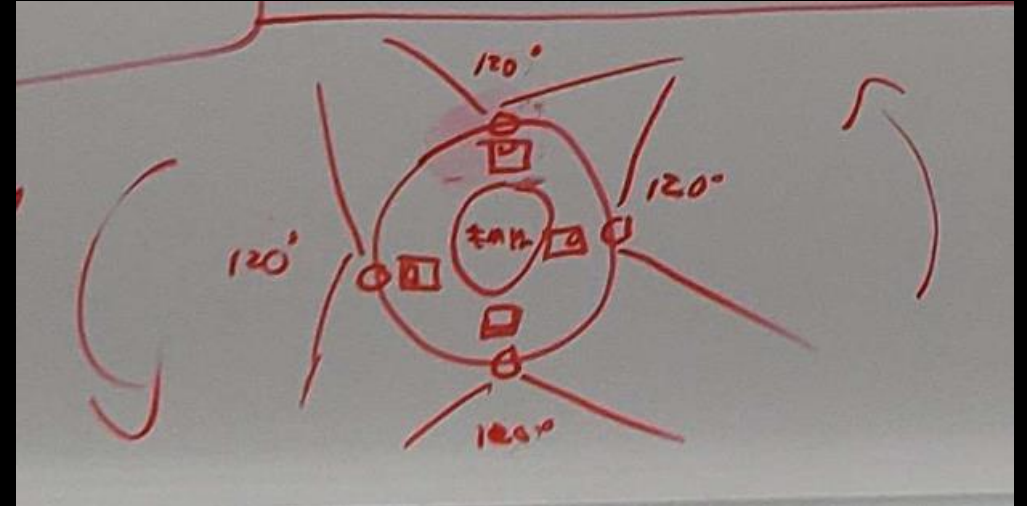
- Physical damage
 - Damage on impact could render communication issues between components or damage to components
- Possible component miscommunication
 - If an incorrect reading is given by the IMU the solar orientation algorithm will be wrong
 - If the Experimental Camera is wired incorrectly or returning the wrong data format that could corrupt all visual data storage/processing
 - If incorrect time is returned by the clock the solar orientation algorithm will be wrong

Software Path Forward

- *Decide on exact components*
- *Compile all documentation for components*
- *Acquire a test-bed to start developing algorithms*

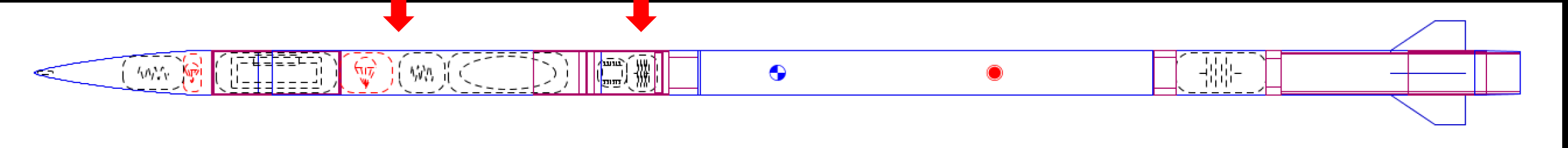
Flight Recording Sub-System

- 360 Camera System
- Quick Disconnect (QD) Real-Time Camera

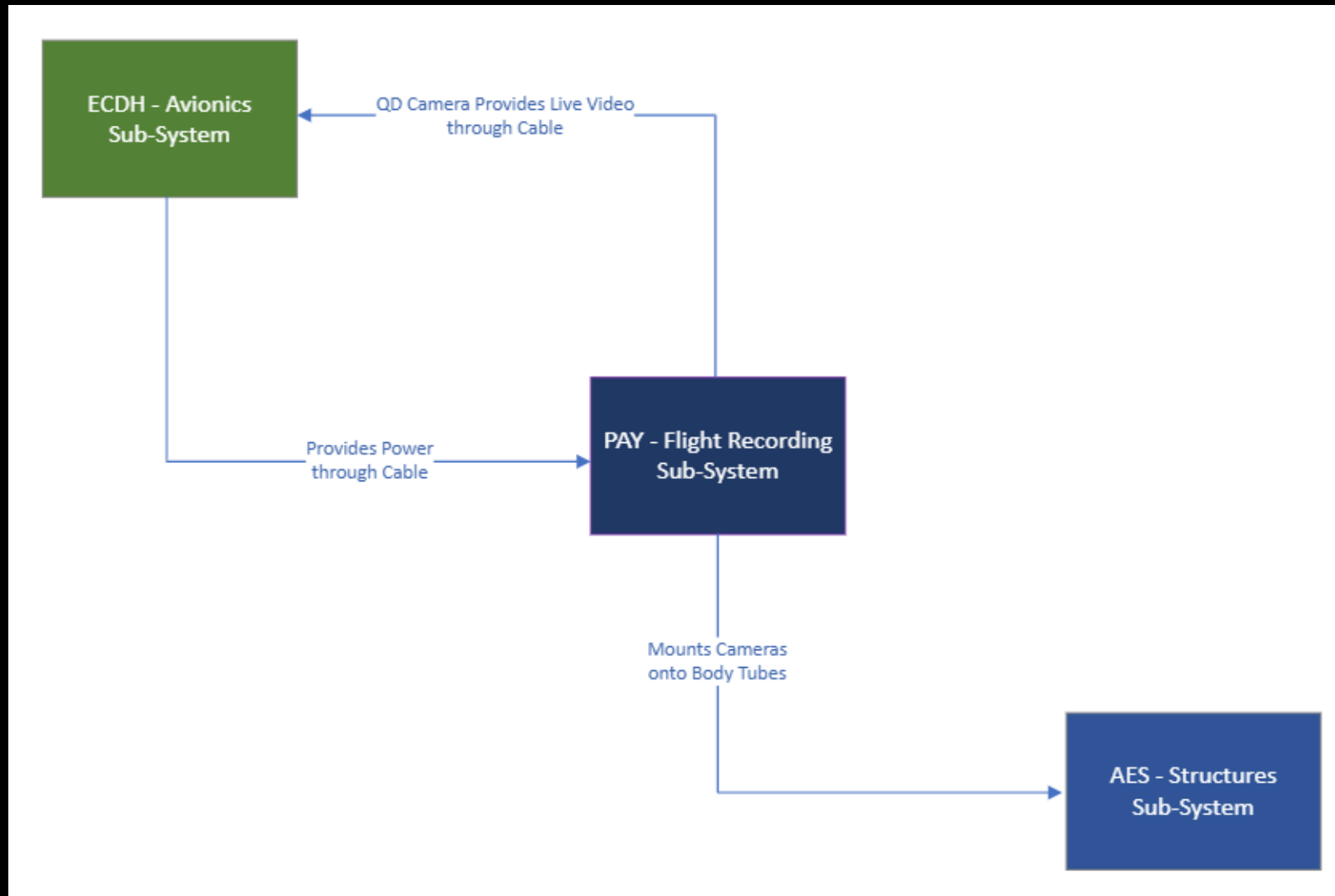


QD CAM

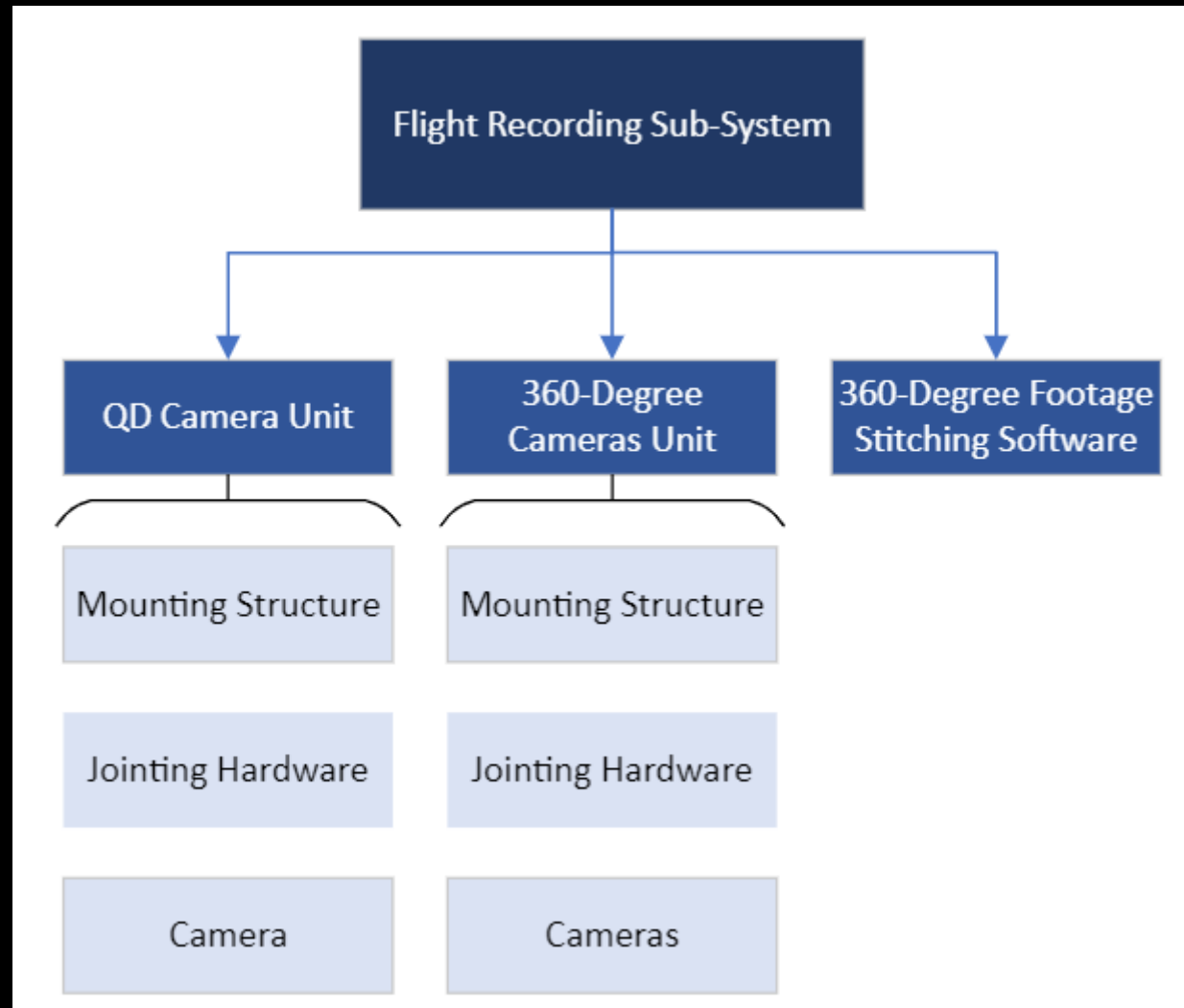
360 CAM



Flight Recording Interface Diagram



Flight Recording Component Breakdown



Flight Recording Requirements

Should be tables including requirements and verification methods

| Requirement | Verification Method |
|---|---------------------|
| The 360-degree camera system shall capture synchronized footage from [4] cameras | Demonstration |
| The QD camera shall capture and transmit live footage of the QD connector | Demonstration |
| The 360-degree camera system shall weigh [TBD lbs] | Inspection |
| The 360-degree camera system shall weigh [TBD lbs] | Inspection |
| Both camera systems shall interface with the aerostructures system | Inspection |
| Both camera systems shall withstand [10 G] of acceleration | Demonstration |

Flight Recording TPMs

| Measure | TPM Value | Units | Verification Method |
|---------------------------------------|-----------|-------------------|---------------------|
| Quick Disconnect Camera System | | | |
| Resolution | [1080p] | Pixels | Test |
| Framerate | [30fps] | Frames per second | Test |
| Field of View | [90°] | Degrees | Test |
| 360-Degree Camera System | | | |
| Resolution | [1080p] | Pixels | Test |
| Framerate | [30fps] | Frames per second | Test |
| Field of View | [90°] | Degrees | Test |

Camera Trade Study

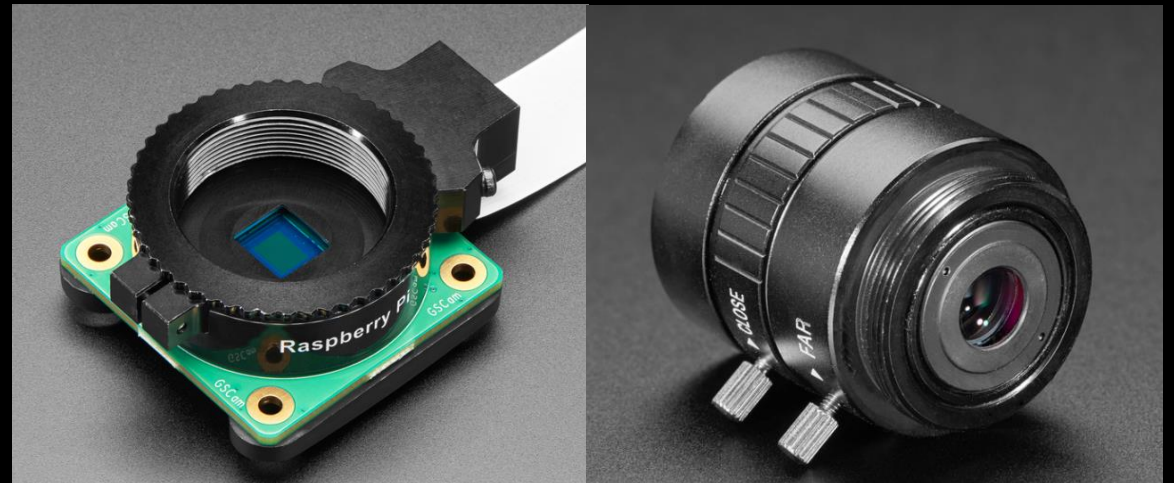
The Top Two According to the decision Matrix were the Raspberry Pi Cameras:

Arducam Camera for Raspberry Pi



Pros: Cheap, Field of View
(120°(D)x88°(H)x55°(V)) , 1080p
Cons: Rolling Shutter, 30 fps

Raspberry Pi Global Shutter Camera + Wide Angle Lens



Pros: Global shutter, can record
1440 x 1080 pixels at 60 fps
Cons: 63 degree angle lens, more
expensive

Camera Trade Study

Looked at 6 different options: 4 stand-alone cameras & 2 Raspberry Pi cameras

| Criteria and Weights | | Options and Scores | | | | | |
|------------------------|--------|--------------------|----------------|------------|-----------------|--------------|----------------|
| Criteria | Weight | AKASO Action | Estes AstroCam | RunCam2 | Mobius Pro Mini | Raspberry Pi | Global Shutter |
| Cost | 0.35 | 3 | 5 | 3 | 3 | 5 | 3 |
| Schedule | 0.1 | 5 | 5 | 5 | 5 | 5 | 5 |
| Risk | 0.2 | 3 | 1 | 3 | 3 | 5 | 5 |
| Performance | 0.35 | 3 | 1 | 5 | 5 | 3 | 5 |
| Weighted Scores | | 3.2 | 2.8 | 3.9 | 3.9 | 4.3 | 4.3 |

Flight Recording Path Forward

- ***Make a final camera selection***
- ***Procure components and begin test-bedding both camera systems***
- ***Create CAD***